NASA/CR-2003-212369



Design and Test of Fan/Nacelle Models Quiet High-Speed Fan Design

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NAS3-27752, NASA AST AOI 14 DESIGN AND TEST OF FAN/NACELLE MODELS QUIET HIGH-SPEED FAN DESIGN REPORT

1.0 SUMMARY

1.1 Purpose

The primary objective of the Quiet High-Speed Fan (QHSF) program was to develop an advanced high-speed fan design that will achieve a 6 dB reduction in overall fan noise over a baseline configuration while maintaining similar performance. The program applies and validates acoustic, aerodynamic, aeroelastic, and mechanical design tools developed by NASA, US industry, and academia. The successful fan design will be used in an AlliedSignal Engines (AE) advanced regional engine to be marketed in the year 2000 and beyond. This technology is needed to maintain US industry leadership in the regional turbofan engine market.

1.2 Conclusions

The AE/QHSF was designed to match the baseline rotor design point performance and yet achieve a 6 dB reduction. Innovative techniques in coupling the aerodynamic and mechanical analyses, along with design of experiments (DOE's) involving multiple engineering disciplines accomplished the effort. The result was a fan stage design with a forward swept rotor with 50 degrees of leading edge sweep that matched the baseline aerodynamic performance and achieved all mechanical and aeroelastic design criteria; and, a stator lean tailored for minimum rotor-stator acoustic interaction.

1.3 Design Procedure

The design technique used for the QHSF involved early design input from the Aerodynamic, Mechanical, Acoustic, and Aeroelastic disciplines through the use of design of experiments. Additional optimization procedures were used to enhance and verify the DOE analyses and results. Four DOE's were completed for the rotor and two for the stator which focused on achieving the design goals without violating established design criteria.

1.4 Design Summary/Perspective

In order to obtain the design goal of a 6 dB reduction, the design concentrated on three aspects. Reduction of the inlet relative normal Mach number, position of the shock within the passage, and minimizing rotor-stator interaction effects. Leading edge effective sweep was used to reduce the inlet relative normal Mach number, while mean-camber line blade angle distributions were used to correctly position the shock in the blade-to-blade passage. Although some design constraints were imposed on the stator design, rotor-stator interaction effects were minimized by controlling the rotor wake impingement on the stator leading edge.

The AE/QHSF is a forward swept rotor designed to match the design point performance of the baseline. It incorporates relatively large amounts of leading edge effective sweep, and yet with the use of statistical analyses and optimization procedures has reduced stress and weight while maintaining adequate aeroelastic and critical vibration margins relative to the baseline. Table 1.4-1 shows a summary of the AE/QHSF design features at the aerodynamic design point.

TABLE 1.4-1 AE/QHSF SUMMARY AERODYNAMIC DESIGN POINT

Performance Parameters:						
Inlet corrected flow						
Bypass ratio 3.78						
	DAW	ES Resu	lts in AX	CAPS	Test	
	QH	ISF	Baseline		Baseline	
	Rotor	<u>Stage</u>	<u>Rotor</u>	<u>Stage</u>	<u>Stage</u>	
Bypass pressure ratio	1.839	1.813	1.839	1.811	1.844	
Core pressure ratio	1.845	1.817	1.777	1.741	1.718	
Bypass efficiency (ad)	0.883	0.860	0.888	0.865	0.889	
Core efficiency (ad)	0.960	0.934	0.961	0.923	0.918	
O/A pressure ratio		1.813		1.796	1.818	
O/A efficiency (ad)		0.874		0.875	0.895	
Rotor Parameters:						
Number of blades						
Corrected RPM		15358	3			
Inlet tip diameter						
Corrected tip speed		1474	ft/s			
Hub - tip ratio	0.35					
Aspect ratio 1.35						
Inlet specific flow						
Hub and tip slope						
Hub work coefficient						
Inlet tip rel Mach no		1.43				
Stator Parameters:						
Number of vanes						
Aspect ratio		2.42				
Exit Mach no. (avg.)		0.52				
Exit air angle (avg.)		0 º				

2.0 INTRODUCTION AND BACKGROUND

2.1 AST Area of Interest 14

Area of Interest 14 specifically addresses two level-three milestones to support "Model Tests for Code Validation/Concept Evaluation" in the Noise Reduction element of the NASA Advanced Subsonic Technology (AST) Program:

- Select second-generation, low-noise concepts for model tests (4Q, FY 1997)
- Complete second-generation model tests for low noise designs (4Q, FY 1999)

A 2-year effort was originally proposed for this effort. During the first year, a calibration of the acoustic and aerodynamic prediction methods was performed and a baseline fan definition was established and evaluated. The second year activities were to include evaluation of several candidate noise reduction concepts using combinations of rotor and stator lean and sweep. The program scope has since been expanded. The program roadmap is illustrated in Figure 2.1-1. Three fan designs are used in the program. The existing NASA-Lewis QF-12 fan and an AE TFE derivative fan serve as baseline configurations. Data from these fans will aid in design tool validation. A new advanced fan was designed by AE. The program will lead to a model scale test using the Ultra High Bypass (UHB) Propulsion Simulator in the NASA-Lewis 9 x 15-foot wind tunnel.

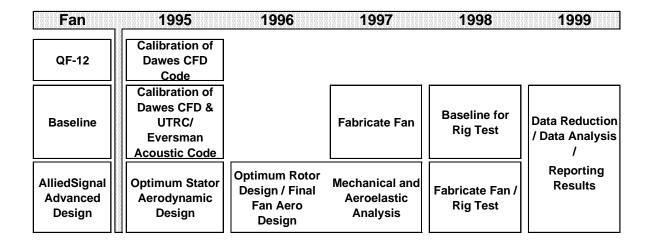


Figure 2.1-1. The Quiet High-Speed Fan Program Roadmap Targets a Wind Tunnel Test in October of 1998.

2.2 Quiet High Speed Fan Program

The QHSF program is a cooperative effort between AE and the NASA Lewis Research Center. Both NASA and AE will design an advanced high-speed fan that will be tested on the Universal Propulsion Simulator in the NASA Lewis 9 x 15 foot wind tunnel, currently scheduled for the third quarter of 1998. An AE scaled TFE derivative fan design will be used as a baseline. A nacelle model will be provided that will be characteristic of a typical, modern regional aircraft nacelle and meet all of the program test objectives.

The quiet high-speed fan is an advanced single-stage fan designed for a 5K to 20K pound thrust turbofan regional airline application. Advanced aerodynamic, mechanical, aeroelastic, and computational fluid dynamic (CFD) tools will be used to meet aggressive performance goals while achieving at least a 6 dB reduction in fan noise at a critical takeoff noise condition. Three fans will be built for evaluation at NASA; a baseline TFE derivative, an advanced NASA design, and an advanced AE design. A 22-in. diameter fan nacelle for the UHB Propulsion Simulator in the NASA-Lewis 9 x 15 foot wind tunnel will integrate NASA and AE aerodynamic designs and instrumentation to fulfill all aerodynamic, mechanical, distortion sensitivity, and acoustic test needs.

2.3 Baseline Fan

The baseline consists of a damperless, low-aspect- ratio, moderately aft swept rotor and full-span aft swept composite stator vanes. The geared fan configuration allows the fan to run at a tip speed optimized for best performance, stall margin, and noise.

The baseline fan was the subject of considerable acoustic evaluation early in the design phase. Blade and vane counts, rotor/stator spacing, and stator vane sweep were selected to minimize the noise signature within the fan design constraints. Fan rig spinning-mode measurements were made to verify the design constraints and provide acoustic treatment design criteria. The acoustic effort was concluded with a full-scale engine acoustic test which verified that the acoustic design goals, including the acoustic treatment, were satisfied.

Installed performance of the baseline fan was a critical issue. Measuring fan component performance accurately during flight was required. The engine nacelle and inlet section of the flight test engine were custom tailored and instrumented for flow measurement. The engine front frame was also instrumented so fan performance could be measured without impacting fan or engine performance. The fan performance measured in flight at appropriate Reynolds numbers during the extensive flight test program agreed very well with rig and engine data acquired on the ground using conventional fluid metering techniques. The experience gained on these programs will be used on the Quiet High-Speed Fan program.

2.4 Historical Background

The presence of a shock at the inlet of the fan rotor in a turbofan engine can result in acoustic phenomena that represent substantial noise sources. Multiple Pure Tone (MPT) noise, for example, results when the pressure disturbances from the inlet shock move upstream out of the rotor blade passage. One approach to reducing these shock-related noise sources is to eliminate the formation of the inlet shock in the fan by tailoring the rotor blade shape. The introduction of sweep in the fan rotor blade can reduce the relative velocity component normal to the blade to subsonic values, much as a swept wing on an aircraft can produce subsonic velocities normal to the wing leading edge, even when the resultant velocity is supersonic.

This noise minimization technique was applied by AlliedSignal Engines (formerly AVCO Lycoming) and Bolt, Beranek and Newman, Inc. (BBN) to the design of the QF-12 quiet high-speed fan, as part of a NASA-sponsored program performed between 1974 and 1977 (Reference 1). The fan rotor featured a compound forward-and-aft sweep to eliminate the leading edge shock. However, although shock-induced MPT noise was reduced, the aerodynamic performance of the fan did not meet design goals. The reasons for the performance deficiencies were not determined, but the source was found to be localized in the rotor. Figure 2.4-1 shows a summary of the results of the acoustic evaluation of the QF-12.

The ability to accurately predict, during the fan rotor design, the speed range over which the MPT noise will occur can provide a valuable tool for acoustically tailoring the design of the rotor to minimize the effect of this noise source.

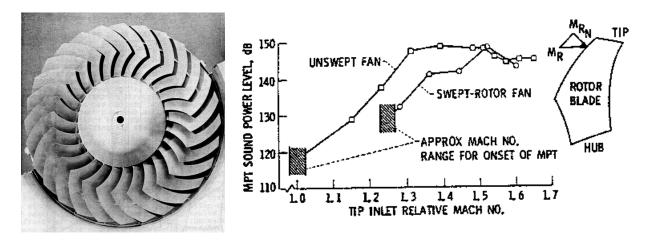


Figure 2.4-1. The QF-12 Fan Demonstrated the Feasibility of Using Blade Leading Edge Sweep to Reduce Fan Noise.

3.0 DESIGN OBJECTIVES / CRITERIA

3.1 Overall

The AE/QHSF stage design included design objectives from the acoustic, aerodynamic, mechanical, and aeroelastic disciplines. Acoustic objectives included a 6 dB reduction in effective perceived noise (EPNdB) at takeoff relative to the baseline. This was to be accomplished without the known benefits of vane count reduction and increased axial spacing (hub and tip) between rotor TE and vane LE. The intent of the design was focused on obtaining noise reductions through the use of more unconventional design techniques such as: controlling the rotor wake impinging on the stator LE, reducing rotor normal inlet relative Mach number, and controlling shocks within blade passage. Aerodynamic objectives were to obtain baseline performance (or better) with a forward swept rotor and acoustically matched vanes. Mechanical and aeroelastic objectives were to obtain AE and NASA criteria for stress and burst margins, proper positioning of mode / harmonic crossings to minimize high vibratory strains, and provide sufficient flutter margin throughout the operating region.

3.2 Specific

Specific aerodynamic design criteria is shown in Table 3.2-1. Mechanical criteria is shown in Table 3.2-2 and Figure 3.2-1.

Table 3.2-1. Specific Aerodynamic Design Criteria

Fan Aero Design Point (100% Nc)

	22" DIA
Wcorr, lbm/s	98.9
Wc/A, lbm/s/ft^2	42.7
Utcorr, ft/s	1474
Bypass ratio	3.8
P/P, overall	1.82
Eff ad, overall	> .895
Stall margin (N _{1C} =C)	15%
Hub/Tip ratio	.35
Rotor blade count	22
Stator vane count	52

Table 3.2-2. Specific Mechanical Design Criteria

- Airfoil and Attachment LCF Life > 10^5 cycles
 - Peak Stress in Attachment and Airfoil
- HCF Life > 10^7 Cycles
 - Frequency, Excitation (Inlet Distortion Harmonics, Speed @Resonance)
- Bird-Ingestion Capability
 - Bird Weight, LE Thickness
- · Flutter Margin Exceeds Baseline
 - Reduced Frequency, Twist/Flex, and Incidence
- Blade Weight Equal to Or Less Than Baseline
- High Speed Resonances Must Be Avoided
 - Rotor
 - Mode 1 / 2E Crossing < 50 percent speed
 - Mode 2 / 4E Crossing < 70 percent speed
- 10% Frequency Margin at Maximum Engine Operating Speed
- Modified Goodman Approach used SF*Se for HCF Calculation

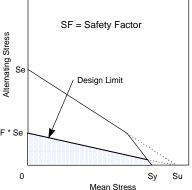


Figure 3.2-1. Mechanical Criteria for High-Cycle Fatigue

4.0 PRELIMINARY DESIGN

4.1 Cycle Point Selection

An engine cycle design point was originally established for the AE/QHSF to correspond to a regional type application for the AE Small Engine Technology (SET) engine concept. However, during the initial study efforts, it became apparent that although the cycle point originally selected would benefit the AST program in terms of a regional application; it would make comparisons relative to the baseline (design and test) much more difficult. Therefore, the baseline design cycle was selected which would provide the best and cleanest 'back-to-back' test to verify Area of Interest 14 technology goals.

4.2 Rotor

4.2.1 Blade Harmonic Selection

Analyses were completed to determine whether the rotor should be a "1/rev" design (similar to the baseline) or a "2/rev". The 1/rev design offers the advantage of less weight but, without a midspan damper, increases the susceptibility to flutter. An optimization process using ANSYS was completed for both designs in order to minimize weight while maintaining all other design criteria.

A parametric model of the airfoil was defined to allow aeromechanical optimization via ANSYS. The value of maximum airfoil thickness (Tmax/C) and its chordwise position were varied at several spanwise locations, including the hub and tip. A cubic spline fit through these control points was used to define these distributions at 5% span increments.

X-cg and Y-cg distributions were defined along 4th-order polynomial functions fixed at the airfoil hub. The X-cg at the tip was fixed at 1.07 inches forward. The coefficients of the polynomials were used as design variables. X-cg and Y-cg were defined at 5% span increments.

The blade geometric parameters were used to create an AXCAPS (AE's radial equilibrium streamline flow solver) input file, with the resulting airfoil geometry automatically meshed and analyzed. These geometric parameters were defined as independent design variables for the optimization procedure. A single objective function is available with ANSYS, thus additional "objectives" were forced with constraints.

The first optimization was for a blade with a 1/rev crossing (1E/4E). The initial analyses were conducted with the Tmax/C and position of Tmax design variables defined at 4 equally-spaced, spanwise sections. The resulting optimized airfoil was approximately 23 percent heavier than the baseline design. The spanwise resolution was subsequently increased to 6 sections. A cubic spline fit through these control points was used to define these distributions at 5% span increments. Using the new spanwise resolution, the optimized blade was only 1 percent heavier than the baseline blade.

Table 4.2.1-1. AE/QHSF Preliminary 1/rev Design

Parameter	QHSF 1/Rev Airfoil (Fixed-root; no root fillet)	Relative to Baseline
Airfoil Weight	1.27 lb	+ 1.2 %
Max Principal Stress	82 ksi	-10.8 %
Max Deflection	0.31 in	-19.5 %
Mode 1/2E crossing	48 percent speed	-28 %
Flutter Margin	-0.44	+10 %
Tip Tmax/C	0.02	Same

Initial optimization analyses for the 2/rev crossing (2E/4E) were also conducted with Tmax/C and position of Tmax design variables defined at 4 equally-spaced, spanwise sections. The best, near-feasible solution obtained was 31 percent heavier than baseline blade. Based on the success of the 1E/4E study, the spanwise Tmax/C and position of Tmax design variables were defined up to 11 sections. A cubic spline fit through these control points was used to define these distributions at 5% span increments. The total number of design variables (DV's) was 29 for the 11 point case even though ANSYS recommends no more than 20 DV's.

The weight of the 2E/4E design was relatively unaffected by an increase in spanwise sections. The best design was slightly low on mode 1 frequency margin, 14 percent vs. the desired 15 percent, but met all other mechanical constraints. The blade was still 22 percent heavier than the baseline blade; and, was obtained only after reducing Tmax/C at the tip to 0.01, well below the design criteria of 0.02.

Table 4.2.1-2. AE/QHSF Preliminary 2/rev Design

Parameter	QHSF 2/Rev Airfoil	Relative to Baseline
	(Fixed-root; no root fillet)	
A . C . 1 X 1 .	1.52.11	. 220/
Airfoil Weight	1.53 lb	+22%
Max Stress	67 ksi	-27%
Max Deflection	0.39 in	0
Mode 2/4E freq margin	14 percent (require 15%)	
Flutter Margin	-0.32	+18%
Tip Tmax/C	0.01	-50%

4.2.2 DAWES Model

The DAWES program is a finite-volume time-marching solver for the 3-D thin-layer Navier-Stokes equations. The version of the program used by AE for the QHSF was the same version previously used on the baseline rotor design. To maintain close compatibility, the same grid and flow size were used for the two designs. All DAWES analyses, rotor and stator, were completed at the baseline engine flow size.

The blade and flowpath geometry descriptions for the fan rotor were obtained from AE's axisymmetric streamline curvature program, AXCAPS. This program provides a discrete-point definition of the blade geometry along specified stream surfaces intersected with the blade surface. Endwall definition is also provided in terms of radial and axial coordinates at discrete points. The AXCAPS model of the flowpath is shown in Figure 4.2.2-1.

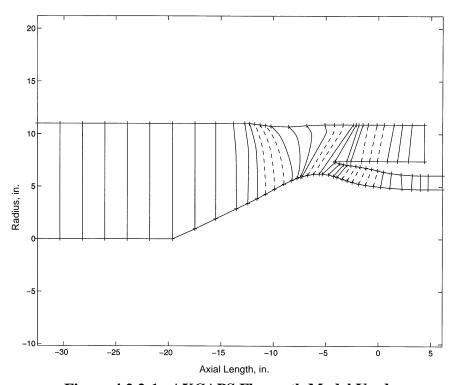


Figure 4.2.2-1. AXCAPS Flowpath Model Used

The geometry represented in the AXCAPS model was the deflected aerodynamic blade shape at 100 percent wheel speed. Hub fillets were not modeled in DAWES; and, the blade tip was modeled as tapered, rather than squared-off, which is required by a gridding limitation within the program.

The computational domain consisted of a single rotor blade passage, with the suction surface of one blade forming one pitchwise boundary, and the pressure surface of the adjacent blade forming the other pitchwise boundary. The inlet to the computational domain was positioned upstream (approximately one tip chord) to reduce impacting the development of the upstream shock structure. The exit of the computational domain was located the same, relative to the rotor TE, as the baseline. Because of the rotor forward sweep, the exit plane did not include the outer 40% span of the stator leading edge. Analyses were completed late in the design with an extended grid which showed there was no impact on the previously completed rotor wake/stator interaction study.

The computational grid for the DAWES program is a structured single-block skewed H-grid. Typical sections of the computational grid are presented in Figure 4.2.2-2. The grid size employed for the final set of rotor analyses was specified as:

Pitchwise: 25 Nodes

Spanwise: 71 Nodes (with 8 cells in the clearance gap)

Streamwise: 131 Nodes

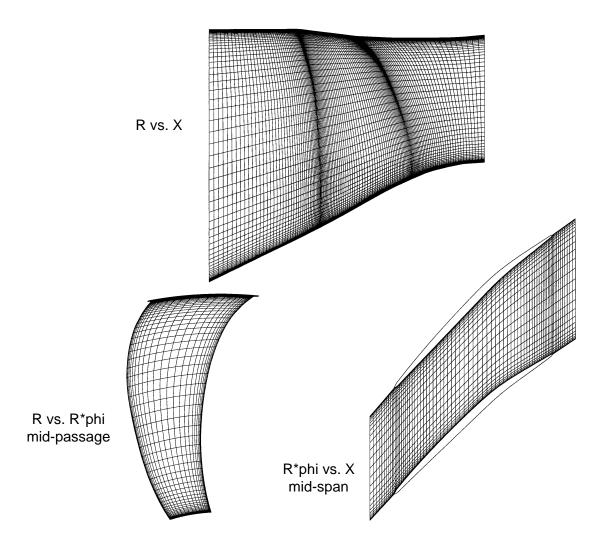


Figure 4.2.2-2. Rotor DAWES Computational Grid

4.2.3 Design of Experiment 1

4.2.3.1 Purpose

The first statistical design of experiment (DOE) was created to determine the extent of forward sweep required to obtain the 6 dB noise reduction with marginal violation of the other performance goals. (It was believed that the performance goals would be achieved in subsequent DOE's planned.) The DOE involved analyses from the Aerodynamic, Mechanical, Acoustic, and Aeroelastic disciplines.

4.2.3.2 Description

Because of the perceived non-linearity of the axial and tangential leans, the DOE was created to consist of 4 factors (Xcg-tip, Ycg-tip, Xcg-midspan, Ycg-midspan) with 3 levels each. The 9 cases are described in Table 4.2.3-1, with the CG orientation shown in Figure 4.2.3-1. Figures 4.2.3-2 and 4.2.3-3 show the CG distribrutions of the nine cases. The blade geometry and the mean flow properties were determined using AXCAPS. The intent was to analyze the cases at design speed (100% Nc) and 80% Nc using DENTON (Denton's 3D inviscid Euler code) to obtain the aerodynamic and acoustic quality characteristics. It was believed, based on past experience, that DENTON could be used to provide accurate solutions for flow, and shock locations; and, because of the fast turnaround time would be advantages for the DOE. For solution verification, several cases were analyzed using DAWES which showed some differences in flow and passage shock locations. The differences were believed to be caused by using convergence factors (time step, pitch and axial smoothing, damping, and inlet smoothing) in DENTON which deviated from experience but were required for case convergence. DAWES was therefore used to analyze the cases; however, because of the CPU time required, only the design speed was analyzed for the DOE.

Table 4.2.3-1. DOE 1 Forward Sweep Analysis X-Factors

Case No.	X-cg tip	Y-cg tip	X-cg mid	Y-cg mid
1	-1.50	-1.0	-0.25	-0.50
2	-1.50	0	0	0
3	-1.50	+1.0	+0.60	+0.50
4	-0.75	-1.0	0	+0.50
5	-0.75	0	+0.60	-0.50
6	-0.75	+1.0	-0.25	0
7	0	-1.0	+0.60	0
8	0	0	-0.25	+0.50
9	0	+1.0	0	-0.50

Dimensions are in inches and relative to Baseline engine size (30.7 in Dia.). Negative X-cg is axial forward direction, positive Y-cg is in direction of rotation

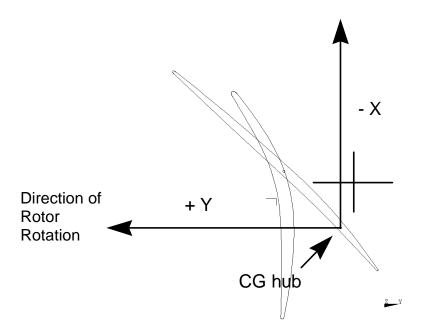


Figure 4.2.3-1. Blade CG Orientation Definition.

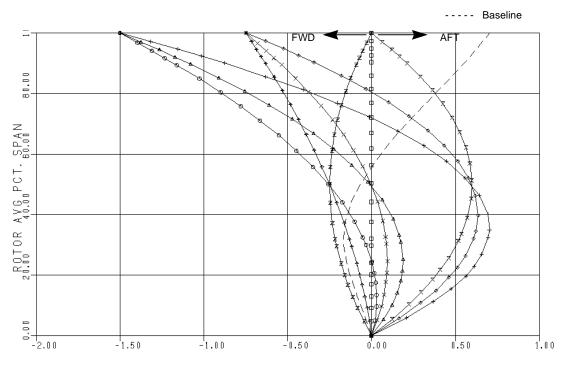


Figure 4.2.3-2. Xcg distributions used in DOE 1.

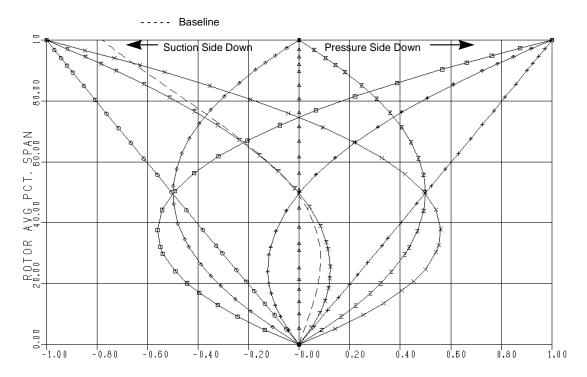


Figure 4.2.3-3. Ycg distributions used in DOE 1.

4.2.3.3 Analysis

Response variables (Y-factors) were compiled based on analyses from the four engineering disciplines. These factors became somewhat dynamic in that some were deleted, while others were added or calculated slightly differently. The final list of Y-factors with a brief description is shown in Table 4.2.3-2 with results tabulated in Table 4.2.3-3. Vector solutions for each case were created at 55.9% corrected fan speed using an off-design model from the design point. This speed corresponds to an approach condition where interaction noise is likely to be the dominant fan noise source (buzzsaw noise cutoff). Predictions of rotor/stator interaction noise were performed using the NASA code V072. The baseline stator was used for all rotor design studies.

Table 4.2.3-2. DOE 1 Forward Sweep Analysis Y-Factors

	Discipline	Y-factor	Description	Analysis Tool	<u>% N1c</u>
1	Acoustic	dB	Interaction noise sound power level	V072	55.9
2	Acoustic	eff swp le	Rotor LE effective sweep	AXCAPS	100.0
3	Acoustic	%c 50	Passage shock location at 50% span	DAWES	100.0
4	Acoustic	%c 60	Passage shock location at 60% span	DAWES	100.0
5	Acoustic	%c 70	Passage shock location at 70% span	DAWES	100.0
6	Acoustic	%c 80	Passage shock location at 80% span	DAWES	100.0
7	Acoustic	%c 90	Passage shock location at 90% span	DAWES	100.0
8	Acoustic	%c 95	Passage shock location at 95% span	DAWES	100.0
9	Acoustic	delz 50	Axial distance RTE - VLE midspan	AXCAPS	
10	Acoustic	delz 100	Axial distance RTE - VLE tip	AXCAPS	
11	Aero	Wc	Inlet corrected flow (top of choke)	DAWES	100.0
12	Aero	PR	Rotor pressure ratio (top of choke)	DAWES	100.0
13	Aero	Eff	Rotor efficiency (top of choke)	DAWES	100.0
14	Aero	Mpeak 50	Suction surface peak Mach 50% span	DAWES	100.0
15	Aero	Mpeak 60	Suction surface peak Mach 60% span	DAWES	100.0
16	Aero	Mpeak 70	Suction surface peak Mach 70% span	DAWES	100.0
17	Aero	Mpeak 80	Suction surface peak Mach 80% span	DAWES	100.0
18	Aero	Mpeak 90	Suction surface peak Mach 90% span	DAWES	100.0
19	Aero	Mpeak 95	Suction surface peak Mach 95% span	DAWES	100.0
20	Aero	ws 50	Wennerstrom shock loss 50% span	AXCAPS	100.0
21	Aero	ws 75	Wennerstrom shock loss 75% span	AXCAPS	100.0
22	Aero	ws 100	Wennerstrom shock loss 100% span	AXCAPS	100.0
23	Aero	spds	Speedline slope out of choke	DAWES	100.0
24	Aeroelastic	flut 1	Empirical flutter parameter mode 1	ANSYS	100.0
25	Aeroelastic	flut 2	Empirical flutter parameter mode 2	ANSYS	100.0
26	Aeroelastic	twf 1	Twist/flex ratio mode 1	ANSYS	100.0
27	Mechanical	ssavg1	Avg suction surface stress 4 - 20%	ANSYS	100.0
28	Mechanical	ssavg2	Avg suction surface stress 21 - 40%	ANSYS	100.0
29	Mechanical	ssavg3	Avg suction surface stress 41 - 60%	ANSYS	100.0
30	Mechanical	ssavg4	Avg suction surface stress 61 - 80%	ANSYS	100.0
31	Mechanical	ssavg5	Avg suction surface stress 81 - 100%	ANSYS	100.0
32	Mechanical	psavg1	Avg pressure surface stress 4 - 20%	ANSYS	100.0
33	Mechanical	psavg2	Avg pressure surface stress 21 - 40%	ANSYS	100.0
34	Mechanical	psavg3	Avg pressure surface stress 41 - 60%	ANSYS	100.0
35	Mechanical	psavg4	Avg pressure surface stress 61 - 80%	ANSYS	100.0
36	Mechanical	psavg5	Avg pressure surface stress 81 - 100%	ANSYS	100.0
37	Mechanical	umax tip	Max tip deflection	ANSYS	100.0
38	Mechanical	freq 1	Frequency margin mode 1	ANSYS	100.0
39	Mechanical	freq 2	Frequency margin mode 2	ANSYS	100.0
40	Mechanical	freq 3	Frequency margin mode 3	ANSYS	100.0
41	Mechanical	fec 1	Placement of 2/rev crossing mode 1	ANSYS	100.0
42	Mechanical	fec 2	Placement of 4/rev crossing mode 2	ANSYS	100.0

The process flowchart used for DOE 1 is shown in Figure 4.2.3-4. The flowchart shows how the traditional "experiment" analysis is coupled with the ANSYS optimization technique.

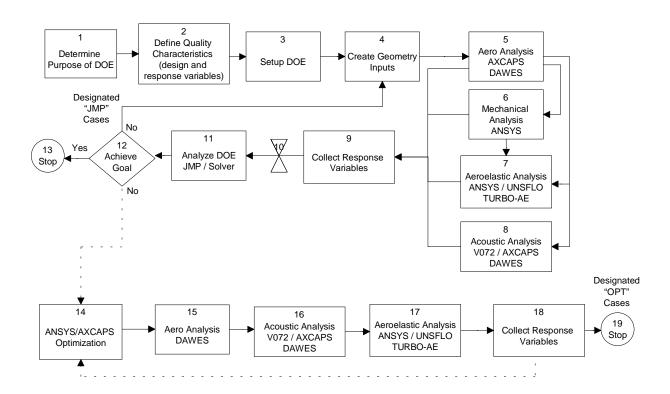


Figure 4.2.3-4. The process flow diagram for the QHSF design shows the combined use of design of experiments and ANSYS parametric optimization.

After all the response variables were collected, the data was input into the statistical program JMP. A second order polynomial was used to fit the data, and coefficients were generated for each Y-factor. Because of some deficiencies in the JMP program, the coefficients were input into EXCEL and used with the SOLVER function. SOLVER works by maximizing, minimizing, or iterating to a specified value for one Y-factor and can extrapolate outside the X-factor envelope. To obtain a feasible solution, limits were set for the other Y-factors and weighting was somewhat controlled by the range of these limits. Various feasible solutions were obtained by optimizing different Y-factors. Several new rotor geometry cases (go-forward cases) were created based on these results and analyzed. Go-forward cases created using EXCEL Solver and the statistical program JMP were designated jmp(number), while cases created from the ANSYS optimization were designated opt(letter).

4.2.3.4 Results

Acoustic response variables, overall power level at 55.9% N1c relative to the baseline rotor and blade effective sweep at the leading edge and at the suction surface impingement point of the adjacent blade, are shown in Figure 4.2.3-5. Although the figure shows only one case (case #4) to have lower noise than the baseline, the purpose of the DOE was to acquire sensitivities of the design variables and response variables. The sensitivities were the objective of the experiment such that go-forward cases could be created. The figure also shows two important aspects of effective sweep: the large effect of tangential lean, and the importance of the suction surface impingement point. Figure 4.2.3-2 showed that only one case (case #8) had a small amount of aft sweep; however Figure 4.2.3-5 shows that cases with large amounts of tangential lean (suction side down) can cause the effective sweep to be in the opposite direction thereby (potentially) mitigating benefits of the forward swept blade.

The blade effective sweep was calculated based on Art Wennerstroms' oblique shock angle (OBA) calculation at the leading edge and the shock impingement on the adjacent airfoil's suction surface. In AE's coordinate system, oblique shock angles greater than 90 degrees represent forward sweep, and are shown in the figure as a negative value (relative to 90 degrees). The desired amount of effective sweep was based on the amount required to reduce the normal component of the inlet relative Mach number at 90% N1c (maximum obtainable fan speed up to 4000 feet elevation) to a Mach number of 1.0. Effective sweep described here is the delta between the angle of the shock surface relative to the engine center line and 90 degrees. The shock surface at the leading edge is created as the shock travels from the leading edge of one blade and impinges on the adjacent blade's suction surface at some distance aft of the leading edge. Therefore, the shock loss (at a particular radius) is a function of the oblique shock angle at the leading edge and the impingement point on the suction surface. Because the shock surface impingement point on the adjacent blade is aft of the blade leading edge, the oblique shock angle at the impingement point will always be less than the oblique shock angle at the leading edge. Therefore, for aft swept and forward swept blades with the same absolute value of leading edge effective sweep, the aft swept blade will have lower shock losses.

Aerodynamic response variables, inlet corrected flow and rotor efficiency, are shown in Figure 4.2.3-6. Several empirical aeroelastic response variables for Mode 1, twist/flex ratio and empirical flutter parameter (EFP), are shown in Figure 4.2.3-7. The EFP is the ratio of the actual reduced frequency and the desired reduced frequency for a twist/flex ratio, based on AE's empirical data base. Figure 4.2.3-8 shows two, of many, mechanical response variables. The two variables shown are the maximum tip deflection and the average bending stress between 41 and 60% radial span. The tip deflection is the vector summation of the radial, axial, and tangential components and is relative to the baseline engine size. The goal of +/- 0.4 inches shown is also relative to the baseline engine size. The dominant component of the deflection vector is axial, while the radial component is much less (approximately 0.060 inches). Positive values of tip deflection indicate bending toward the suction side. The bending stress goal of less than +/- 40 ksi on the suction and pressure surfaces was used to keep the maximum principal stresses on the blade below 85 ksi.

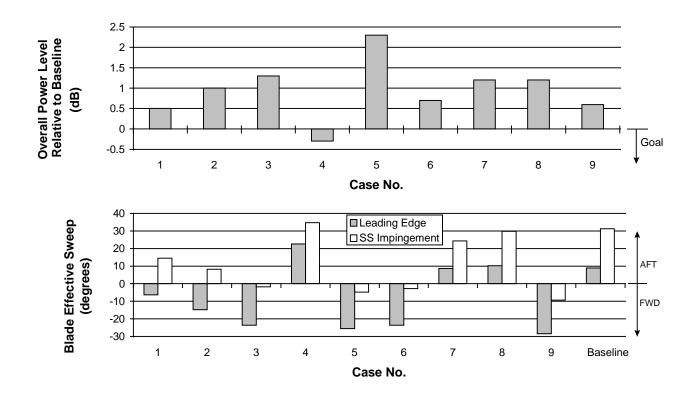


Figure 4.2.3-5. Result of the Acoustic Assessment of the Configurations for Rotor DOE 1.

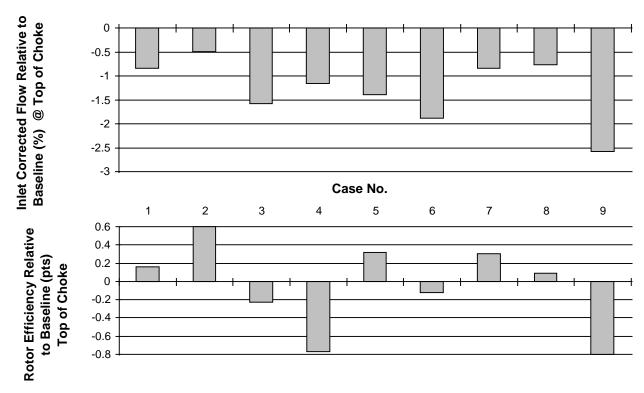


Figure 4.2.3-6. Result of the Aerodynamic Assessment of the Configurations for Rotor DOE 1.

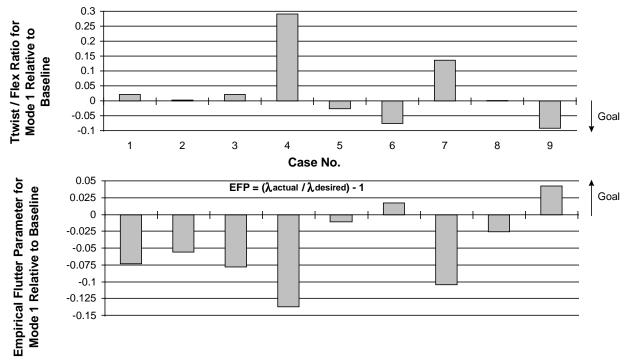


Figure 4.2.3-7. Result of the Aeroelastic Assessment of the Configurations for Rotor DOE 1.

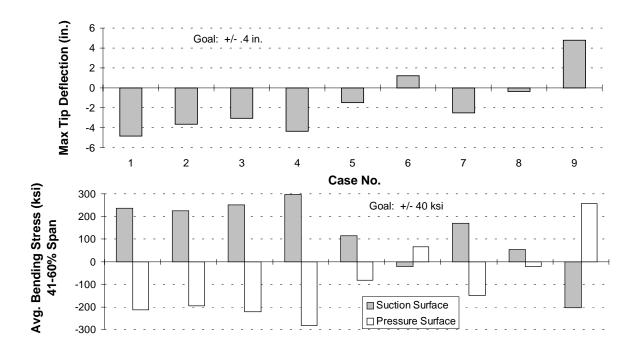


Figure 4.2.3-8. Result of the Mechanical Assessment of the Configurations for Rotor DOE 1.

 Table 4.2.3-3. Tabulated Results of DOE 1 Forward Sweep Analysis

	Config #	<u>1</u>	<u>2</u>	<u>3</u>	<u>4</u>	<u>5</u>	<u>6</u>	<u>7</u>	<u>8</u>	9
1	dB	132.4	132.8	133.0	131.4	133.8	132.1	133.1	132.8	132.1
2	eff swp le	-6.27	-14.70	-23.53	22.61	-25.50	-23.57	8.73	10.28	-28.40
3	%c 50	28.80	31.26	33.66	35.85	31.00	26.49	35.79	30.91	21.83
4	%c 60	33.61	36.29	39.09	43.65	36.19	33.56	46.53	38.36	28.30
5	%c 70	41.13	47.06	47.42	48.86	44.33	43.87	52.07	46.42	38.11
6	%c 80	46.65	55.76	59.04	51.82	53.06	52.46	60.46	52.40	46.26
7	%c 90	55.96	66.79	69.56	55.31	66.93	68.87	68.49	55.96	65.99
8	%c 95	58.55	73.37	75.69	55.48	73.39	73.26	70.63	58.83	70.62
9	delz 50	3.827	3.601	3.024	3.526	3.004	3.757	2.944	3.767	3.516
10	delz 100	6.862	6.784	6.737	6.228	5.950	6.036	5.298	5.465	5.235
11	Wc	191.01	191.69	189.59	190.39	189.94	188.99	191.02	191.16	187.67
12	PR	1.899	1.901	1.894	1.811	1.898	1.890	1.852	1.854	1.891
13	Eff	.9038	.9082	.8999	.8945	.9054	.9010	.9052	.9031	.8942
14	Mpeak 50	1.429	1.464	1.470	1.470	1.453	1.404	1.480	1.422	1.423
15	Mpeak 60	1.452	1.480	1.491	1.496	1.483	1.432	1.504	1.441	1.440
16	Mpeak 70	1.454	1.464	1.475	1.503	1.468	1.432	1.545	1.444	1.448
17	Mpeak 80	1.445	1.460	1.475	1.514	1.459	1.426	1.550	1.454	1.429
18	Mpeak 90	1.467	1.527	1.535	1.510	1.554	1.545	1.574	1.517	1.503
19	Mpeak 95	1.493	1.534	1.532	1.465	1.586	1.631	1.571	1.488	1.613
20	ws 50	.023	.028	.034	.016	.029	.024	.019	.019	.026
21	ws 75	.055	.046	.031	.045	.043	.040	.053	.046	.039
22	ws 100	.081	.076	.055	.040	.048	.045	.077	.059	.029
23	spds	-9.5e-4	2.79e-3	3.90e-3	-2.84e-3	3.01e-3	9.34e-3	-1.09e-3	7.35e-4	-9.5e-4
24	flut 1	-0.563	-0.546	-0.568	-0.627	-0.501	-0.473	-0.594	-0.516	-0.448
25	flut 2	-0.105	-0.008	-0.129	-0.181	-0.005	-0.005	-0.008	0.319	-0.200
26	twf 1	0.581	0.562	0.581	0.851	0.533	0.484	0.696	0.561	0.468
27	ssavg1	152.5	71.9	-14.5	92.2	21.4	-1.3	48.1	21.2	-50.3
28	ssavg2	221.0	164.1	110.5	232.5	49.6	-25.5	120.6	54.8	-152.9
29	ssavg3	235.1	223.6	250.7	296.9	115.8	-21.4	169.4	54.0	-205.0
30	ssavg4	173.5	184.0	241.9	200.6	113.6	-12.1	149.1	28.4	-138.9
31	ssavg5	39.5	45.1	63.4	48.3	31.8	-3.3	38.0	4.0	-35.7
32	psavg1	-110.6	-14.8	92.9	-45.5	30.6	69.8	3.4	44.1	108.7
33	psavg2	-191.9	-126.8	-66.3	-207.8	-6.5	82.0	-87.6	-9.8	213.9
34	psavg3	-211.6	-195.9	-221.4	-283.4	-81.2	65.3	-150.1	-22.5	256.3
35	psavg4	-152.1	-160.0	-215.2	-192.9	-88.6	38.2	-133.8	-6.9	165.3
36	psavg5	-32.4	-38.1	-57.0	-45.2	-24.5	11.8	-31.7	3.6	44.1
37	umax tip	-4.83	-3.64	-3.05	-4.35	-1.48	1.22	-2.50	-0.35	4.78
38	freq 1	1.478	1.491	1.461	1.478	1.559	1.504	1.606	1.586	1.527
39	freq 2	3.541	3.630	3.448	3.242	3.775	3.751	3.640	3.827	3.166
40	freq 3	0.692	0.533	0.802	0.785	0.423	0.589	0.498	0.304	0.958
41	fec 1	0.420	0.457	0.436	0.423	0.556	0.518	0.579	0.610	0.407
42	fec 2	0.724	0.784	0.673	0.522	0.899	0.896	0.828	0.930	0.560

Two cases emerged from DOE 1 as the go-forward designs: jmp08 and optc. A comparison of some mechanical and geometry parameters is shown in Table 4.2.3-4. Also shown in the table is another optimization design, opta, because it revealed an important aspect of integrating all the design disciplines. Although it had some highly desirable features in terms of stress and sweep, rotor-stator interaction noise results from the NASA - Acoustic code V072 showed the case to be unacceptable. (A comparison of the opta, optc and jmp08 results is shown with the DOE 2 results, Section 4.2.4.4). Results from DOE 1 go-forward cases; jmp08, opta, and optc showed that at the conclusion of DOE 1, the aerodynamic performance was below the design goal, and would have to be increased in the other DOE's planned. Although the results of these cases looked promising, additional blade effective sweep was required.

Table 4.2.3-4. DOE 1 Go-Forward Case Comparison

	Desired	OPTA	OPTC	JMP08
Optimization		Acoustic Performance	Aero Performance	DOE #1
Stresses	< 85 Ksi	93 Ksi	73 Ksi	120 Ksi
Defln.	< 0.4 in.	0.35 in.	0.47 in.	0.55 in.
YCG* Tip		1.475 in.	1.774 in.	1.343 in.
YCG* Mid		-0.443 in.	-0.152 in.	-0.446 in.
XCG** Tip	Fwd (-)	-1.5 in.	-1.871 in.	-1.871 in.
XCG** Mid	Fwd (-)	+0.281 in.	-0.333 in.	-0.279 in.
OBA075	> -34°	-39°	-25°	-22°
OBA100	> -41°	-50°	-49°	-41°

^{*} Positive Toward Pressure Side

Late in the design process (after completion of the rotor design and half of the stator design effort), it was discovered that the V072 input parameter ACLS, the relative flow angle at the stator leading edge, was being input incorrectly. Time allowed for recalculation of the baseline engine calibration, the original rotor DOE 1, and all pertinent stator DOE cases. Although incorrect predictions were used to direct the rotor design process, the general conclusions derived from these analyses were supported by the corrected predictions.

^{**} Positive Toward Aft

V072 predictions were generated for the rotor DOE 1 cases 1-9 and the optimized cases jmp08 and optc. All the rotor DOE cases used the baseline stator design. The inlet and aft duct power levels for 2 and 3 times the bladepass frequency are shown in Figure 4.2.3-9 as deltas from the baseline predicted power levels.

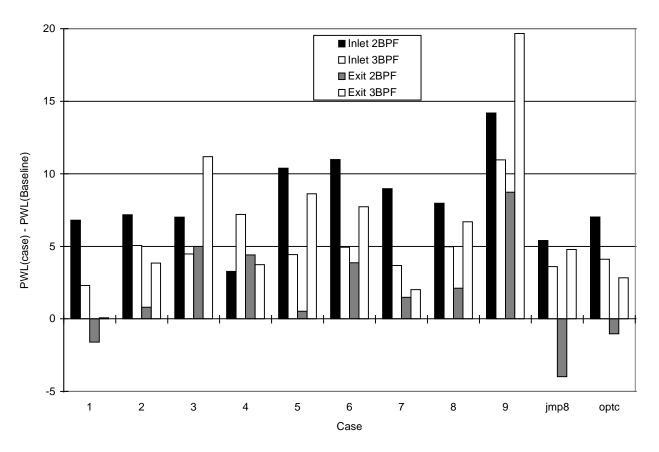


Figure 4.2.3-9. V072 Predictions for Cases in Rotor DOE 1, All Use Baseline Stator (1*BPF cutoff)

4.2.4 Design of Experiment 2

4.2.4.1 Purpose

The second DOE was created to minimize flutter. Although the goal was to use aeroelastic programs (FREPS, UNSFLO, and TURBO-AE) to calculate the aerodynamic damping at various points on the performance map, unfortunately the codes either had problems running or were considered not well enough calibrated to use in the DOE. Empirical data was used as the aeroelastic quality factors and direct the blade design.

4.2.4.2 Description

The second DOE consisted of 7 factors (position of max thickness at hub, mid, and tip; Tmax/C at hub, mid, and tip; and Ycg-tip) with 2 levels each. The 8 cases are described in Table 4.2.4-1. Although DOE 1 had shown that Ycg had little impact on aeroelastic empirical data, it was included to alleviate potential weight increase produced by thickness increases required to reduce stress and not flutter. Values of Ycg tip were selected based on DOE 1 results. Because the DOE 1 go-forward case optc was designed slightly after jmp08, and the need to initiate DOE 2; jmp08 was used as the QHSF baseline in DOE 2 even though the peak stresses and maximum tip deflection were above desired levels. Ranges for the other values in the table were selected based on previous analyses using the ANSYS optimization process completed to determine whether the blade design should be a 1/rev or a 2/rev, (Section 4.2.1). Spanwise thickness and position of max thickness for these analyses, compared to the baseline, are shown in Figures 4.2.4-1 and 4.2.4-2.

Table 4.2.4-1. DOE 2 Aeroelastic Analysis X-Factors

Case No. Hub	Tmax Loc Mid	Tmax Loc Tip	Tmax Loc Hub	Tmax/c Mid	Tmax/c Tip	Tmax/c Tip	Y cg*
1	0.25	0.30	0.50	0.04	0.065	0.030	1.80
2	0.25	0.30	0.70	0.08	0.030	0.015	1.80
3	0.25	0.65	0.50	0.08	0.030	0.030	1.00
4	0.25	0.65	0.70	0.04	0.065	0.015	1.00
5	0.60	0.30	0.50	0.08	0.065	0.015	1.00
6	0.60	0.30	0.70	0.04	0.030	0.030	1.00
7	0.60	0.65	0.50	0.04	0.030	0.015	1.80
8	0.60	0.65	0.70	0.08	0.065	0.030	1.80

^{*}Y-cg are in inches and relative to Baseline engine size (30.7 in Dia.) positive in direction of rotation

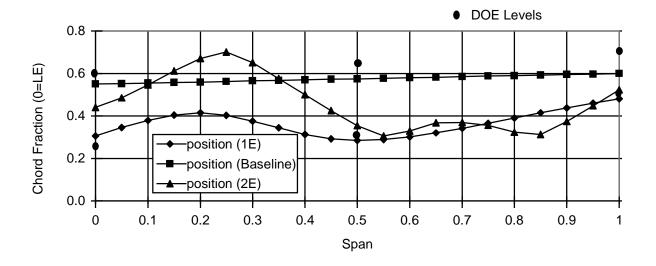


Figure 4.2.4-1. Levels Selected for Position of Maximum Thickness

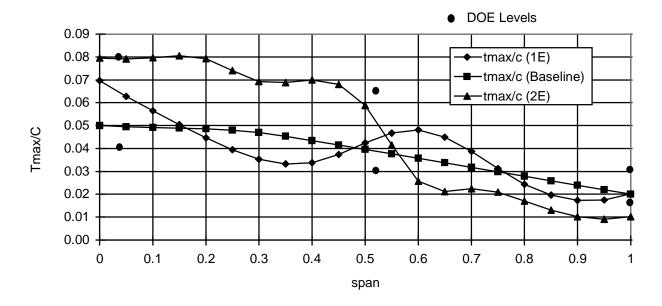


Figure 4.2.4-2. Levels Selected for Maximum Thickness

4.2.4.3 Analysis

The DOE 2 cases were analyzed using the same techniques and same response variables (Y-factors) as DOE 1, shown in Table 4.2.3-2.

4.2.4.4 Results

For consistency, the same response variables are shown as presented in Section 4.2.4.4. Included also in the figures are the go-forward cases from DOE 1; jmp08, opta, and optc. Again, it is the sensitivities of the design and response variables which were the objective of the DOE. Results from DOE 2, as well as jmp08, opta, and optc are shown in Figures 4.2.4-3 through 4.2.4-6. Acoustic response variables for interaction noise and MPT's are shown in Figure 4.2.4-3. Rotor-stator interaction noise was again calculated using V072 at 55.9% N1c, and was compared relative to the baseline such that a negative value was desired. The figure shows why opta, which looked good based on the other discipline design criteria, was not used as a go-forward design. The reason why it had such high interaction noise levels was that in an effort to create more effective sweep the blade was allowed to sweep aft at midspan prior to sweeping forward. The reduction in rotor-stator axial spacing caused the increased noise. The figure shows the large amount of effective sweep being considered relative to the aft-swept baseline rotor.

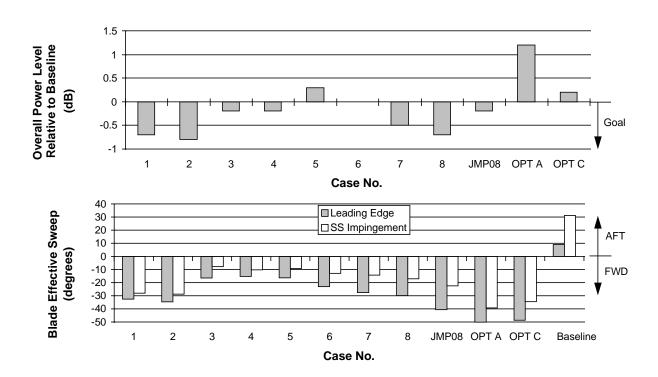


Figure 4.2.4-3. Result of the Acoustic Assessment of the Configurations for Rotor DOE 2.

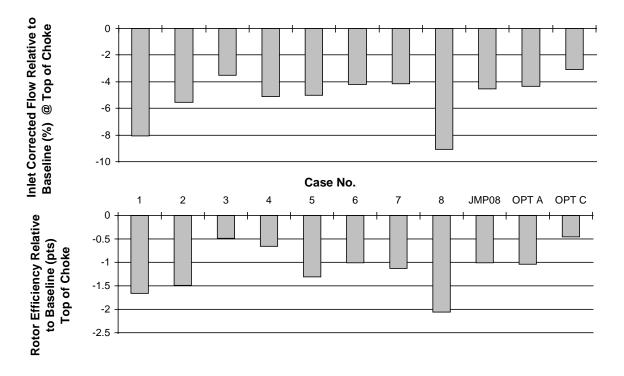


Figure 4.2.4-4. Results of the Aerodynamic Assessment of the Configurations for Rotor DOE 2.

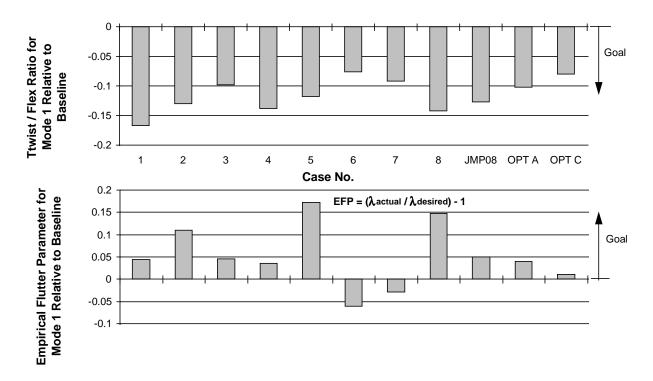


Figure 4.2.4-5. Results of the Aeroelastic Assessment of the Configurations for Rotor DOE 2.

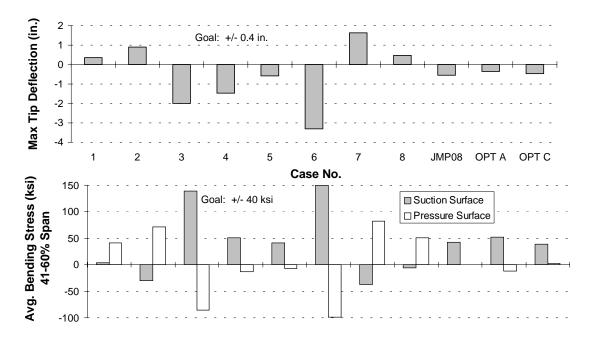


Figure 4.2.4-6. Results of the Mechanical Assessment of the Configurations for Rotor DOE 2.

A new methodology was required to complete rotor DOE 2. Because the blade geometry was created by stacking airfoil section centers of gravity (cg) on a radial line from the engine center line through the hub section cg, any change to the hub section maximum thickness (tmax), or position of tmax would cause the hub cg to shift and the airfoil geometry to change - in particular, the leading edge (LE) metal angle. Because the LE metal angle was changing between the cases, the aerodynamic quality characteristics (flow, efficiency, etc.) could not be directly related to the x-factors. A long and unsuccessful attempt was made to stack the optc airfoil from DOE 1 on the LE which would, if successful, allow direct comparisons between the x-factors and y-factors of DOE 2. Although this technique was successful for the stator, the large hub slope of the rotor proved too much for AXCAPS and back-to-back geometry generation cases could not be duplicated.

In order to solve this problem, a technique was developed which generated the blade geometry in an ANSYS script using the airfoil generation routines from AXCAPS. This procedure forced the airfoil section mean camber lines to be the same as optc. Ranges were determined for the x-factors, and the internal ANSYS optimization procedure was used to minimize flutter. This was essentially the same optimization procedure described in the process flowchart (Figure 4.2.3-4) and used in DOE 1 and the first part of DOE 2, except AXCAPS was not used to generate the airfoil. Go-forward cases, based on the mechanical and empirical aeroelastic criteria, were then analyzed using DAWES to obtain the aerodynamic quality characteristics. The final go-forward case from DOE 2 was designated optm. Results of optm relative to the two go-forward cases from DOE 1 (jmp08 and optc) are shown in Figure 4.2.4-7. The figures show that relative to the baseline, substantial improvements in empirical flutter parameters were achieved while reducing weight; without any additional flow reduction.

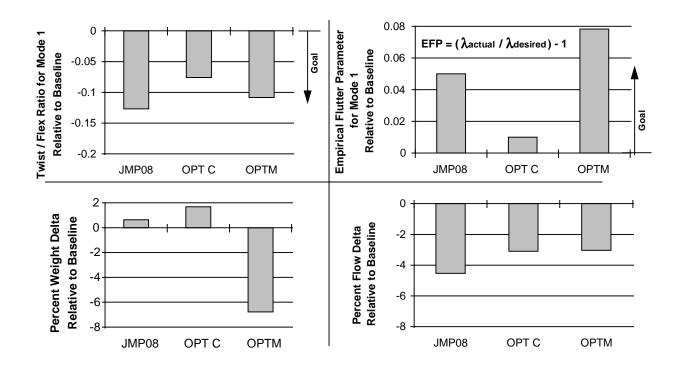


Figure 4.2.4-7. Rotor summary from DOE 2

4.2.5 Design of Experiment 3

4.2.5.1 Purpose

The purpose of DOE 3 was to increase flow and achieve desirable efficiency characteristics by optimizing the incidence.

4.2.5.2 Description

The base level used for DOE 3 was optm. In order to optimize the incidence across the radial span, five spans were chosen (0%, 25%, 50%, 75%, and 95%) with three levels at each span as shown in Table 4.2.5-1. The tip (100% span) was not included because of an inlet boundary layer which was being modeled in the analysis; however, the tip was adjusted to remain consistent with the 95% span level. The magnitude in incidence was also reduced from hub to 95% span to satisfy an additional flutter criteria involving excessive positive tip incidence. The minimum fractional factorial orthogonal matrix for 5 factors with 3 levels produced 27 different geometry's; and, because four or more runs were required for each geometry to determine the

speedline shapes, it resulted in over 108 runs. Because of the time required to run such a large number of cases, DENTON (3-D inviscid analysis) was again reviewed as a possible analysis tool.

Table 4.2.5-1. DOE 3 Design Point Rotor Performance Analysis

Incidence Values in Degrees (Values represent differences from base case - OPTM)

Case #	<u>0% Span</u>	<u>25%</u>	<u>50%</u>	<u>75%</u>	<u>95%</u>
1	-4.0	-3.5	-3.0	-2.5	-2.0
2	-4.0	-3.5	0.0	2.5	2.0
3	-4.0	-3.5	3.0	0.0	0.0
4	-4.0	0.0	-3.0	2.5	2.0
5	-4.0	0.0	0.0	0.0	0.0
6	-4.0	0.0	3.0	-2.5	-2.0
7	-4.0	3.5	-3.0	0.0	0.0
8	-4.0	3.5	0.0	-2.5	-2.0
9	-4.0	3.5	3.0	2.5	2.0
10	0.0	-3.5	-3.0	2.5	0.0
11	0.0	-3.5	0.0	0.0	-2.0
12	0.0	-3.5	3.0	-2.5	2.0
13	0.0	0.0	-3.0	0.0	-2.0
14	0.0	0.0	0.0	-2.5	2.0
15	0.0	0.0	3.0	2.5	0.0
16	0.0	3.5	-3.0	-2.5	2.0
17	0.0	3.5	0.0	2.5	0.0
18	0.0	3.5	3.0	0.0	-2.0
19	4.0	-3.5	-3.0	0.0	2.0
20	4.0	-3.5	0.0	-2.5	0.0
21	4.0	-3.5	3.0	2.5	-2.0
22	4.0	0.0	-3.0	-2.5	0.0
23	4.0	0.0	0.0	2.5	-2.0
24	4.0	0.0	3.0	0.0	2.0
25	4.0	3.5	-3.0	2.5	-2.0
26	4.0	3.5	0.0	0.0	2.0
27	4.0	3.5	3.0	-2.5	0.0

4.2.5.3 Analysis

DENTON was run with the optc geometry and speedlines were compared with those previously completed using DAWES. Although the total pressure and efficiency levels were considerably different (as expected), choke flow and speedline shapes of efficiency and pressure compared reasonably well. Therefore DENTON was used to complete DOE 3. After completing the analysis, the JMP program and EXCEL Solver were used to predict go-forward cases which would achieve the desirable results. The y-factors were choke flow, a flow delta between deep choke and design-point, and a qualitative assessment of the efficiency speed line shape.

4.2.5.4 Results

Statistical correlations of the data were not very good, but several go-forward cases were completed. The final go-forward case from DOE 3 was designated optm3 and is shown compared to three representative cases from the DOE: Cases 1, 9, and 19 in Figure 4.2.5-1. The figure shows that, in general, cases with large amounts of choke flow (Case 9) had poor efficiency shapes that reduced in level from choke to stall, while those with lower amounts of choke flow (Case 1) had efficiency shapes that increased in level from choke to stall. The go-forward case optm3 was selected to provide maximum flow while still exhibiting good efficiency speedline shape characteristics.

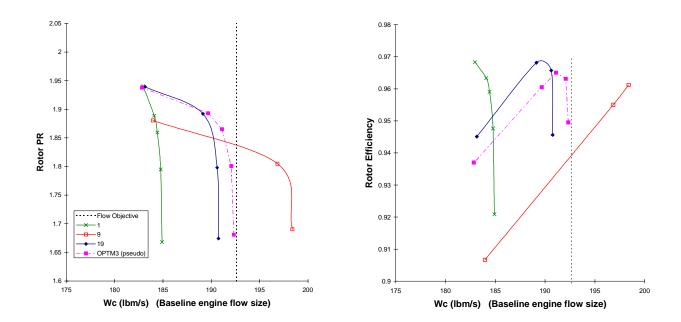


Figure 4.2.5-1. Representative Results from DOE 3 Compared to the Go-Forward Case OPTM3.

4.2.6 Design of Experiment 4

4.2.6.1 Purpose

The purpose of DOE 4 was to increase design point flow and efficiency by optimizing the blade metal angle (beta) distributions along the mean camber.

4.2.6.2 Description

A fractional factorial L-16 was used incorporating fifteen x-factors with two levels each. (Although 3 levels for each x-factor would have been preferred, completion time associated with using DAWES prevented it.) Go-forward case optm3 (from DOE 3) was used as the base level. DAWES analyses performed on optm3 showed that efficiency levels could be increased across the span and therefore dictated the complete spanwise range for DOE 4 design factors. A description of the x and y-factors selected are included in Table 4.2.6-1 along with the matrix description in Table 4.2.6-2. The x-factors are shown graphically in Figure 4.2.6-1 as ranges from the base configuration (optm3). For the blade metal angles on streamlines between those being modified, the betas were linearly interpolated for a smooth transition.

Table 4.2.6-1. DOE 4 Design Point Rotor Performance Analysis Quality Characteristics

(Values represent differences from base case - OPTM3)

X-factors		<u>Y-factors</u>
1 Beta SL1 LE 2 Beta SL1 25% m 3 Beta SL1 50% m 4 Beta SL12 LE 5 Beta SL12 25% m 6 Beta SL12 50% m 7 Beta SL12 75% m 8 Beta SL24 LE 9 Slope of Beta to x-f 14 loc. 10 Beta SL24 50% m 11 Beta SL24 75% m 12 Beta location of x-factor 5 13 Beta location of x-factor 9 15 Beta location of x-factor 10	+/- 3.0 deg. +/- 3.0 deg. +/- 3.0 deg. +/- 2.5 deg. +/- 2.5 deg. +/- 2.5 deg. +/- 2.5 deg. +/- 2.0 deg. 0, -20.5 deg. +/- 2.0 deg. +/- 2.0 deg. 15%, 25% 40%, 60% 10%, 25% 40%, 60%	Design point flow Peak efficiency Design point efficiency radial profile (10, 30, 50, 70, 90% radial spans) Aerodynamic loadings (10, 30, 50, 70, 90% radial spans) Suction surface passage shock location (50, 70, 90% radial spans) Surge margin assessment Part-speed efficiency

Table 4.2.6-2. DOE 4 Design Point Rotor Performance Analysis X-Factor Levels

(Values represent differences from base case - OPTM3)

Case #	X1	X2	Х3	X4	X5	X6	X7	X8	X9	X10	X11	X12	X13	X14	X15
1	-3.0	-3.0	-3.0	-2.5	2.5	-2.5	-2.5	2.0	-20.5064	2.0	2.0	4.0	13.0	6.0	13.0
2	-3.0	-3.0	-3.0	2.5	-2.5	2.5	2.5	-2.0	0.0	-2.0	-2.0	4.0	13.0	6.0	13.0
3	-3.0	-3.0	3.0	-2.5	-2.5	2.5	2.5	-2.0	-20.5064	2.0	2.0	6.0	9.0	3.0	13.0
4	-3.0	-3.0	3.0	2.5	2.5	-2.5	-2.5	2.0	0.0	-2.0	-2.0	6.0	9.0	3.0	13.0
5	-3.0	3.0	-3.0	-2.5	-2.5	2.5	-2.5	2.0	0.0	-2.0	2.0	6.0	9.0	6.0	9.0
6	-3.0	3.0	-3.0	2.5	2.5	-2.5	2.5	-2.0	-20.5064	2.0	-2.0	6.0	9.0	6.0	9.0
7	-3.0	3.0	3.0	-2.5	2.5	-2.5	2.5	-2.0	0.0	-2.0	2.0	4.0	13.0	3.0	9.0
8	-3.0	3.0	3.0	2.5	-2.5	2.5	-2.5	2.0	-20.5064	2.0	-2.0	4.0	13.0	3.0	9.0
9	3.0	-3.0	-3.0	-2.5	-2.5	-2.5	2.5	2.0	0.0	2.0	-2.0	6.0	13.0	3.0	9.0
10	3.0	-3.0	-3.0	2.5	2.5	2.5	-2.5	-2.0	-20.5064	-2.0	2.0	6.0	13.0	3.0	9.0
11	3.0	-3.0	3.0	-2.5	2.5	2.5	-2.5	-2.0	0.0	2.0	-2.0	4.0	9.0	6.0	9.0
12	3.0	-3.0	3.0	2.5	-2.5	-2.5	2.5	2.0	-20.5064	-2.0	2.0	4.0	9.0	6.0	9.0
13	3.0	3.0	-3.0	-2.5	2.5	2.5	2.5	2.0	-20.5064	-2.0	-2.0	4.0	9.0	3.0	13.0
14	3.0	3.0	-3.0	2.5	-2.5	-2.5	-2.5	-2.0	0.0	2.0	2.0	4.0	9.0	3.0	13.0
15	3.0	3.0	3.0	-2.5	-2.5	-2.5	-2.5	-2.0	-20.5064	-2.0	-2.0	6.0	13.0	6.0	13.0
16	3.0	3.0	3.0	2.5	2.5	2.5	2.5	2.0	0.0	2.0	2.0	6.0	13.0	6.0	13.0

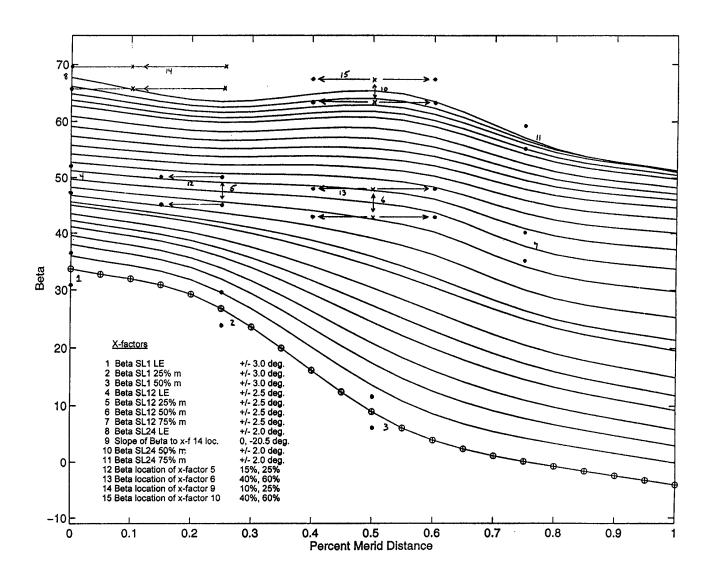


Figure 4.2.6-1. Graphical Description of the X-Factors Used in Rotor DOE 4.

4.2.6.3 Analysis

DAWES results were analyzed and compared for the 64 runs (16 cases with 4 different backpressures) at the design speed. Y-factors of flow, efficiency, pressure ratio, and work were calculated and plotted. Spanwise efficiency profiles were compared and tabulated at 10, 30, 50, 70, and 90 percent span. Aerodynamic blade loadings were also compared at the same spans and were qualitatively assigned values for the DOE. The loadings were judged on suction surface

peak Mach number, suction surface diffusion rate, leading edge assessment (choke/stall), and pressure surface Mach number distribution. Passage shock locations (on the suction surface) were calculated and tabulated at 50, 70, and 90 percent span. Once the y-factors had been tabulated, JMP and EXCEL were used to create go-forward cases.

One potential problem in using DOE's with two levels is that it assumes that the solution is linear, which is typically not the case. This results in extrapolated values with excessive ranges which would not be feasible. To address this, constraints were imposed on the amount that the design factors could permeate. This forced the optimization procedure to find a feasible solution (all design criteria met) within the specified range.

4.2.6.4 Results

The tabulated results are shown in Table 4.2.6-3. A representative sample of the cases (low flow, high flow, and max efficiency) are shown at the design point relative to the baseline rotor DAWES analysis in Figure 4.2.6-2.

Table 4.2.6-3. DOE 4 Tabulated Results

Case	Wc 17.0	Eff pk	Eff 10%	Eff 30%	Eff 50%	Eff 70%	Eff 90%	L 10%	L30%	L 50%	L 70%	L 90%	S L 50%	S L 70%	S L 90%	SM
1	183.78	0.8914	0.9797	0.9415	0.8992	0.8784	0.8136	13	9	2	8	2	0.247	0.343	0.649	-0.0215
2	188.24	0.8931	0.9767	0.9279	0.8829	0.8831	0.8451	12	14	11	10	13	0.246	0.170	0.701	-0.0094
3	185.40	0.8772	0.9762	0.9055	0.8425	0.8595	0.8783	16	16	16	16	8	0.166	0.189	0.595	0.0000
4	180.95	0.8832	0.9819	0.9474	0.8956	0.8556	0.6982	14	6	6	13	10	0.291	0.418	0.596	-0.0074
5	183.33	0.8845	0.9790	0.9225	0.8844	0.8734	0.8294	15	15	13	6	12	0.185	0.343	0.650	-0.0215
6	188.68	0.9040	0.9802	0.9493	0.8948	0.8916	0.8405	11	3	14	2	16	0.339	0.210	0.700	-0.0128
7	190.02	0.8994	0.9803	0.9458	0.8983	0.8931	0.8603	10	5	3	1	5	0.269	0.296	0.699	0.0000
8	186.04	0.8965	0.9795	0.9306	0.8830	0.8855	0.8616	9	11	12	5	7	0.225	0.189	0.675	-0.0322
9	184.90	0.8859	0.9775	0.9300	0.8840	0.8775	0.8416	4	12	4	4	3	0.205	0.295	0.649	-0.0196
10	188.76	0.9072	0.9790	0.9462	0.8925	0.8836	0.8589	1	7	15	12	9	0.316	0.210	0.700	-0.0120
11	186.55	0.8984	0.9801	0.9456	0.8944	0.8689	0.8431	5	2	10	15	15	0.248	0.134	0.700	-0.0149
12	187.91	0.8909	0.9773	0.9355	0.8830	0.8771	0.8479	3	13	7	14	1	0.291	0.319	0.699	-0.0139
13	183.81	0.8995	0.9786	0.9518	0.9034	0.8795	0.8649	8	1	1	11	4	0.248	0.418	0.675	-0.0314
14	189.80	0.8982	0.9803	0.9419	0.8897	0.8926	0.8353	7	8	8	3	14	0.292	0.189	0.722	-0.0098
15	194.05	0.8970	0.9791	0.9380	0.8848	0.8925	0.8766	6	10	5	7	6	0.247	0.250	0.674	0.0000
16	177.01	0.8938	0.9797	0.9458	0.8957	0.8612	0.6605	2	4	9	9	11	0.269	0.446	0.700	-0.0087

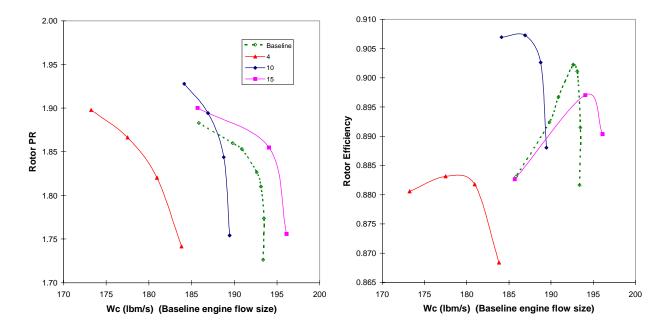


Figure 4.2.6-2. Representative Results from Rotor DOE 4 Compared to Baseline Rotor

As Table 4.2.6-3 shows, the calculated efficiency and the qualitative value given to the aerodynamic loading, for the same spanwise location, varied considerably in some instances. Therefore, go-forward cases were created based on the other y-factors combined with either the efficiency at radial spans or the loadings. DAWES results for the go-forward cases at the design point are shown in Figure 4.2.6-3 compared to the starting case from DOE 3 (optm3) and the performance objective (baseline rotor). The initial two cases (gf01, and gf02) were completed using EXCEL and were predicted to match the design flow objective; however, DAWES results showed insufficient flow. Y-factors from these cases were added to the model and two new go-forward cases were created, gf03 and gf04. The four cases were also analyzed at 85% N1c to verify good part-speed performance with results shown in Figure 4.2.6-4. The results indicated a reduction in part-speed efficiency with increased choke flow at 100% N1c. In order to best match the baseline rotor performance goals, and provide the best back-to-back test for acoustic comparisons, go-forward case 3 (gf03) was selected as the final design (details reported in Section 5.0).

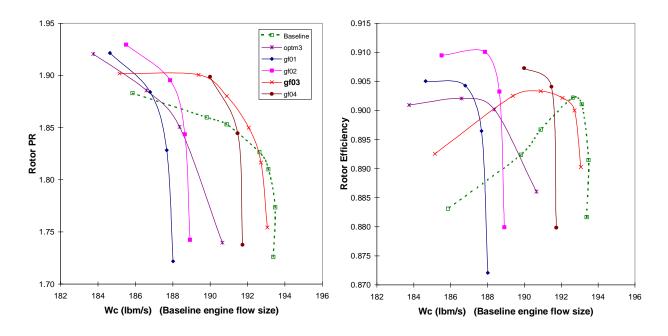


Figure 4.2.6-3. Effect of Beta Distribution on Flow and Efficiency on DOE 4 Go-Forward Cases at 100% N1c.

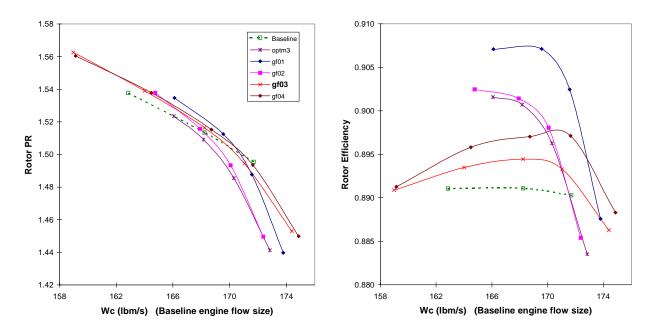


Figure 4.2.6-4. Effect of Beta Distribution on Flow and Efficiency on DOE 4 Go-Forward Cases at 85% N1c.

4.2.7 Bird Ingestion Analysis

A preliminary Foreign-Object-Damage (FOD) assessment performed on the QHSF blade indicated leading-edge thickness parameters to be well within successful AE experience. This analysis is comparative in nature and considers only spanwise geometric characteristics, LE thickness distribution, blade count, metal angles, bird weight, as well as other parameters. The computed damage tolerance factors are compared with a database consisting of very successful, marginal, and poor designs. This preliminary analysis however, can not account for the high degree of forward sweep and tangential lean built into the QHSF, which will be accounted for using NOSAPM (Section 5.1.4.6).

4.3 Stator

4.3.1 Design Ground Rules and Goals

The stator design emphasizes the reduction of rotor wake interaction noise while meeting aerodynamic, mechanical, and producibility goals. The ground rules required:

- 1.) Use the baseline vane count (52 vanes). The acoustic effects of varying vane count would confuse the analysis of the effects of vane design.
- 2.) Don't sweep the stator aft or move it downstream relative to the baseline stator. The fan will be able to replace the baseline fan without a change in engine dimensions.
- 3.) Use the baseline front frame unchanged.
- 4.) Use the baseline stator solidity. The inflow from the AE/QHSF was similar to the baseline fan so that the baseline solidity would provide good loadings.
- 5.) Control vane stacking by specifying the shape of the leading edge. Since this is what affects wake trace speed, it seemed simpler to specify the shape of the leading edge.
- 6.) Design for composite construction. This requirement puts some restrictions on leading and trailing edge radius and maximum thickness.

The stator was to reduce rotor wake interaction noise to the greatest extent possible while still having acceptable aerodynamic loadings, losses similar to the baseline stator, and acceptable stress, frequency margin, flutter margin, and composite producibility.

4.3.2 Strategy

The vane stacking would be set first since this was important acoustically and since it was likely to be radical enough that it would be good to find out up front about any aerodynamic or mechanical problems it caused. DOE 1 would be used to experiment with different vane stacks. As originally planned, this DOE varied vane stacking tangentially and axially. Axial stacking changes were outside the ground rules, but knowledge of these trends was desirable anyway. Acoustic, aerodynamic, and mechanical quality factors would be collected and their trends with vane stacking would be used to design the final vane stack. The vane incidence, vane section mean camber line angle chordwise distributions, deviation, chord, and thicknesses for this DOE would come from the baseline vane in the beliefs that it was a good starting point. Aerodynamic quality characteristics would be calculated by the DAWES 3D viscous CFD code.

Once the stacking had been decided, the second vane DOE would be used to optimize vane incidence and mean camber line angle chordwise distributions for good aerodynamic performance with this stacking.

The vane trailing edge angle would then be set to produce axial (zero swirl) stator exit flow. This would not require a DOE. The vane trailing angle would be set to remove any residual DAWES-predicted exit swirl. This angle usually produces exit swirl acceptably close to axial in just a few iterations. Loadings and loss are then re-checked to make sure they're still acceptable in spite of any increased flow turning required to produce axial flow.

The vane leading and trailing edge radius and maximum thicknesses would then be adjusted to meet composite vane producibility criteria. Composite vanes require control of the vane thickness as measured normal to the vane suction and pressure surfaces, as opposed to the design tools which specify thickness in the tangential direction. The tangential thickness varies from the normal thickness by the cosine of the lean angle. Since the lean angle everywhere but the leading edge is affected by both the vane stacking and the vane section stagger angles, the thickness modification is performed after the stagger angles have been determined. After thickness has been adjusted, the loadings, loss, and exit swirl are re-checked.

The composite construction provides flexibility to produce a vane with combinations of high axial and tangential leans with minimum weight penalty. The current production vanes on the baseline stator have symmetric lay-up or taped construction. Recently, a braided vane has been developed at AE as a part of the ENCORE (Engine Cost Reduction) program. In addition, the braided vane has better FOD characteristics than the taped vane. However, it has lower flutter margins due to decreased torsional rigidity. The QHSF exit vane will be a braided composite vane. A preliminary layout of the vane is shown in Figure 4.3.2-1.

The AE in-house design tool CADS (Composite Airfoil Design System) will be used for the composite vane design. As shown in the flowchart in Figure 4.3.2-2, CADS uses the airfoil geometry from the aero bank file to generate an ANSYS input (prep7) file. This system was used to design the baseline stator.

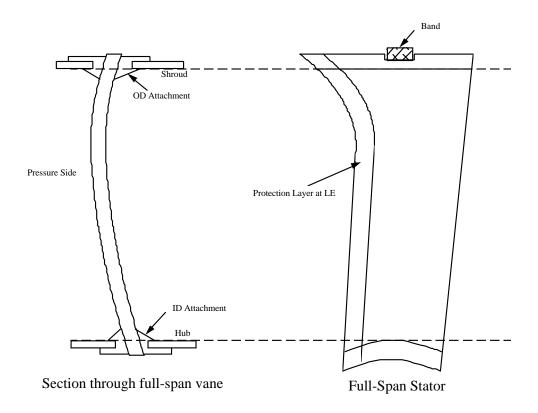


Figure 4.3.2-1. Preliminary Layout of the Composite Vane

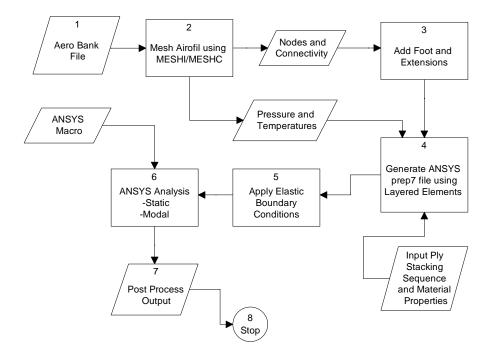


Figure 4.3.2-2. The AE Composite Stator Design System will Design a Composite Vane that Will Incorporate the Optimum Aerodynamic and Acoustic Features.

4.3.3 Stacking DOE

The vanes used in the stacking DOE were designed to operate behind the "optc" fan rotor. Initially, an L18 Taguchi matrix was chosen with 2 levels for axial sweep at the vane midspan and tip and 3 levels for the tangential stacking at the midspan and tip. The 2 sweep levels represented the baseline vane axial sweep and a 14 degree increase. This DOE couldn't be completed because the great variation in rotor-stator axial spacing between the hub and the shroud created convergence problems in the axisymmetric flow solver (AXCAPS). The matrix was modified so the sweep now varied between the baseline sweep and a radial leading edge, which would still provide information about the effects of sweep. Work on the modified DOE was abandoned due to schedule constraints and since decreasing axial sweep had noticeable acoustic and aerodynamic penalties.

The baseline axial sweep was then chosen for the vane and only the effect of tangential lean and bow was studied. Tangential stacking was controlled at the midspan and the shroud with three levels at each location representing lean with the direction of rotor rotation, no lean, and lean against the direction of rotor rotation. Since this only produced 9 different vanes, a full factorial was used to examine all 9 vanes.

In full factorial stator DOE 1 (FF 1), tip tangential displacement equivalent to 30 deg of lean from the radial were tried. This had acoustic benefits but vanes leaned 30 deg with the rotor rotation (i.e., with the suction surface leaned down facing the hub) showed serious flow separation near the hub in DAWES. Since the 30 deg lean was infeasible from a performance standpoint, a second full factorial, FF 2, was used which had vane section tangential tip displacements equivalent to 15 deg of lean from the radial. The leading edges are shown in Fig. 4.3.3-1 and tabulated in Table 4.3.3-1. This figure also shows rev_3, which was the stacking eventually chosen after vane iterations using the acoustic, aerodynamic, and mechanical results of FF 2.

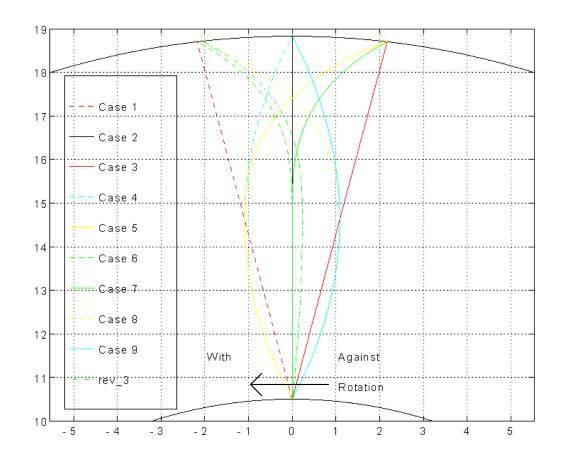


Figure 4.3.3-1. Stator Stacking Full Factorial 2 Vane Leading Edges

Table 4.3.3-1. Stator Stacking Full Factorial 2 CG Shifts in Baseline Engine Scale

Case	Midspan Ycg (in)	Shroud Ycg (in)	
1	-0.879	-1.758	15 deg lean with rotation
2	0.000	0.000	No lean
3	0.879	1.758	15 deg lean against rotation
4	-0.879	0.000	Bow with rotation
5	-0.879	1.758	Compound bow w/ and against rotation
6	0.000	-1.758	Compound lean with rotation
7	0.000	1.758	Compound lean against rotation
8	0.879	-1.758	Compound bow against and w/ rotation
9	0.879	0.000	Bow against rotation

Ycg: +ve=shifted against direction of rotation 0=unshifted

-ve=shifted with direction of rotation

The quality characteristics used for the vane stacking iterations were:

Acoustic: V072-predicted rotor wake-stator interaction sound power levels (SPL) at 2 and 3 times blade passing frequency (BPF), radiated towards the fan inlet and exit, at a representative approach point at 55.9% corrected fan design speed were used. The stator inlet velocity triangles used in V072 were predicted by an off-design model of the optc rotor in AXCAPS. The acoustic results are shown in Figures 4.3.3-2 through 4.3.3-5 for inlet 2*BPF, inlet 3*BPF, exit 2*BPF, and exit 3*BPF respectively. These figures also show the acoustic predictions for rev_3, the final vane stack.

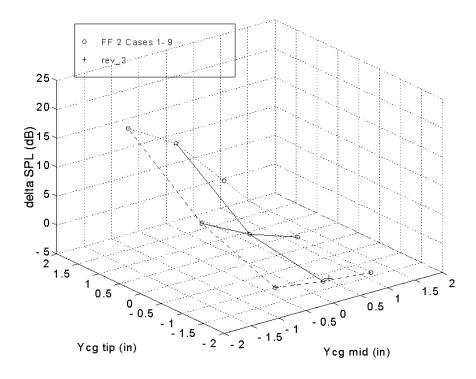


Figure 4.3.3-2. Stator Stacking Full Factorial 2 Inlet 2*BPF SPL Predictions

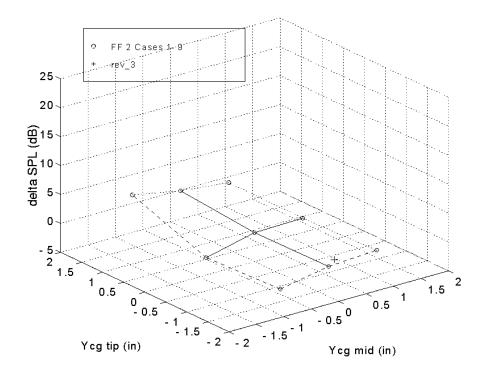


Figure 4.3.3-3. Stator Stacking Full Factorial 2 Inlet 3*BPF SPL Predictions

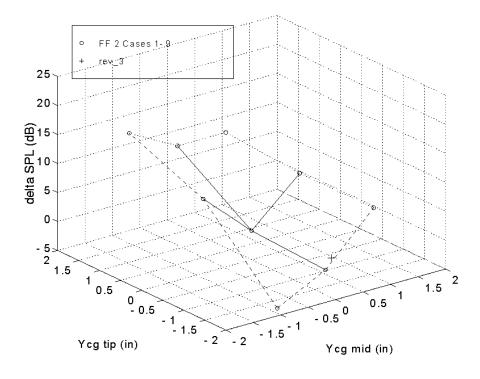


Figure 4.3.3-4. Stator Stacking Full Factorial 2 Exit 2*BPF SPL Predictions

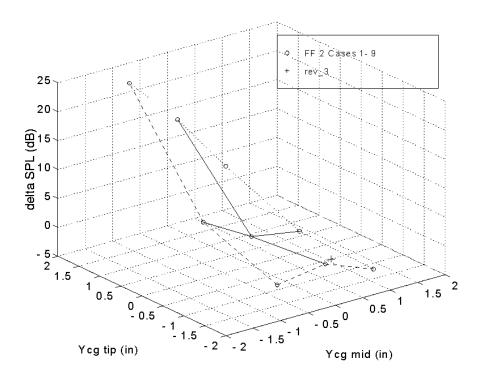


Figure 4.4.3-5. Stator Stacking Full Factorial 2 Exit 3*BPF SPL Predictions

The acoustic results were difficult to optimize. Generally, any stacking that reduced one sound power level increased others. A reduction in inlet sound power level was favored over a reduction in exit because the exit radiated sound can be more easily attenuated with acoustic treatment. The inlet 2*BPF results favored leaning the midspan against rotation and the tip with rotation as in rev_3.

<u>Aerodynamic:</u> Design point stator loss and stator exit swirl as estimated by DAWES were used as quality factors. Stator loadings were used as a qualitative guide. Figure 4.3.3-6 shows the DAWES predicted loss for the 9 stators in FF 2 as well as for the rev_3 stator.

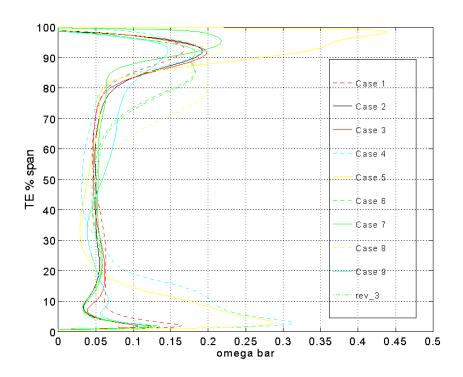


Figure 4.3.3-6. Stator Stacking Full Factorial 2 Stator Loss

The loss profiles show that leaning the suction surface towards the endwalls should be avoided because of flow separation from the suction surface and high hub loss (Cases 1, 4, and 5) and shrould loss (Cases 5 and 7). This result was an important aerodynamic driver in the stacking iterations.

The loss profiles also show a high-loss region from about 60% to 80% span for Cases 6, 8, and (to a lesser extent) 9. At first, this appeared to be a problem with stacks that lean with rotation over the outer 50% span but examination of the flow vectors in the vane-to-vane surface in this region showed flow separation from the pressure surface due to large negative incidence. Figure 4.3.3-7 shows incidence calculated using DAWES leading edge swirl angles. Figures 4.3.3-8 and 4.3.3-9 show the vane leading edge geometric angles and the vane leading edge swirl angles that resulted in this incidence.

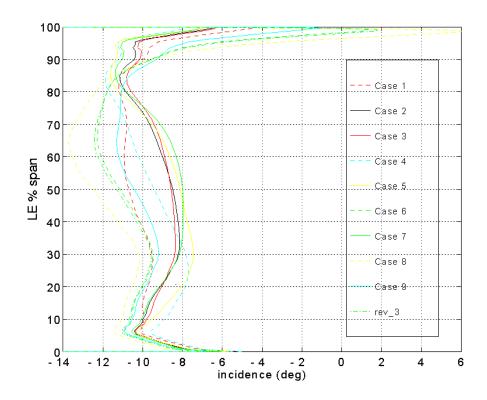


Figure 4.3.3-7. Stator Stacking Full Factorial 2 Stator Incidence

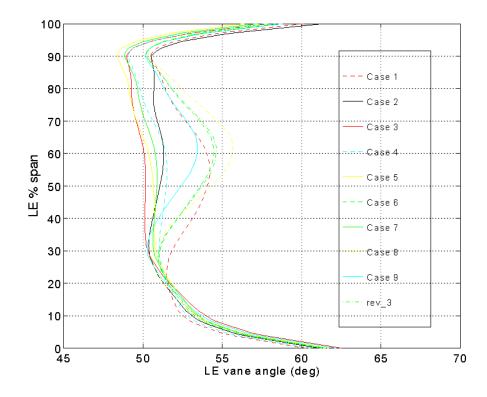


Figure 4.3.3-8. Stator Stacking Full Factorial 2 Vane LE Angle

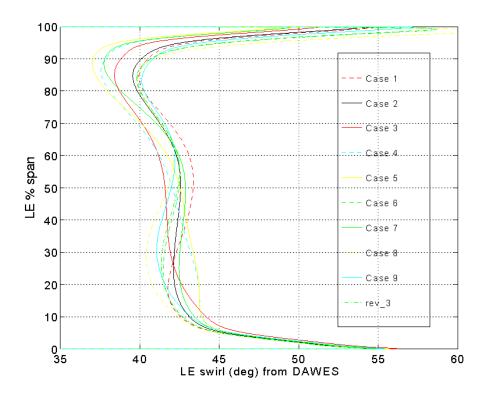


Figure 4.3.3-9. Stator Stacking Full Factorial 2 Stator LE Swirl Angle

The vane leading edge geometric angles in Figure 4.3.3-8 were calculated by AXCAPS. The intended stator incidence for FF 2 was the baseline design point stator incidence. AXCAPS was used to calculate the required vane leading edge angles to achieve this incidence given the rotor exit swirl angles from the OPTC rotor DAWES model. The varying vane stacks produced varying stator leading edge swirl profiles in AXCAPS since AXCAPS applies vane body radial forces to shift the flow radially, which changes the stator leading edge meridional velocity profile according to how the vane is stacked. Since all 9 cases have the same incidence, the variation in vane leading edge geometric angle in Figure 4.3.3-8 represents the variation in leading edge swirl angle in AXCAPS, which can be slightly over 5 degrees between 60% and 80% span.

The air angles in Figure 4.3.3-9 were obtained from the stator DAWES models. Each DAWES model used the exit swirl profile predicted by the OPTC rotor DAWES model at the design point. The swirl profile variation from case to case at the stator leading edge represents DAWES' estimate of how vane stacking affects the radial distribution of flow upstream of the stator. In DAWES, the variation due to vane stacking is only 2-3 degrees.

The high loss in this region results from setting the vane leading edge angles too high for these cases. This happened because the streamline code which was used to generate vane geometry over-estimated the effect of vane stacking on the stator leading edge swirl angles. Stator loss in this region was, therefore, not considered in the vane stacking evaluation. Figure 4.3.3-6 shows that the final vane stack, rev_3, also has high loss. This issue was resolved in the next DOE where incidence would be optimized.

Figure 4.3.3-10 shows stator exit swirl estimated by DAWES. The ultimate intent of the design was to set this as close to axial (0 deg) as practical, which was done in the baseline vane design. The vane trailing edge geometric angles for this DOE were taken from the baseline vane and were not adjusted to reduce the exit swirl. Exit swirl was still used as a quality characteristic since stators with swirl already close to axial would require less trailing edge angle adjustment.

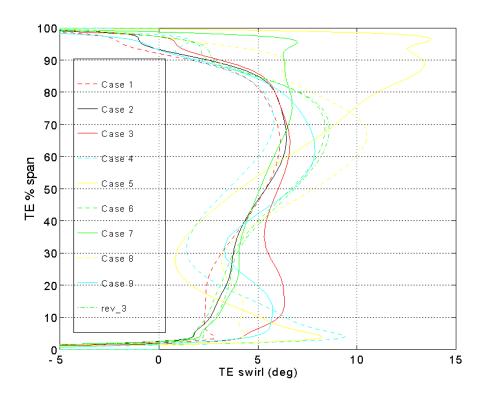


Figure 4.3.3-10. Stator Stacking Full Factorial 2 Stator TE Swirl Angle

Since all the vanes in this study have the same trailing edge geometric vane angle, Figure 4.3.3-10 shows the expected result that vanes with higher loss have higher deviation. Although exit swirl was examined in the design process, it was not used directly in the DOE since it is another way of representing loss.

Vane displacements, static strains on pressure and suction surfaces, vibration and flutter margins were used in the DOE as mechanical inputs. In addition, the static stresses and strains in the vane foot were monitored for abnormalities. The strain and displacement limits were defined based on AE experience. As seen from Figures 4.3.3-11 and 4.3.3-12, there were no surprises with both strain levels and the maximum displacements. Further, the maximum strains and displacements were within acceptable limits for all the go-forward cases that emerged out of the stator DOE.

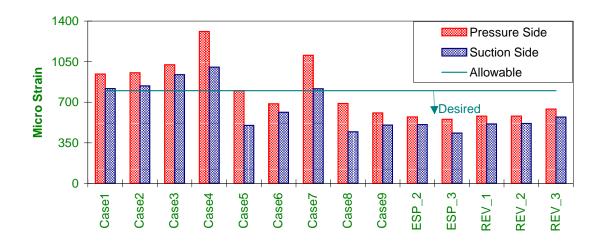


Figure 4.3.3-11 Stator Stacking Full Factorial Maximum Static Strain Predictions

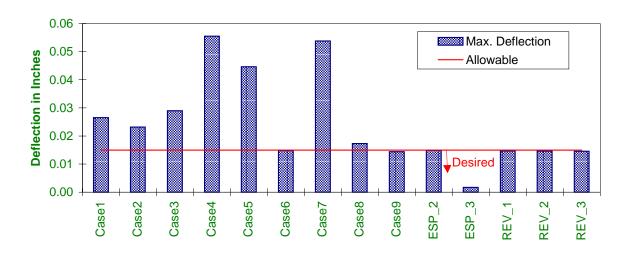


Figure 4.3.3-12 Stator Stacking Full Factorial Maximum Deflection Predictions

Satisfying the AE empirical flutter criteria for vanes conflicted with the aerodynamic and acoustic design. From an aerodynamic/acoustics perspective, a high degree of tangential bow is desirable. However, the addition of bow causes a significant change in the mode shape of the fundamental vibration mode. With small amounts of bow this mode is predominantly a flexure mode, while a larger degree of bow changes it into a torsion mode (see Figure 4.3.3-13). The problem arises because the flutter criteria based on reduced frequency, fc/V, is dependent on mode shape, with the torsion mode being much more difficult to satisfy. Fundamental mode flutter parameters were found to be at least 25% lower than AE recommended values for most of the go-forward designs (see Figure 4.3.3-14).

A number of approaches were investigated to increase the frequency of the bowed stator vane, such as thickness increases, changing airfoil chord, attachment boundary conditions, etc. Table 4.3.3-2 summarizes the flutter predictions for all the cases investigated. Initial attempts to increase flutter margin by thickening the vanes resulted in rapidly increasing stator weight with some moderate aerodynamic loss increases predicted by DAWES. For example, the total weight of the case_6 stator (52 vanes) increased from 5.68 to 7.07 lb. to drive the torsion mode flutter from 0.89 to 0.96. Increasing the chord further increased the weight with negligible change to the flutter margins. The weight issue was considered a problem and the low flutter margin was not necessarily a problem. There was no data for stators that implied flutter was guaranteed at this level of the flutter parameter, it was simply below previous experience. A compromise was reached where the vanes would be left at the rev_3 flutter parameter level and the vanes would be strain gauged in the rig to watch for flutter. Vane hub and shroud attachment modifications have been identified analytically that will increase flutter margin to baseline levels if vane flutter is seen in the rig. The modification is to provide two retention bands instead of the conventional one-band retention scheme used in the previous designs. The approach was tested on a number of cases and was found to consistently increase the flutter parameter by about 10 to 13% (see This approach helps keep stator weight under control and provides new knowledge about the limits of vane flutter parameter while providing an acceptable alternative if flutter actually occurs.

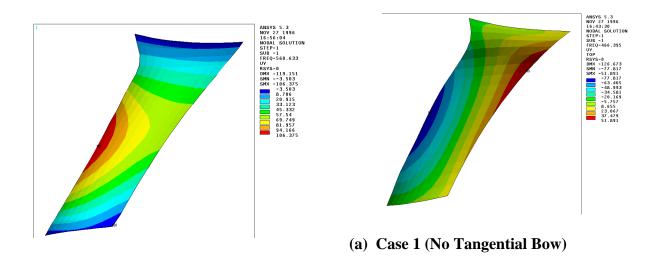


Figure 4.3.3-13 Effect of Tangential Bow on the Mode Shape of Fundamental Vibration Mode

(b) **Rev_3**

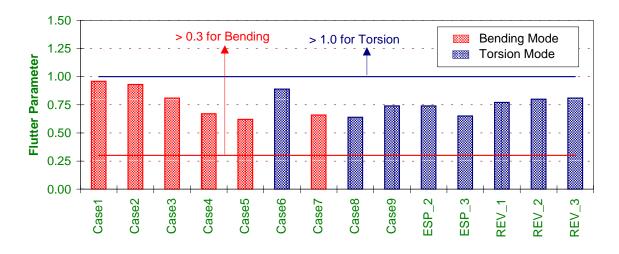


Figure 4.3.3-14. Stator Stacking Full Factorial Fundamental Mode Flutter Predictions

Table 4.3.3-2. Summary of Fundamental Modal Frequencies for Stator Vane

DOE Case		Mode 1		Total Wt. (lb.)	Comment
	Frequency	Type*	Flutter		
	(N/rev)		Parameter		
Case_1	3.31	В	0.96	5.64	
Case_2	3.28	В	0.93	5.60	
Case_3	2.96	В	0.81	5.63	
Case_4	2.39	В	0.67	5.63	
Case_5	2.26	B/T	0.62	5.56	
Case_6	3.01	T	0.89	5.68	
Case_6_Bnd	3.29	T	0.97	5.68	Two Bands
Case_6a	3.27	T	0.96	7.07	Inc. Thickness
Case_7	2.37	В	0.66	5.62	
Case_8	2.14	T	0.64	5.73	
Case_8c	2.50	T	0.76	8.45	Inc. Thickness
Case_9	2.56	T	0.74	5.65	
ESP_2	2.49	T	0.74	5.71	
ESP_2a	2.83	T	0.85	7.60	Inc. Thickness
ESP_2b	2.82	T	0.84	8.00	Inc. Thk & Chord
ESP_3	2.18	T	0.65	5.72	
REV_1	2.62	T	0.77	5.68	
REV_1_Bnd	2.94	T	0.87	5.68	Two Bands
REV_2	2.72	T	0.80	5.67	
REV_2_Bnd	3.04	T	0.89	5.67	Two Bands
REV_3	2.75	T	0.81	5.69	
REV_3_Bnd	3.11	T	0.92	5.69	Two Bands

^{*} **Type:** B → Bending Mode; T → Torsion Mode

4.3.4 Incidence DOE

The next step has to set the incidence and mean camber line angle distribution. The vane mean camber line angle distributions were taken from the baseline vane. Although the mean camber line angle distribution optimization would render reduced stator loss and improved stator loadings, the stator losses predicted by DAWES for the rev_3 stator were already close to those predicted by DAWES for the baseline stator. Since the final rotor design (gf03) was available, the AXCAPS model was modified to use this rotor with its DAWES-predicted exit velocity triangles for the remaining stator design work.

The rev_3 stator behind the gf03 rotor was used as the base case for the incidence study. The vane leading edge angles were set to produce the baseline design point incidence with the stator leading edge swirl angles from the new gf03 rotor. A 9-case full factorial DOE was performed where vane leading edge angle was varied over three levels (-3 deg, 0 deg, and +3 deg) from the baseline at the vane hub, midspan, and tip. The vane leading edge angle was specified directly rather than allowing AXCAPS to try to calculate it from incidence. Figure 4.3.4-1 shows the vane leading edge geometric angles for the 9 cases (Case 5 is the baseline case) and for the rev_3 vane. Figure 4.3.4-1 also shows the vane leading edge angles for inc_1, which is the leading edge chosen from the results of this study. Figure 4.3.4-2 shows the resulting incidence profiles using DAWES stator leading edge swirl angles.

The effects of varying the vane leading edge angle on the mechanical and acoustic quality factors were insignificant so the leading edge angle was chosen based on stator loss and loading. Exit swirl was examined, but for a constant vane trailing edge angle, the deviation largely followed the loss as in the stacking study. Figures 4.3.4-3 and 4.3.4-4 show the stator loss and exit swirl estimated by DAWES.

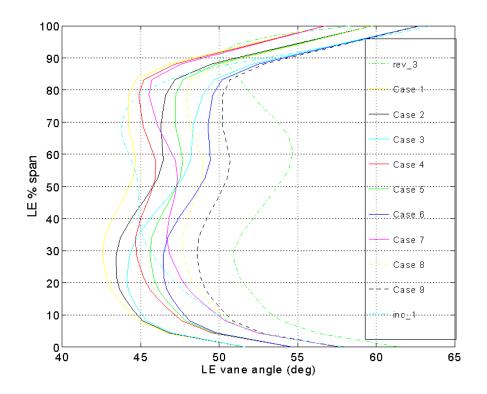


Figure 4.3.4-1. Stator Incidence Full Factorial Incidence

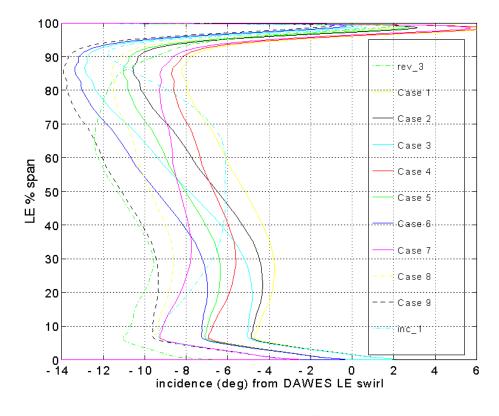


Figure 4.3.4-2. Stator Incidence FF Stator Incidence

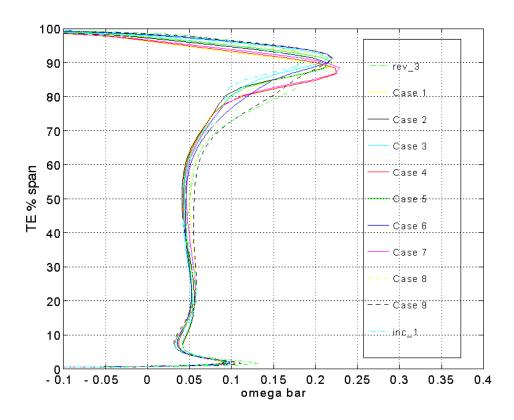


Figure 4.3.4-3. Stator Incidence FF Stator Loss

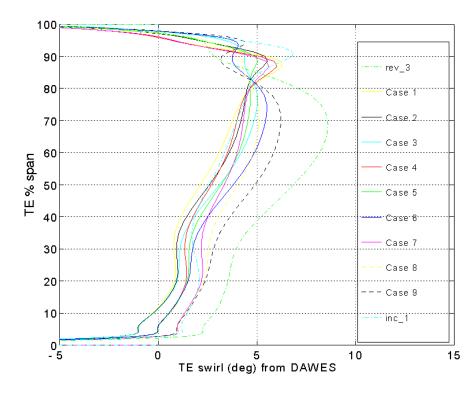


Figure 4.3.4-4. Stator Incidence FF Stator TE Swirl Angle

The loss was reduced from or equal to rev_3 over much of the span for all 9 cases, indicating that the biggest benefit of the incidence full factorial was due to correcting the way in which incidence was set. The leading edge angle was set by looking at curves of loss vs. leading edge angle and at vane loadings at different spanwise locations. Generally, leading edge angles that improve loss also improve loadings. If a section showed trends in loss but little effect on loading, loss was favored in setting the leading edge. The resulting leading edge angles are shown in Figure 4.3.4-1 as inc_1. Relative to the baseline (Case 5), the leading edge is overcambered 3.5 degrees at the hub and shroud and uncambered 3.5 degrees from 55% to 75% span. The inc_1 losses and exit swirl are shown in Figures 4.3.4-3 and 4.3.4-4.

The high-loss bump around 90% span was largely insensitive to incidence. Particle traces completed in the DAWES solution (Figure 4.3.4-5) showed that the high loss was coming from flow from the shroud boundary layer, the vane suction surface boundary layer, and the suction surface-shroud intersection. This flow was migrating down in span probably due to the vane lean and was unaffected by incidence at 90% span. This spanwise flow would be difficult to eliminate and may not be a problem relative to other stators. A similar loss source is probably present in all stators but isn't so distinctive since it would stay closer to the shroud for unleaned vanes.

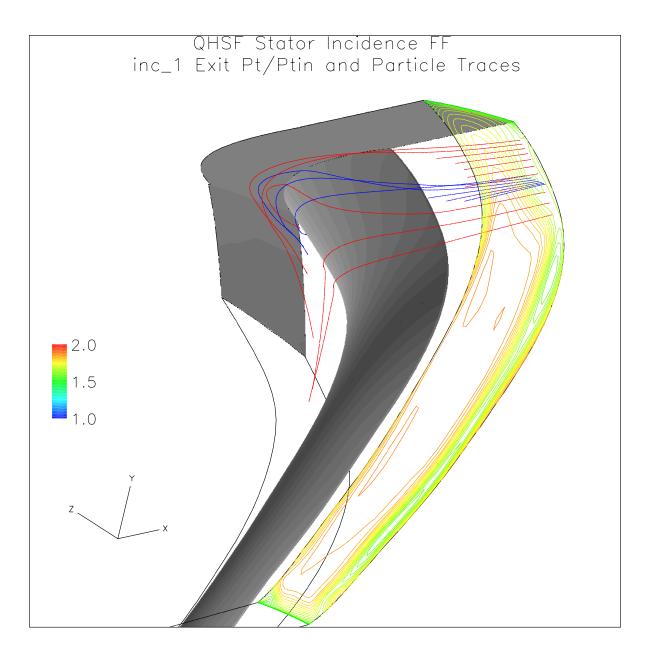


Figure 4.3.4-5. Stator Loss Bump

4.3.5 Trailing Edge Angle Setting

The vane trailing edge angle was adjusted to turn the flow as close to axial as practical. The trailing edge swirl angles from inc_1 were used to adjust the inc_1 vane trailing edge angle, DAWES was used to estimate the new trailing edge swirl, and within a couple of iterations the exit swirl was acceptably close to axial. Figure 4.3.5-1 shows the exit swirl for inc_1 and the exit swirl with the reset vane trailing edge (dev_2).

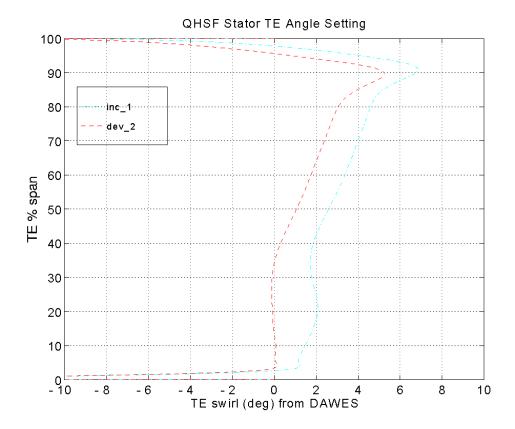


Figure 4.3.5-1. QHSF Stator Trailing Edge Angle Setting Swirl

The swirl is close to zero in the inner 30% span. Zero swirl in this region is important since the flow goes through a decreasing-radius transition duct to the core. Close control of the fan stator exit swirl in the core flow region is desirable to avoid mis-matching the compressor. The swirl farther out in span is considered acceptably close to zero. The swirl peak around 90% span is due to the shroud/vane intersection flow discussed previously and does not represent the actual stator flow turning in this region.

Figures 4.3.5-2 and 4.3.5-3 show the vane trailing edge angles and the deviation. Figure 4.3.5-4 shows the stator loss before and after the trailing edge angle was modified. Figure 4.3.5-4 shows that the loss changes caused by resetting the trailing edge angle were minor.

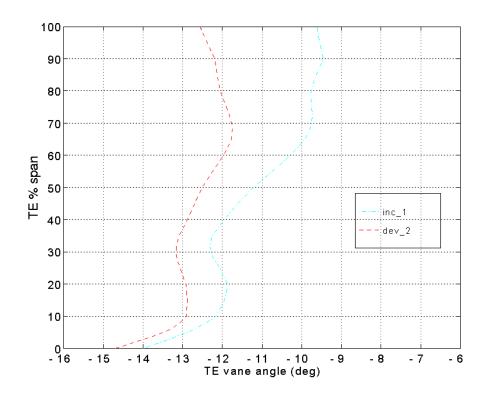


Figure 4.3.5-2. QHSF Stator Trailing Edge

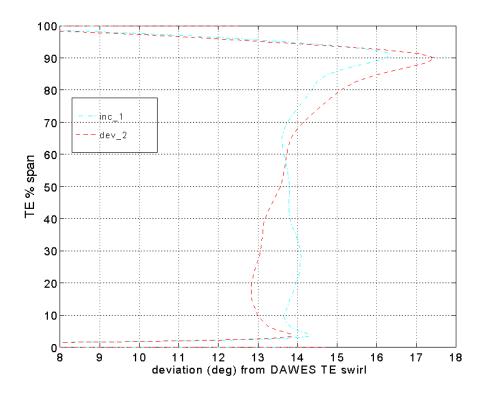


Figure 4.3.5-3. QHSF Stator Trailing Edge Angle Setting Deviation

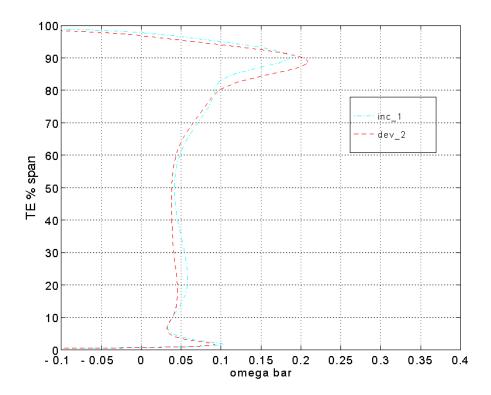


Figure 4.3.5-4. OHSF Stator Trailing Edge Angle Setting Loss

4.3.6 Composite Vane Thickness

The vane thicknesses were adjusted in the direction normal to the suction and pressure surfaces to allow for composite construction. The thickness adjustment was defined by the cosine of the local lean angle so it was necessary to consider it last since the lean angles depend on stacking and on vane section stagger.

Figures 4.3.6-1 and 4.3.6-2 show stator loss and exit swirl before (dev_2) and after the thickness change. Meeting the composite vane thickness criteria had only minor effects on loss and exit swirl. Examination of the vane section loadings showed little difference. This result was unexpected considering how much the tangential thicknesses were increased. This may be because the flow was not constrained to follow 2-dimensional streamsurfaces so the effect of the tangential thickness increase was reduced. The "effective" vane section thicknesses are probably somewhere between the tangential and the normal thicknesses. Figure 4.3.6-3 shows normal thickness distribution of the vane before (DEV_2) and after (THK_2d) thickness changes along with the recommended thickness distribution.

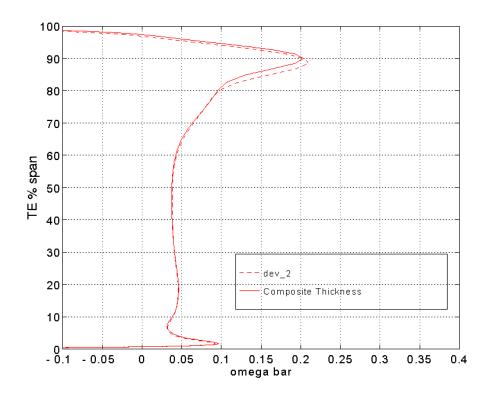


Figure 4.3.6-1. QHSF Stator Composite Thickness Loss

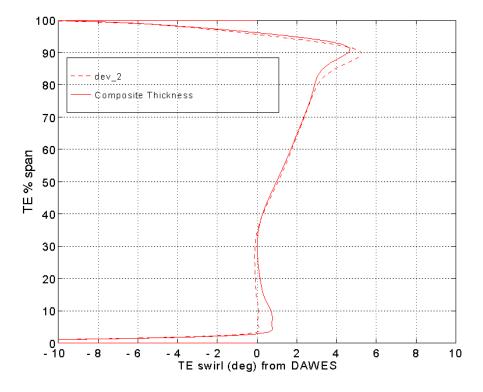


Figure 4.3.6-2. QHSF Stator Composite Thickness Exit Swirl

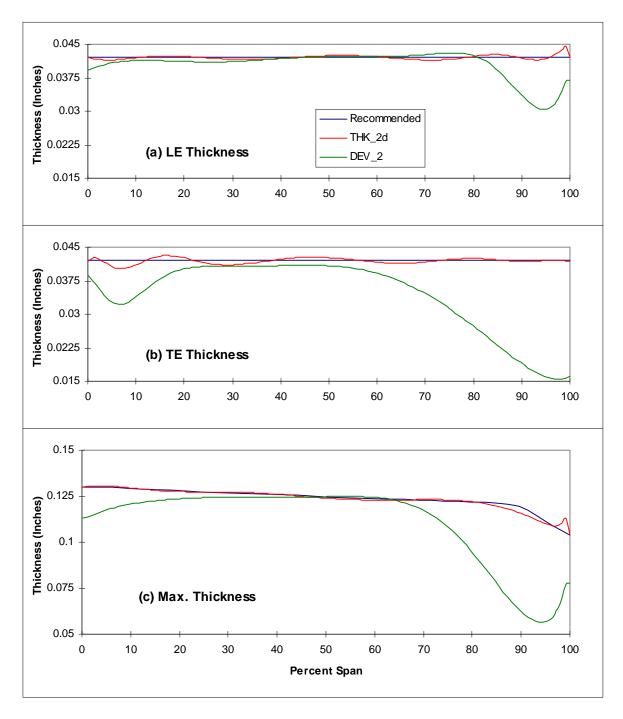


Fig. 4.3.6-3. Vane Normal Thickness Distribution for QHSF Stator Vane

4.4 Disk / Attachment

4.4.1 Initial Design Phase

Since both the AE/QHSF and baseline fans will be tested in the rig, it is desirable to have the same fan disk for both. The primary intent during this phase was to determine the feasibility of using the baseline disk for QHSF with minor modifications which include changes required to match the disk dimensions with the NASA rig dimensions.

The following points summarize the initial design parameter inputs for the disk design:

TFE engine size disk scaled down to NASA rig size (Scale Factor = 22/30.698)

- a rectangular cross section (no under cuts)
 Disk width and bore radius fixed to match the NASA rig dimensions
 (width = 3.4"; bore radius = 2.2")
- titanium blades and attachment.
- blade airfoil load based on OPTM blade
- blade attachment load assumed from baseline rotor configuration

4.4.2 Cyclic-Symmetric Model

A 2-D wedge section normal to the broach line was built using the baseline disk and dovetail geometry data. The bore diameter of the disk was trimmed to accommodate a 0.15" thick torque sleeve. The 2-D geometry of the disk/dovetail was imported to HyperMesh in IGES format for the finite element (FE) meshing. The FE model from HyperMesh is output in ANSYS prep7 format for stress analysis.

Boundary Conditions: The nodes on the two edges of the wedge were positioned at identical radial locations, and the corresponding nodes on two edges were coupled both in radial and tangential directions to enforce cyclic symmetry in the disk. The blade and disk were coupled by constraining blade and disk nodes at the contact plane to have same displacement in the direction normal to the contact surface. The blade attachment and airfoil CF loads were applied as shown in Figure 4.4.2-1. The attachment geometry acts as a load transfer mechanism and was assumed to have zero density such that the attachment CF loads were not duplicated. The effects of slot broach were simulated by including a side load at the contact plane (see Figure 4.4.2-2). Finally, the nodes at the bore were constrained to have zero displacement in the tangential direction.

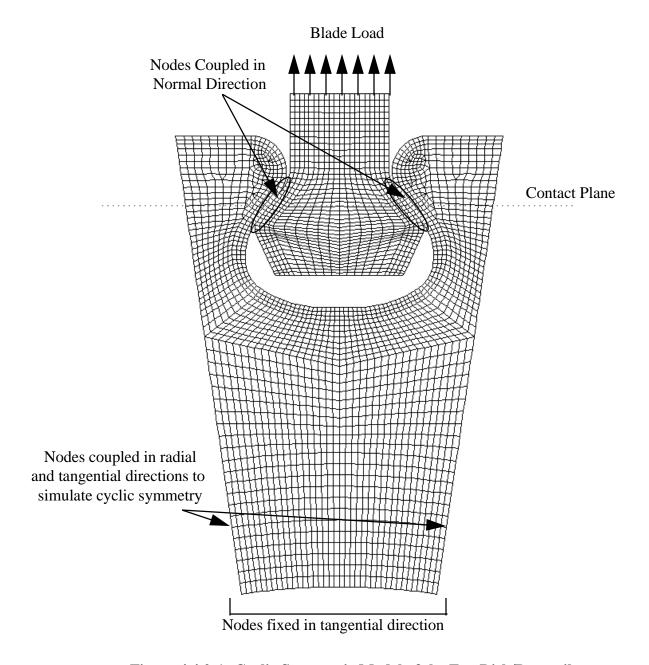


Figure 4.4.2-1. Cyclic Symmetric Model of the Fan Disk/Dovetail

4.4.3 Selection of Disk Material

Two different materials — titanium and C250 steel — were considered for the disk during test runs. Analysis was performed at three points:

- *Mechanical Design Point:* 100% physical speed (T_{tin} = 65 F) scaled to rig, 15444 RPM
- Max. Test Speed: 110% Design Speed, 16988 RPM
- *Max. Operating Speed:* 110% Max. Test Speed, based on NASA specification = 121% Design Speed, 18687 RPM

Table 4.4.3-1. Material Properties for C250 Steel and Titanium at 75 F

Material Type	Iron	Titanium	
Material Name	Maraging Steel C250	TI-6AL-4V	
Process/Forming	Wrought	Wrought	
Heat Treatment	Double ST+Aged	Annealed	
Yield Strength (Ksi)	238.3	121.5	
Ultimate Strength (Ksi)	247.2	129.1	
Density (lb/in ³)	0.289	0.161	

The results from 2-D cyclic symmetric analyses are presented in Table 4.4.3-2. With the titanium disk, the maximum radial stress was within the safe limits at the design speed. However, at maximum test speed, the maximum radial stress reached unacceptable limits (> 100 Ksi) as did the burst margin. The C250 steel disk had adequate stress as well as burst margins at all speed conditions. Based on this analysis, it was planned to proceed with the disk design using C250 steel. Figure 4.4.2-2 shows the principal stress distribution in the disk/dovetail at maximum operating speed for C250 steel disk case.

Table 4.4.3-2. Maximum Stresses for Various Cases – Rig-Size Model

Titanium Disk							
	Design Speed	Max. Test Speed	Max Operating				
	$(100\% N_{phy})$	$(110\% N_{phy})$	Speed (121% N _{phy})				
RPM	15444	16988	18687				
Max. Deflection (inch)	0.0134	0.0162	0.0196				
Max. Principal (Ksi)	87	105	127				
Max. Radial Stress (Ksi)	84	101	122				
Max. Tangential (Ksi)	71	86	104				
Av. Tangential (Ksi)	57.8	69.9	84.6				
Burst Margin*	1.42	1.29	1.18				
C250 Steel Disk							
Max. Deflection (inch)	0.0105	0.0127	0.0154				
Max. Principal (Ksi)	89.53	108.36	131.09				
Max. Radial Stress (Ksi)	85.51	103.47	125.20				
Max. Tangential (Ksi)	83.88	101.49	122.81				
Av. Tangential (Ksi)	68.34	82.70	100.06				
Burst Margin*	1.76	1.603	1.46				

^{*} Burst Margin requirements: AE > 1.25

NASA > 1.345 (when translated to AE criteria)

AE and NASA have different definitions of burst margin (BM); this merits some comments.

$$BM_{AE} = \sqrt{\frac{0.85 \times F_{tu}}{\sigma_{average}}} \geq 1.25$$

$$BM_{NASA} = \frac{0.7 \times F_{tu}}{\sigma_{average}} \ge 1.5$$

where F_{u} is the ultimate strength of the material.

The AE design practice recommends a material utilization factor of 0.85, whereas NASA recommends a value of 0.7. The AE burst factor provides a margin on rotor speed (25% over speed) and NASA factor provides a 50% margin on average tangential stresses. When translated to the over speed condition, the NASA criterion is equivalent to $BM_{AE} \ge 1.345$.

4.4.4 Attachment Stresses and Post Unzip Capability

The margins of safety in various disk and attachment sections were obtained based on simple hand calculations. The sections represent the cross section that was most likely to fail under the given load. For example, the blade direct stresses were calculated at the blade minimum neck cross section. The margin of safety (MS) was defined (based on NASA guidelines) as

$$MS = \frac{Allowable\ Stress}{SF \times Calculated\ Stress} - 1$$

where SF = Safety Factor

Table 4.4.4-1 provides the margins of safety for various disk and attachment sectional stresses. The stresses are presented at Mechanical Design point (100% N_{phy}) and the margins of safety are presented at 100%, 110% and 121% N_{phy} .

Post unzip capability represents the failure resistance of the disk post due to combined bending and direct stresses during blade out condition. In other words, it is the capability of the disk to resist unzipping or losing adjacent blades. Figure 4.4.4-1 shows the unzip capability for the QHSF disk. The figure is analogous to well-known "Goodman diagram". In the figure, the y-axis represents a pure bending case and x-axis the pure tension case. For pure bending, failure occurs when bending stress exceeds 1.5 times yield strength of the material and for the pure tension case it occurs when the direct stress exceeds ultimate strength. For combined bending and direct stress case, failure occurs if the stresses are above the solid line or above the dotted line when safety factors are included. As seen in the figure, the QHSF disk post is safe under all three speed conditions.

4.4.5 Go-Forward Design

In summary, the preliminary design showed the baseline disk design can be used for the QHSF and should be manufactured with C250 steel material. It had adequate margins of safety, burst margin and unzip capability, and met both AE and NASA design criteria.

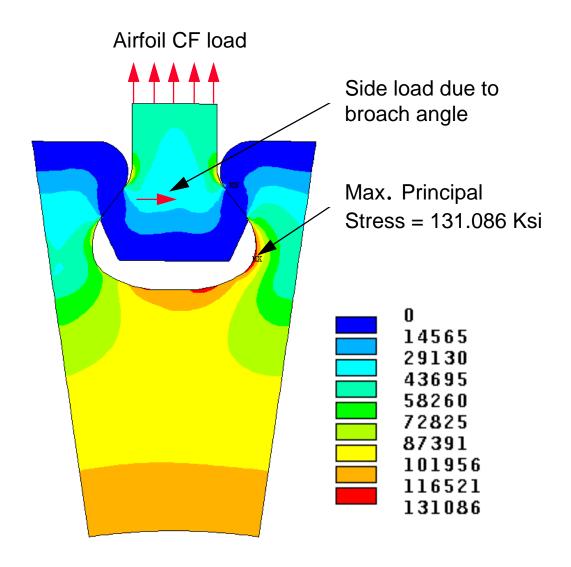


Figure 4.4.2-2 Principal Stress in disk/dovetail at 121% N_{phy}

Table 4.4.4-1. Attachment/Disk Stress Safety Margins

	Stress(Ksi)	Margin of Safety		fety
Percent Speed	100%	100%	110%	121%
Contact Stress	65.0290	0.5630	0.2917	0.0676
Blade Shear Stress	17.8972	2.2769	1.7082	1.2382
Blade Direct Stress	31.4828	2.2285	1.6682	1.2051
Blade Lobe Bending Stress	14.7769	5.8784	4.6846	3.6980
Disk Shear Stress	18.2269	5.6909	4.5297	3.5700
Disk Direct Stress	32.5291	5.4975	4.3699	3.4379
Disk Lobe Bending Stress	15.3263	12.7906	10.3972	8.4192
Yield Safety Factor (SF _y) =	1.1*			
Ultimate Safety Factor $(SF_u) =$	1.5*			

^{*} Based on NASA guidelines

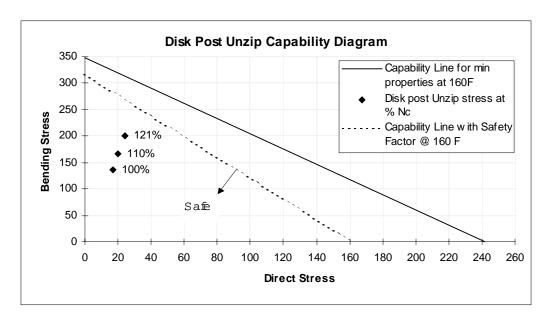


Figure 4.4.4-1 QHSF Disk is Designed to Withstand Unzipping During Blade-Out Condition

5.0 DETAILED DESIGN

5.1 Rotor

5.1.1 Geometry

The final QHSF design configuration was chosen based on the analyses from the "go-forward" cases of DOE 4. Go-forward case 3 was selected as the QHSF rotor design. Final blade geometry spanwise distributions are shown in Figure 5.1.1-1 through 5.1.1-5. Tabular geometry data of the blade is included in Appendix I. Values in the figures represent the NASA rig rotor design (22 inch diameter at the rotor leading edge tip). The incidence was calculated using DAWES at the design point. The large incidence increase at the tip was due to the endwall modeling of the boundary layer at the rotor inlet.

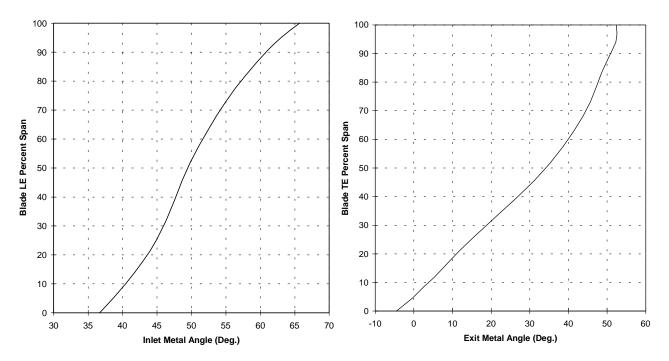


Figure 5.1.1-1. Blade Inlet and Exit Metal Angle

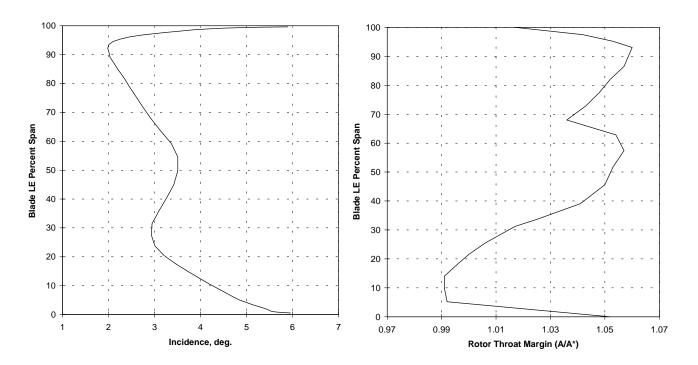


Figure 5.1.1-2. Blade Incidence and Throat Margin

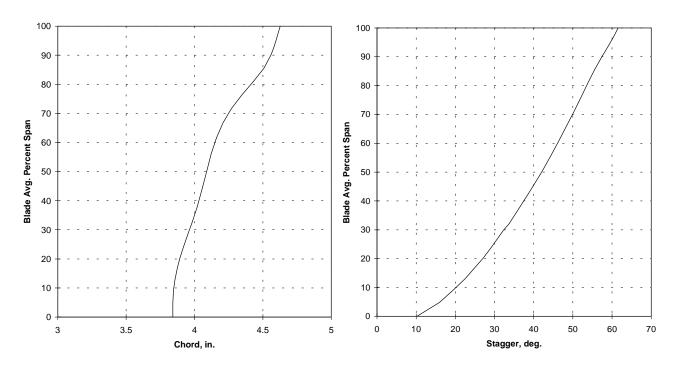


Figure 5.1.1-3. Blade Chord and Stagger Angle

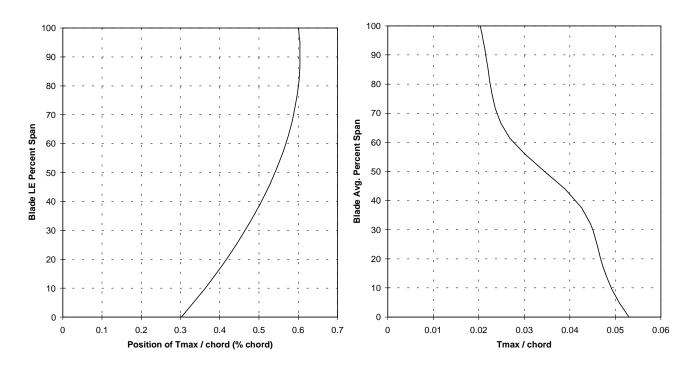


Figure 5.1.1-4. Blade Position of Maximum Thickness and Maximum Thickness / Chord Ratio

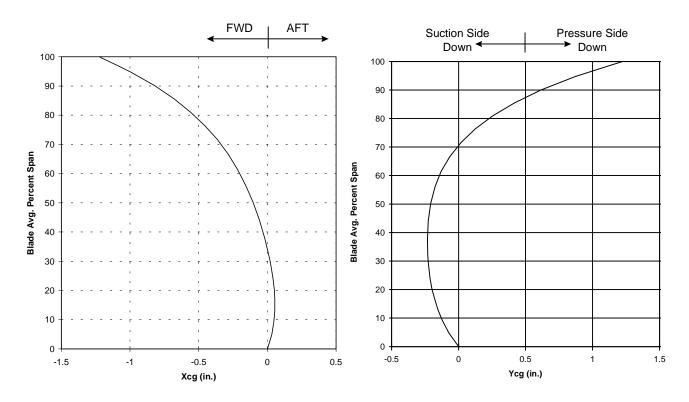


Figure 5.1.1-5. Blade CG Stacking

5.1.2 Aerodynamic Results

The design concentrated on achieving a close performance match relative to the baseline such that the acoustic benefits of the forward swept technology could be easily deduced from the test data. DAWES analyses at aerodynamic design point showed only a slight shift in speedline characteristics relative to the baseline. At the design efficiency goal, the flow was 0.3% lower than the design goal, while the pressure ratio was 1.3% higher. At the design flow goal, the efficiency was 0.2 points lower while the pressure ratio was 0.3% lower. However, analyses completed from choke to stall showed slightly higher peak efficiency than the baseline and approximately one point higher as the rotor was throttled up from peak efficiency to stall. Spanwise distributions of pressure ratio, temperature ratio, efficiency, deviation, omega-bar, and D-factor are shown in Figures 5.1.2-1 through 5.1.2-3, respectively. As previously mentioned, an endwall total pressure loss was being modeled at the rotor leading edge and the effect is evident in the figures.

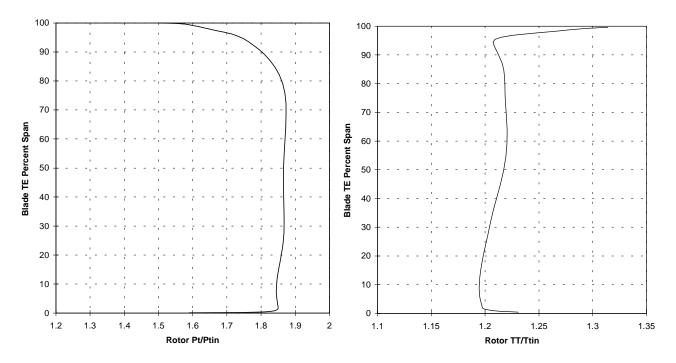


Figure 5.1.2-1. DAWES Calculated Rotor Pressure Ratio and Temperature Ratio at the Aerodynamic Design Point

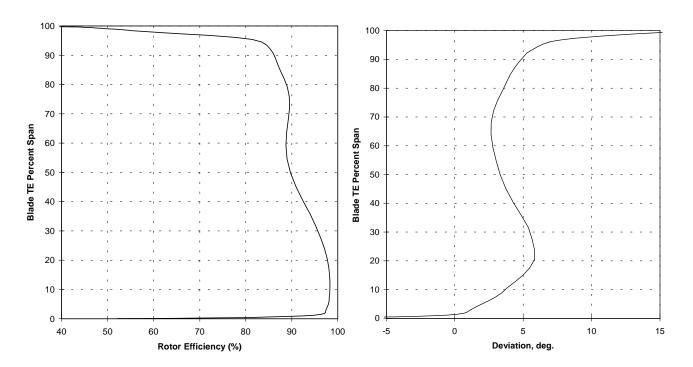


Figure 5.1.2-2. DAWES Calculated Rotor Efficiency and Deviation at the Aerodynamic Design Point

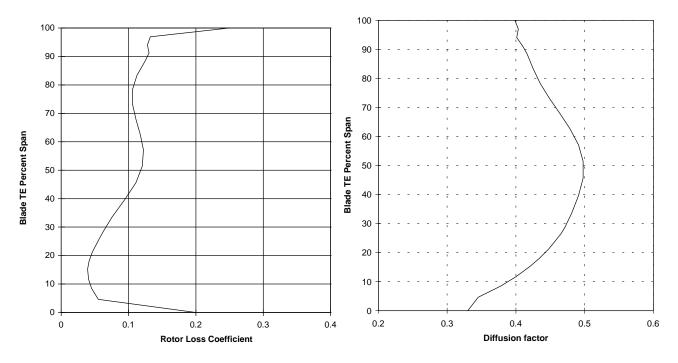


Figure 5.1.2-3. Rotor Omega-bar and D-factor at the Aerodynamic Design Point

The increase in loss coefficient near mid-span was due to an increase in shock loss as the blade effective sweep approached zero degrees. Effective sweep described within is the oblique shock angle which the inlet flow passes coming into the passage. The oblique shock angle is a function of stream surface angle, blade lean angle, station lean angle (blade sweep), and relative flow angle. When the shock angle is considered a shock surface, both the blade leading edge and the impingement point on the suction surface of the adjacent blade (across the passage) must be accounted. Figures 5.1.2-4 and 5.1.2-5 show the difference in the effective sweep calculation (in terms of normal inlet relative Mach number) between leading edge only and a simple average calculation of the leading edge and the suction surface impingement point. The figures also show an inherent difference between the forward swept QHSF and the aft swept baseline in that the shock surface will always reduce the benefit of a forward swept blade, while it will always increase the benefit of an aft swept blade. Although the QHSF design had significant benefits near the tip, it was slightly worse than the baseline near the mid-span.

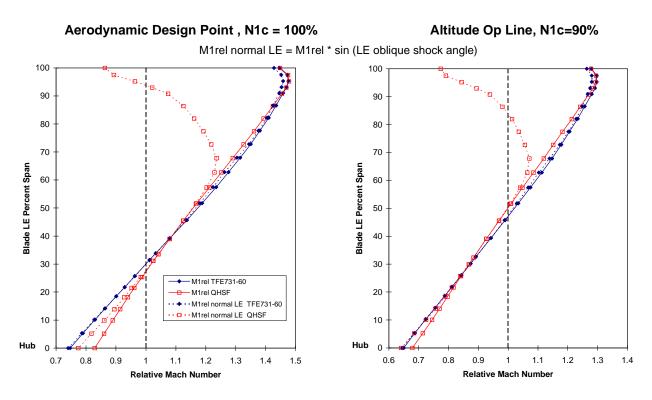


Figure 5.1.2-4. Effect of Blade Leading Edge Effective Sweep on Inlet Relative Normal Mach Number at 90 and 100% N1c

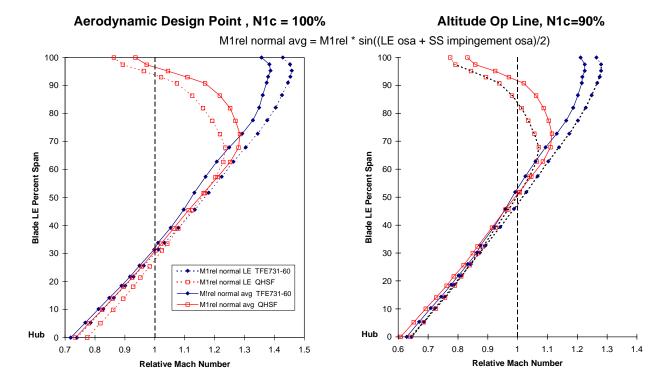


Figure 5.1.2-5. Effect of the Combination of Blade Leading Edge and Suction Surface Impingement Effective Sweep on Inlet Relative Normal Mach Number at 90 and 100% N1c

Airfoil loadings, represented as isentropic Mach number distributions along the suction and pressure surfaces, and the corresponding blade-to-blade Mach number contour plots are shown for 10, 30, 50, 70, and 90 percent spans at the aerodynamic design point in Figures 5.1.2-6 through 5.1.2-10. As previously mentioned, the QHSF design was chosen based on the design and part-speed performance results from DOE 4. In the DOE, many quality characteristics including airfoil loadings were examined and optimized for an overall best design. The loadings at 30, 50, and 90 percent spans show excessive diffusion with unnecessary re-acceleration on the suction surface. Additional work could be completed to improve these loadings and improve efficiency.

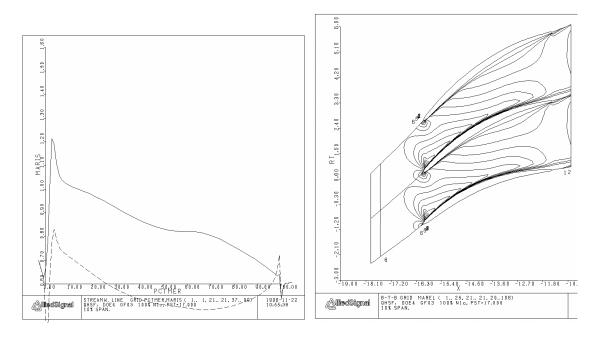


Figure 5.1.2-6. DAWES Calculated Isentropic Blade Loadings and Blade-to-Blade mach Number Contours at 10% Span at the Design Point

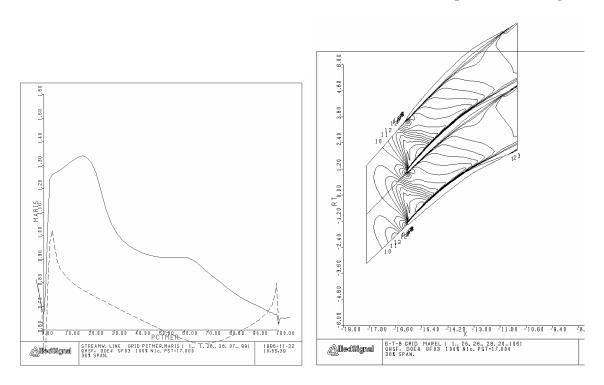


Figure 5.1.2-7. DAWES Calculated Isentropic Blade Loadings and Blade-to-Blade mach Number Contours at 30% Span at the Design Point

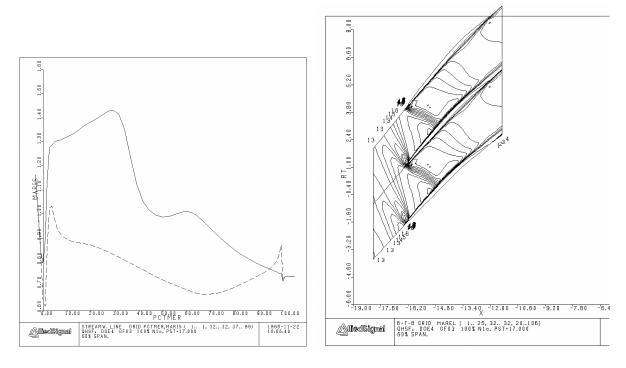


Figure 5.1.2-8. DAWES Calculated Isentropic Blade Loadings and Blade-to-Blade mach Number Contours at 50% Span at the Design Point

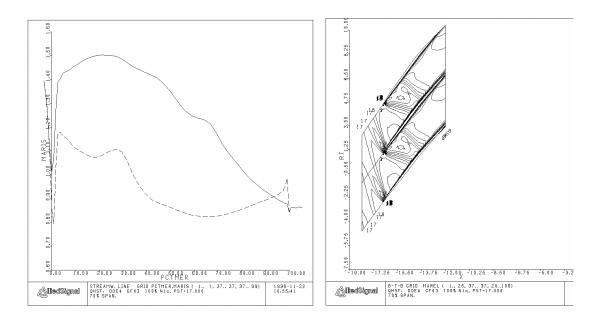


Figure 5.1.2-9. DAWES Calculated Isentropic Blade Loadings and Blade-to-Blade mach Number Contours at 70% Span at the Design Point

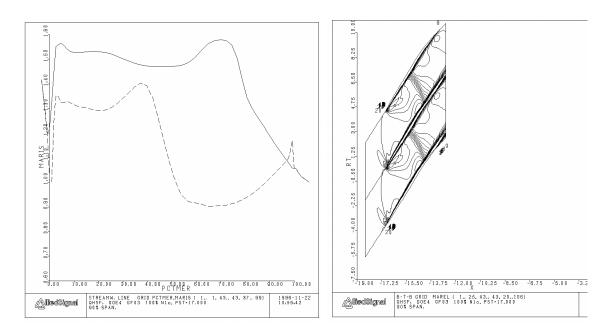


Figure 5.1.2-10. DAWES Calculated Isentropic Blade Loadings and Blade-to-Blade mach Number Contours at 90% Span at the Design Point

Blade-to-blade contour plots are shown for 50, 70, and 90 percent spans at choke, design point, and near stall in Figures 5.1.2-11 through 5.1.2-13, respectively. Relative to the baseline, the amount of blade effective sweep significantly reduced the strength of the shock. The figures show that the passage shock was well contained at the tip although it moved forward as span decreased and was positioned upstream of the leading edge at 70% span. Attempts in DOE 4 to keep the shock positioned back in the passage had negative impact on rotor efficiency.

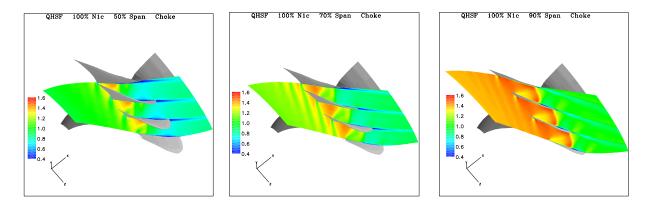


Figure 5.1.2-11. DAWES Calculated Blade-to-Blade Relative Mach Number Contours at 100% N1c Choke for 50, 70, and 90% Spans

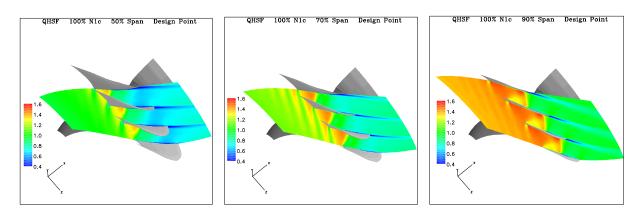


Figure 5.1.2-12. DAWES Calculated Blade-to-Blade Relative Mach Number Contours at 100% N1c Design Point for 50, 70, and 90% Spans

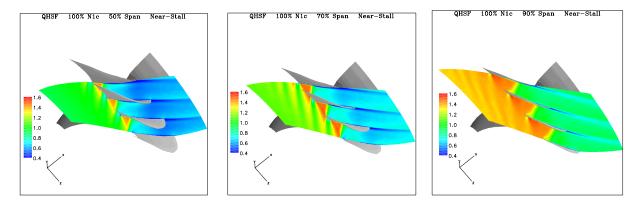


Figure 5.1.2-13. DAWES Calculated Blade-to-Blade Relative Mach Number Contours at 100% N1c Near Stall for 50, 70, and 90% Spans

A rotor-only predicted map for the QHSF is shown in Figures 5.1.2-14 through 5.1.2-16. DAWES analyses were completed at the points indicated. The map agrees very well with DAWES analyses completed with the baseline rotor. Relative to test data of the baseline rotor, flow and work agreed very well but the efficiency would be adjusted higher approximately 1.5 points.

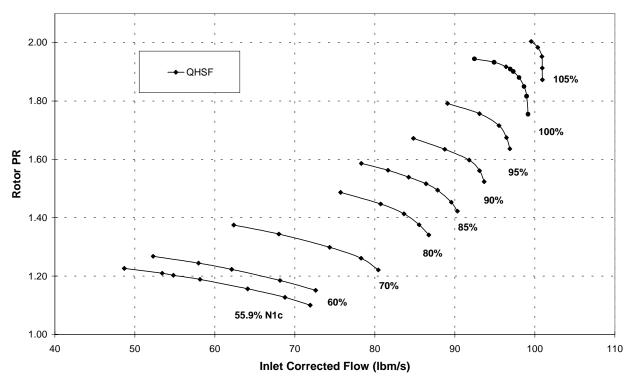


Figure 5.1.2-14. QHSF Predicted Map (Wc vs. PR) Based on DAWES Analyses

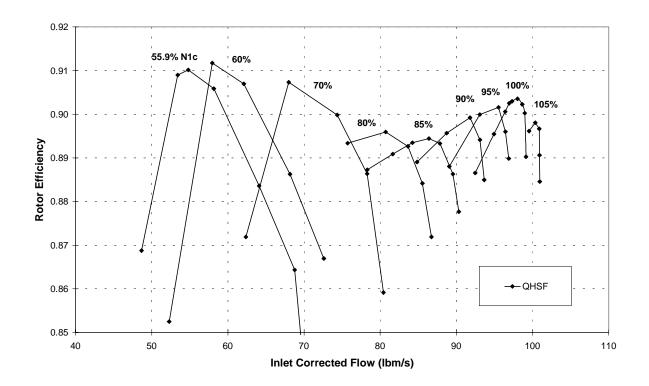


Figure 5.1.2-15. QHSF Predicted Map (Wc vs. Eff) Based on DAWES Analyses

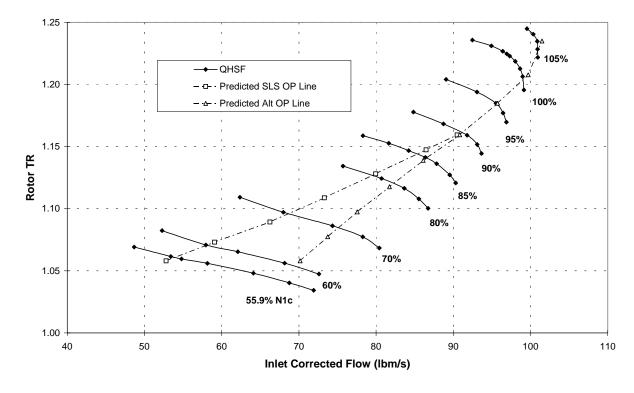


Figure 5.1.2-16. QHSF Predicted Map (Wc vs. TR) Based on DAWES Analyses with Predicted Operating Lines

5.1.3 Acoustic Results

5.1.3.1 Design Philosophy

The design philosophy employed in reducing multiple pure tone (MPT) noise below that of the baseline rotor was to use rotor forward sweep to control the strength and position of the passage shock emanating from the suction surface of the airfoil. It was assumed that use of forward sweep would delay the onset of strong shocks to higher fan speeds and allow the capture of shocks within the blade passages at lower fan speeds, thereby affecting buzzsaw noise as shown in Figure 5.1.3-1.

5.1.3.2 Baseline Fan

It became apparent that the geometry and operating conditions of the baseline were such that the dominant shock structure consisted of a normal shock emanating from the suction side of the rotor, herein called a "passage" shock. At all conditions except 100% corrected fan speed (see Figure 5.1.3-2), the presence of a propagating passage shock (not contained within the blade passage) precluded the need for a bow shock at the blade leading edge since the flow was subsonic at this location (see Figure 5.1.3-3). The volume of literature gathered to date seem only to discuss the effects of blade geometry and stagger angle variation on the strength and position of a bow shock (References 2 to 13). Evidently these studies were conducted on rotors which operated closer to choke where the passage shock was swallowed. For this effort it was assumed that the geometric variations classically deemed responsible for MPT noise generation, due to their effect on bow shocks, in a similar way cause variations in the strength and propagation direction of the expelled (propagating) passage shocks.

Using the 3-D thin-layer Navier-Stokes equation solver DAWES, the location of the passage shock relative to the leading edge of the following blade was predicted at a number of fan speeds for the baseline rotor. Figure 5.1.3-4 shows the linear relationship between shock location and wheel speed.

A correlation was developed between the predicted rotor passage shock position relative to the leading edge of the downstream blade and the measured buzzsaw noise in the 500 Hz, 1 kHz and 2 kHz octave bands which contain harmonics of shaft speed below the BPF. In an attempt to capture noise level differences due to buzzsaw cuton, inlet sound pressures (measured from 10-90° from the inlet engine centerline) were summed over the 3 octave bands mentioned above and normalized by the level at 75.3% corrected fan speed (the speed above which MPT noise was cuton). Reference 14 contains measured buzzsaw noise spectra for the baseline.

Figure 5.1.3-5 shows the normalized inlet noise versus passage shock position normalized by the rotor chord. As fan speed increases, the passage shock strength increases (see Figure 5.1.3-6) as it moves closer to the leading edge of the following blade, and buzzsaw noise levels increase. Between 75.3% and 86.5% speed the variation of measured sound pressure level is nearly linear with shock location following the equation

$$SPL = -45.6(x) + 13.0$$

where *SPL* is the increase in level after MPT noise is cuton, and *x* is the shock location divided by the rotor chord. From 86.5% to 90.0% speed, however, the correlation deviates from a linear behavior, showing little increase in noise level with increasing rotor speed and decreasing shock stand-off distance.

At 90.0% speed the shock location, predicted assuming symmetric rotor blades, is very near the leading edge of the downstream blade. It may be that in actuality, with asymmetric blade shapes and stagger angles, the leading edge of the following blade interferes with the propagation of the passage shock to the far-field. From Figure 5.1.3-4 it can be inferred that near 94% speed the passage shock is completely swallowed and buzzsaw noise is cutoff. This is confirmed by the fact that in cruise (100% corrected fan speed) the baseline engine generates no buzzsaw noise (see Figure 5.1.3-2). The transition between conditions of passage shock propagation and swallowing (as corrected speed is increased) may involve some hysterisis effects, but it appears to begin between 86.5% and 90.0% corrected fan speed for the baseline.

5.1.3.3 **OHSF**

As shown in Figure 5.1.3-7, the spanwise distribution of effective sweep causes the passage shock location to vary with span at 100% speed. A strong shock is swallowed at 90% span while a weaker shock is expelled at 70% span. As fan speed is reduced to near takeoff conditions (85% corrected speed, Figure 5.1.3-8), the strong tip shock dissipates leaving only a weak shock at 70% span.

Examination of the shock position at 2 radial locations along the rotor reveals that, at 100% speed and 90% span, the QHSF passage shock is severely swallowed (see Figure 5.1.3-9). The streamwise shock position at 90% span is not recorded for speeds below design because the computed pressure gradient at that location does not suggest the presence of a shock, but merely a gradual pressure rise (see Figure 5.1.3-8). At 70% span, the QHSF passage shock is expelled, but the strength of this shock is much less than that of the baseline at 75% speed where buzzsaw noise is nearly cutoff (see Figure 5.1.3-10). The shock at 90% span is relatively strong, but its position near the streamwise center of the rotor passage suggests buzzsaw noise cutoff.

The main objectives have been achieved to reduce buzzsaw noise: passage shock strength has been greatly reduced from that of the baseline rotor and strong shocks are swallowed within the rotor passage. Buzzsaw noise levels are predicted to be much lower for the QHSF than for the baseline, if not completely eliminated.

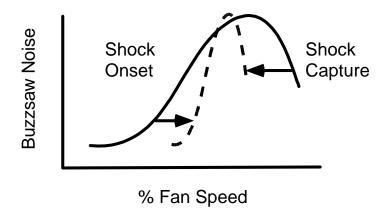


Figure 5.1.3-1. Potential Buzzsaw Noise Control Accomplished Using Forward Sweep

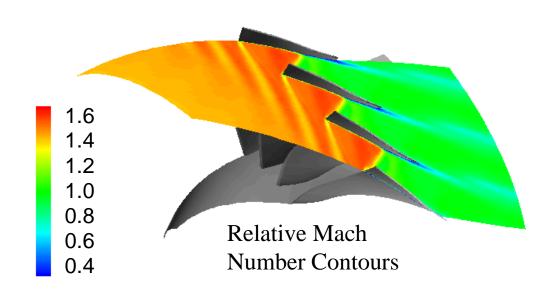


Figure 5.1.3-2. Passage Shock Position for Baseline at Design Point Showing Swallowed Shock

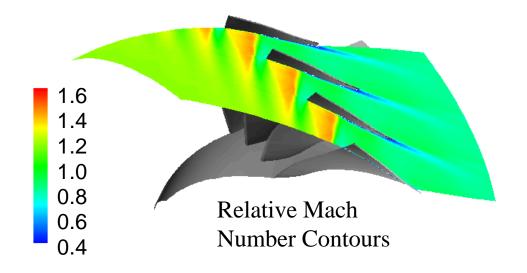


Figure 5.1.3-3. Passage Shock Position for Baseline at Take-Off (86.5% N1c)

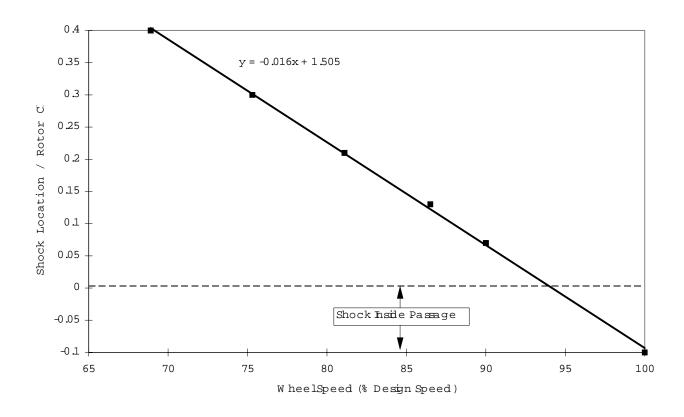


Figure 5.1.3-4. Relationship Between Shock Location and Fan Rotational Speed for Baseline

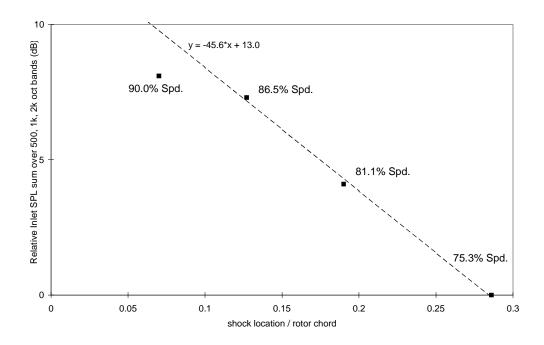


Figure 5.1.3-5. Correlation Between Passage Shock Position and Buzzsaw Noise For Baseline

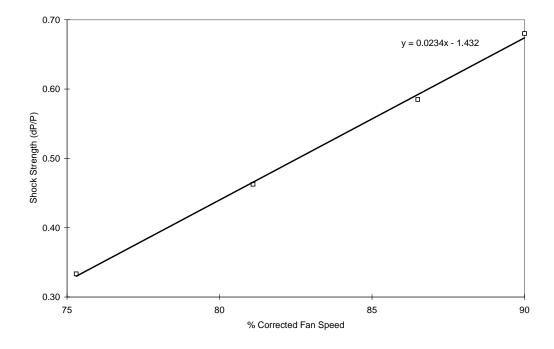


Figure 5.1.3-6. Predicted Strengths of Expelled Passage Shocks at the Rotor Corrected Speeds For Which Buzzsaw Noise Was Measured in the Baseline (strengths extracted from pressure distribution along stagnation streamline)

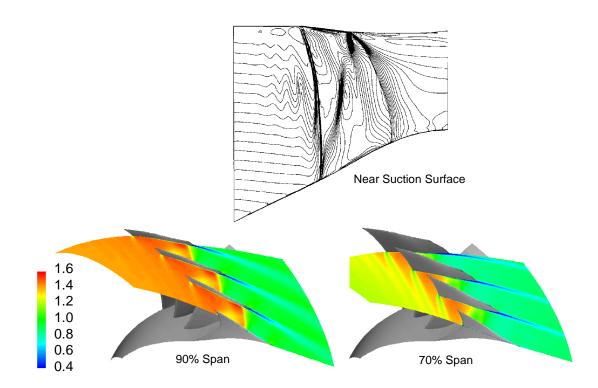


Figure 5.1.3-7. Mach Number Contours for QHSF at 100% Speed

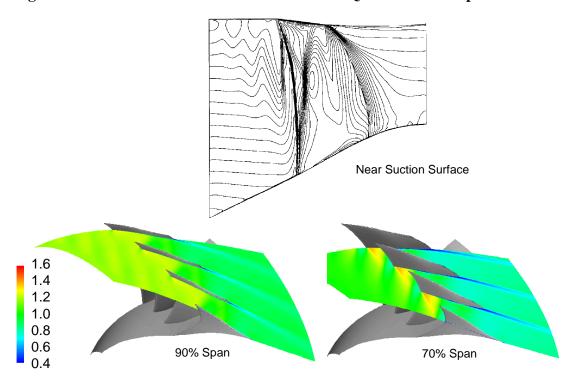


Figure 5.1.3-8. Mach Number Contours for QHSF at 85% Speed

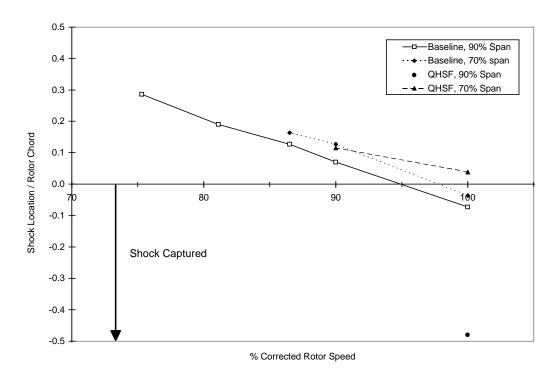


Figure 5.1.3-9. Comparison of Shock Locations for Baseline and QHSF

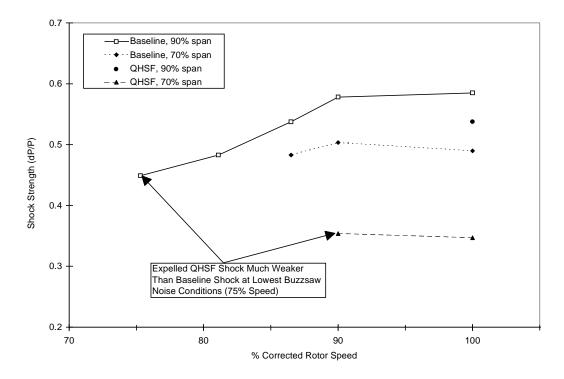


Figure 5.1.3-10. Comparison of Shock Strengths for Baseline and QHSF (strengths extracted from pressure distribution along 50% pitch streamline)

5.1.4 Mechanical Results

This section presents the results of the Mechanical Analysis of the QHSF Fan Rotor Blade.

5.1.4.1 Design Criteria

Mechanical design criteria were selected to satisfy both AE and NASA requirements. While the NASA criteria primarily addresses rig safety, the more stringent AE criteria is intended to ensure long-term field reliability. The QHSF blade was designed to achieve state-of-the-art noise reduction while maintaining commercial viability.

- Peak steady-state stresses in the blade and attachment are at levels providing unlimited Low-Cycle Fatigue (LCF) life.
- Adequate frequency margin exists to preclude resonant vibration at maximum power conditions.
- Leading edge thickness and stress levels are within range of successful AE experience.
- Flutter parameters involving reduced frequency and incidence are within AE experience.

5.1.4.2 Finite-Element Model Description

A fully 3D finite element model was constructed of the entire rig scale (22-inch diameter) fan rotor, including the airfoil, blade attachment, and disk. The complete model is shown in Figure 5.1.4-1. The airfoil and blade attachment were modeled separately and connected via an AE multi-point constraint equation technique. Due to cyclic symmetry, the disk model consisted of a single, slotted wedge. Figure 5.1.4-2 shows the detail of the blade model; the disk model will described in a later section.

The QHSF fan airfoil has roughly the same geometric parameters, such as chord, hub/tip ratio, etc., as an existing AE fan. To reduce design costs, the platform/ blade attachment from the AE fan was used for the QHSF rotor. Optimization of the circumferential position of the airfoil on the platform was done to minimize the peak stress in the attachment.

For computational efficiency, the disk and attachment were modeled as linear substructures. Blade to disk interaction at the dovetail was modeled using 3D contact surface elements.

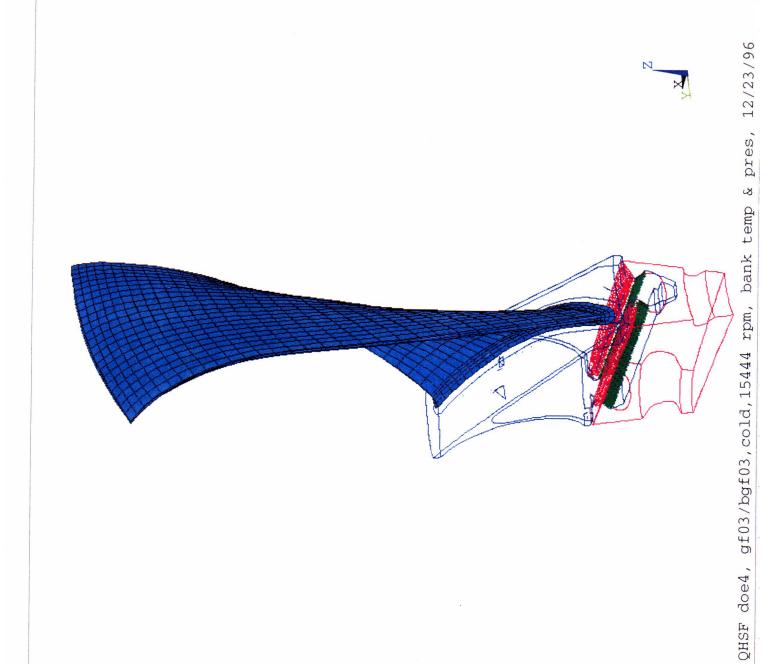


Figure 5.1.4-1.

and Disk in Rig Scale.

Three Dimensional Finite-Element Model of the Blade, Blade Attachment,

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ANSYS 5.3 FEB 6 1997 08:12:17 PLOT NO. 1 ELEMENTS TYPE NUM XV = .8334 YV = .8286 ZV = .1615 DIST=4.826 XF = .297485 YF = .692017 ZF = .692017 ZF = .692017 ZF = .297485 YF = .692017 ZF = .297485

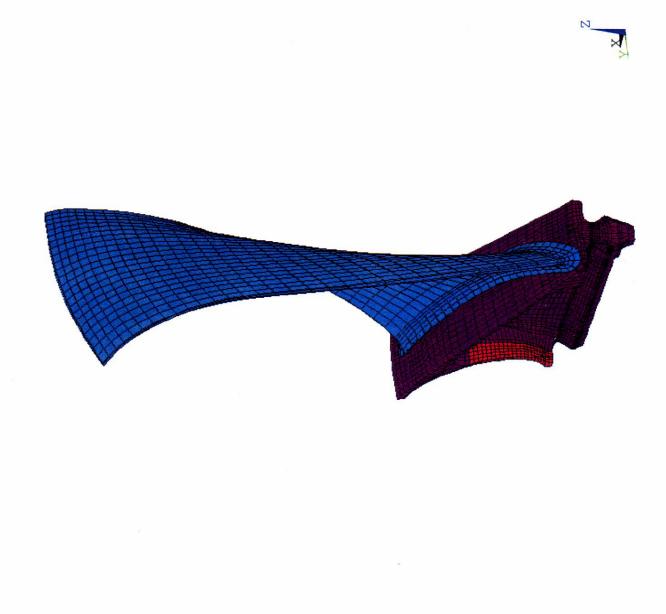


Figure 5.1.4-2. Detail of the Three-Dimensional Finite-Element Model of the Blade.

Material Selection and Properties

The blade material was chosen to be Ti-6Al-4V STOA with the nominal properties summarized in Table 5.4.1-1.

Table 5.1.4-1 Ti-6AL-4V STOA Mechanical Properties

Property	75F	250F
Modulus, Mpsi	17.5	17.1
Weight Density, lb/in ³	0.161	0.161
0.2% Yield Strength, ksi	137.7	111.4
Ultimate Strength, ksi	146.2	127.3

Boundary Conditions

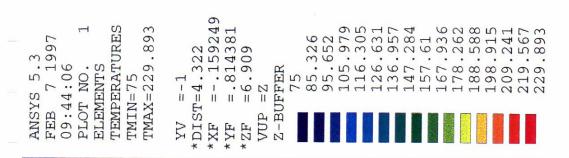
A physical speed of 15444 rpm was applied to the rotor along the x-axis. The cut wedge faces were coupled together cylindrically to enforce cyclic symmetry. Axial (thrust) and tangential (torque) loads were reacted on the forward annular face representing the disk bolt flange.

Figure 5.1.4-3 shows metal temperatures corresponding to the aerodynamic design point as mapped to the airfoil model. A uniform metal temperature of 75 degrees F was applied to the platform/attachment and 85 degrees F was applied to the disk to simulate operating conditions. Design point suction and pressure-side static pressure distributions were mapped to the corresponding element faces on the airfoil (Figure 5.1.4-4).

5.1.4.3 Static Analysis Results

As constructed, the airfoil model represented the at-speed, design point geometry ("hot" shape). The first phase of the static analysis is to calculate the manufactured shape ("cold"), that will, under design point conditions of speed, temperature, and pressure, deflect the blade into the desired "hot" shape. This transformation was done using an iterative procedure within ANSYS where the maximum error between the deflected cold geometry and the desired hot geometry is reduced to less than 0.001 inch. For undampered fans of this type a non-linear, large-deflection analysis is required.

At the conclusion of the hot-to-cold analysis the nonlinear, steady-state stresses and deflections are available. Figure 5.1.4-5 shows the maximum principal stress distribution on the pressure and suction sides of the airfoil. Figure 5.1.4-6 shows the Equivalent (Von Mises) stress distribution.



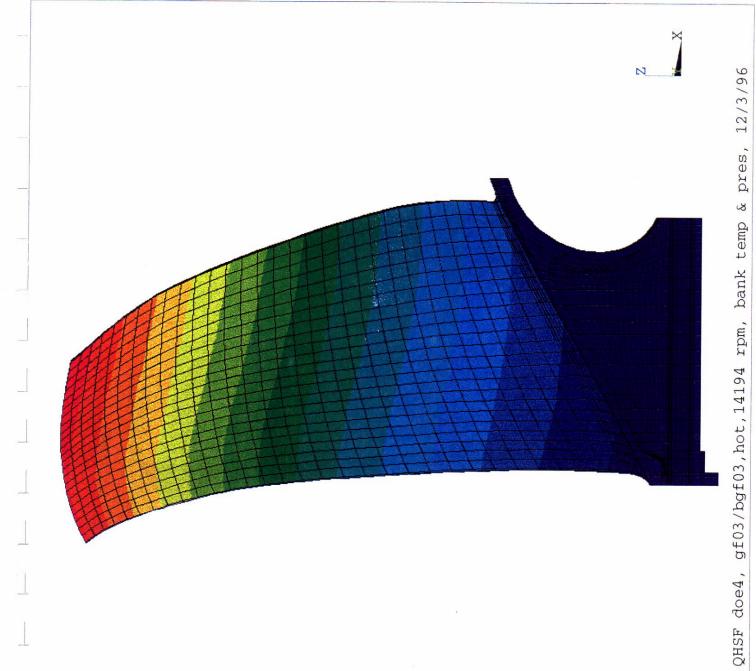
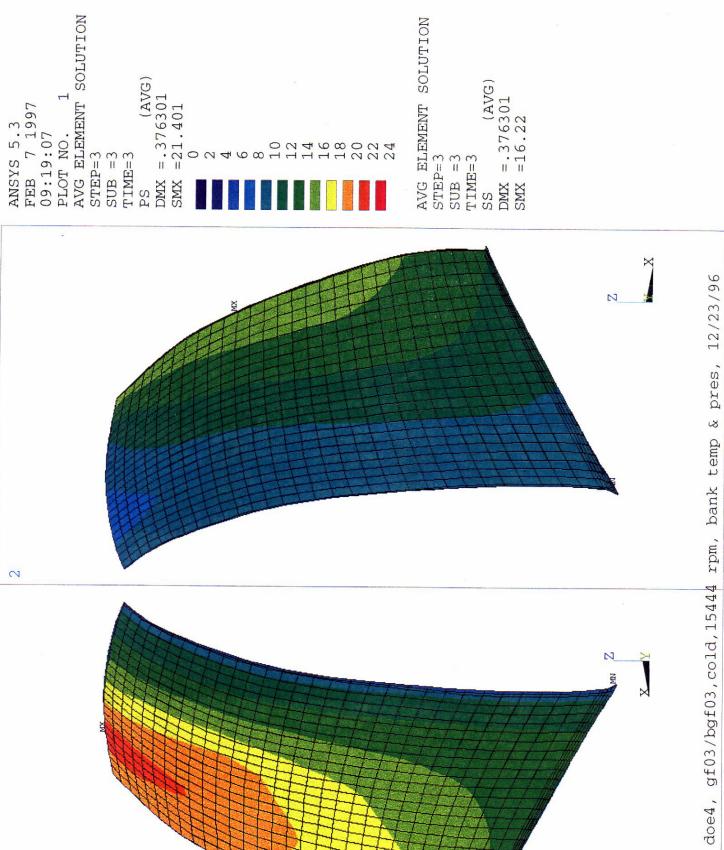
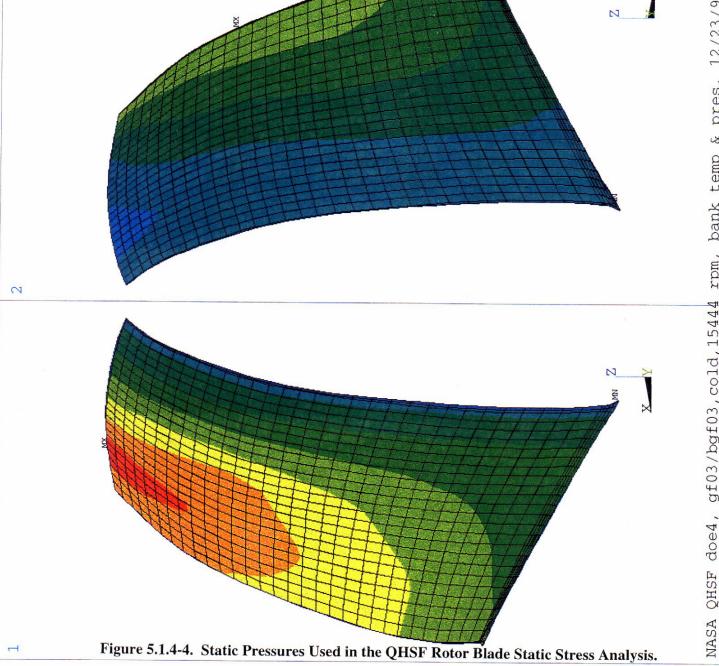
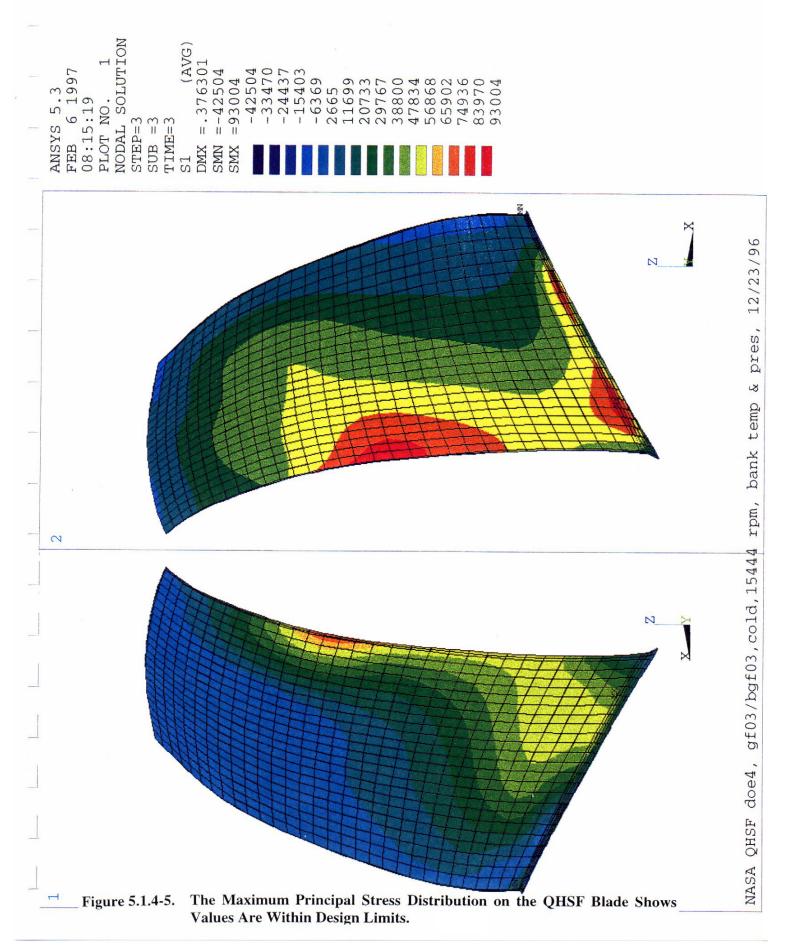


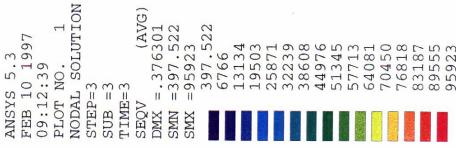
Figure 5.1.4-3. Metal Temperatures Used in the QHSF Rotor Blade Static Stress Analysis.

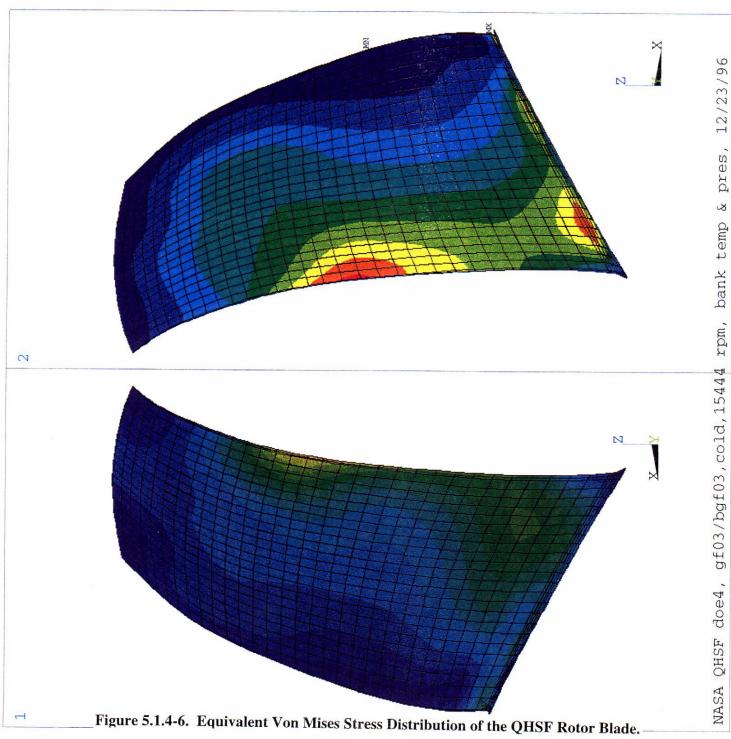
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Airfoil deflections at the Design point are shown next. Figure 5.1.4-7 shows the true-scale displaced shape of the airfoil (shown as element grid) along with the undeformed, cold geometry (shown in outline) as viewed along the tip chord. Quantitative plots of airfoil deflection are presented in the next four figures. Figure 5.1.4-8 shows the vector sum (scalar) displacement of the airfoil. The maximum airfoil deflection was predicted to be 0.376 inch.

Radial displacements (positive outward from center) are shown in Figure 5.1.4-9. A maximum radial deflection of 0.051 inch occurred at the leading-edge. Figure 5.1.4-10 shows the airfoil tangential deflection (positive toward pressure-side) and Figure 5.1.4-11 shows the axial (positive forward) component.

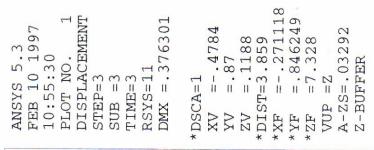
The substructure displacements at the airfoil root and the disk contact faces were used to expand the attachment stress and displacement solutions. Pressure and suction side attachment principal and equivalent stress distributions are shown in Figures 5.1.4-12 & 5.1.4-13 respectively. Figure 5.1.4-14 shows the displacement contours for the attachment and platform.

Inspection of the principal stress plot (Figure 5.1.4-12) and the displacement plot (Figure 5.1.4-14) showed a significant amount of bending in the attachment due to the large tangential sweep of the blade. Stress levels, however, were well within acceptable range, and the effect of the bending displacement was accounted for in the cold blade geometry.

5.1.4.4 Modal Analysis Results

Modal analysis was performed on the QHSF blade using the same model as was used for the static analysis (Figure 5.1.4-2). The effect of the disk was ignored based on the relative rigidity of the disk combined with past experience with similar designs.

All nodes on the contact faces of the blade dovetail were fixed in the three translational degrees-of-freedom (DOF). To define the Campbell diagram two analyses were performed. The first was done assuming uniform room-temperature, with no pressure loading, and no rotational speed. The second analysis assumed ADP metal temperatures (Figure 5.1.4-3), static pressures (Figure 5.1.4-4), and a rotational speed of 15444 rpm. Natural frequencies for the first five modes are tabulated in Table 5.1.4-2. Also included are the frequencies for the fixed-root condition (attachment effects neglected) under identical conditions.



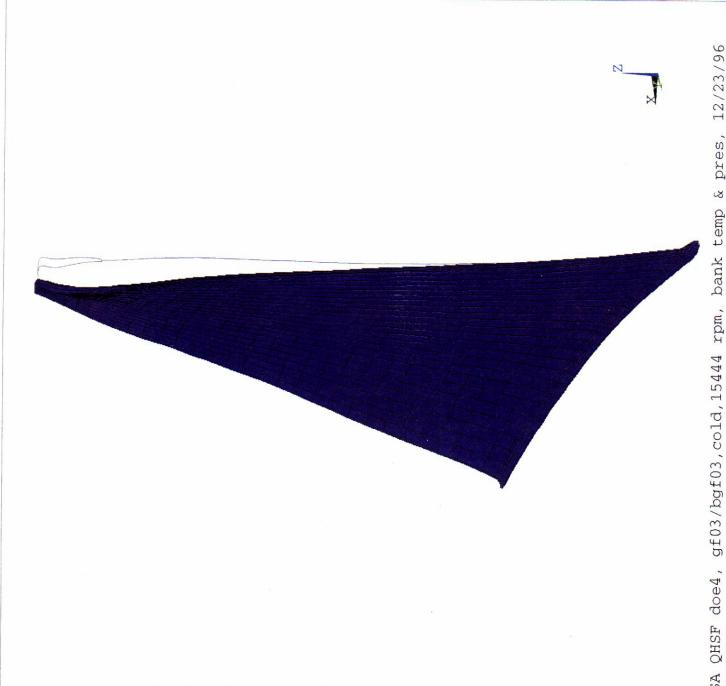
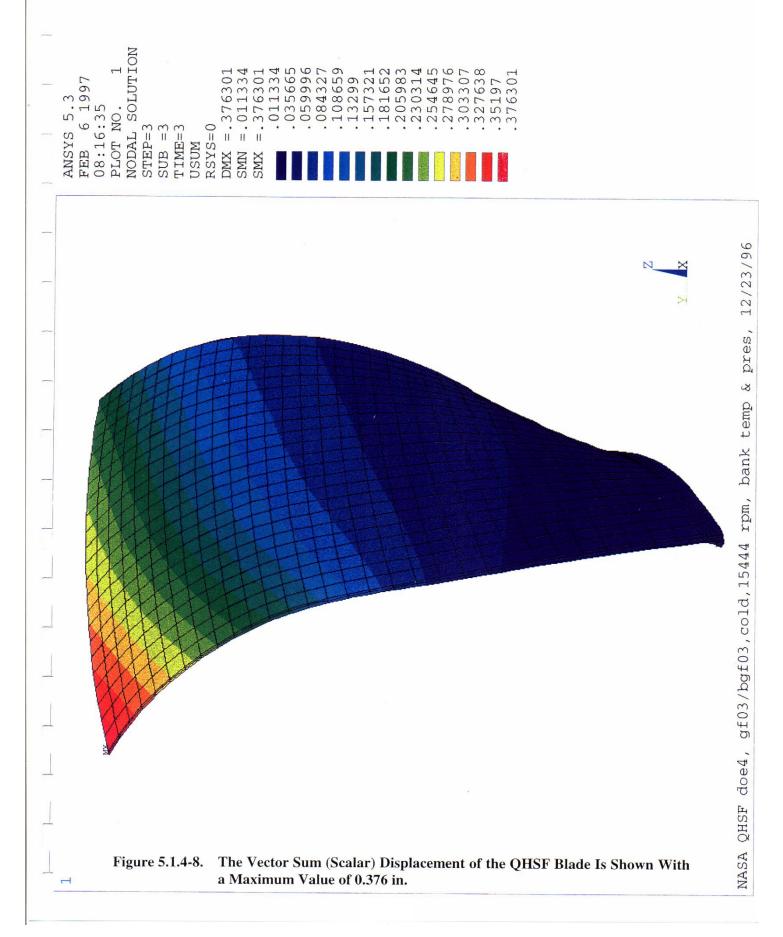
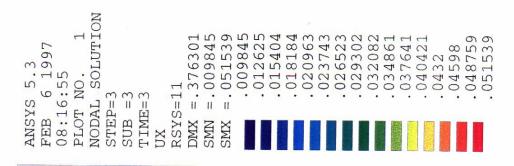
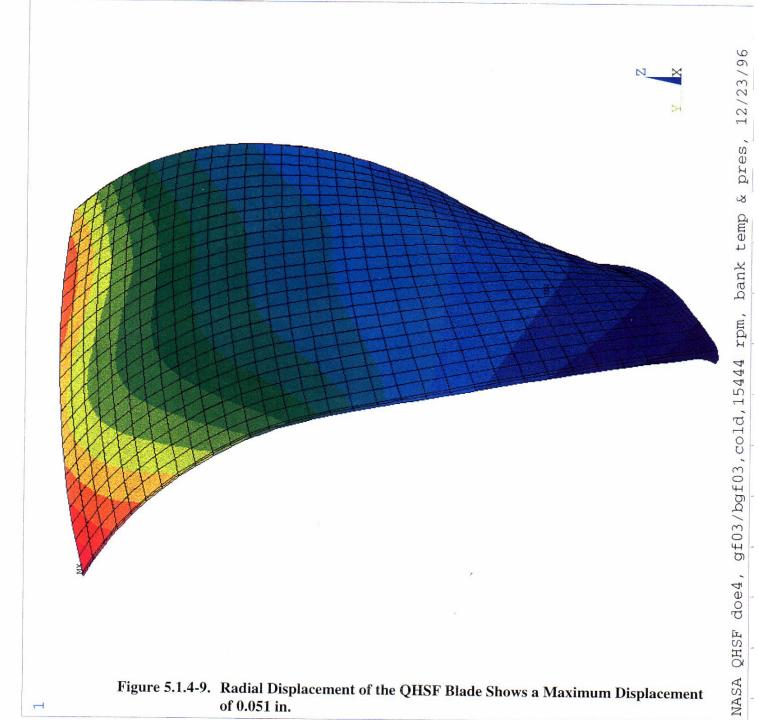


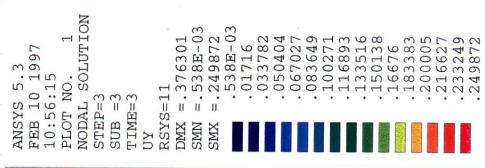
Figure 5.1.4-7. The True-Scale Displaced Shape of the Blade Is Compared With the Cold Geometry.

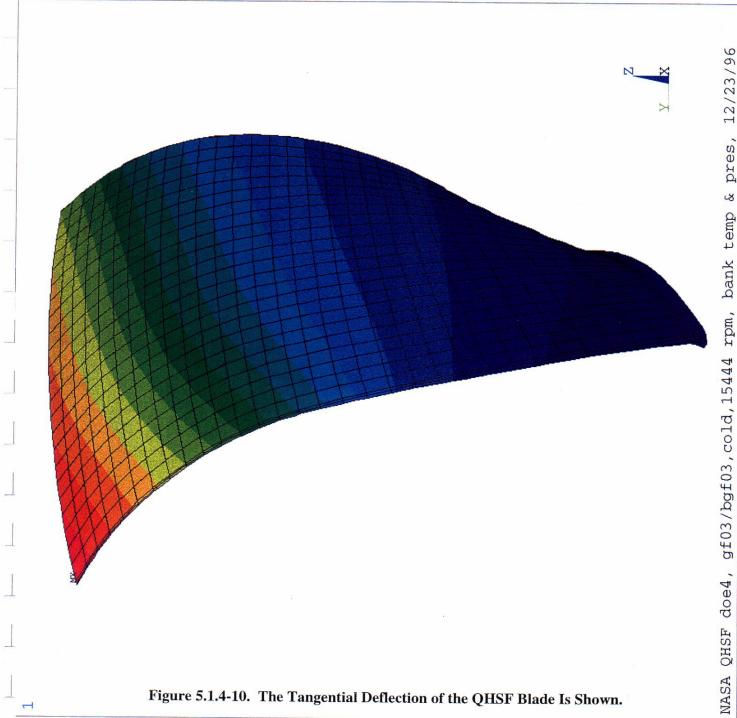


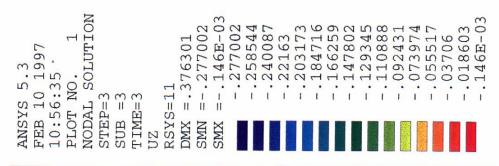


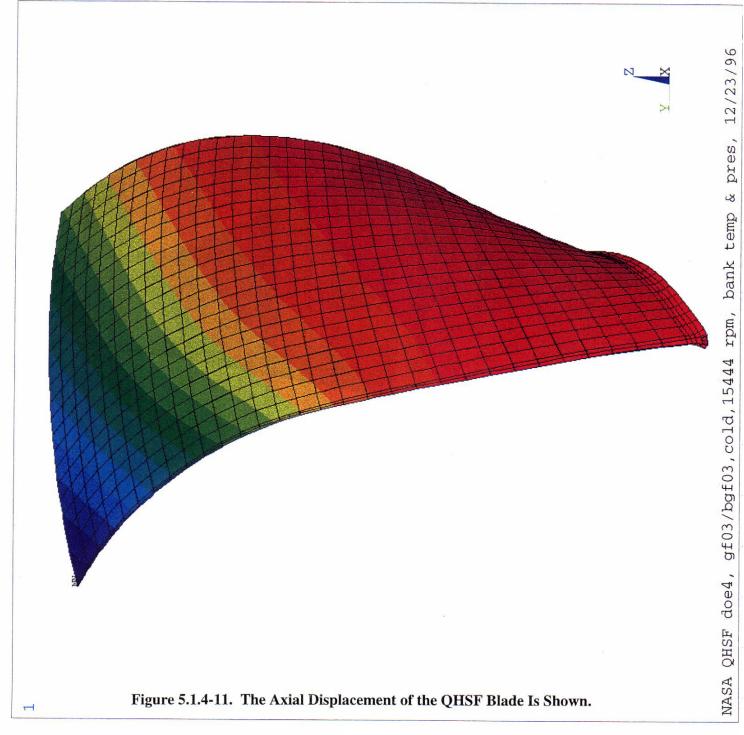


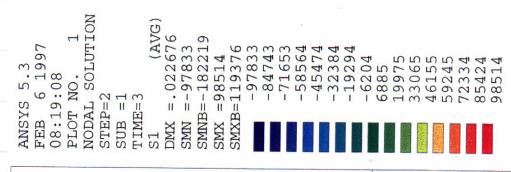
of 0.051 in.

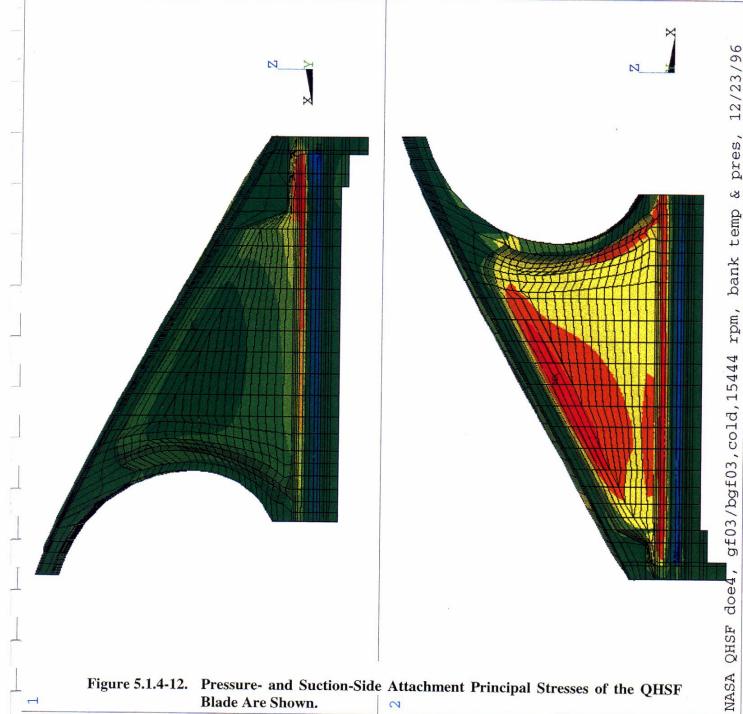


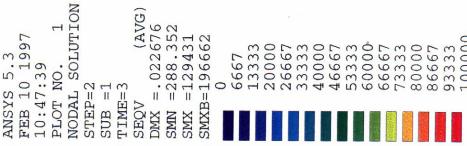


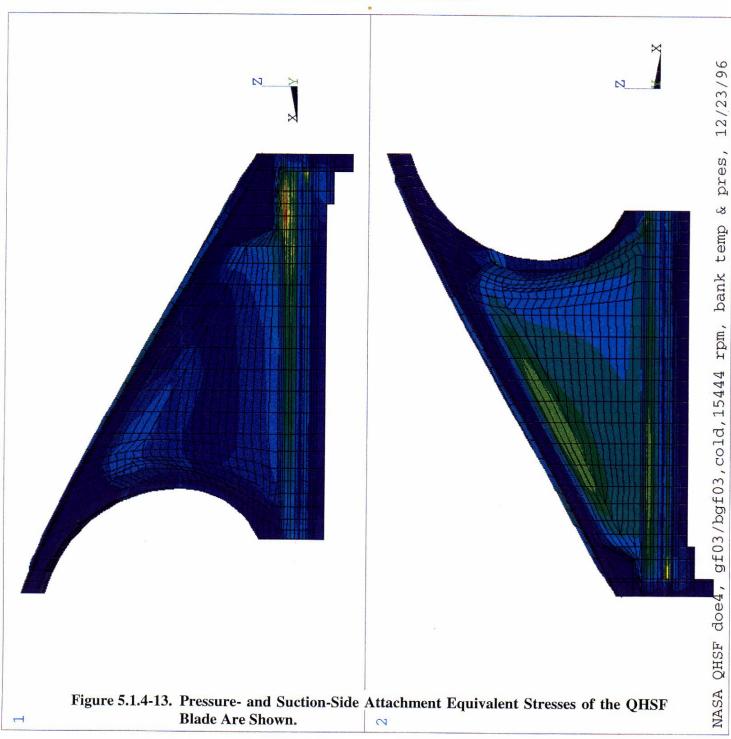


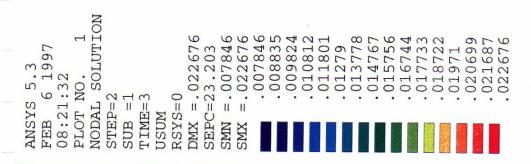












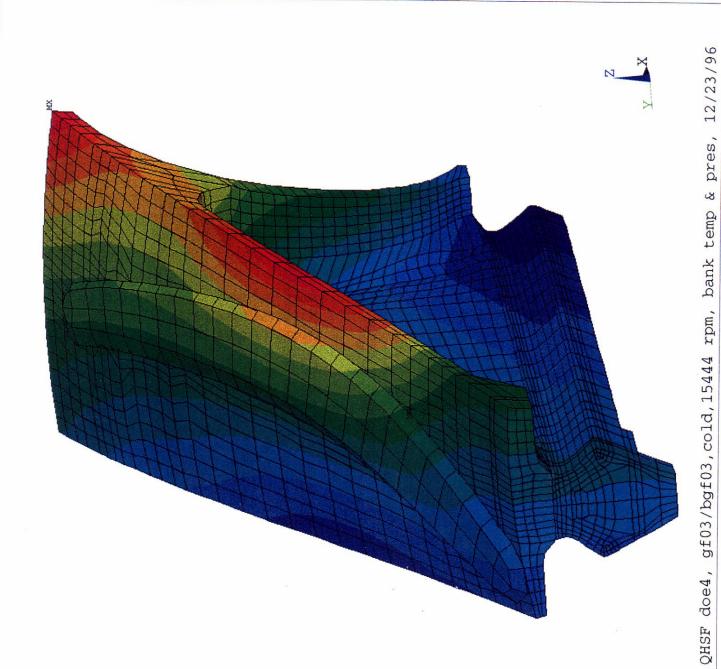


Figure 5.1.4-14. Displacement Contours of the Attachment and Platform for the QHSF Blade Are Shown.

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Table 5.1.4-2 QHSF Vibration Results - Natural Frequencies

	Mode					
Conditions	1	2	3	4	5	
Room temp., 0 rpm	172	520	879	1136	1521	
ADP temp, 15444 rpm	349	704	984	1485	1568	
Fixed-Root, 0 rpm	177	599	879	1333	1556	
Fixed Root, 15444 rpm	373	791	1020	1600	1682	

The resulting Campbell diagrams are shown in Figure 5.1.4-15 along with the first six excitation orders. Vertical lines highlight the design speed of 15444 rpm as well as the maximum test speed of 110 percent. Design speed frequency margin with respect to the nearest excitation order for the first five modes is shown in Table 5.1.4-3.

Table 5.1.4-3 QHSF Frequency Margin Summary

	Mode/Engine Order					
Conditions	1/2E	2/3E	3/4E	4/6E	5/6E	
ADP temp, 15444 rpm	58%	15%	5%	6%	2%	

As the Campbell diagrams illustrate, a significant frequency shift was experienced for modes 2 and 4 after adding the attachment to the model. The shift in mode 2 frequency of 11 percent was especially relevant as it moved the 3E crossing to within the operating range.

Figures 5.1.4-16 through 5.1.4-20 show contour plots of the first 5 modes at design point conditions with mode shapes normalized to unit generalized mass. Normalized vibratory stress (maximum absolute value principal stress) contours for each mode are shown in Figures 5.1.4-21 through 5.1.4-25.

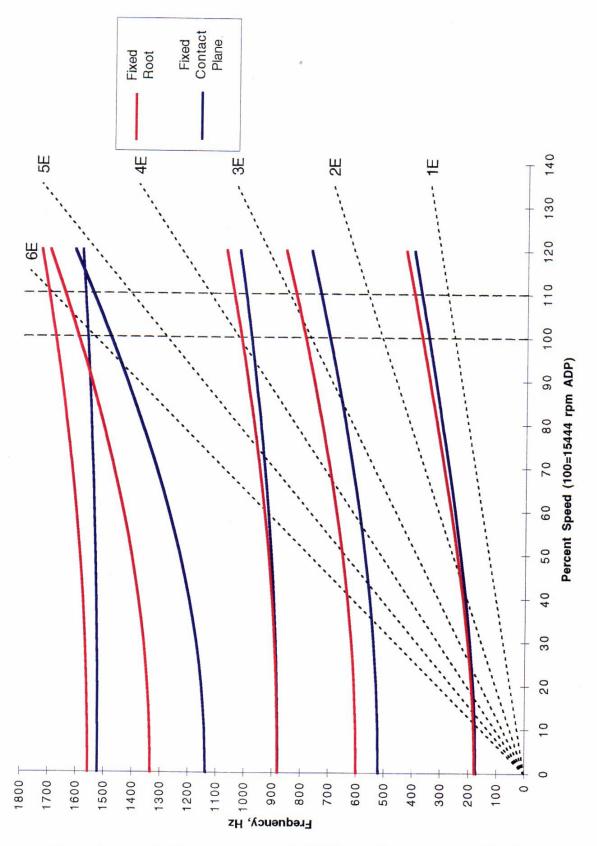
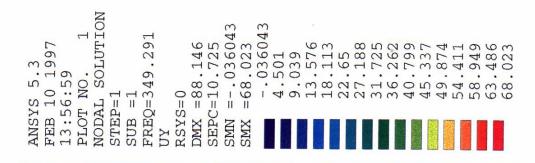
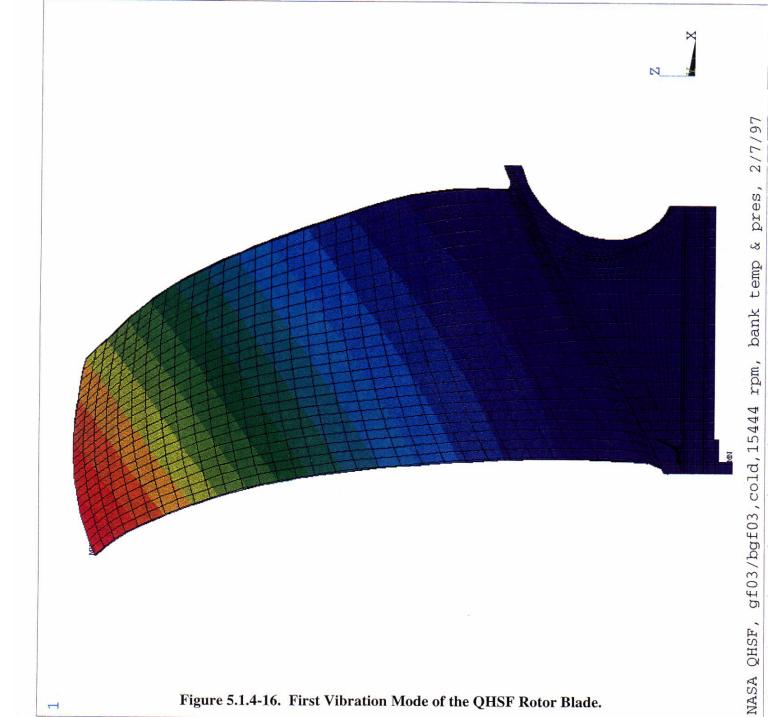
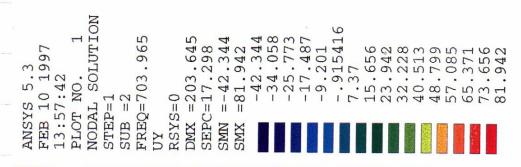
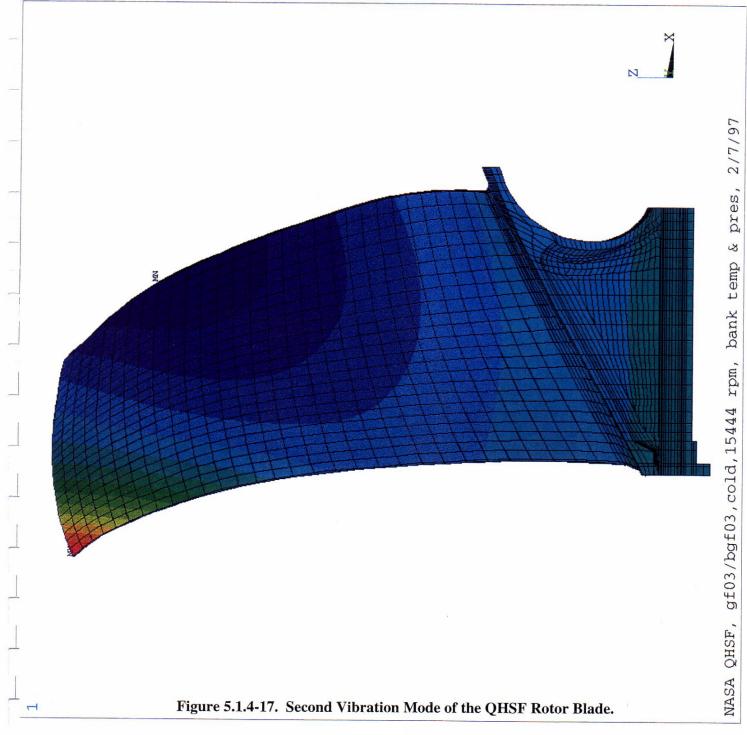


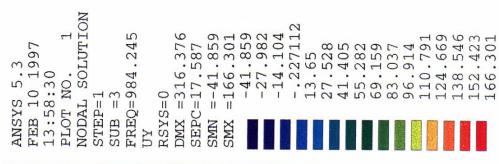
Figure 5.1.4-15. Cambell Diagram for the QHSF Rotor Blade Shows the Mode 3 Crossing in the Fan Operating Range.

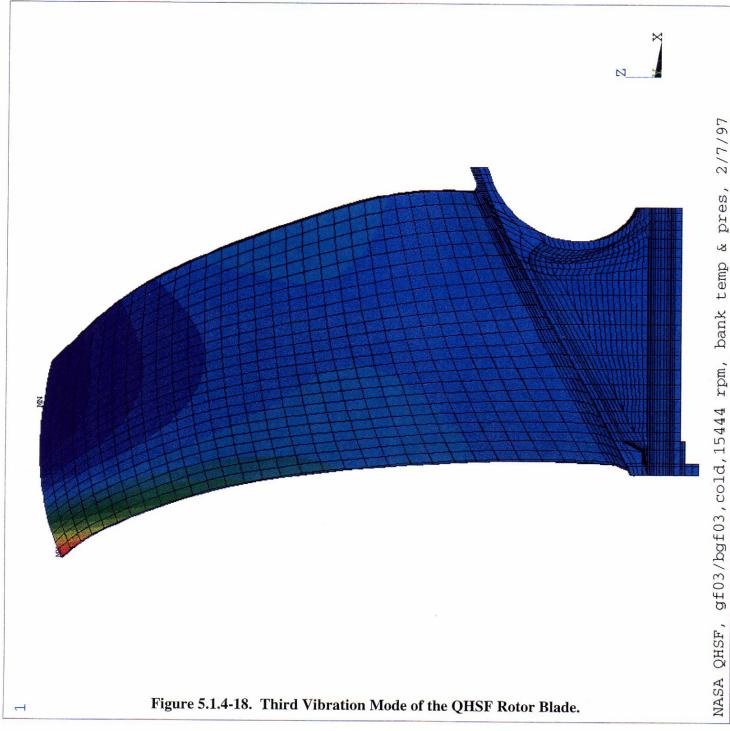


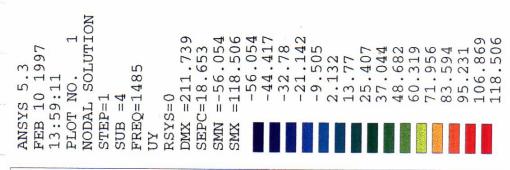


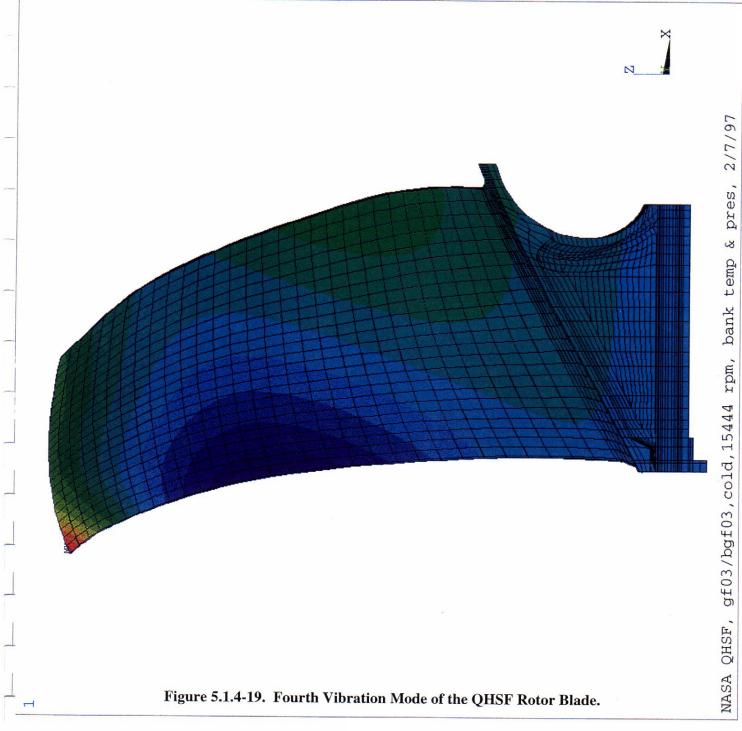


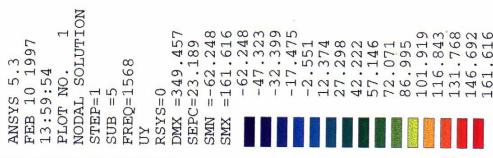


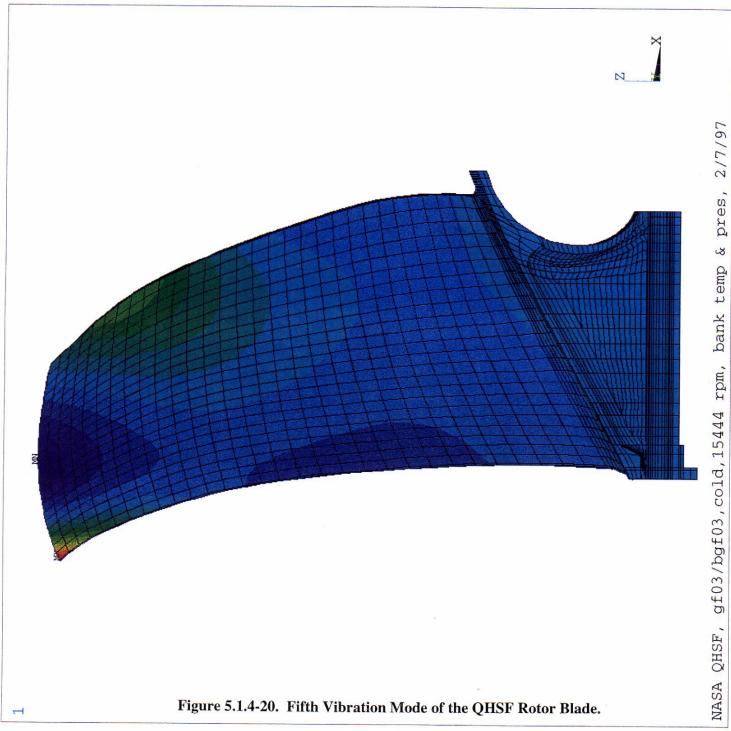


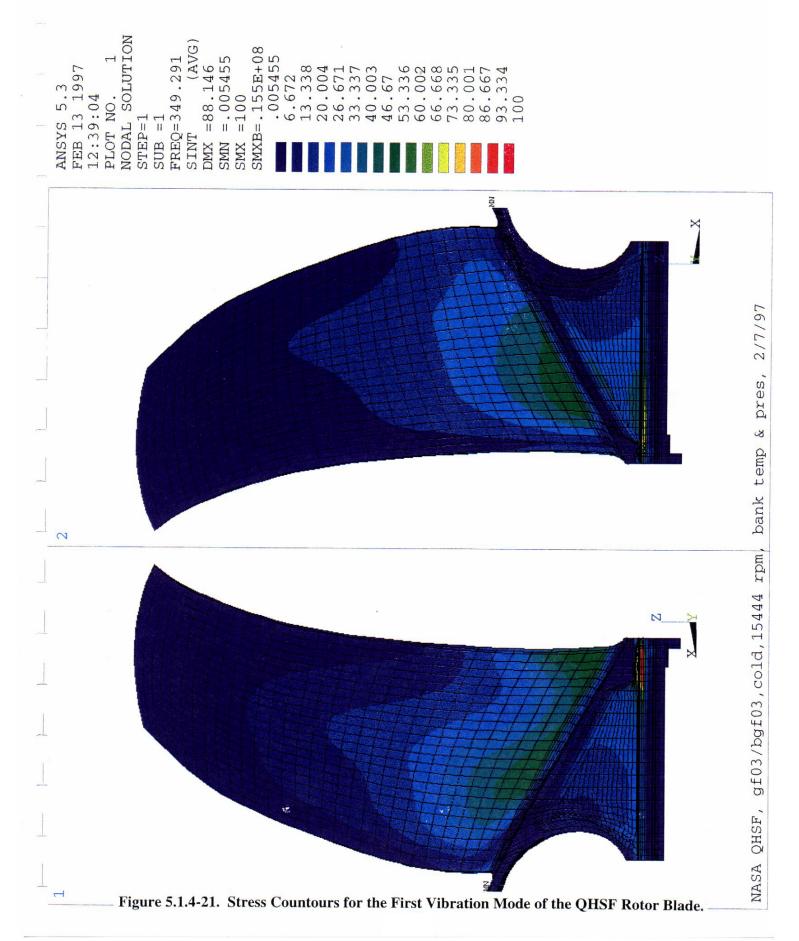


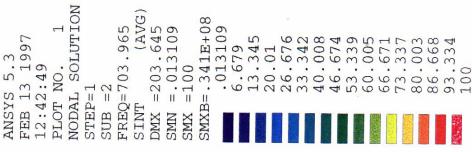


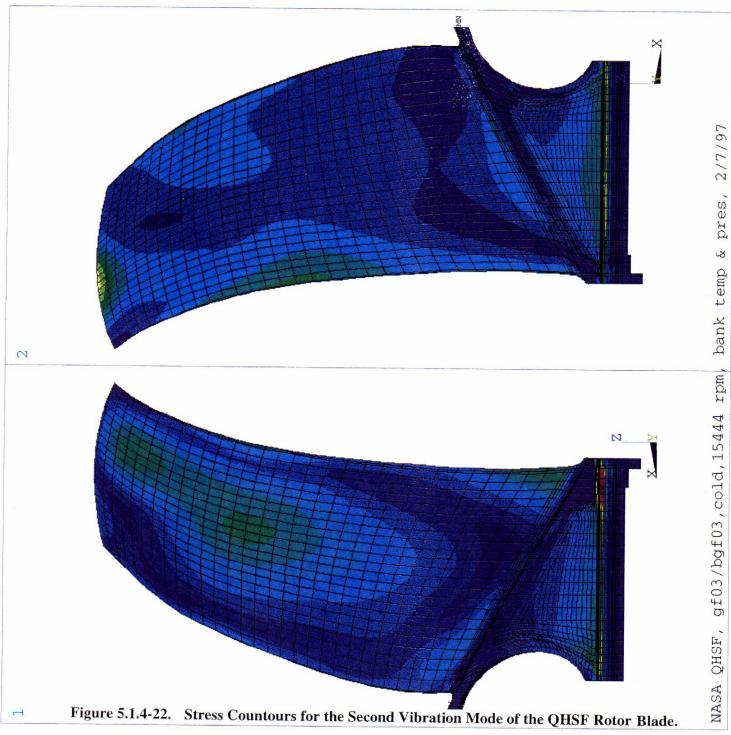


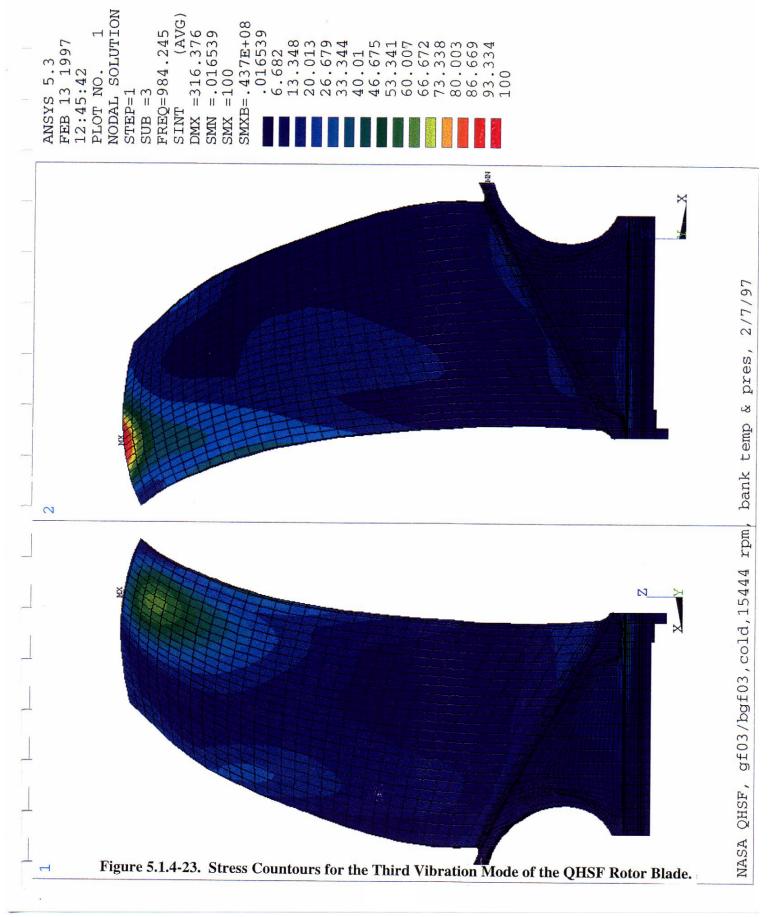


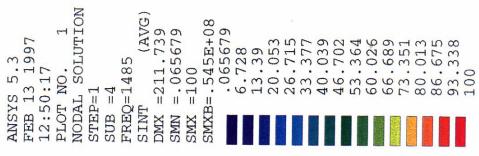


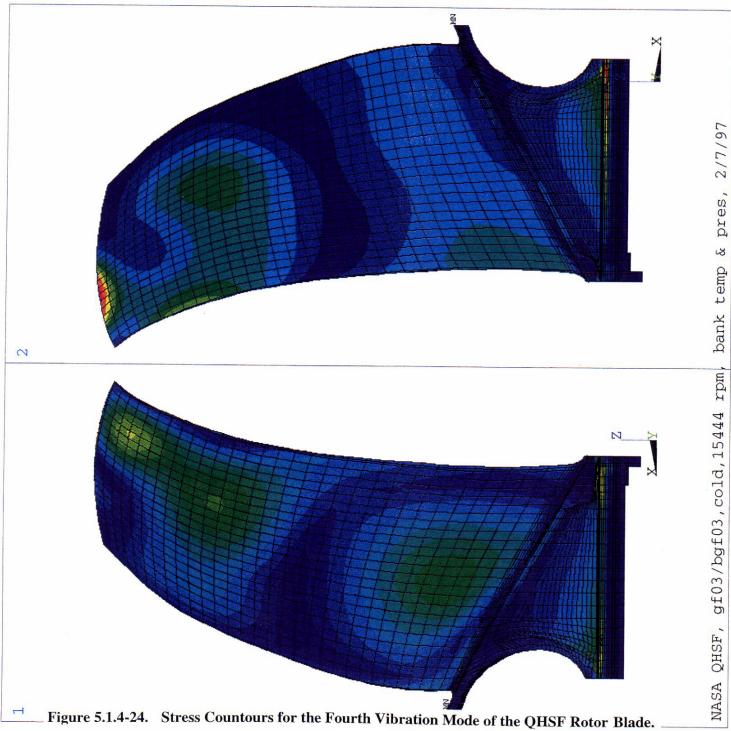


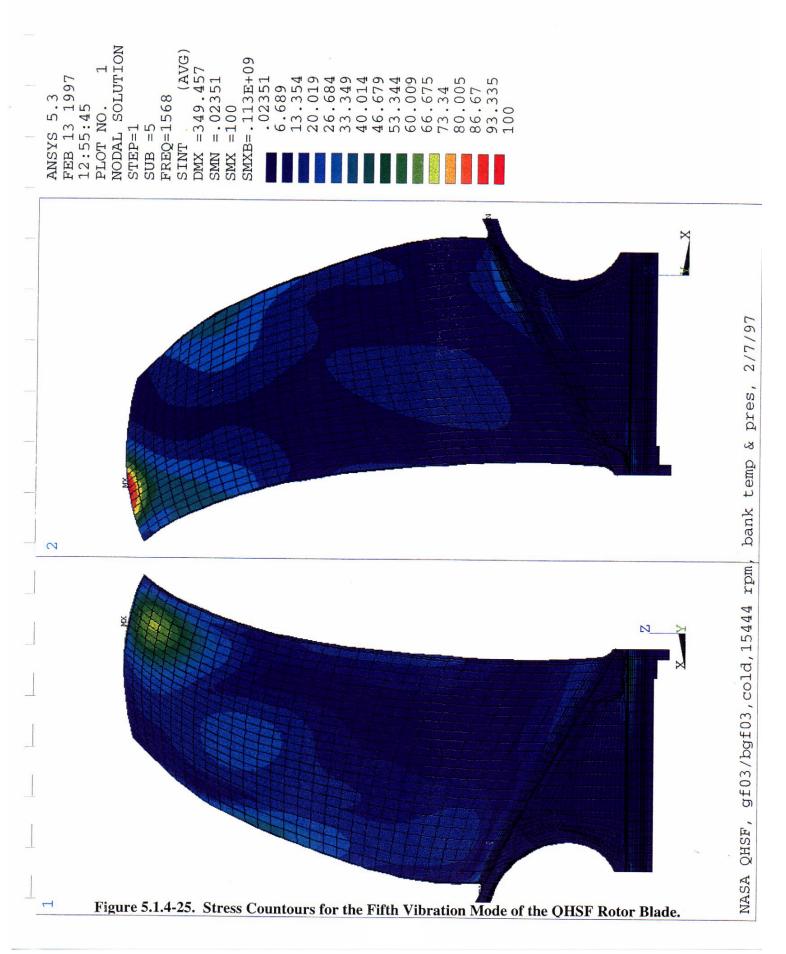












5.1.4.5 Flutter Parameters

AE criteria for flutter is based on reduced frequency (fc/V), where f is the blade natural frequency, c is the chord at 75 % span, and V is the design-point relative velocity at 75 % span. The twist-to-flex ratio is used to characterize the mode as being flexure-dominate (small ratio) or torsion-dominant (large ratio).

The reduced frequency flutter parameter calculations are shown in Table 5.1.4-4.

Parameter Twist/Flex Mode f (Hz) c (in) V (ft/sec) fc/V 75% 95% 349 4.31 1435 0.45 0.37 0.6 2 704 4.31 4.8 1.1 1435 1.1 3 984 4.31 1.3 1435 1.6 1.8 4 1485 4.31 1435 2.3 1.2 0.62 5 1568 4.31 1435 2.5 0.82 0.42

Table 5.1.4-4 QHSF Flutter Parameter Summary

As can be seen in the above table, mode 1 was predominantly flexure, and the reduced frequency of 0.6 exceeded AE's minimum design criteria for bending mode. Modes 2 and 3 both displayed significant torsional activity hence AE's minimum design criteria for torsion mode was applied. Mode 2 appeared to be moderately aggressive, due in part to the frequency shift associated with the attachment.

Dynamic response measurements, either with strain-gages or by other means, are highly recommended to ensure flutter free operation.

5.1.4.6 Bird Ingestion

A preliminary Foreign-Object-Damage (FOD) assessment performed on the QHSF blade indicated leading-edge thickness parameters to be well within successful AE experience. This analysis is comparative in nature and considers only spanwise geometric characteristics, LE thickness distribution, blade count, metal angles, bird weight, as well as other parameters. The computed damage tolerance factors are compared with a database consisting of very successful, marginal, and poor designs. This preliminary analysis however, can not account for the high degree of forward sweep and tangential lean built into the QHSF, which will be accounted for using NOSAPM.

A more accurate prediction of ingestion damage was obtained using the NOSAPM code developed for the Air Force Wright Aeronautical Laboratories (Reference 15). NOSAPM can perform an inelastic, transient, impact response analysis on the rotating blade, and predict with good success, the resulting permanent deformations of the airfoil. A key characteristic of the code is a loading model which allows an interactive determination of the pressure distribution on the impacted blade based on the blade's instantaneous, deflected shape. With simple user input defining the geometry and initial conditions, i.e., rotor rpm, bird weight and trajectory, etc., the program solves for the blade transient response.

Figure 5.1.4-26 shows the impact model in detail. Since the impact analysis is nonlinear, the model was constructed using the "cold" geometry as described above. The shaded region represents the predefined set of impacted elements centered upon the designated impact radius.

As certification requirements vary with the inlet area of the fan, this analysis was performed at baseline fan scale. The bird weight, impact radius, as well as the rotor rotational speed were chosen to match conditions defined for a FAA certification test of the baseline fan. These conditions were defined as follows:

Rotor Speed 88.8%
Bird Weight 1.5 pounds
Aircraft Forward Speed 138 knots
Impact Radius 56% span

Transient response of the blade due to the bird impact is illustrated in Figure 5.1.4-27 with time histories of the tip leading and trailing edges. The maximum displacement of approximately 3.1 inches occurs at t=0.011 seconds at the tip trailing edge. The maximum displacement is overlayed on the undeformed blade in Figure 5.1.4-28. The undeformed blade shown includes the effects of rotational speed.

At the conclusion of the transient (t=0.053 seconds) the permanent deformations are readily visible. Figure 5.1.4-29 shows the deformation contours overlayed on the undeformed blade. Damage appears to be localized to the leading edge tip where the maximum deformation is 0.36 inches. The remainder of the blade appears to have been unaffected by the impact. Figure 5.1.4-30 presents another view of the deformation superimposed on the undeformed geometry.

An identical analysis performed on the baseline blade revealed contrasting results. Instead of localized deformation like the QHSF, the baseline blade exhibited an overall gross, albeit very small, torsional deformation (restagger).

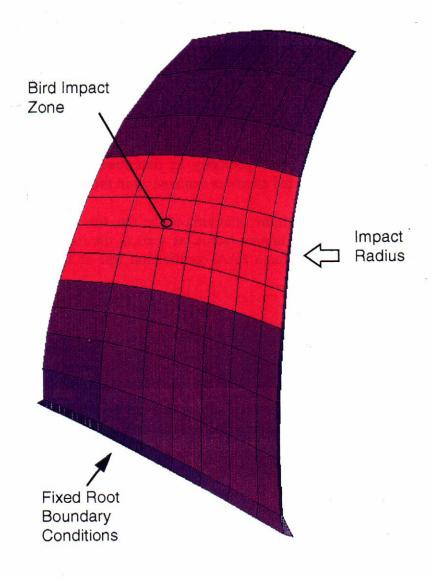
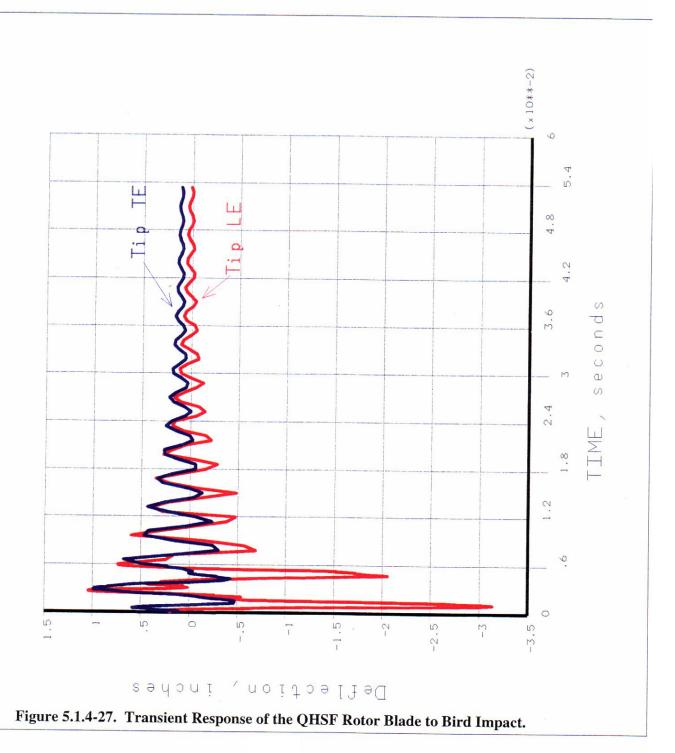


Figure 5.1.4-26. Impact Model for Assessing the Bird Strike Capability of the QHSF Rotor.

ANSYS 5.3 FEB 14 1997 09:40:07 PLOT NO. 1 DATA ZV =1 DIST=.75 XF =.5 YF =.5 Z-BUFFER



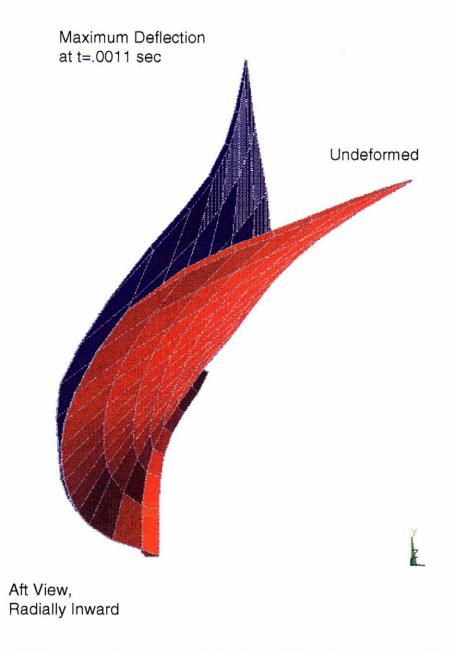


Figure 5.1.4-28. Maximum Displacement of the QHSF Rotor Blade Due to Bird Impact.

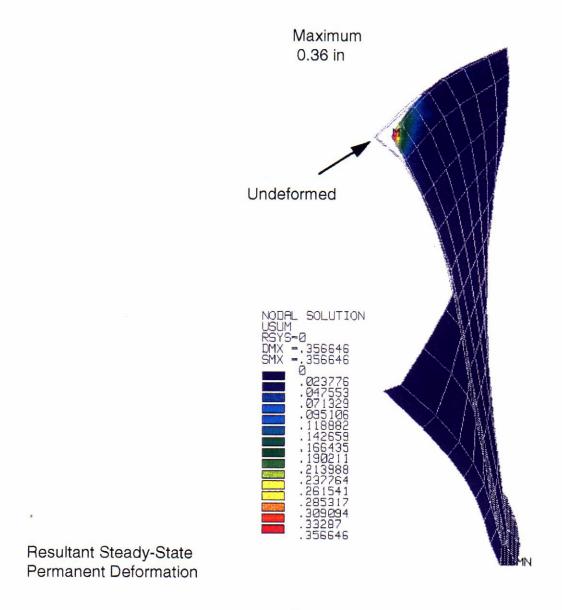


Figure 5.1.4-29. Permanent Deformation of the QHSF Rotor Blade After Bird Strike Compared to the Undeformed Blade.



Resultant Steady-State Permanent Deformation

Figure 5.1.4-30. Tip View of Permanent Deformation of the QHSF Rotor Blade After Bird Strike Compared With the Undeformed Blade.

5.2 Stator

The analytical tools used in the stator aerodynamic design and analysis were the AXCAPS streamline curvature code and the DAWES 3D viscous CFD code. AXCAPS was used to perform a 2D streamline solution, calculating velocity triangles based on user-specified blade row performance profiles and generating airfoil section coordinates for use in mechanical and CFD analysis. These airfoil sections are generated according to user specifications of chord, leading and trailing edge angles (or incidence and deviation), mean camber line angles, stacking, and thickness. In this design, AXCAPS was used mainly as a geometry generator since the extreme vane stacking was felt to make some of the 2D calculations questionable. An example is the disagreement in leading edge swirl angle seen between AXCAPS and DAWES in the vane stacking FF 2.

The rotor used in the stator design AXCAPS model changed as the stator design proceeded. The stator stacking DOE was done using the "optc" rotor which was the most current available at the time. The rest of the design was done using the final "gf03" rotor. The rotor exit velocity triangles which set the stator inlet velocity triangles were taken from the rotor DAWES solution at the design point. Some profile smoothing and adjustment was needed near the endwalls since the AXCAPS streamline curvature code cannot impose a no-slip condition at flowpath and airfoil surfaces. The flow used in the stator design AXCAPS cases was the flow predicted by the rotor DAWES model.

The actual stator performance calculations used during the stator design were performed with the DAWES code. The DAWES calculation grid used for this stator was as similar as possible to the grid used in the baseline stator DAWES model, with 41 nodes pitchwise, 71 nodes spanwise, and 121 nodes streamwise.

The model was run through enough timesteps to reach acceptable convergence. One of the boundary conditions in the DAWES model is the exit static pressure which, if the inlet boundary conditions are held constant, sets the flow. For the design point runs in the latter part of the design the model exit static pressure was adjusted to match the flow predicted by DAWES in the stator model to the flow predicted by DAWES in the rotor model to within 0.5%.

5.2.1 Stator Geometry

The final vane geometry was reached when the composite vane thickness criteria were met. This geometry is tabulated in Appendix I.

The vane leading and trailing edge, stagger, and camber angles are shown in Figure 5.2.1-1. The stator solidity, which is the same as the baseline stator, is shown in Figure 5.2.1-2. Vane maximum thickness/chord, where maximum thickness is defined as tangential thickness, is shown in Figure 5.2.1-3. The increase in thickness around 90% span was needed to maintain a desired normal thickness with an increasing vane lean angle. The waves in the thickness were not completely understood, but they were necessary to meet the required normal thickness profile; and, similar waves were seen in the required tangential leading and trailing edge

thicknesses. They were at least partially due to the fact that the vane lean angle at the maximum thickness location and at the trailing edge was slightly wavy.

The maximum thickness location is shown in Figure 5.2.1-4. The maximum thickness was located at 60% of the mean camber line length as in the baseline vane. This was set by mechanical, aerodynamic, and producibility considerations. The leading and trailing edge radii (tangential), shown in Figure 5.2.1-5, are the radii of semicircles that form the vane section leading and trailing edges. They were scaled appropriately for the 22-inch rig. Edge thickness was approximately twice these radii. This was not exactly true for the leading edge since the vane leading edge was formed by a cubic generated using the semicircular leading edge as a starting point. The edge radii show some waviness similar to the maximum thickness. These were what was necessary to match the normal edge thicknesses required for composite producibility.

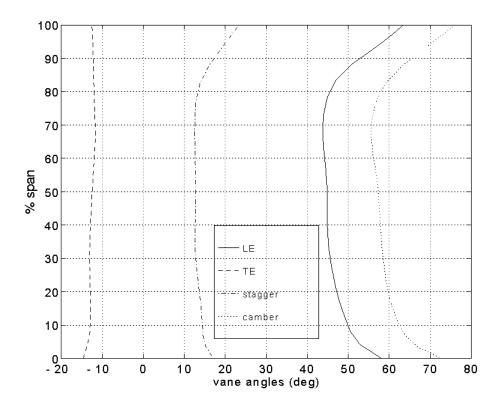


Figure 5.2.1-1. QHSF Stator Angles

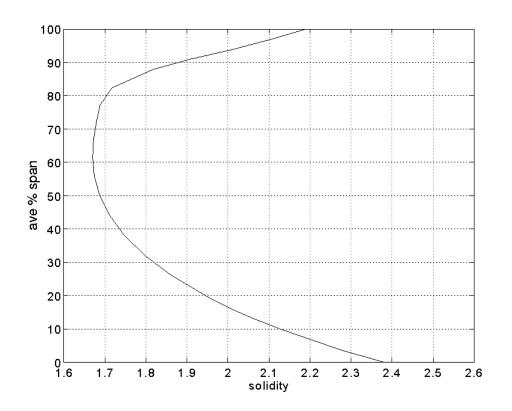


Figure 5.2.1-2. QHSF Stator Solidity

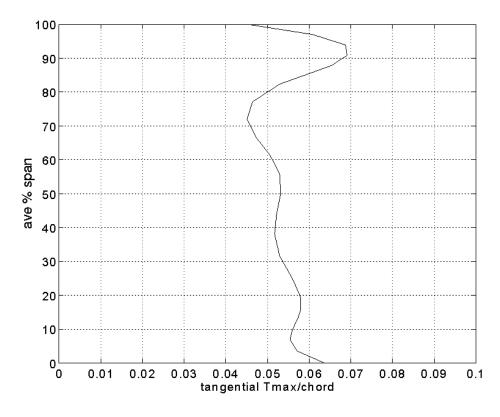


Figure 5.2.1-3. QHSF Stator Tmax/c

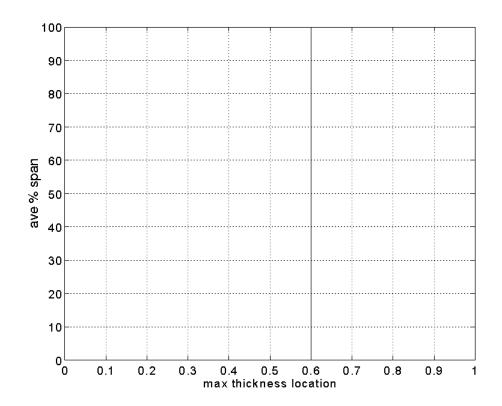


Figure 5.2.1-4. QHSF Stator Max Thickness Location

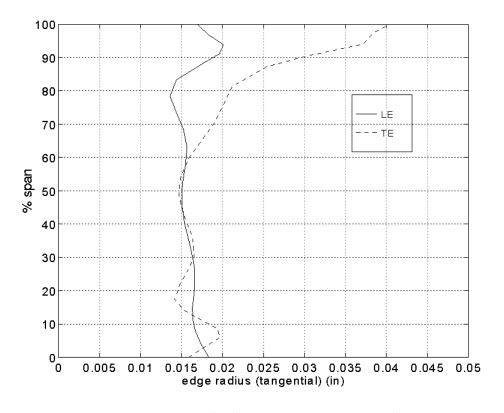


Figure 5.2.1-5. QHSF Stator Edge Radii (Rig Size)

The vane chord, scaled for the rig, is shown in Figure 5.2.1-6. The chord is determined from the solidity and vane count, which were maintained from the baseline stator. Figure 5.2.1-7 shows the vane section leading edge tangential shifts applied to produce the required vane tangential stack. Figures 5.2.1-8 and -9 show the chordwise distribution of mean camber line angle in dimensional (Fig. 5.2.1-8) and non-dimensional (Fig. 5.2.1-9) form.

These plots show how the vane camber was distributed from the leading edge (0) to the trailing edge (1). Figure 5.2.1-8 shows actual angles. At the leading and trailing edge, these are the hub, midspan, and tip angles from Figure 5.2.1-1. This figure provides a view of the actual curvature for different areas of the vane. Figure 5.2.1-9 shows the non-dimensionalized version which is how these curves are usually used in design. This allows the leading and trailing edge angles (or incidence and deviation) to be varied without the need to change these curves. The non-dimensional curves were retained from the baseline stator. The curves in Figure 5.2.1-8 were slightly different from the baseline stator since the leading and trailing edge vane angles were different.

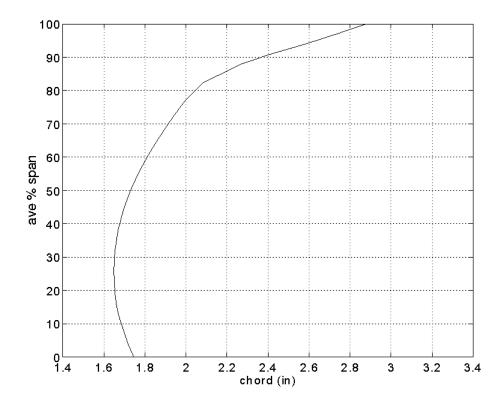


Figure 5.2.1-6. QHSF Stator Chord (Rig Size)

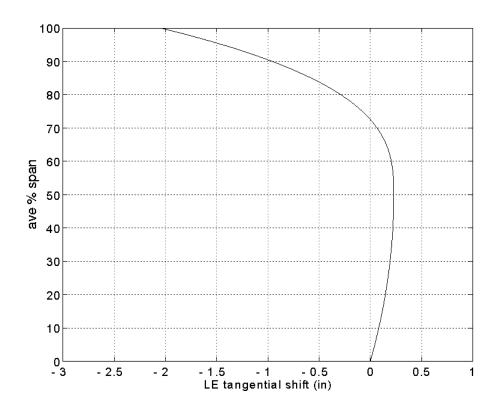


Figure 5.2.1-7. QHSF Stator LE Tangential Shifts (Rig Size)

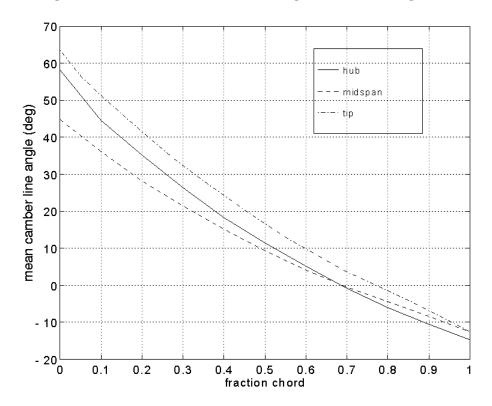


Figure 5.2.1-8. QHSF Stator Mean Camber Line Angles

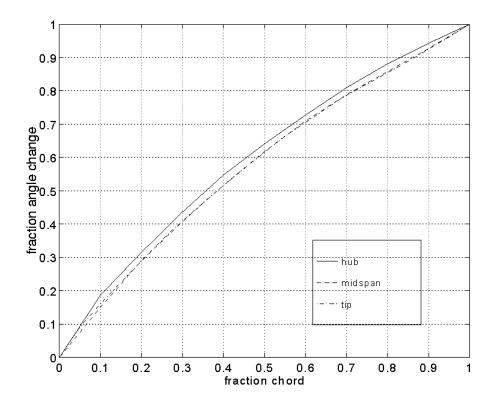


Figure 5.2.1-9. QHSF Stator Mean Camber Line Angles

5.2.2 Aerodynamic Results

Stator Streamline Curvature Velocity Triangles

After the final vane geometry was obtained, the design point loss estimated by DAWES was used in the AXCAPS model along with the design point rotor pressure ratio and temperature ratio profiles and flow predicted by DAWES. This procedure was done to calculate performance information that cannot be easily extracted from DAWES results.

Stator loss and D-factor are shown in Figures 5.2.2-1 and 5.2.2-2. This loss was the same as shown in Figure 4.3.6-1 except for some local modification at the endwalls necessary for AXCAPS convergence. The bump in D-factor around 25% span was caused by the flow splitter downstream of the vane trailing edge in AXCAPS. The splitter causes the streamtube area to increase as the flow moves radially inward and outward to get around the splitter, resulting in a local region of low stator exit velocity. The splitter was not modeled in DAWES. The D-factor peak at 90% span was caused by the bump in the loss profile from DAWES. Since this loss bump was due to radial migration of corner flow, the D-factor peak did not really indicate locally high stator loading. All D-factor values were in the acceptable range.

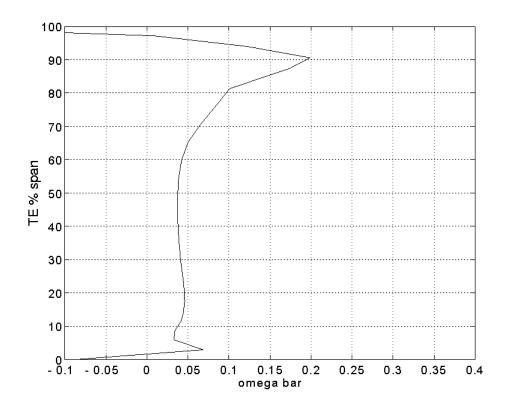


Figure 5.2.2-1. QHSF Stator Design Point Loss

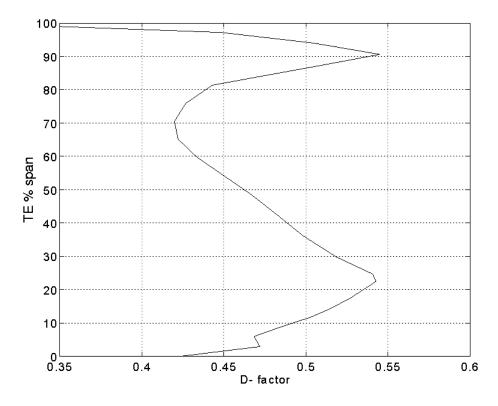


Figure 5.2.2-2. QHSF Stator design point D-factor

Figure 5.2.2-3 shows leading edge swirl from the AXCAPS calculation. Since AXCAPS distributes flow radially differently than DAWES, the swirl from DAWES was slightly different. In the AXCAPS model, the small amount of swirl at the stator exit was neglected and stator exit flow was assumed to be axial; therefore, Figure 5.2.2-3 also shows the flow turning.

Figure 5.2.2-4 shows the leading and trailing edge Mach number profiles from AXCAPS. These were slightly different than the DAWES profiles, particularly at the exit where DAWES does not include the effect of the splitter. These Mach numbers were within previous AE experience at the stator hub leading edge and at the core inlet region of the stator exit.

Figures 5.2.2-5, -6, and -7 show profiles of pressure ratio, temperature ratio, and efficiency at the stator trailing edge. These profiles combine the DAWES predictions of rotor performance and of stator loss and reflect the performance of the stage as a whole.

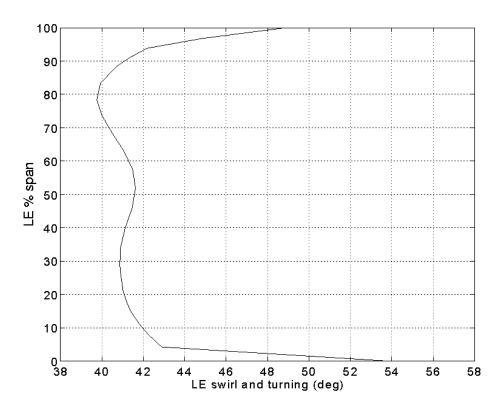


Figure 5.2.2-3. QHSF Stator Design Point LE Swirl and Turning

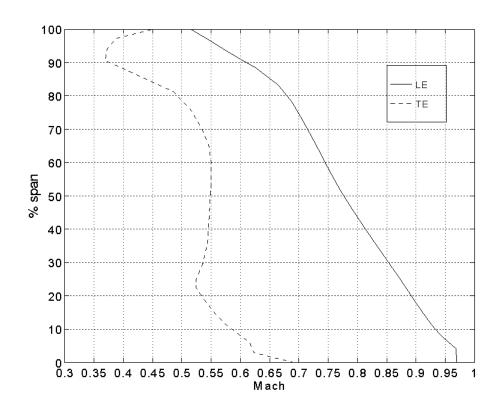


Figure 5.2.2-4. QHSF Stator Design Point LE and TE Mach

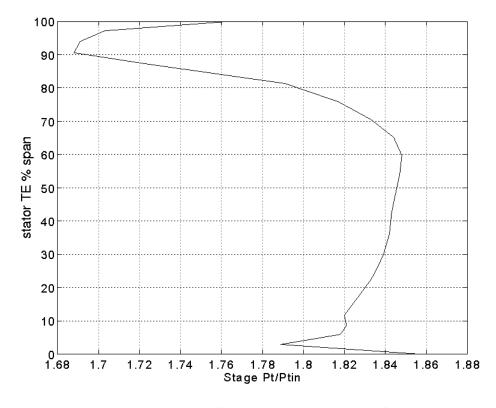


Figure 5.2.2-5. QHSF Stage Design Point Pt/Ptin

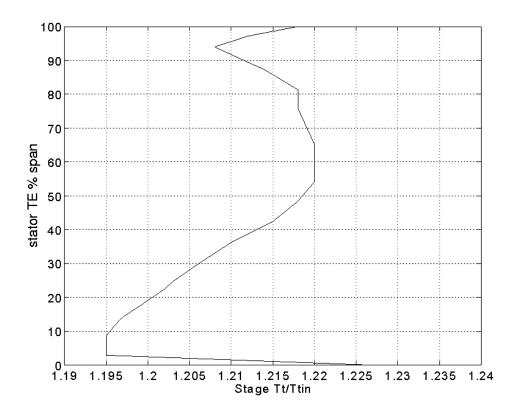


Figure 5.2.2-6. QHSF Stage Design Point Tt/Ttin

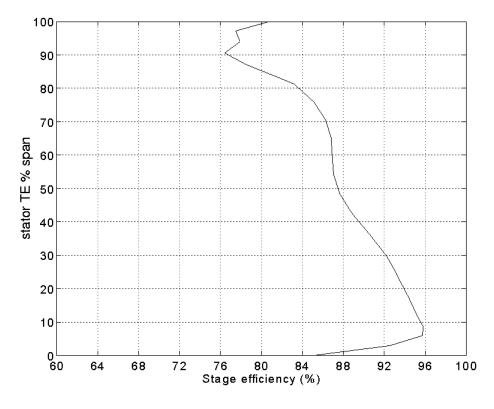


Figure 5.2.2-7. QHSF Stage Design Point Efficiency

Stator DAWES Analysis

Figures 5.2.2-8 through -12 show design point Mach number contours on the vane-to-vane surface at 10%, 30%, 50%, 70%, and 90% spans. Figures 5.2.2-13 through -17 show stator loadings as isentropically-calculated pressure and suction surface Mach numbers at 10%, 30%, 50%, 70%, and 90% span. There were two things considered when evaluating these loadings:

- 1.) Judging blade row quality by looking at suction and pressure surface Mach numbers on a grid surface assumes that flow mostly follows that grid surface. This is not strictly the case for highly three-dimensional blade rows.
- 2.) The isentropic Mach number is a convenient quantity to make it easier to compare the results of viscous CFD codes to past experience with inviscid analyses.

The loadings were biased towards choke near the hub which was a guideline used in setting the incidence, which past experience had shown to be a good strategy at the design point.

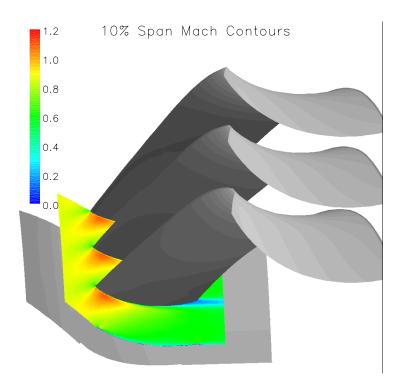


Figure 5.2.2-8. QHSF Stator Design Point Mach Contours 10% Span

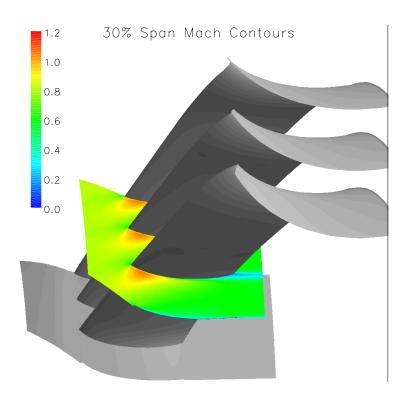


Figure 5.2.2-9. QHSF Stator Design Point Mach Contours 30% Span

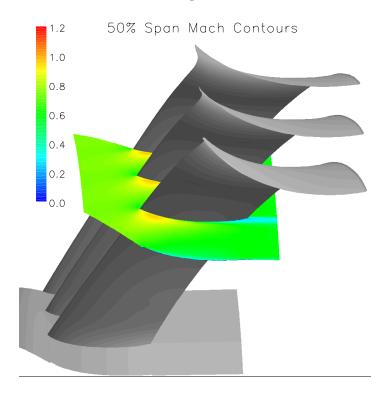


Figure 5.2.2-10. QHSF Stator Design Point Mach Contours 50% Span

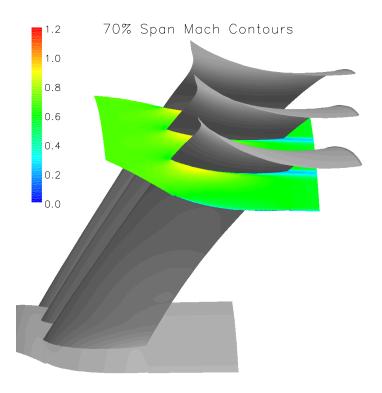


Figure 5.2.2-11. QHSF Stator Design Point Mach Contours 70% Span

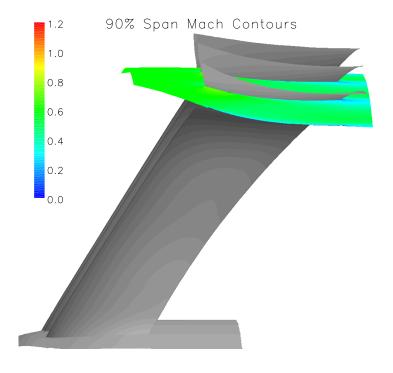


Figure 5.2.2-12. QHSF Stator Design Point Mach Contours 90% Span

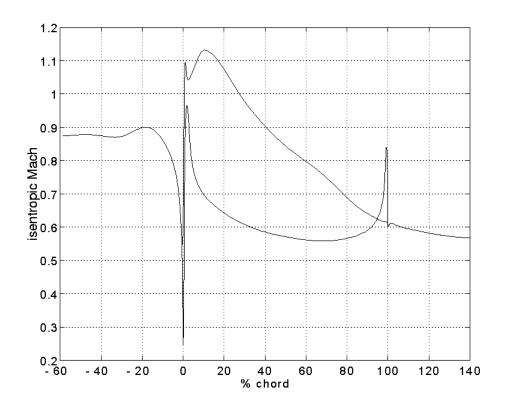


Figure 5.2.2-13. QHSF Stator Design Point Loading 10% Span

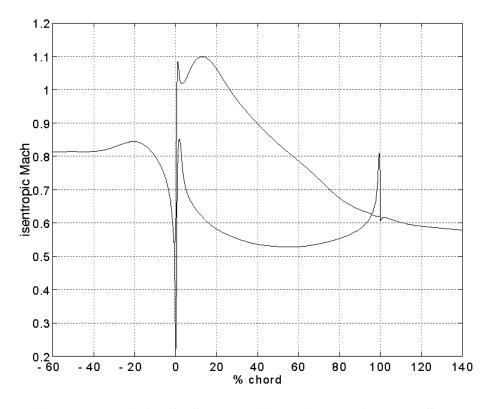


Figure 5.2.2-14. QHSF Stator Design Point Loading 30% Span

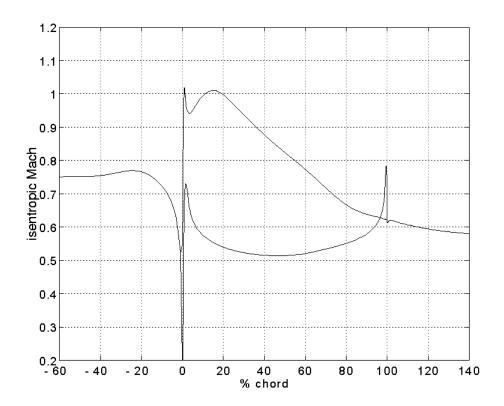


Figure 5.2.2-15. QHSF Stator Design Point Loading 50% Span

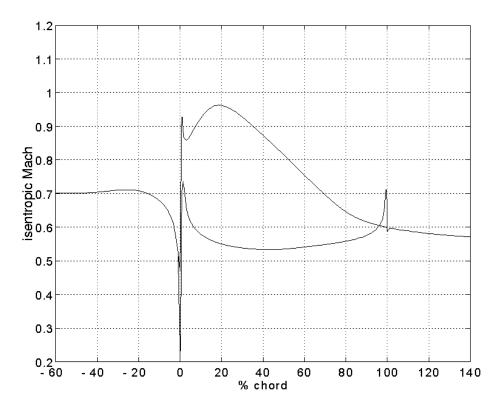


Figure 5.2.2-16. QHSF Stator Design Point Loading 70% Span

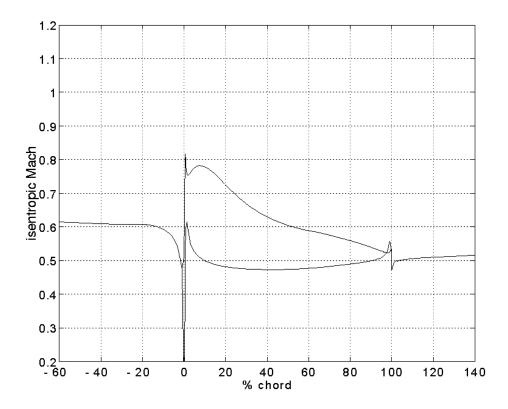


Figure 5.2.2-17. QHSF Stator Design Point Loading 90% Span

The stator performance at several off-design conditions was evaluated using DAWES. The points analyzed were on the sea level operating line at 55.9%, 70%, and 80% fan corrected design speed and on the altitude operating line at 90%, 100%, and 105% speed. The 55.9% speed point is a representative approach point where most of the acoustic V072 analysis had been performed. The 100% speed operating line point was slightly lower on the speed line than the design point so an additional run was completed. The stator inlet conditions for the stator DAWES model came from rotor exit velocity triangles predicted by the rotor DAWES model at these points. Figures 5.2.2-18 through -24 show stator profiles at these off-design conditions. Stator loss was well-behaved at part speed conditions. The loss bump increased between 100% and 105% speed and grew to affect more of the span. If efficiency at overspeed conditions (like max climb points) is important, this may need to be studied. The stator leading edge Mach numbers remained subsonic, even at 105% speed. The tip of the stator had an exceptionally large incidence swing throughout this speed range. The vane leading edge angle at the shroud was set as a compromise between design and part-speed shroud swirl. The stator exit swirl was adequately close to axial throughout. The importance of the locally high exit swirl accompanying the loss bump at 105% speed would have to be assessed considering the importance of overspeed performance.

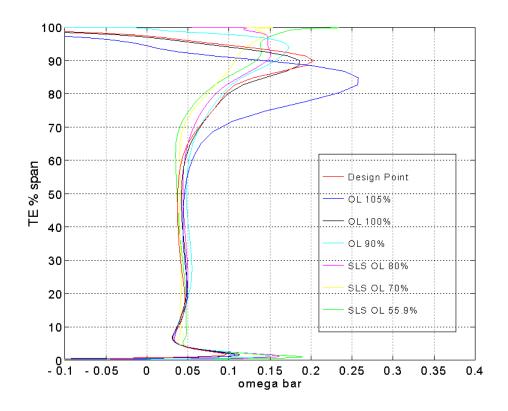


Figure 5.2.2-18. QHSF Stator Off-Design Loss

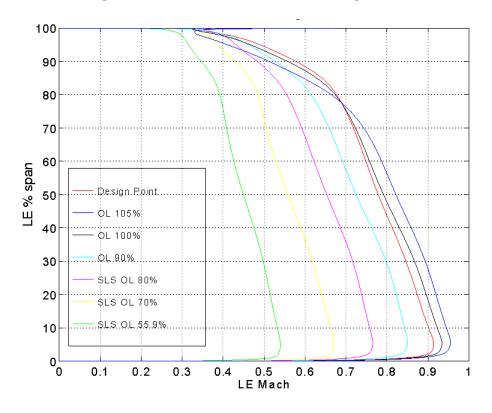


Figure 5.2.2-19. QHSF Stator Off-Design LE Mach

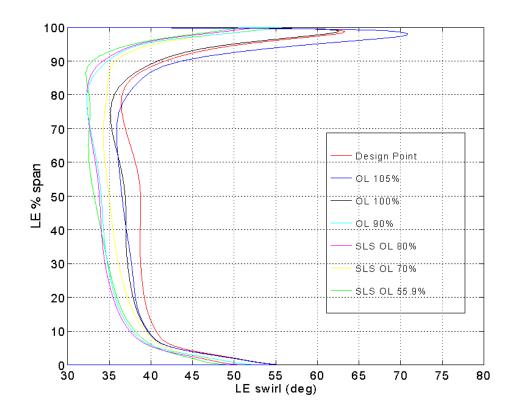


Figure 5.2.2-20. QHSF Stator Off-Design LE Swirl

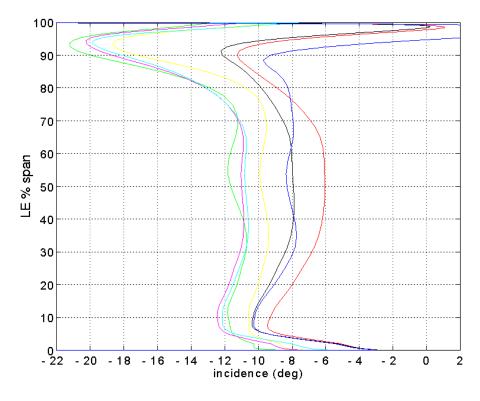


Figure 5.2.2-21. QHSF Stator Off-Design Incidence

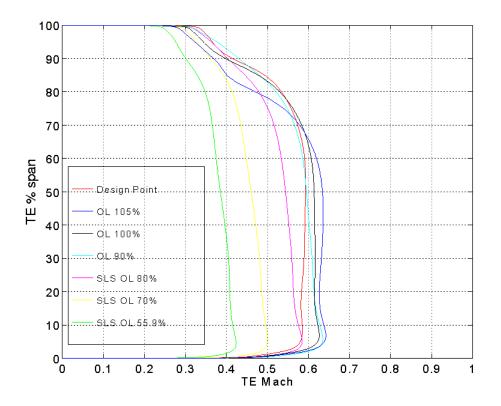


Figure 5.2.2-22. QHSF Stator Off-Design TE Mach

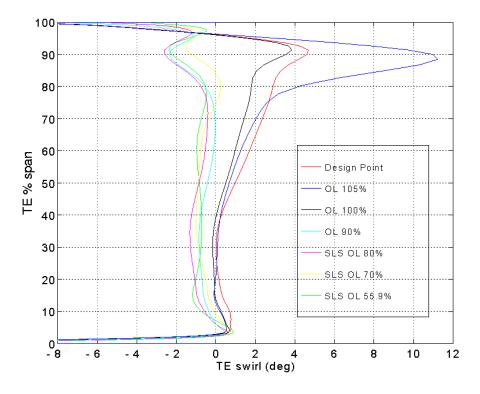


Figure 5.2.2-23. QHSF Stator Off-Design TE Swirl

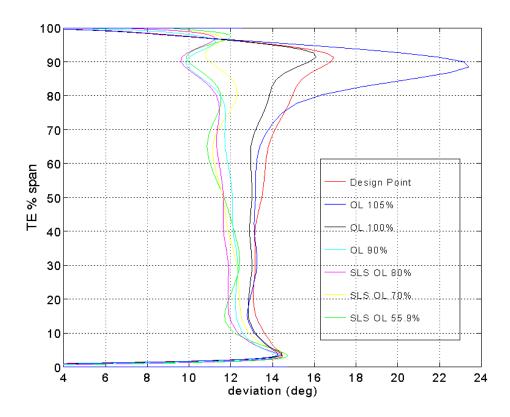


Figure 5.2.2-24. QHSF Stator Off-Design Deviation

5.2.3 Acoustic Results

Interaction of the fan rotor wake with the leading edge of the fan stator represents a significant source of noise at certain fan speeds. The noise associated with this interaction is most noticeable at approach conditions, when other noise sources are not dominant. Rotor-stator interaction tones are caused as the rotor wake impinges on the vane leading edge. The wakes emanating from the rotor are skewed and rotate circumferentially as they move downstream. Based on theory, the rotor wake/stator interaction noise occurs when the wake trace speed over the stator leading edge is supersonic. Thus, to minimize the interaction noise, the wake trace speed should be subsonic.

If the shape of the rotor wake is known, then the wake trace speed may be controlled during the design of the stator. This can be accomplished through vane axial and/or tangential lean. Increased axial lean (or sweep) increases the distance between the stator and rotor, causing a longer time increment for the circumferential wake pattern to impact the leading edge tip relative to the hub. Increased tangential lean (pressure side down) increases the delta in slope between the vane leading edge and wake trace, thereby reducing the amount of simultaneous impingement.

5.2.3.1 Objective

The objective was to use the QHSF fan rotor wake structure predicted by DAWES to help direct the design of the stator with the leading edge configured to minimize the acoustic effects of rotor wake/stator interaction, without appreciably degrading performance. The V072 acoustic prediction program was used to confirm the effect of the design on the interaction noise, and the stator design was evaluated with DAWES to determine the overall performance relative to the baseline stator.

5.2.3.2 Approach

Interaction of the fan rotor wake with the stator leading edge was assessed using a graphical technique. A contour plot of the rotor wake was produced on a surface formed by tangentially sweeping the stator leading edge curve (Figure 5.2.3-1). By superimposing the stator leading edge curve on the rotor wake plot, the time required for the wake to traverse a given distance along the leading edge was computed, thereby obtaining the wake trace speed.

5.2.3.3 Design Philosophy

The design philosophy employed in reducing interaction noise below that of the baseline was to use rotor forward sweep and stator aft sweep and tangential lean. The goal was to increase rotor/stator spacing to allow greater rotor wakes diffusion, and to increase the tangential lean of the rotor wakes relative to the stators thereby reducing wake trace speed and increasing phase variation along the stator span.

5.2.3.4 Stator DOE

Many stator configurations involving increased sweep and lean compared to the baseline were analyzed. It was found that leans of 30° and sweeps beyond the approximately 30° of the baseline stator did not provide significant acoustic benefit. In addition to the lack of acoustic benefit achieved by large leans and sweeps, DAWES results showed large flow separations near the hub for these cases. Consequently, more conservative leans were investigated.

For stator DOE1 involving variations on a 15° leaned vane, interaction noise predictions showed that the compound bow (leaning with the rotation direction at the midspan and against rotation at the tip) of case 5 produced high levels while the straight and compound leans with rotation of cases 1, 6 and 8 produced benefits in some harmonics (Figure 5.2.3-2). The aero results showed that separation and high hub losses resulted from leaning the suction surface towards the endwall. Case 6 was selected as a go forward case.

5.2.3.5 Final Configuration

The final QHSF stage was predicted to be 3 to 5 dB quieter than the baseline in all but the forward-propagating 3*BPF harmonic (Figure 5.2.3-1). Recall that the 1*BPF tone was cut-off at this speed. Although the predicted wake velocity deficit was greater for the quiet fan, the wake width was also greater and may at least partially offset the effect of the increased deficit. Reduced interaction noise was likely due to increased phase variation across the stator leading edge as shown in the wake traces. The pressure contours were extracted from a DAWES solution and revealed that the 2-D kinematic analysis of V072 may predict greater wake leans than exist in a 3-D flow. However, variation between the V072 and DAWES predictions appeared to be the same for both the baseline and the QHSF.

The objective of reducing interaction noise using rotor forward sweep and stator aft sweep and tangential lean was accomplished. Large reductions in the 2*BPF harmonic will translate into significantly lower flyover effective perceived noise levels.

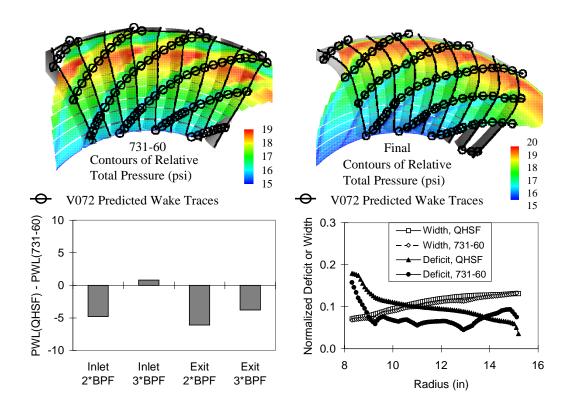


Figure 5.2.3-1. V072 Predictions for Final QHSF Rotor and Stator at 55.9% Speed

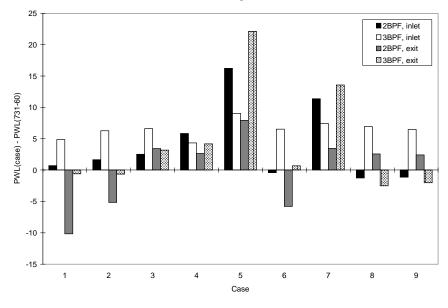


Figure 5.2.3-2. V072 Results for Stators with Moderate Leans, all with Baseline Sweep (30°)

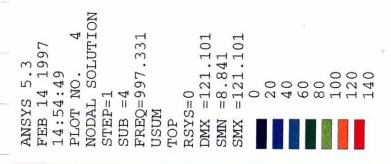
5.2.4 Mechanical Results

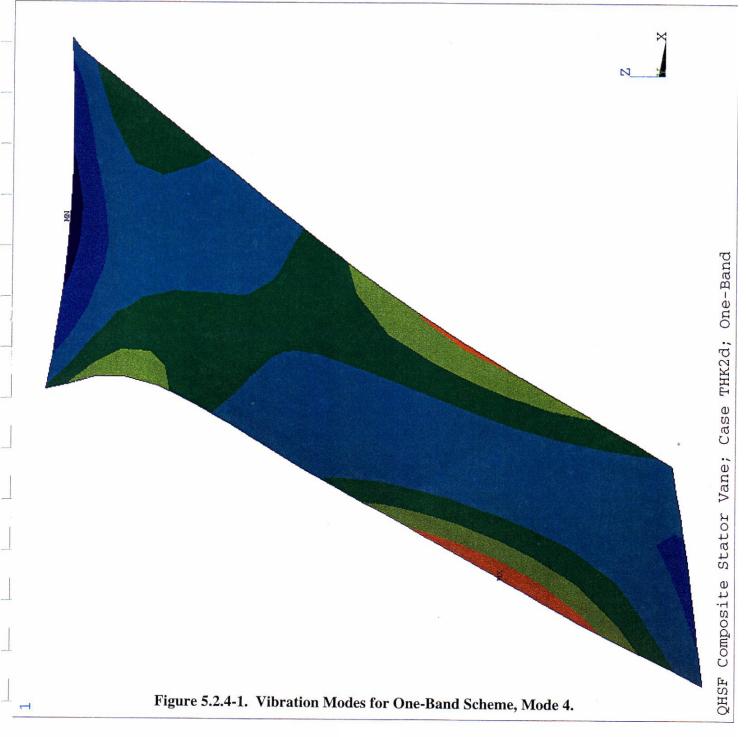
The static and vibration results for the final QHSF stator design are discussed in this section. As mentioned in section 4.3.3, two different retention schemes were recommended — the conventional single-band scheme and a two-band scheme as a backup in case flutter is observed in the rig. The two configurations have no aerodynamic impact; however, they have different mechanical behavior due to the change in the boundary conditions.

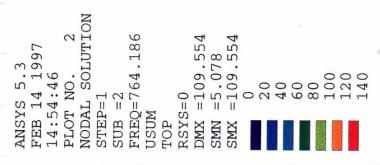
Table 5.2.4-1 Effect of Retention Scheme on Modal Frequencies

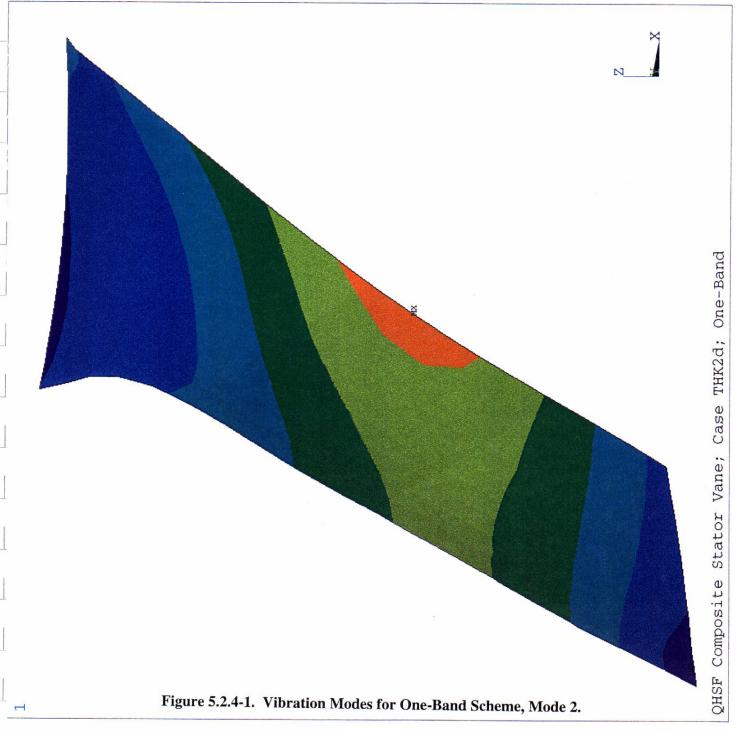
Frequency (Hz)			
Mode	One-Band Scheme	Two-Band Scheme	% Change in Frequency
1	505.52	579.66	14.67
2	764.19	740.23	-3.14
3	861.4	867.59	0.72
4	997.33	1086.7	8.96
5	1296.1	1381.7	6.60

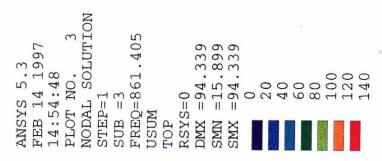
Table 5.2.4-1 shows the frequency predictions for the two retention schemes. The mode shapes for the first five modes are presented in Figures 5.2.4-1 and 5.2.4-2, respectively, for one-band and two-band schemes. In both schemes, the first mode was a torsion mode with frequency for the two-band scheme about 14% higher than the one-band scheme. However, second mode frequency was lower by about 3%. The flutter parameter for one-band scheme was 0.85, and 0.98 for the two-band scheme. The flutter parameter for baseline vane was 1.03 with taped construction and 0.96 with the braided construction. The Campbell diagram for two-band case is shown in Figure 5.2.4-3. Stress analyses completed for the two schemes, Figures 5.2.4-4 to 5.2.4-7, showed all design criteria was achieved.

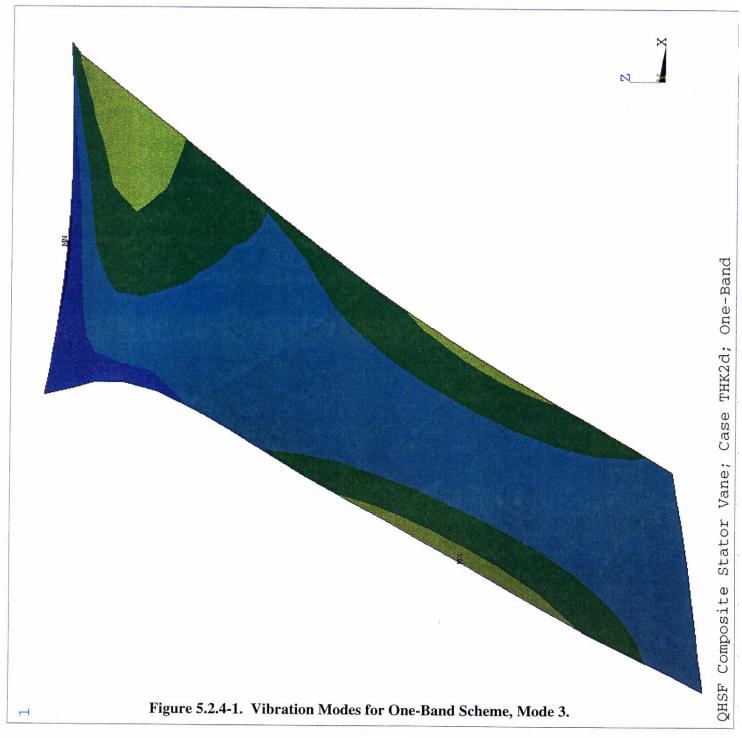


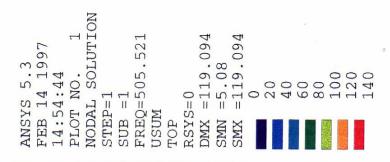


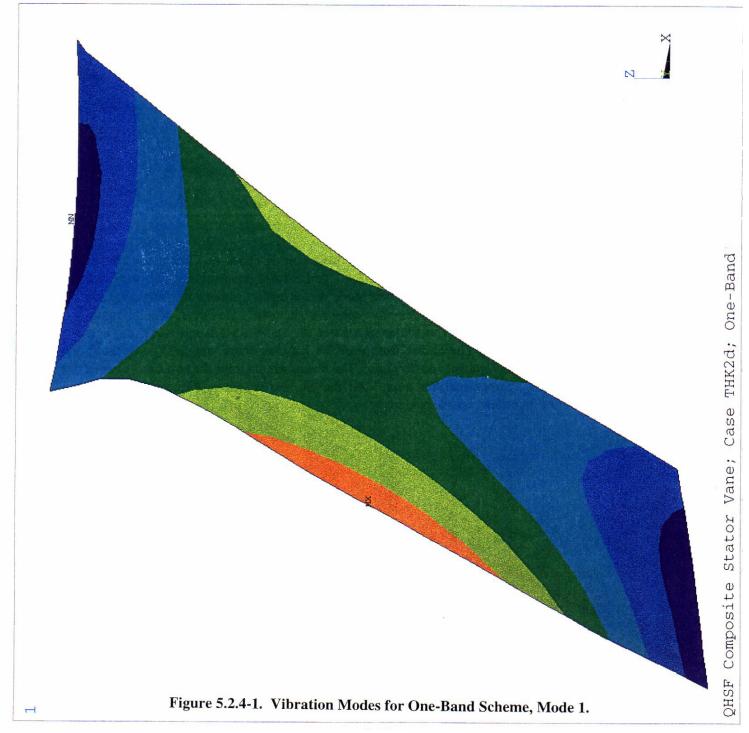


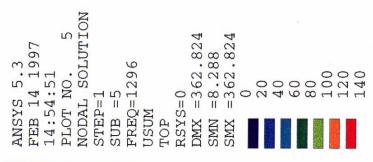


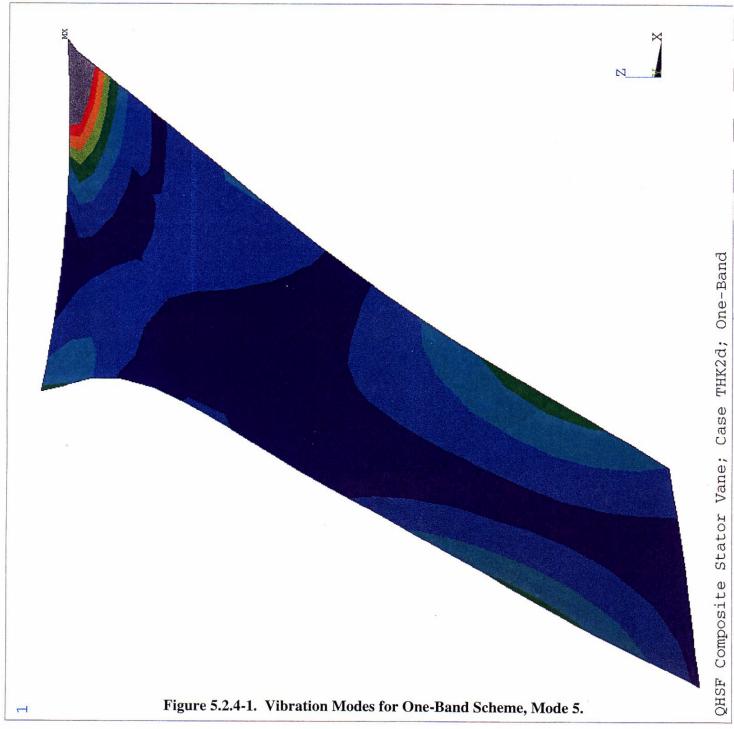


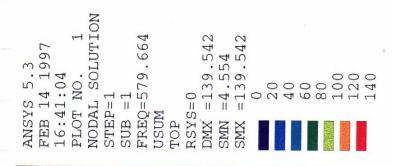


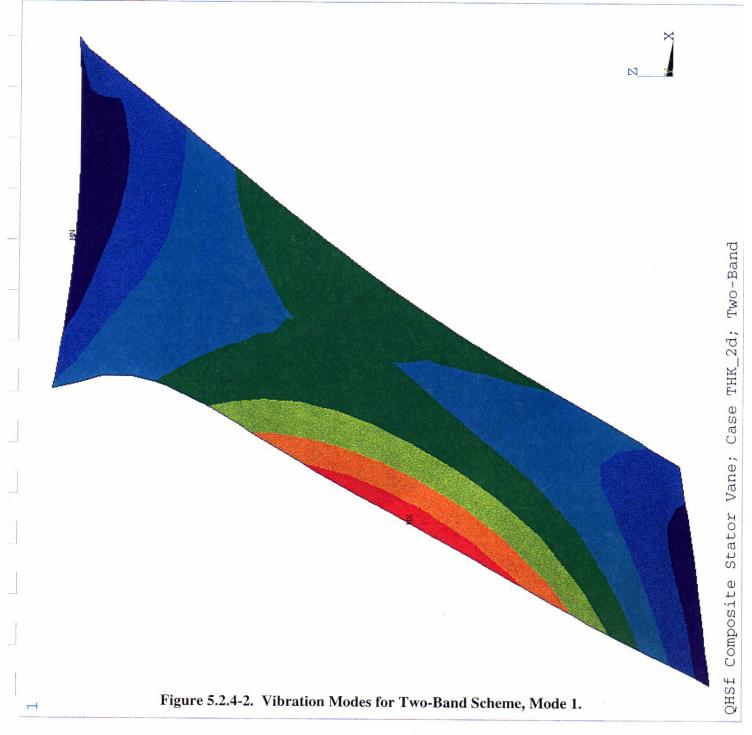


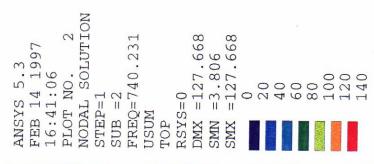


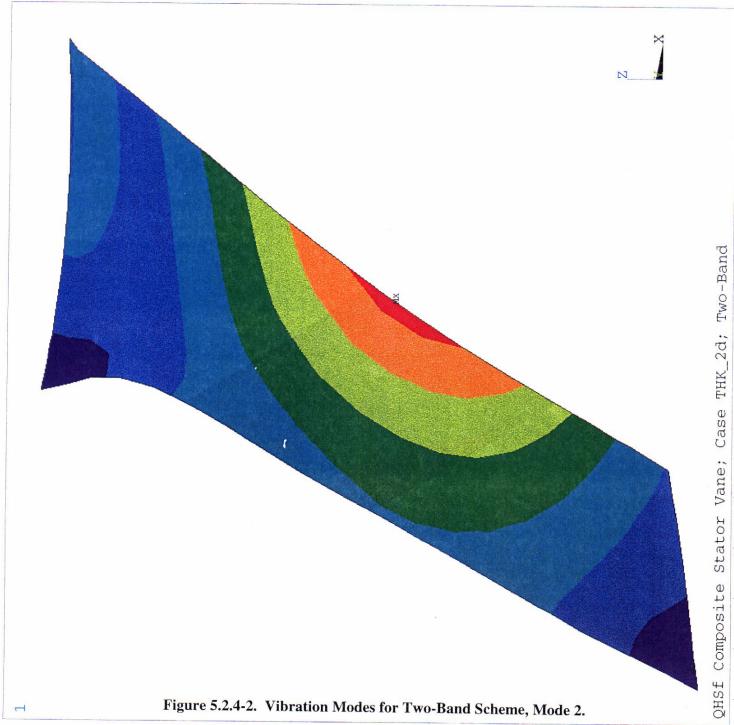


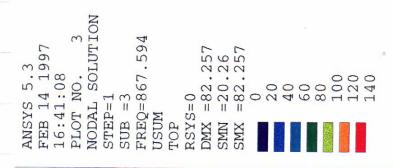


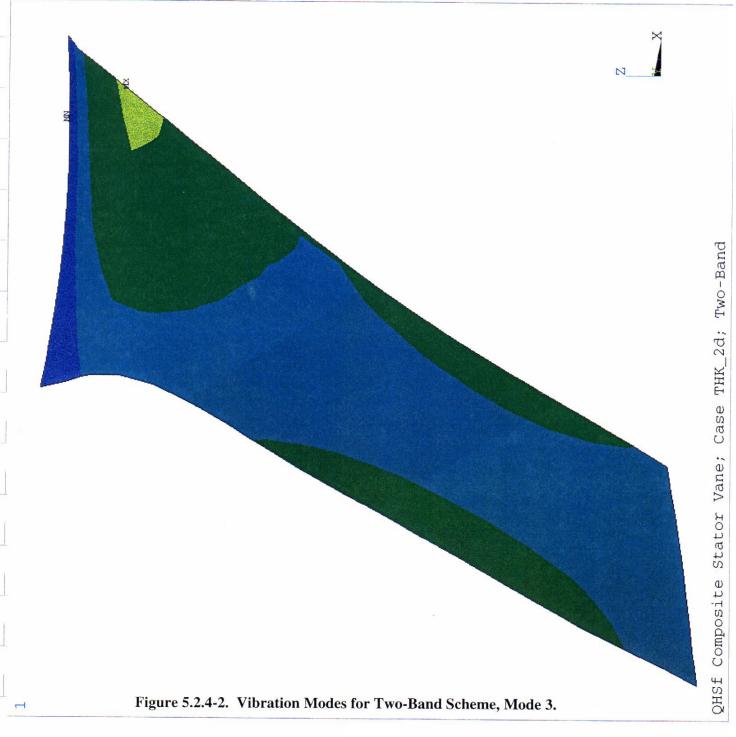


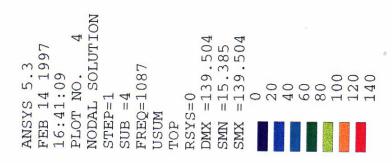


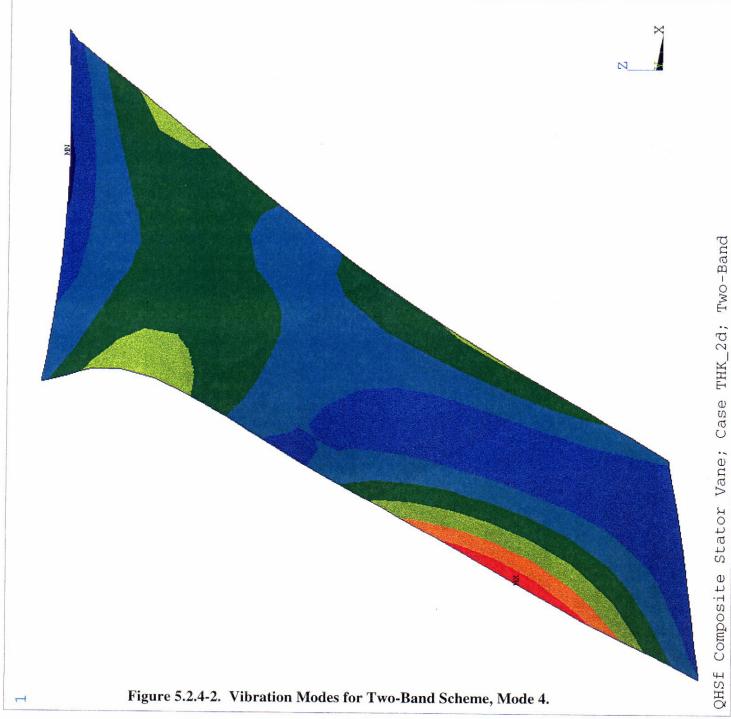


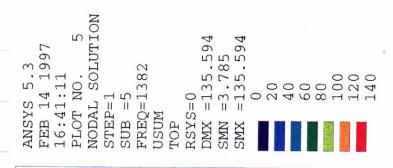


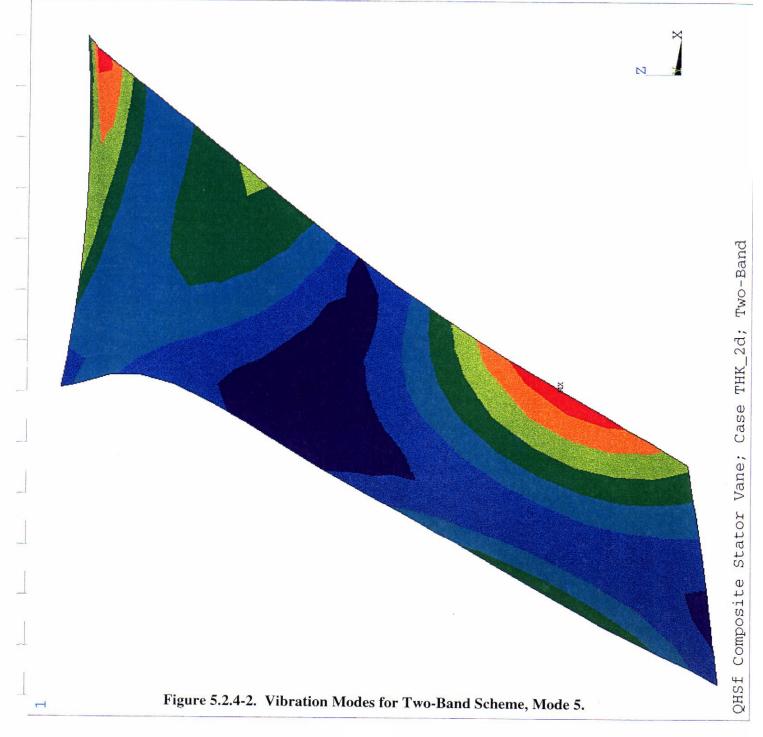














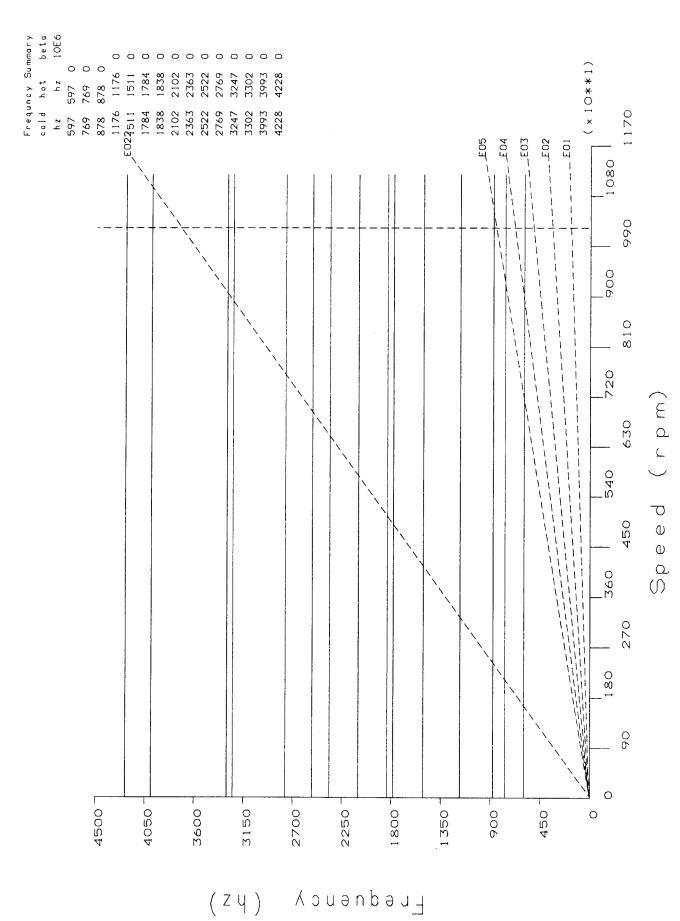
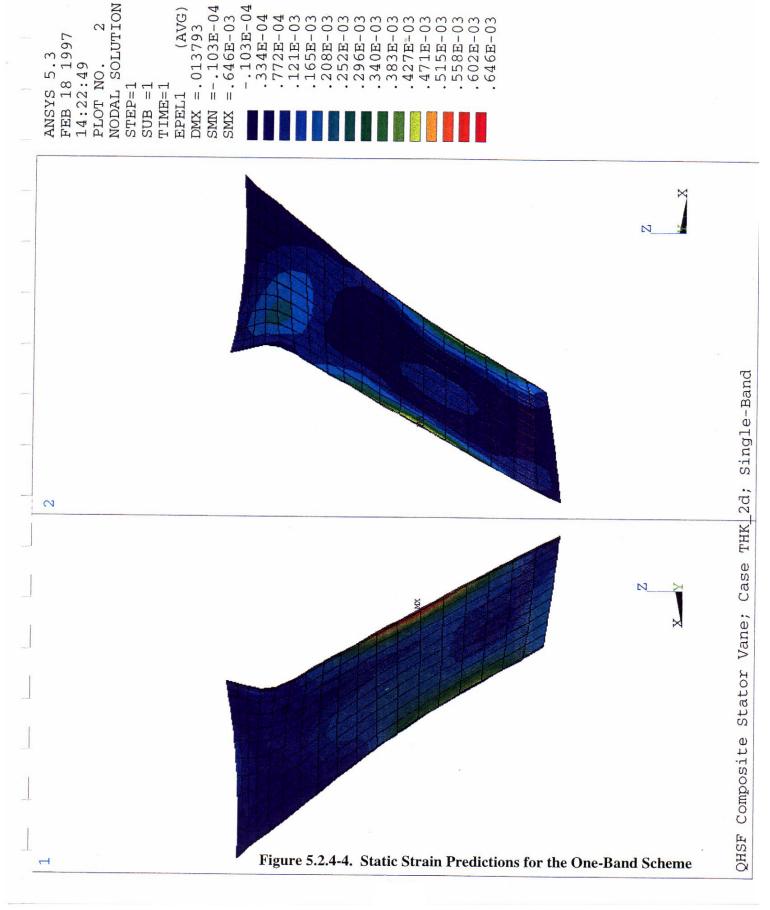
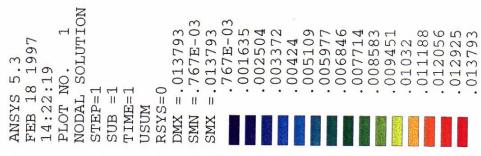
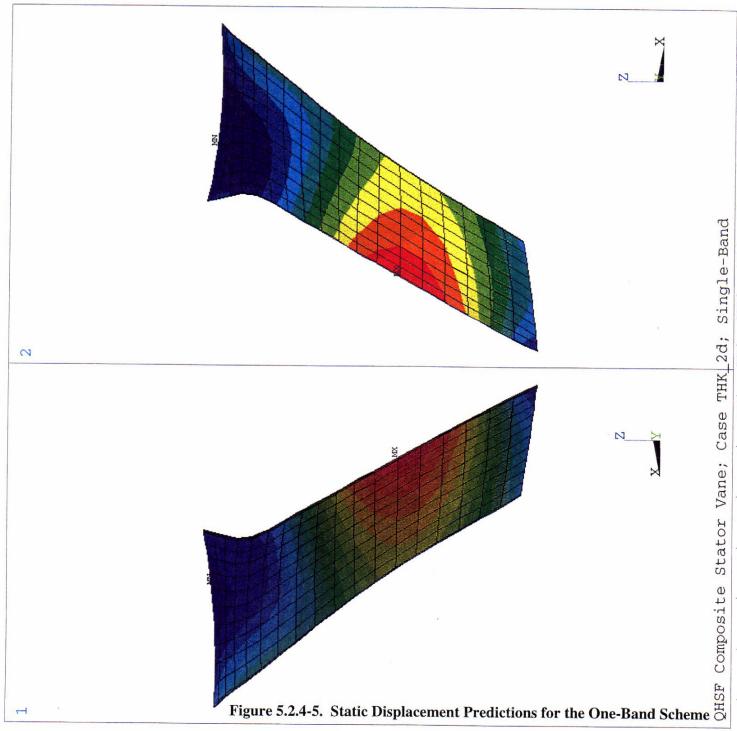
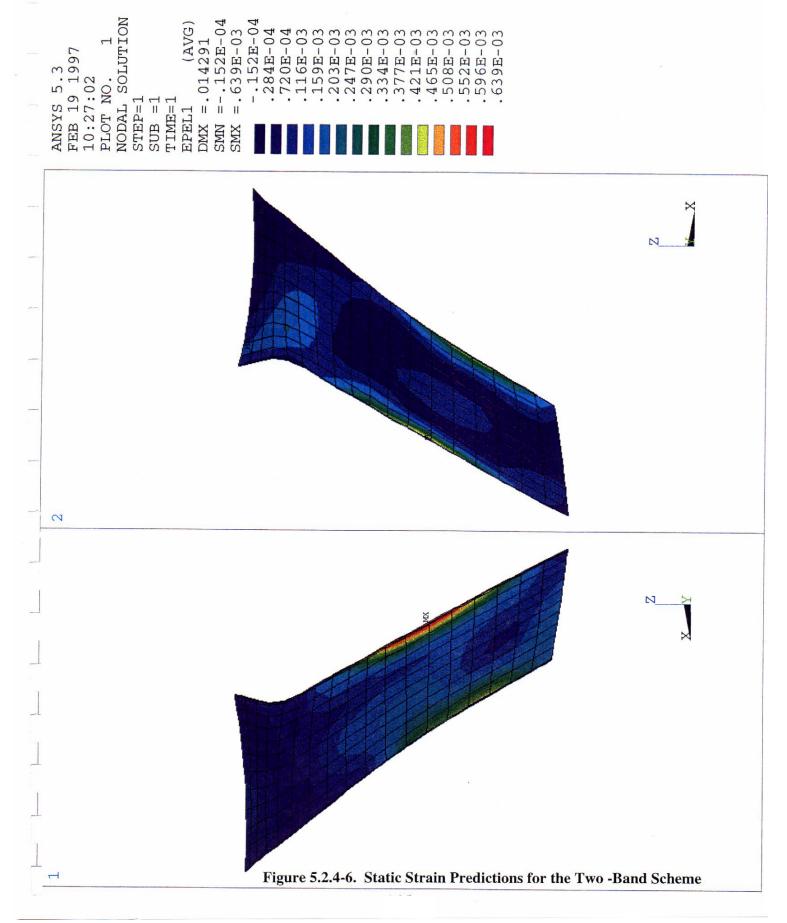


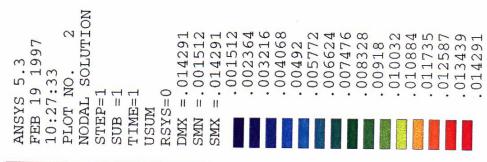
Figure 5.2.4-3. Campbell Diagram for Two-Band Scheme

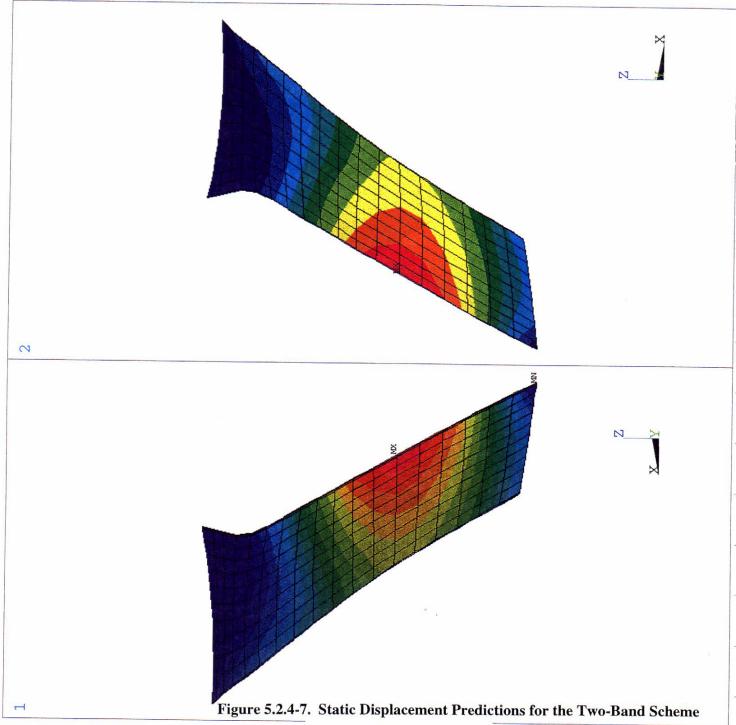












5.3 Disk

5.3.1 2-D Analysis

The preliminary design (PD) of the rig disk was reexamined with the optm3 airfoil loads. The results from the 2-D wedge model showed adequate burst margins and acceptable stresses in the disk. However, a 2-D axisymmetric analysis of the disk showed excessive roll at the rim with both baseline rotor and optm3 airfoil loads. It may be recalled that the PD of the disk was assumed to have rectangular cross section; i.e., no undercuts.

The rim roll and the tangential stress distribution along the rim (slot bottom) were balanced by providing undercuts in the disk. Also, the disk width was reduced to 3.08" to conform with the blade attachment width. The bore radius was increased to 2.3" to accommodate a 0.25" thick torque sleeve. Figure 5.3.1-1 shows the layout of the disk with undercuts and the torque sleeve mounted on the NASA balance. The tangential stress distribution in the disk hub at design speed is shown in Figure 5.3.1-2 for the two airfoil load cases. Figures 5.3.1-3 and 5.3.1-4, respectively, show the variation of tangential stress and radial displacement at the slot bottom.

5.3.2 3-D Analysis

A solid model of the 3D disk was constructed in ANSYS using Boolean operators. The 2D disk profile was swept about the centerline by an angle of 360/22= 16.36°. A solid defined by the slot contour was then subtracted from the swept disk wedge, resulting in the final slotted disk sector; as shown in Figure 5.3.2-1. The finite element mesh of the final model consisted almost entirely of brick elements, with tetrahedra only in regions of low stress. Figures 5.3.2-2 and 5.3.2-3 show the maximum principal stresses and the tangential stresses in the disk at design speed for QHSF airfoil loads.

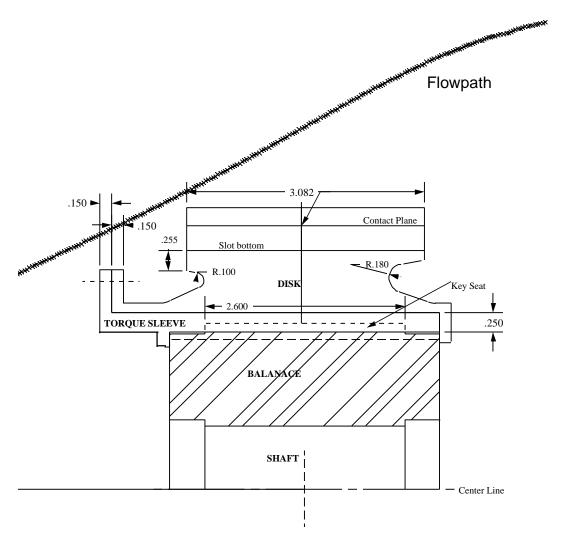


Figure 5.3.1-1. QHSF Disk Layout with Torque Sleeve Mounted on NASA Balance

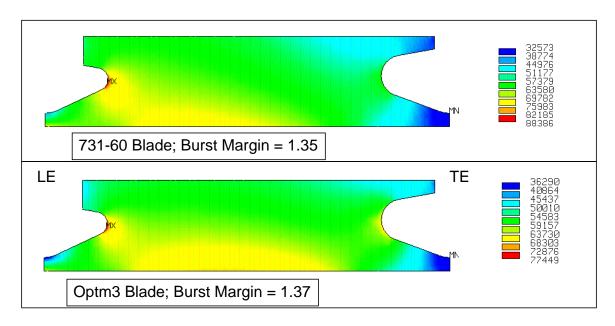


Figure 5.3.1-2. Tangential Stresses in Disk Hub from 2-D Axisymmetric Analysis

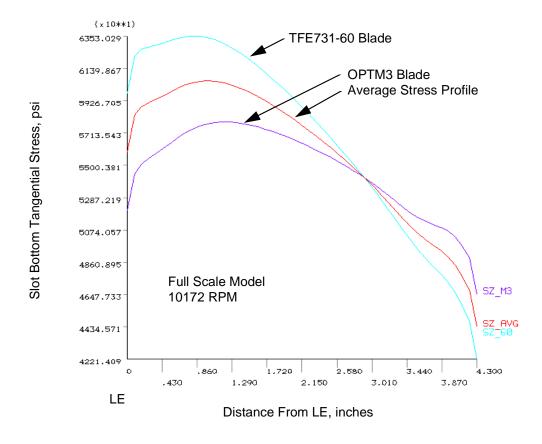


Figure 5.3.1-3. Rig Disk Optimized to Perform Well With Both Blades

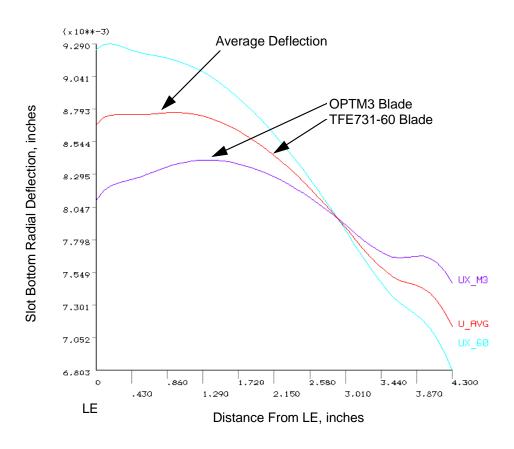
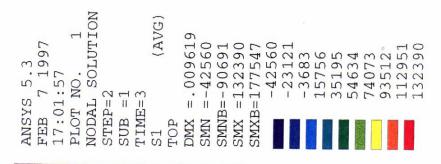


Figure 5.3.1-4. Rim Rolling Well Balanced for Both Blades





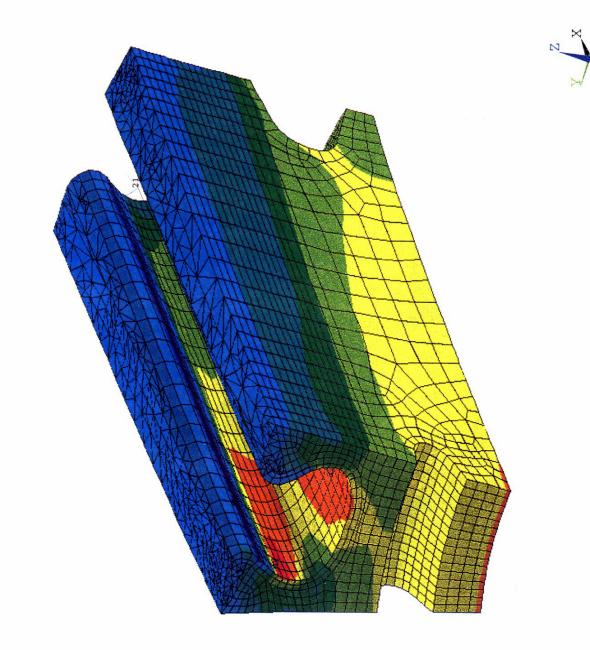
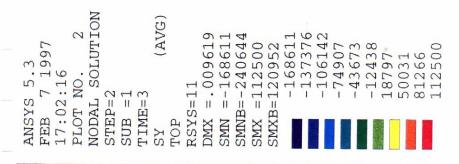


Figure 5.3.2-2. Maximum Principal Stresses in the Rig-Size Disk at Mechanical Design Speed (15444 RPM) with QHSF Airfoil Loads



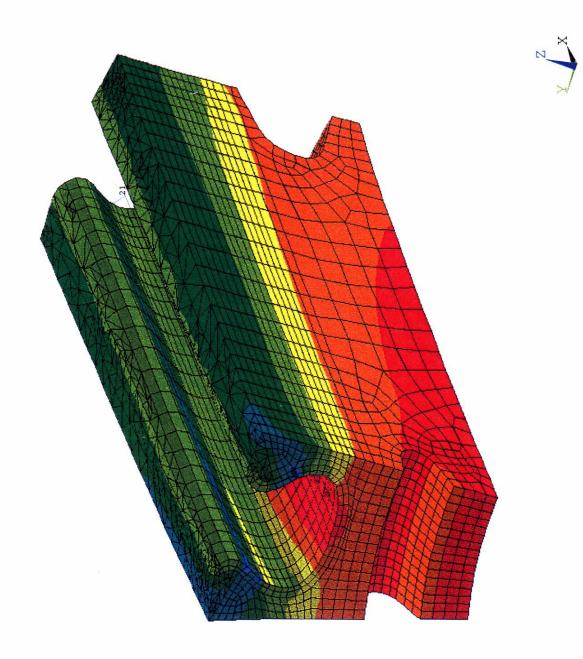


Figure 5.3.2-3. Tangential Stresses in the Rig-Size Disk at Mechanical Design Speed (15444 RPM) with QHSF Airfoil Loads

Highlights of QHSF Disk are summarized below:

- Disk Material is C250 steel
- Disk/dovetail profile is a derivative of the baseline disk design
- Dimensions modified to fit into the NASA balance and to accommodate .25" thick torque sleeve
- Meets both NASA and AE design criteria
- Disk has adequate burst margins, unzip capability and factors of safety
- Optimized to perform well with both QHSF and baseline rotors
- Well balanced to minimize rim roll by providing undercuts
- 3-D stress analysis performed to qualify design

6.0 CONCLUSIONS

The QHSF has been designed to meet aggressive acoustic goals while maintaining performance and mechanical qualities equal to or better than the baseline fan. The design was achieved through innovative use of advanced design techniques and close cooperation between the different disciplines on the team. Validation of this technology in the upcoming rig test will allow it to be used to meet the stringent acoustic requirements of the next generation of turbofan engines.

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APPENDIX I

COMPUTER LISTING OF AEROTHERMODYNAMIC PARAMETERS FOR QHSF ROTOR, STATOR, AND FLOWPATH DESIGN

This appendix contains a computer listing of the aerothermodynamic output data for the QHSF fan stage design. All units are in the English system and headings on the columns are self-explanatory. Rotor and stator geometry and loss parameters are included in Tables A-1 to A-3, respectively. Tables A-4 to A-6 contain rotor and stator performance and vector output from spinner leading edge (station 7) to stator trailing edge (station 22).

Table A1. Summary of Quiet High Speed Fan Design

			3.783				
	WORK	1.006	RATIO				
	PRESS COEF	.907	BYPASS				
97/03/07 15:44:24	FLOW	631	SE	.215			
6 H	d CORR.	15357	 ERALL EFFI TEMP CIENCY RISE	88.15 .2 85.83 .2		HORSE POWER	3149.
	RPM PHYS.	15444.	0V SS. IO	1.837 88 1.811 85		H D(ж
ct loss.	INLET SPEC. FLOW	42.35	-BYPASS TEMP PRE RISE RAT	.215 1.8	IO TIP 1.008 1.334	DODGE STALL MARGIN	14.8 17.2
Point. 22-in rotor LE tip diameter. Final core duct flowpath w/ -60 duct nce from DAWES models.	CORRECTED TIP SPEED	1474.	STAGE EFFI- CIENCY	88.31 85.99	VM RATIO JB MEAN 34 .931	DODGE SURGE MARGIN	20.4
LE tip owpath wels.	NO VANE HUB ABS.	968	 PRESS. RATIO	1.839	HUB 90 .924 91 1.246	MID SPAN BETA	57.41 41.39
1 rotor luct flo TES mode	MACH ROTOR TIP REL.	1.423	INLET CORR. FLOW	78.11	SOLIDITY MEAN TIP 1.848 1.490 1.709 2.191	MEAN SOLID -ITY	1.872
. 22-ir L core c From DAM	TEMP	3 .212	 - TEMP	199 . 199	SOLJ HUB ME 2.849 1.8 2.381 1.3	ASPECT RATIO	1.35
yn Point . Fina rmance f	OVERALL PRESS. EFFI- RATIO CIENCY	4 90.16 3 87.43	OVERALL PRESS. EFFI- RATIO CIENCY) 96.41 2 93.85	TIP 8.8 48.7 2	LOAD COEF.	.589
n. Desig eometry r perfo		2 1.844 2 1.813	と 田	9 1.850 9 1.822	TURNING MEAN 21.5 41.4	DELTA P OVER Q FREE	.418
NASA/AE QHSF Fan. Design Point. 22-in rotor LE Design report geometry. Final core duct flowpa Rotor and stator performance from DAWES models.	STAGE EFFI- TEMP CIENCY RISE	90.15 .212 87.43 .212	COi STAGE . EFFI- TEMP CIENCY RISE	95.96 .199 93.40 .199	T TIP HUB .399 60.1 .291 54.0	DELTA P OVER Q	.470
NASA/ Desig Rotor	STAGE PRESS. EFFI- RATIO CIENCY	1.844	S S PRESS. RATIO C	1.845	D-FACTOR MEAN .498 .	TWO THETA FREE	8.7
	INLET P CORR. R FLOW	98.65	INLET P CORR. R	20.57	D- HUB .504	TWO	8.9
ARY		1 1		1 1	пп		пп
* SUMMARY		ROTOR		ROTOR	ROTOR		ROTOR

Table A2. Quiet High Speed Fan Rotor Blade Airfoil Geometry

1*AIRFOIL GEOMETRY NASA/A Design Rotor	ROTOR 1 AT STATION 16 HAS INCIDENCE RULEMETAL AN	STAG CAMBER SOLID -GER -ITY	61.55 13.30 1.490	0.47 11.63 1.50	9.48 10.74 1.	8.42 10.67 1.54	7.42 10.70 1.	55.56 10.82 1.587	2.23 10.60 1.	0.50 11.16	.64 12.26 1.	.66 13.87 1.	15.93 1.	2.12 18.61 1.	.32 22.06 1	6.27 25.99 2.01	3.72 28.98 2.08	2.21 30.28 2.12	9.25 33.12 2.21	7.10 34.54 2.28	5.09 35.54 2.34	2.50 36.71 2.43	0 38.05 2.5	.91 39.12 2.66	.21 41.15 2.84
NASA/AE QHSF Fan. Design report gec Rotor and stator	S 22	TMAX/ TH CHORD /S	.0203	•	•	•	•	.0220		•	•	•	•	•	•	•	•	•	.0462		<#	ω.	194	. 60	.0530
Design metry. performa	AI ION RUL	THROAT CHORD	4270 4.626	8 4.	0 4.5	4.5	4	5092 4.507	1 4	5	5	4	4	5	Q 4.	82 4	m	40 3	3	5	ω	1 3.85	4 3.	404 3.84	7485 3.841
Point. 22-i: Final core nce from DA	RFOILS EMETAL A	INLET METAL ANGLE	65.69	64.21	63.01	62.00	61.09	59.42	56.30	54.88	53.55	52.28	51.01	49.76	48.66	47.59	46.69	46.26	45.07	44.12	43.22	41.9	40.	N	36.66
22-in rotor LE tip d core duct flowpath w/		EXIT METAL MEAN ANGLE LINE	52.39 5.5	.58 2.	.28 1	.34 2.	.39 2.	48.60 3.12	J 44	.73 4			35.08 5.15		4		17.71 3.01		11.96 2.14	9.58 1.86	ij.	.21 1.	ij.	29 1.7	-4.49 3.9
tip d ath w/		INCIDENCE- N SUCTION FCM E SURFACE	2	1.1	9.		П	12 1.77	2 6	3	3.5	3.1	7	C/I			1	1		-3.0	-3	-3.4	-3.8	- 0	6 -3.47
iameter. -60 duct los:		 	3.81	.92	. 25	. 62	. 77	1.63	3.12	•	•	•	•	•	2.62	•	1.56	1.34	.84	.50	.20	.15	.14	60	.19
ů.		INLET DELTA REL. MACH	.00 1.42	•	ή.	ij	i.	.00 1.415	; ;	ij.		Ļ.	.00 1.202	Н	Н	П	1.03	1.01	.00	.00.	.91	٠	•		.00.
97/03/07 15:44:24		T PASSAGE ANGLE	Ŋ	3 58.38		56.		5 54.03	51.				2 43.23				~	Ю			0	6 23.	4	4 1	2 16.08
. 24		MIN. E A/A* NS LC	1.018	1.042	1.053	1.060	1.058	1.057	1.048	1.043	1.036	1.054	1.057	1.053	1.050	1.041	1.025	1.017	1.006	1.000	966.	.991	.991	. 992	1.052
		PASSAGE ANGLE LOCATION I	009.	.587	.574	.561	.551	.531	. 494 494	.473	.449	.428	.400	.372	.341	.310	.283	.272	.245	.230	.214	.193	7	.156	.132
		MAXIMUM THICKNESS LOCATION	.601	.602	.604	.604	.604	.604	.597	7	3	8	.560 1	4		Н		.469	.443 2		7		.363 2		.302 3
		CRE	8		8.	8.		9		.0	30:	. 13	. 22	31	44	. 55	75	. 82	2.02	3.16	2.32	2.58	2.84	3.20	3.78

.834 .854 .886 .886 .986 .910 .910 .938 .938 .1087 .10

Table A2. Quiet High Speed Fan Rotor Blade Airfoil Geometry (cont.)

INCI- DENTAL ANGLE		
НД	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
SUCTION SURFACE MACH NO	$\begin{array}{c} 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ 1\\ $	
MAX CAMBER LOC	C 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
DELTA S		
SHOCK		
 RADIAL	100.7 10	
TES REAR AXIAL	-10 . 199 -10 . 199 -9 . 9042 -9 . 9043 -9 . 466 -9 . 313 -9 . 181 -9 . 181 -9 . 183 -8 . 825 -8 . 238	
EDGE COORDINATES FRONT REA AXIAL RADIAL AXI	11. 013 10. 824 10. 5641 10. 5641 10. 3494 9. 393 9. 393 9. 717 7. 552 7. 119 6. 658 6. 107 6. 658 7. 118 8. 343 7. 119 7. 119 8. 343 7. 119 8. 343 8. 344 8. 344 8. 344 8. 344 8. 345 8. 345 8	
EDGE C FRONT- AXIAL	-12.402 -12.2314 -12.2314 -12.175 -12.174 -11.924 -11.924 -11.924 -11.636 -11.636 -11.639 -11.639 -11.649 -11.543	
Y-CG	11. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1	
X-CG	-1. 225 -1. 286 -1. 986 -1. 986 -1. 986 -1. 659 -1. 1433 -1. 1433	
CONE	7.00 7.00	04
STACK RADIUS	10011100111001100110011001100110011001	-10.1104
TRAIL -ING RADIUS	. 0087 . 0091 . 0093 . 0098 . 0103 . 0113 . 0113 . 01142 . 0136 . 0142 . 0136 . 0142 . 0142 . 0142 . 0151 . 0169 . 0169 . 0182 . 0182 . 0182 . 0169 . 0169	STACKING AXIS COORDINATE IS
LEAD -ING RADIUS	. 0142 . 01442 . 01443 . 01443 . 01452 . 0152 . 0199 . 0213 . 0237 . 0237 . 0351 . 0351 . 0352 . 0420 . 0442	S COORD.
-DEVLATION- MEAN DELTA LINE		ING AXL
-DEVI MEAN LINE	100 0	STACK
STL NO.	22 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	THE

Table A2. Quiet High Speed Fan Rotor Blade Airfoil Geometry (concluded)

	SHOCK	OMEGA	BAR	000.	000.	000.	000.	000.	000.	.003	.010	.012	.016	.021	.027	.035	.043	.048	.042	.030	.019	600.	.002	000.	000.	000.	000.
		MACH NO	ET S.S	32 1.014				.919 1.250										80 1.468									25 1.289
		M	INLET	7.	8.	ω.	ω.	0.	o.	ο.	1.0	1.0	1.0	1.117	1.1	1.2	1.2	1.2	1.3	1.3	1.3	1.4	1.4	1.4	1.4	1.4	1.4
ΤŅ	OBLQ	SHOCK	ANGLE	55.63	58.48	61.38	63.48	64.99	66.11	68.10	70.39	70.80	71.81	73.21	74.85	77.52	82.42	88.78						126.77	129.94	134.81	136.25
IMPINGEMENT POINT	BLADE	LEAN	ANGLE	1.87										3.79										-27.11		-32.54	-33.18
MPINGEM	REL	FLOW	ANGLE											44.82												63.10	64.37
SHOCK I		STL	ANGLE											13.44												-5.03	-7.00
	STA	SWEEP	ANGLE	13.27	12.00	10.40	9.44	00.6	8.66	7.37	7.05	7.04	6.62	6.59	6.03	4.56	1.77	-1.93	-5.61	-10.52	-13.86	-17.01	-21.30	-24.46	-27.17	-31.14	-32.22
	OBLQ	SHOCK	ANGLE	71.38	73.65	77.17	80.57	83.03	84.04	86.31	88.94	90.11	91.60	93.19	94.99	97.93	101.79	107.28	113.96	119.71	124.53	129.14	133.74	137.21	140.22	142.58	142.64
ΤΞ	BLADE	LEAN	ANGLE	-6.94	-6.90	-7.60	-9.28	96.6-	-9.18	-9.67	-11.14	-11.61	-11.20	-11.59	-12.08	-13.61	-16.12	-20.23	-25.57	-29.99	-33.68	-37.27	-40.38	-43.35	-45.42	-46.86	-46.86
INLET	REL	FLOW												53.12											64.92	96.99	71.21
		STL	7											13.48											-1.1	-1.9	-3.2
	STA	SWEEP	ANGLE	4.55	3.28	1.55	.37	52	-1.43	-2.58	-2.43	-2.72	-3.51	-3.18	-3.37	-4.38	-5.82	-8.01	-10.71	-13.16	-15.02	-16.76	-19.56	-21.13	-23.34	-25.33	-24.94
		STL	NO	П	7	3	4	2	9	7	∞	0	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24

Table A3. Quiet High Speed Fan Stator Vane Airfoil Geometry

)3	*- -212369	ÞН	ωz					18	37																	
	AIRFOI	VANE 1 AT INCIDENCE	STL ST. NOG	24 23	3 2	2	1 17	20 16	ν α Σ Γ Γ	7 12	6 12	5 12	4 12	3 12	2 1	1 1	0 12	13	13	1	1	5 14	П	3 14	2 15	Н
	1*AIRFOIL GEOMETRY	_ 124	STAG CA -GER	•	.51 7	.61 6	.81 6	.07	. XI . V	.57	.52 5	.60 5	.75 5	.82 5	. 79 5	.63 5	.67 5	.02 5	. 22 5	.73 5	.10	. 21	1.43 6	•	•	7 14 7
	ŒTRY		CAMBER	76.05	2.8	ο.	9	m	ν α	N	Ω	2	9	57.34	_	_	∞	∞	σ	88.69	•	.3		53.86		72.84
	NASA/A: Design Rotor	ION 22 HAS METAL AN	SOLID -ITY	2.191	.10	2.014	1.905	•	1./1/	.67	.67	1.670	1.674	1.686	1.709	1.745	1.799	1.854	1.882	1.954	2.008	.05	.12	2.197	. 27	2,381
	NASA/AE QHSF Fan. Design report gec Rotor and stator	52 A	TMAX/ CHORD	.0453	.0609	.0687	O	.0654	1840.	.0452	.0471	.0505	.0529	.0532	.0523	.0517	2	.0551	.0562	.0579	.0580	.0574	.0560	.0554	.0571	0637
	dan. geo	AXIARB DEVIATION	THROAT /SPACE	.5802	07	.6408	.6794	.7166	7007	.7943	.7940	.7896	.7843	•		.7775		.7623			.7379	.7313	.7214	.7112	.6871	6256
	מ	AIRFO RULE	CHORD	ω	-		2.413	2.270		1.939		•	٠.	1.733	1.6	1.6	1.6	1.6		1.6	1.662	1.670	1.686	1.703	1.721	1,748
,	Point. 22. Final core	FOILS EMETAL	INLET METAL ANGLE	63.49	0	57.35	4.	51.1	46.94	44.0	4	4	44.45	44.78	44.85	44.90	45.19	45.77		46.98		48.46	49.52	50.57	52.82	58.14
)		ø	EXIT METAL ANGLE	-12.56-1	-12.4	-12.3	-12.22	-12.16	-12.08	-11.7	-11.	-11.98	-12.28	-12.56	-12.78	-13						-12.90	-13.01	-13.29	-13.97	-14.70
_	otor LE ti flowpath models.		 MEAN LINE	-14.77	2	5-15.26	2.9	0 0	-7. IU		-3.35	-3.03	-3.06	-3.22	-3.43	-3.79	-4.30	-4.87	-5.18	-5.91	-6.44	-6.99	-7.66	-8.26	-9.86	-4 15
			INCIDENCE SUCTION FCI	-17.08	-19.22	-19.10	-16.73	-14.10	1 . 80	-6.01	-5.20	-5.05	-5.26	-5.45	-5.53	-5.67	-7.86	-8.52	-8.88	-9.75	-10.31	-10.78	-11.29	-11.77	-13.35	ر م
	T)		ENCE	-7.90	-8.60	-11.61	-9.54	-6.82	-6.03	-2.62	-2.84	-2.69	-2.44	-2.25	-2.20	-2.39	-2.47	-3.94	-3.85	-3.39	-4.94	-4.80	-4.58	-4.48	-5.84	-1 43
	loss.		 DELTA	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	0
•			INLET REL. MACH	.516	.547	.574	.600	.627	. 000.	.706	.722	.737	. 753	.771	.791	.814	.839	.858	.867	.887	006.	.911	.926	.942	996.	890
	97/03/07 15:44:24		PASSAGE ANGLE	25.46	χ,	22.50	•	19.53	17.44	16.13	16.00	16.01	16.10	16.11	16.05	15.95	16.02	16.35	16.54	17.04	17.41	17.78	18.27	18.67	19.46	21 72
	/07 :24		MIN. A/A* NS	1.092	07	1.076	10	1.112	1.121 1 123	1.121	1.117	1.110	1.101	1.090	1.076	1.060	1.040	1.021	1.013	.993	086.	.971	.957	.946	. 923	977
			PASSAGE ANGLE LOCATION	.302	.297	. 293	. 288	.285	5/2.	. 259	.260	.259	.262	.263	.260	.256	.253	.251	.248	.246	.244	.242	.237	.234	.233	245
			MAXIMUM THICKNES	.600	.600	. 600	.600	. 600	009.	009.	.600	. 600	. 600	. 600	.600	.600	.600	.600	.600	.600	. 600	.600	. 600	. 600	.600	009
			f iss v cri	1.6	1.68	1.70	1.73	1.75	٦ -		1.92	1.92	1.90	1.9(1.92	1.9	1.9	1.99	2.0.	2.03	2.04		2.13	2.13	2.14	2.0

1.658 1.658 1.708 1.

Table A3. Quiet High Speed Fan Stator Vane Airfoil Geometry (cont.)

INCI- DENTAL ANGLE	5.57 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 3.32 4.74 4.74 4.74 6.08 6.08 6.11 6.08	
НД	E C 7 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	
SUCTION SURFACE MACH NO	1.303 1.	
MAX CAMBER LOC	0 0 0 1 8 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
DELTA S	88 88 88 88 88 88 88 88 88 88 88 88 88	
SHOCK		
 RADIAL	10.889 10.757 10.607 10.293 10.293 10.293 10.293 10.293 10.293 10.293 10.326	
 R	$\begin{smallmatrix} 2 & -1 & -1 & -2 & -2 & -1 & -1 & -1 & $	
EDGE COORDINATES FRONT REA AXIAL RADIAL AXI	10.889 10.730 10.586 10.586 10.069 9.825 9.825 9.067 8.796 8.796 8.796 7.924 7.924 7.925 6.994 6.823 6.994 6.823	
EDGE CO FRONT AXIAL 1	-5.075 -5.046 -5.046 -4.999 -6.2382 -6.2382 -6.2382 -6.528 -6.528 -6.528 -6.528 -6.528 -6.528 -6.528 -7.076 -7.076	
Y-CG	2. 13 1. 1. 8116 1. 1. 1144 1.	
X-CG	3 3 3 3 4 4 4 8 8 8 8 8 8 8 8 8 8 8 8 8	
CONE		23
STACK RADIUS	900004W9U0000WWWUUHU49000WUU	-/.2/23
TRAIL -ING RADIUS	. 0404 . 0383 . 0372 . 0306 . 0254 . 0212 . 0191 . 0147 . 0153 . 0163 . 0163 . 0163 . 0163 . 0163 . 0163 . 0163	NATE IS
LEAD - ING RADIUS		S COORDINATE
-DEVIATION- MEAN DELTA LINE		STACKING AXIS
STL -DEVIANO. MEAN		THE STACK

Table A3. Quiet High Speed Fan Stator Vane Airfoil Geometry (concluded)

	SHOCK	OMEGA	BAR	.029	.002	000.	.003	.003	.002	.002	.001	.001	.001	000.	000.	000.	000.	000.	000.	000.	000.	000.	000.	000.	000.	000.	000.
		NO	s.s	1.497	1.308	1.303	1.289	1.290	1.287	1.285	1.281	1.280	1.270	1.252	1.225	1.202	1.178	1.150	1.120	1.077	1.035	086.	.884	.812	.735	.687	.653
		MACH	INLET	.968	996.	.942	.926	.911	006.	.887	.867	.858	.839	.814	.791	.771	.753	.737	.722	.706	.689	.665	.627	009.	.574	.547	.516
H	OBLQ	SHOCK	ANGLE	70.24	66.16	62.34	61.31	60.92	61.27	61.43	61.43	60.79	60.67	60.61	60.80	60.94	86.09	59.52	57.50	54.74	52.15	55.56	60.63	64.86	62.90	59.45	57.16
IMPINGEMENT POINT	BLADE	LEAN	ANGLE	-15.68	-12.09	-6.99	-5.02	-4.30	-4.13	-3.84	-3.37	-2.51	-1.54	23	. 73	1.33	2.46	5.71	10.97	18.34	28.27	42.13	52.31	58.31	60.01	60.68	60.85
PINGEME	REL	FLOW	ANGLE	38.14	34.65	33.06	32.47	31.55	31.07	30.40	29.48	29.04	28.36	27.88	27.88	27.65	27.33	27.00	26.65	26.91	27.40	28.70	31.89	34.44	37.07	39.73	41.89
SHOCK IM		STL	ANGLE											2.57													
Ø	STA	SWEEP	ANGLE	29.02	29.63	29.94	30.18	30.59	30.47	30.53	30.84	31.25	31.32	31.31	31.26	31.34	31.16	31.52	31.64	31.98	31.75	23.32	8.72	-7.42	-10.21	-8.49	-7.15
	OBLQ	SHOCK	ANGLE	73.18	67.26	66.63	66.42	66.36	66.23	66.17	66.40	66.10	66.07	66.49	67.02	67.07	66.29	64.25	61.17	57.19	53.89	55.91	63.10	66.29	64.67	60.41	57.25
H	BLADE	LEAN	ANGLE	-5.40										-1.06											52.06	53.48	53.67
INLET	REL	FLOW	ANGLE	53.99	42.96	42.30	41.86	41.47	41.26	41.07	40.93	40.90	40.89	41.11	41.42	41.55	41.39	40.95	40.41	39.92	39.66	39.84	40.63	41.28	42.09	44.72	48.72
		STL	ANGLE	13.29	11.68	10.51	9.55	8.55	7.82	6.99	5.86	5.42	4.49	3.46	2.58	1.82	1.14	. 52	06	57	96	-1.08	50	.12	. 79		
	STA	SWEEP	ANGLE	25.36	26.25	27.31	27.72	28.15	28.50	28.83	29.22	29.57	29.69	29.61	29.47	29.36	29.35	29.46	29.71	30.09	28.80	18.90	.64	-12.01	-13.56	-10.98	-10.44
		STL	NO	П	7	٣	4	2	9	7	∞	0	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24

Table A4. Quiet High Speed Fan Rotor Blade Performance

1. Part Pa		&	* INCID DENTAL 6.41 2.91 2.91 2.91 2.16 2.10 2.29 4.11 4.11 3.87 3.42 2.02 1.43 1.21 1.21 1.21	
PASA/AE CHSF Fan. Design Point. 22-in rotor LE tip diameter. 15:44:24 15:44		Е 83.	** MEAN INC	0.7.6
PASSA/AE OHSF Fan. Design Point. 22-in rotor LE tip diameter. 15:44:24 Point and stator performance from DAWES wodels. 15:44:24 Point and stator performance from DAWES wodels. 15:44:24 Point and stator performance from DAWES wodels. 15:45 Point and Stator Performance from DAWES wodels.		SPEED	* E	
NASA/AE QHSF Fan. Design Point. 22-in rotor LE tip diameter. Rotor and stator performance from DAWES models. RATION 16 HAS 22 AXIARB AIRFOILS. A I R F O I L P R O P E R T I E S * * * * * * * * * * * * * I N C I D E N C E AAAACGER CAMBER SOLID TWAX THROAT CHORD MACH NUMBER 2-90 100 110 110 110 110 110 110 110 110 1	/03/0	FLOW	$\begin{array}{c} \mathbb{Z} \ 1 \ \Omega \\ \mathbb{Z} \ 1 \ \Omega \\ \mathbb{Z} \ 0 \\ \mathbb{Z} \ $. 93 999.0 . 17 999.0 . 83 999.0
ANGER CAMBER SOLID THROAT CHORD MASCH NICE TIP diameter. ANGER CAMBER SOLID THAX THROAT CHORD MACH NUMBER 2-D INLET CHORD LISS 150 1.549 0.020 4.27 4.626 1.455 0.814 0.89 6.69 1.450 0.81 0.80 0.81 0.80 0.81 0.80 0.81 0.80 0.81 0.80 0.81 0.80 0.81 0.80 0.82 0.84 0.84 0.85 0.85 0.84 0.85 0.85 0.85 0.85 0.85 0.85 0.85 0.85	o u	o. SI T	C E E E E E E E E E E E E E E E E E E E	5.24-999.0 2.20-999.0 0.53-999.0 0.62-999.0
. NASA/AE OHSF Fan. Design Point. 22-in rotor LE tip di Design report geometry. Final core duct flowpath W/Rotor and stator performance from DAWES models. TATION 16 HAS 22 AXIARB AIRFOLLS AGGER CAMBER SOLID TWAX THROAT CHORD MACH NUMBER NACH 1.55 13.30 1.490 .020 .427 4.626 1.425 .814 ANG -ITY /C /S ILSG .021 .458 4.596 1.465 .839 8.42 10.67 1.559 .021 .458 4.596 1.459 .845 5.56 10.82 1.559 .021 .472 4.580 1.459 .845 5.56 10.62 1.626 .023 .559 4.313 1.383 .855 5.56 10.63 1.626 .023 .554 4.354 1.351 .855 5.56 11.16 1.649 .024 .575 4.275 1.316 .854 8.64 12.26 1.683 .025 .594 4.124 1.202 .835 5.57 11.16 1.649 .024 .575 4.275 1.316 .855 6.66 15.93 1.782 .031 .675 4.090 1.160 .823 5.22 0.6 1.925 .040 .655 4.057 1.117 .811 6.27 22.98 2.08 2.08 .045 .668 3.919 .973 .789 7.10 34.54 2.282 .047 .705 3.893 .944 .788 5.09 35.54 2.282 .047 .705 3.893 .944 .789 5.09 35.54 2.282 .047 .705 3.893 .944 .789 6.06 38.05 2.533 .049 .730 3.846 .854 .789 6.07 38.05 2.533 .049 .730 3.841 .732 .731	ct los		INLET INLET INLET INLET INLET INLET INLET INLET IN C 1 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2	680.7
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ASA/AE QHSF Fan. Design Point. 22-in rot Design report geometry. Final core duct Rotor and stator performance from DAWES marton 16 HAS 22 AXIARB AIRFOILS A I R F O I L P R O P E R T I E S * * * * AGGER CAMBER SOLID TWAX THROAT CHORD IN ANG -ITY /C /S I 1.509 .021 .444 4.610 1.490 .020 .427 4.626 1.491 .020 .427 4.626 1.491 .020 .427 4.626 1.491 .020 .427 4.639 1.492 .021 .448 4.596 1.491 .020 .472 1.633 1.607 .022 .532 4.438 1.330 1.607 .022 .532 4.438 1.330 1.607 .022 .532 4.438 1.330 1.607 .022 .532 4.438 1.330 1.607 .023 .554 4.354 1.330 1.607 .024 .575 4.212 1.261 1.683 .025 .594 4.312 1.261 1.683 .025 .594 4.314 1.226 1.683 .025 .594 4.314 1.226 1.683 .025 .594 4.314 1.226 1.683 .025 .594 4.314 1.226 1.683 .025 .594 4.318 1.331 1.32 2.06 1.925 .040 .655 4.057 1.166 .025 .031 .627 4.124 1.226 .025 .031 .627 4.124 1.226 .031 .627 4.124 1.226 .031 .025 .031 .627 4.124 1.226 .031 .025 .031 .627 4.124 1.226 .031 .025 .031 .627 4.124 1.226 .031 .025 .031 .627 4.124 1.236 .031 .025 .031 .035 .031 .025 .031 .035 .031 .031 .031 .031 .031 .031 .031 .031				0 00 00
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NASA/AE QHSF Fan. Design Poin Design report geometry. Fina Rotor and stator performance Rotor and stator performance and stator performance ALTION 16 HAS 22 AXIARB AIRFOI ANG	, O Q	លី	Ω Ω 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	
NASA/AE QHSF Fan. Design report gec Rotor and stator TATION 16 HAS 22 AXIAR AGGER CAMBER SOLID T ANG ANG 11.63 1.509 9.48 10.74 1.526 8.42 10.67 1.543 7.42 10.70 1.559 9.48 10.74 1.526 8.42 10.67 1.543 7.42 10.70 1.559 9.48 10.74 1.526 8.42 10.67 1.543 7.42 10.70 1.559 9.48 10.63 1.607 1.59 11.16 1.649 8.64 12.26 1.683 8.64 12.26 1.683 8.64 12.26 1.683 8.64 12.26 1.683 9.50 11.16 1.489 9.32 22.06 1.925 9.32 22.06 1.925 9.32 22.06 1.925 9.32 22.06 1.925 9.32 22.06 1.925 9.25 33.12 2.214 7.10 34.54 2.282 9.25 38.71 2.437 9.60 38.05 2.533 9.60 38.05 2.849	Poin Fina nce		HFI	. 730 . 740 . 749
. TAT TATE TO THE TATE TATE TO THE TATE TATE TO THE TATE TATE TATE TATE TATE TATE TATE	ometry.	ARB	TWAX /C	.049
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*LOSS AND DEV. ROTOR 1 AT STATION 5444. * * * * * * A I STL PCT STAGGER NO. SPAN ANG 24 100.00 61.55 23 95.96 60.47 22 92.44 59.48 21 89.22 58.42 20 86.12 57.42 19 80.42 55.56 18 75.13 53.88 17 69.97 55.23 16 64.80 50.50 18 75.13 53.88 17 69.97 52.23 10 31.35 36.27 11 37.22 39.32 10 31.35 36.27 9 26.71 33.72 10 31.35 36.27 9 26.71 33.72 10 31.35 36.27 9 26.71 33.72 10 31.35 36.27 9 26.71 33.72 10 31.35 36.27 9 26.71 32.20 1 19.96 29.25 6 16.86 27.10 5 14.34 25.09 4 11.10 22.50 3 8.13 19.60 2 4.59 15.91	NASA/AE (Design re Rotor and	16 HA	PO I CAMBER ANG ANG 113.30 111.63 110.74 110.74 110.70 110.82 110.82 110.82 110.82 113.87 115.93 112.26 113.87 115.93 112.38 112.38 112.38 113.33 112.38 113.33 112.38 113.33 113	8.0 9.1 1.1
*LOSS AND D ROTOR 1 AT 5444. * * * * * * STL PCT NO. SPAN 24 100.06 23 95.96 22 92.44 21 89.22 20 86.12 19 80.42 19 80.42 11 89.22 20 86.12 11 89.22 12 86.12 13 42.98 11 37.22 10 31.35 11 31.35 12 42.98 11 31.35 12 42.98 11 31.35 12 42.98 13 42.98 14 44.69 7 19.96 6 16.86 6 16.86 7 19.96 1 11.10	EV.		* A I STAGGER ANG 61.55 60.47 60.47 59.48 59.48 55.56 55.55 56.23 644.50 48.64 44.50 42.12 39.32 33.72 33.21 29.25 27.10 25.09	0.00
* (<u>r. lf</u>) *	LOSS AND E	OR 1	STL PCT NO. SPAN 24 100.00 23 95.96 22 92.44 22 86.12 20 86.12 19 75.13 48.63 11 37.22 10 31.35 9 26.71 9 26.71 9 26.71 9 6.86 11 37.22 10 31.35 9 26.71 9 6.86 11 37.22 10 31.35 9 26.71 9 6.86 11 37.22 10 31.35 9 26.71 9 2	4.5 4.5 0.

Table A4. Quiet High Speed Fan Rotor Blade Performance (cont.)

* * * *	REY	X 10**-	26.74	28.01	28.12	27.87	27.48	26.57	25.51	24.39	23.31	22.31	21.39	20.50	19.62	18.73	17.79	17.01	16.67	15.85	15.31	14.89	14.35	13.86	13.30	11.88
* * *	DEVIATION)	INPUT)	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
* * *	(DELTA DE	MACH NO	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
MARY	SP 36 (_	5.06	5.31	4.92	4.73	4.52	4.24	4.06	4.03	4.18	4.52	4.99	5.57	6.29	7.04	7.86	8.41	8.48	8.89	8.96	8.89	8.83	8.67	8.17	7.60
SUM	CARTER	RULE	4.46	5.44	5.13	4.94	4.66	4.34	4.19	4.23	4.48	4.93	5.54	6.26	7.15	8.21	9.46	10.33	10.41	10.90	10.74	10.44	10.26	10.62	10.95	11.69
N O I	OVER-	LORD	4.14	5.14	5.36	5.45	5.54	5.65	5.78	5.97	6.27	6.73	7.27	7.90	8.63	9.48	10.32	10.80	10.87	10.57	9.94	9.19	7.90	6.41	5.45	66.9
IAT	APPLIED	DEV	10.06	7.91	6.42	5.50	4.55	3.28	2.39	1.77	1.42	1.33	1.41	1.60	2.05	2.78	3.60	3.93	4.04	4.12	3.77	3.35	2.61	2.07	. 62	-14.99
* D E V	ANGLES	AIR	62.45	60.49	58.70	56.84	54.94	51.88	49.59	47.46	45.15	42.61	39.81	36.67	33.19	29.38	25.21	21.63	20.01	16.08	13.35	11.03	7.82	4.62	.33	-19.48
* *	EXIT	METAL	52.39	52.58	52.28	51.34	50.39	48.60	47.19	45.69	43.73	41.29	38.41	35.08	31.15	26.60	21.60	17.71	15.97	11.96	9.58	7.68	5.21	2.55	29	-4.49
*	INPUT	TABLE	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.
* * * X	APPLIED	LOSS	.0819	.1321	.1279	.1300	.1245	.1121	.1057	.1061	.1110	.1175	.1221	.1205	.1110	.0941	.0752	.0628	.0578	.0464	.0414	.0393	.0408	.0451	.0550	.3923
MAR	TOTAL A	LOSS	.1462	.1199	.0962	.0783	.0629	.0534	.0628	.0747	.0880	.0948	.0912	.0835	.0751	.0687	.0621	.0570	.0550	.0457	.0420	.0401	.0414	.0524	.0673	.1234
SUM	MIG.	LOSS	0960.	.0714	.0502	.0324	.0158	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	.0034	.0151	.0307	.0593
L N	REY NO	LOSS	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.
PONENT	SHOCK	LOSS	0000.	0000.	0000.	.0002	.0020	.0093	.0187	.0299	.0421	.0478	.0431	.0350	.0268	.0213	.0158	.0116	.0102	.0026	.0004	0000.	.0002	0000.	0000.	0000.
COM		LOSS	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.
Ω Ω	DRAG	RISE	.0023	.0022	.0020	.0020	.0019	.0019	.0018	.0018	.0019	.0020	.0021	.0021	.0022	.0022	.0022	.0022	.0021	.0020	.0018	.0015	.0011	.0007	.0002	0000.
0 7 *	ROFILE	LOSS	.0479	.0463	.0440	.0437	.0431	.0423	.0423	.0430	.0440	.0451	.0460	.0464	.0461	.0452	.0441	.0432	.0427	.0411	.0398	.0386	.0367	.0366	.0364	.0640
	STL P NUM.	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	S	4	æ	7	Н

Table A4. Quiet High Speed Fan Rotor Blade Performance (concluded)

DELTA	IMIN	3.12	3.19	3.23	3.50	3.78	4.36	5.02	5.29	5.33	5.49	5.71	5.95	6.12	5.73	4.87	4.08	3.47	2.68	1.81	1.20	. 25	10	75	-2.34
	T2CR	.958	.978	.037	.062	.083	.136	.174	.345	.592	.709	.633	.432	.258	.173	.310	.445	.533	.834	.139	.269	.499	.313	2.9198	.382
	CR	50	58	99	87	09	56	90	57	09	9	03	37	55	29	84	18	44	64	87	11	45	.8809	2	1.0834
STL							19										Q	∞	7	9	2	4	3	7	П

(X) TO THE RIGHT OF A NUMBER IN THE ABOVE COLUMNS INDICATES A PROGRAM DEFAULT IS IN EFFECT NOTE.... THE LETTER

Table A5. Quiet High Speed Fan Stator Vane Performance

Table A5. Quiet High Speed Fan Stator Vane Performance (cont.)

* * *	REY	X 10**-		8.76	8.72	8.53	8.38		8.10	8.00	7.87	7.74	7.61	7.53	7.50	7.52	7.59	7.66	7.70	7.80	7.87	7.94	8.05	8.17	8.32	7.91
* * *	DEVIATION)	INPUT)	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
* * *	DELTA DE	MACH NO	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
MARY	SP 36 (11.58	10.82	10.26	9	9.61	•	8.72	9.	۲.	8.96		9.26		8.95	•	8.57	8.51	8.41	8.34	•	8.00	∞.	8.06	10.53
SUM	CARTER	RULE	12.92	12.46	11.99	11.60	11.18	10.55	10.18	10.01	9.98	10.01	10.20	10.27	10.24	10.14	10.04	66.6		96.6	9.98	10.00	10.05	10.12	10.44	11.49
N O I	OVER-	LORD	24.86	21.63	18.75	16.19	14.00	11.47	10.43	10.01	9.87	9.98	10.20	10.36	10.40	10.42	10.56	10.85	11.03	11.49	11.89	12.32	12.95	13.61	15.14	19.45
IAT	APPLIED	DEV	12.56	12.46	12.35	12.22	12.16	12.08	11.93	11.78	11.77	11.98	12.28	12.56	12.78	13.03	13.17	13.06	13.00	12.90	12.89	12.90	13.01	13.29	13.97	14.70
DEV	ANGLES	AIR	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
* * *	EXIT A	METAL	-12.56	-12.46	-12.35	-12.22	-12.16	-12.08	-11.93	-11.78	-11.77	-11.98	-12.28	-12.56	-12.78	-13.03	-13.17	-13.06	-13.00	-12.90	-12.89	-12.90	-13.01	-13.29	-13.97	-14.70
*	INPUT	TABLE	0000.	0000.		0000.	0000.		0000.	0000.	0000.				0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	.0000
* * * 2	APPLIED	LOSS	3218	.0085	.1222	.1988	.1730	.1010	.0832	.0654	.0503	.0423	.0386	.0373	.0374	.0386	.0409	.0435	.0447	.0459	.0446	.0421	.0343	.0328	.0677	0876
MARY	TOTAL A	LOSS	.2645 -	.3094	.2429	.1271	.0475	.0322	.0388	.0431	.0467	.0502	.0538	.0573	.0625	.0738	.0841	.0816	.0740	.0548	.0494	.0492	.0492	.0567	.0930	.1320 -
SUM	MIG.	LOSS	.0641	.0379	.0227	6600.	.0005	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	.0100	.0321	.0580
[L	REY NO	LOSS	.0000	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.
P O N E	SHOCK REY NO	LOSS	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	0000.	.0001	.0005	.0010	.0012	.0017	.0022	.0026	.0031	.0001	.0018	.0292
COM	INC.	LOSS	.1731	.2338	.1774	.0730	.0079	.0021	.0110	.0162	.0197	.0221	.0240	.0255	.0277	.0350	.0416	.0356	.0270	8900.	.0007	0000.	.0001	.0011	.0111	.0025
S	DRAG	RISE	0000.	0000.	0000.	0000.	0000.	0000.	0000.	.0001	.0002	.0004	9000.	6000.	.0011	.0014	.0018	.0020	.0021	.0022	.0022	.0022	.0022	.0022	.0023	.0020
0 1 *	PROFILE	LOSS	.0273	.0377	.0428	.0442	.0390	.0300	.0278	.0268	.0267	.0277	.0292	.0310	.0337	.0372	.0402	.0430	.0438	.0442	.0443	.0444	.0438	.0434	.0456	.0404
* *	STL P	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	2	4	3	7	1

Table A5. Quiet High Speed Fan Stator Vane Performance (concluded)

DELTA	IMIN	ω.	.5	.5	-5.54	4.	-8.01	-8.85	4.	. 7	-9.29	. 7	4.	4.	-9.12	.5	ς.	-9.34	9.	-8.80	-9.11	4.	9.	-9.45	-8.73
	T2CR	.146	.993	.114	.298	.472	.845	.244	.710	.952	.833	.642	.587	.698	.916	.963	.735	.587	.255	.098	.029	.997	2.9716	27	2.3887
	껖	.126	.175	.224	.270	.308	.364	.384	.379	.369	.354	.339	.326	.316	.338	.356	.367	.382	.390	.398	.435	.473	.511	25	.501
STL	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	Q	∞	7	9	2	4	3	7	1

TO THE RIGHT OF A NUMBER IN THE ABOVE COLUMNS INDICATES A PROGRAM DEFAULT IS IN EFFECT (x)NOTE.... THE LETTER

Table A6. AXCAPS Design Point Data for Quiet High Speed Fan (30 pages)

			.00 1.0
	m	는 물	00.
97/03/07 15:44:24	4 1.40083	***	0.
97/1	49 6 GAMMA	H H H	12.900
loss.	PRESSURE 12.49 TEMPERATURE 524.6	ABSOLUTE MACH NO. . 522 . 522 . 522 . 522 . 522 . 522 . 522 . 522 . 522 . 524 . 527 . 528 . 529 . 529 . 549 . 549 . 499 . 496 . 486 . 463	.409
tip diameter. th w/ -60 duct	TOTAL PRESSURE TOTAL TEMPERAT	AATUURE STATURE 4997. 1007.	507.6
tip di ath w/		TEMPE TOTAL TOTAL TOTAL TOTAL 524.0 5224.0 5225.0 5224.0 5226.0	524.6
otor LE flowpa models	W 98.65 W 37.49	SURE STATHC 10.38 10.38 10.38 10.37 10.37 10.37 10.38 10.36 10.50 10.46 10.46 10.50 10.50 10.50 10.50 10.50	11.14
t. 22-in rotor LE tig 1 core duct flowpath from DAWES models.	CORR. FLOW SPEC. FLOW	<u> </u>	12.49
rt Fr	8 CC 9	* \$\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	451.
gn · rma	83.3	M * * M	451.
NASA/AE QHSF Fan. Design Design report geometry. Rotor and stator performs	FLOW AREA	TAN	.0
LE QHSF report and sta		TTIES RADIA 0 0 1 1 1 1 1 1 2 2 3 3 3 3 3 3 4 0 0 0 0 0 0 0 0 0 0 0 0 0	06
NASA/A Design Rotor		VELOC AXIAL 571. 571. 571. 571. 571. 571. 571. 571.	442.
ORS	ON 7	0 0 1 1 8 1 9 1 9 1 9 1 9 1 9 1 9 1 9 1 9 1	.000-19.617
1*VECTORS	STATION	NOU NO 24. 24. 22. 23.4 11. 11. 11. 11. 11. 11. 11. 11. 11. 1	Н

		FLOW			CORR. FLOW	W 98.65	TOTAL	AL PRESSURE		. 49	1 40083	~	
84. 0.		ARE	584.	584	•	رد 10.2	524		2	.6 GAIMINA 15.053		100.00	100.00 1.0000
.845.	5	0		5	12.49	10.28	524.6	496.2	.535	15.053	0.	98.36	96.94 1.0000
	5	0		28	12.49	10.28	524.6	496.2	.535	15.053	0.	96.70	93.88 1.0000
	كا	0		. 584.	12.49	10.28	524.6	496.2	.535	15.053	0.	95.01	90.83 1.0000
	М	0		28	12.49	10.28	524.6	496.2	.535	15.053	0.	93.29	87.77 1.0000
	ή.	0		. 584.	12.49	10.28	524.6	496.2	.535	15.053	0.	89.77	81.65 1.0000
	3.	0		. 584.	12.49	10.28	524.6	496.2	.535	15.053	0.	86.12	75.54 1.0000
	. 9	0		. 584.	12.49	10.28	524.6	496.2	.535	15.053	0.	82.31	69.42 1.0000
		0		. 583.	12.49	10.29	524.6	496.2	.534	15.053	0.	78.35	63.30 1.0000
	4.	0			12.49	10.29	524.6	496.3	.533	15.054	0.	74.20	57.19 1.0000
		0		. 581.	12.49	10.31	524.6	496.5	.532	15.054	0.	69.82	51.07 1.0000
	3.	0		. 579.	12.49	10.32	524.6	496.7	.529	15.054	0.	65.18	44.96 1.0000
		0		. 576.	12.49	10.34	524.6	497.0	. 526	15.055	0.	60.20	38.84 1.0000
	5.	0		. 571.	12.49	10.37	524.6	497.4	.523	15.056	0.	54.82	32.72 1.0000
		0		. 566.	12.49	10.41	524.6	497.9	.517	15.057	0.	48.91	26.61 1.0000
		0		. 560.	12.49	10.45	524.6	498.4	.512	15.058	0.	43.88	21.91 1.0000
	1.	0		. 557.	12.49	10.47	524.6	498.7	. 509	15.058	0.	41.57	19.91 1.0000
		0		. 550.	12.49	10.52	524.6	499.4	.501	15.059	0.	35.87	15.37 1.0000
539. 64.	4.	0	. 543.	. 543.	12.49	10.56	524.6	500.0	.495	15.061	0.	31.81	12.50 1.0000
		0		. 537.	12.49	10.61	524.6	500.6	.489	15.062	0.	28.28	10.25 1.0000
	6	0		. 527.	12.49	10.67	524.6	501.4	.480	15.064	0.	23.40	7.50 1.0000
	9	0		. 516.	12.49	10.74	524.6	502.4	.469	15.067	0.	18.41	5.12 1.0000
		0		. 497.	12.49	10.86	524.6	504.0	.451	15.074	0.	11.36	2.50 1.0000
435. 203.	3.	0	. 480	. 480.	12.49	10.97	524.6	505.4	.435	15.258	0.	00.	.00 1.0000

STATION	0			FLOW	83.38	S		98.6	TOTAL			.49				
			,	AREA	364.9	SP	SPEC. FLOW	1 38.92	TOTAL	AL TEMPERATUR	囝	524.6 GAMMA 1	.40083			
24 11	.001 - 15.490	434.	.0	0	434.	434.	11.15	10.03	524.6	508.9	.392	17.026	0.	100.00	100.00	.9680
23 10	.801-15.490	567.	-3.	0	567.	567.	12.04	10.03	4.	497.8	.518	17.027	0.	97.79	96.94	9886.
22 10	.629-15.490	602.	-3.	0	602.	602.	12.34	10.03	524.6	494.4	.552	17.027	0.	95.90	93.88	.9965
21 10	.462-15.490	614.	-2.	0	614.	614.	12.45	10.04	524.6	493.2	.564	17.027	0.	94.07	90.83	.9991
10	.295-15.490	617.	-1.	0	617.	617.	12.48	10.04	524.6	492.9	. 566	17.027	0.	92.22	87.77	7666.
σ	.954-15.490	619.	3.	0	619.	619.	12.51	10.05	524.6	492.7	. 569	17.027	0.	88.46	81.65 1	.0004
18 9	.602 - 15.490	620.	7.	0	620.	620.	12.54	10.06	524.6	492.5	.570	17.027	0.	84.59	75.54 1	.0010
Q	.238-15.490	619.	12.	0	619.	619.	12.54	10.07	524.6	492.6	. 569	17.027	0.	80.58	69.42 1	.0011
ω	.858-15.490	618.	17.	0	618.	618.	12.55	10.08	524.6	492.8	. 568	17.027	0.	76.39	63.30 1	.0012
∞	5	615.	23.	0	615.	615.	12.55	10.10	•	493.0	. 565	17.028	0.	72.02	57.19 1	.0012
∞	.042 - 15.490	612.	29.	0	612.	612.	.5	10.12	4	493.3	.562	17.029	0.	67.40	51.07 1	.0013
13 7	.599-15.490	608.	36.	0	.609	.609	12.55	10.15	4.	493.7	. 559	17.030	0.	62.52	44.96 1	.0013
12 7	.126 - 15.490	602.	44.	0	604.	604.	12.55	10.19	524.6	494.2	. 554	17.032	0.	57.31	38.84 1	.0013
11 6	.615-15.490	596.	53.	0	598.	598.	12.55	10.23	524.6	494.8	.548	17.035	0.	51.67	72 1	.0013
10 6	.057-15.490	587.	63.	0	591.	591.	12.55	10.28	524.6	495.5	.541	17.039	0.	45.52	26.61 1	.0013
5	.584-15.490	579.	72.	0	583.	583.	12.55	10.33	524.6	496.2	.534	17.043	0.	40.32	21.91 1	.0013
2	.369-15.490	575.	77.	0	580.	580.	12.55	10.36	524.6	496.6	.531	17.045	0.	37.94	19.91 1	.0013
7 4	.839-15.490	563.	. 68	0	570.	570.	12.55	10.43	24.	497.5	.521	17.052	0.	32.10	15.37 1	.0013
6 4	.466-15.490	554.	. 66	0	563.	563.	12.55	10.48	524.6	498.2	.514	17.058	0.	28.00	12.50 1	.0014
5 4	.147-15.490	546.	108.	0	556.	2	12.55	10.53	524.6	498.8	. 508	17.065	0.	24.48	10.25 1	.0014
4 3	.712 - 15.490	533.	121.	0	546.	546.	12.55	10.59	524.6	499.7	.498	17.077	0.	19.69	7.50 1	.0014
ω	.280 - 15.490	518.	137.	0	536.	536.	12.55	10.66	524.6	500.7	.488	17.095	0.	14.93	5.12 1	.0014
2	.708-15.490	495.	164.	0	522.	522.	12.56	10.76	•	501.9	.475	17.138	0.	8.63	2.50 1	.0014
1 1	.925-15.490	401.	196.	0	446.	446.	12.11	10.82	524.6	508.0	.404	17.454	0.	00.	00.	.9911
1*VECTORS	ro.	NASA/AE	QHSF	Fan. Design	sign Po		22-in rotor	Ę	tip diameter	neter.		97/03/07	07			
		Design :	Design report geometry.	eomet		Final c	core duct	: flowpath	/M	-60 duct lo	loss.	15:44:	: 24			
		Rotor ar	and stato	r per	stator performance	se from	m DAWES	models.								

		α		.9680	9886.	.9965	.9991	7666.	1.0004	1.0010	1.0011	1.0012	1.0012	1.0013	1.0013	1.0013	1.0013	1.0013	1.0013	1.0013	1.0013	1.0014	1.0014	1.0014	1.0014	1.0014	0011
		PERCENT	FLOW	100.00	96.94	93.88	90.83	87.77	81.65 1	75.54	69.42	63.30 1	57.19 1	51.07	44.96	38.84	32.72	26.61	21.91	19.91	15.37	12.50	10.25 1	7.50 1	5.12 1	2.50 1	0
	8	PERCENT	SPAN	100.00	97.62	95.55	93.55	91.54	87.45	83.26	78.93	74.44	69.77	64.87	59.71	54.22	48.33	41.95	36.61	34.19	28.32	24.26	20.84	16.29	11.93	6.49	00
	GAMMA 1.40083	R*VU		0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	C
12.49	9.	MERIDINAL	DISTANCE	18.699	18.718	18.735	18.751	18.768	18.801	18.835	18.871	18.906	18.935	18.958	18.975	18.988	19.000	19.012	19.023	19.029	19.046	19.063	19.080	19.109	19.149	19.234	19,650
	TEMPERATURE 524	ABSOLUTE	MACH NO.	.418	.539	.572	.585	. 588	. 593	. 596	. 598	. 600	. 600	. 599	. 597	. 594	. 590	.584	.579	.576	. 569	.564	. 559	.552	.546	.539	479
AL PRESSURE		ATURE	STATIC	506.8	495.8	492.3	491.0	490.6	490.1	489.7	489.5	489.3	489.3	489.4	489.7	490.0	490.4	491.0	491.6	491.8	492.6	493.2	493.7	494.4	495.0	495.7	501.5
TOTAL	TOTAL	TEMPERATURE	TOTAL	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6
98.64		URE	STATIC	9.89	9.89	9.88	9.88	9.87	98.6	9.85	9.84	9.84	9.84	9.85	98.6	9.89	9.92	96.6	10.00	10.02	10.07	10.11	10.15	10.20	10.25	10.30	10.35
R. FLOW	C. FLOW	PRESSURE	TOTAL	11.15	12.04	12.34	12.45	12.48	12.51	12.54	12.54	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.56	12.11
CORR.	SPEC.	* *	ABS.	462.	588.	623.	635.	639.	643.	647.	649.	650.	.059	649.	647.	644.	640.	635.	629.	627.	620.	614.	.609	602.	.965	588.	526.
83.38	349.5	* * *	MERID.	462.	588.	623.	635.	639.	643.	647.	649.	650.	650.	649.	647.	644.	640.	635.	629.	627.	620.	614.	.609	602.	596.	588.	526.
FLOW	AREA	* * * *	TANG.	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
			RADIAL	0	5.	. &	11.	14.	20.	27.	34.	41.	49.	59.	. 69	80.	92.	107.	119.	125.	140.	152.	162.	176.	191.	214.	228.
		VELOCITIES	AXIAL B	462.	588.	623.	635.	639.	643.	647.	648.	649.	649.	647.	644.	639.	634.	626.	618.	614.	603.	595.	587.	576.	564.	548.	474.
FON 10		RADIUS	CORD.	11.001-13.818	10.808-13.799	10.641-13.782	10.479-13.766	10.316-13.749	9.985-13.716	9.645-13.683	9.294-13.648	8.930-13.613	8.552-13.585	8.154-13.565	7.735-13.551	7.290-13.541	6.811-13.535	6.293-13.532	5.860-13.530	5.663-13.529	5.187-13.527	4.857-13.524	4.580-13.523	4.210-13.521	3.856-13.520	3.415-13.518	2.888-13.517
STATION		STL	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	2	4	3	7	Н

		9680	9886	9962	9991	2666	0004	0010	0011	0012	0012	0013	0013	0013	0013	0013	0013	0013	0013	0014	0014	0014	0014	0014	9911
		00	94 . 9	88	83	5. 77	65 1.0	54 1.0	42 1.0	30 1.0	19 1.0	07 1.0	96 1.0	84 1.0	72 1.0	61 1.0	91 1.0	91 1.0	37 1.0	50 1.0	25 1.0	50 1.0	12 1.0	50 1.0	00
		100.	96.	93.8	90.8	87.	81.6	75.	69.	63.	57.	51.	44.	38.8	32.	26.6	21.9	19.	15.3	12.5	10.	7.1		2	Ŭ.
		100.00	97.61	95.49	93.43	91.37	87.18	82.89	78.47	73.91	69.18	64.21	58.99	53.44	47.49	41.07	35.70	33.28	27.42	23.39	20.01	15.53	11.28	6.05	00.
	1.40083	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
	GAMMA	.785	.813	.837	.862	.887	. 935	985	.036	.085	124	.151	.169	180	.187	194	.201	.205	.221	. 238	. 256	. 286	328	420	. 853
12.49	524.6	19.	19.	19.	19.	19.	19.	19.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.	20.
呂		.465	.578	.612	.625	.630	.634	.637	.638	.639	.639	.638	.634	.629	.623	.614	909.	.601	.591	.583	.576	.567	. 558	.549	.488
IL PRESSURE	I TEMPERATURE	502.8	491.7	488.0	486.4	485.9	485.5	485.2	485.0	484.9	484.9	485.1	485.5	486.0	486.8	487.7	488.7	489.1	490.3	491.2	491.9	492.9	493.8	494.8	500.7
TOTAL	TOTAL	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6	524.6
98.65	42.02	9.61	09.6	9.58	9.56	9.55	9.54	9.54	9.53	9.53	9.53	9.54	9.57	9.61	99.6	9.73	9.79	9.83	9.91	6.97	0.02	60.0	0.16	0.23	0.29
. FLOW	. FLOW	1.15	2.04	2.34	2.45	2.48	2.51	2.54	2.54	2.55	2.55	2.55	2.55	2.55	2.55	2.55	2.55	2.55	2.55	2.55	2.55 1	2.55 1	2.55 1	2.56 1	2.11 1
CORR	SPEC	11. 11	8. 1	63. 13	6. 1		85. 13	88. 13	89. 13	90. 13	90. 13	89. 13	85. 13	80. 13	74. 13		56. 13	52. 13	41. 13	Ξ.	26. 13	17. 13	08. 13	98. 13	35. 13
83.38	38.1	<u>г</u>	9	9	9	9	685. 6	9	9	9	9	9	9	9	9	55. 6	9	9	9	33. 6	9	9	9.80	98. 5	35. 5
							0. 68																		
FLOW	AREA																								
		-1	-2	-1	П	4	12.	21	31	41	52	63	74	87	101	117	132	139	156	169	179	194	210	230	236
		511.	628.	663.	676.	681.	685.	687.	689.	689	688.	.989	681.	675.	.999	654.	643.	637.	622.	611.	.009	586.	571.	552.	481.
N 11		11.001-12.732	10.820-12.704	10.659-12.680	10.504-12.656	10.348-12.631	10.030-12.583	9.705-12.534	9.370-12.484	9.023-12.437	8.663-12.402	8.286-12.379	7.889-12.367		7.015-12.365		6.118-12.380	5.935-12.384	.48	5.183-12.395	4.926-12.399	4.585-12.405	4.262-12.412	3.865-12.420	3.405-12.431
STATION							19 1										σ	∞	7	9	Ŋ	4	m	7	П

		.9680	9886.	.9965	.9991	7666.	.0004	.0010	.0011	.0012	.0012	.0013	.0013	.0013	.0013	.0013	.0013	.0013	.0013	.0014	.0014	.0014	.0014	.0014	,
		100.00	96.94	93.88	90.83	87.77	81.65 1	75.54 1	69.42 1	63.30 1	57.19 1	51.07 1	44.96 1	38.84 1	32.72 1	26.61 1	21.91 1	19.91	15.37 1	12.50 1	10.25 1	7.50 1	5.12 1	2.50 1	0
		100.00	97.36	95.04	92.84	90.64	86.22	81.75	77.18	72.49	67.63	62.55	57.21	51.56	45.55	39.13	33.83	31.46	25.77	21.90	18.65	14.40	10.39	5.53	0
	1.40083	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	(
49	.6 GAMMA	20.224	20.288	20.352	20.397	20.451	20.537	20.623	20.713	20.791	20.856	20.905	20.942	20.973	21.005	21.041	21.077	21.093	21.138	21.173	21.200	21.243	21.297	21.401	
12.	URE 524	.459	.582	.621	.638	.645	.652	.658	.663	999.	.667	.668	699.	699.	.670	699.	.667	999.	.660	.655	.650	.642	.632	.618	!!!
. PRESSURE	. TEMPERATURE	503.4	191.3	186.9	485.0	484.3	483.4	482.7	482.1	481.8	181.6	481.5	481.4	481.4	481.3	481.4	481.6	181.8	182.4	483.0	183.6	184.6	185.7	187.3	,
TOTAL	TOTAL	524.6	524.6	524.6	524.6 4	524.6	524.6	524.6 4	524.6	524.6 4	524.6	524.6	524.6	524.6	524.6	524.6 4	524.6	524.6	524.6	524.6	524.6 4	524.6	524.6 4	524.6	
98.65	43.45	9.65	9.58	9.51	9.47	9.44	9.40	9.37	9.34	9.32	9.31	9.30	9.29	9.29	9.29	9.29	9.31	9.32	9.36	9.41	9.45	9.51	9.59	9.70	
R. FLOW	EC. FLOW	11.15	12.04	12.34	12.45	12.48	12.51	12.54	12.54	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.55	12.56	,
CORR	SPEC	504.	63	672.	68	.969	703.	709.	714.	716.	-	719.	-	720.	-	720.	-	717.	711.	.904	701.	693.	683.	. 699	1
83.38	327.0	504.	632.	672.	.689	.969	703.	709.	714.	716.	718.	719.	720.	720.	720.	720.	718.	717.	711.	.902	701.	693.	683.	.699	1
FLOW	AREA	0	0.	0	0	0	0	0	0	0	0	0	0	0.	0	0	0	0	0	0	0	0	0	0	•
		-29.	-22.	-14.	-5.	5.	24.	43.	63.	80.	97.	114.	131.	149.	168.	188.	205.	212.	230.	242.	251.	262.	272.	285.	1
														705.									626.		
STATION 12		11.000-12.293	10.815-12.229	10.654-12.166	10.500-12.121	10.345-12.067	10.033-11.982	9.716-11.896	9.392-11.808	9.058-11.732	8.710-11.671	8.346-11.627	7.963-11.596	7.557-11.574	7.125-11.555	6.663-11.537	6.282-11.520	6.111-11.514	5.702-11.500	5.424-11.492	5.190-11.493	4.884-11.496	4.596-11.503	4.247-11.516	
STATI		24												12											

		O	ł	.9641	\sim	.9905	93	.9942	.9956	9966.	6966.	8966.	.9967	9966.	8966.	.9973	0866.	8866.	.9993	.9995	0000.	.0002	.0003	.0003	.0003	.0001	.9830
		PERCENT	FLOW	100.00	6.94	3.88	0.83	77	1.65	75.54	.42	63.30	57.19	51.07	44.96	38.84	72	26.61	i.	19.91	15.37 1.	.50 1	.25 1	.50 1	•	2.50 1.	00.
	Ø	PERCENT	SPAN	100.00	7	•	3.1	90.91	4	81.98	77.36	72.60	۲.	62.55	57.21	51.59	45.60	9	(v)	31.41	•	•	18.51	14.26	7	5.47	00.
/03/07 :44:24	1.40029	R*VII		1179.7	1108.1	œ.	1075.5	1064.9	1067.2	1074.5	1084.4	1106.5	1131.3	98	1259.2	1308.4	1341.1	1365.5	1389.3	1428.3	1456.0	1470.8	4		2	1604.4	1931.9
97/03. 15:44	14.80 552.7 GAMMA	MERIDINAL	DISTANCE	20.741	0.81	•	0	.99	21.093	21.193	21.292	21.384	.46		21.602	21.662	72	21.779	21.830	21.853	21.915	1.96		22.072	.14	22.286	22.770
loss.	PRESSURE 14 TEMPERATURE 55	ABSOLUTE	MACH NO.	.656	.703	. 709	.708	.701	. 689	.678	999.	.657	.651	.654	.659	.664	999.	.667	.668	.672	.674	.675	.677	.682	.688	.700	969.
ameter. -60 duct	TOTAL PRESTOTATION	ATURE	STATIC	507.4	500.1	498.5	498.7	499.3	500.9	502.5	504.2	505.7	507.0	508.0	508.6	509.0	509.4	509.9	510.2	510.6	510.8	•	511.2	511.1	•	510.6	517.8
		TEMPERATUR	TOTAL	551.1	49	48	48.	48	548.6	548.7	548.9	549.4	550.0	551.5	552.9	554.0	L)	555.3	L)	556.7	557.3	57	558.0	558.7	59	560.6	568.0
or LE flowpa nodels.	85.46	URE	STATIC	9.78	96.6	10.11	10.21	10.30	10.47	10.62	10.76	10.87	10.96	11.03	11.09	11.14	11.20	11.26	•	11.34	11.38	4.	11.42	11.42	4.	11.37	11.24
t. 22-in rotor LE 1 core duct flowps from DAWES models.	CORR. FLOW SPEC. FLOW	PRESSURE	TOTAL	13.06	ω.	14.13		14.31	14.38	14.45	14.48	14.53	14.57	14.71	14.85	14.97	•	15.18	2	15.35	15.44	4.	15.53	15.59	15.66	15.77	15.54
nt. al c fro		* * !	ABS.	724.	770.	.922	.977	768.	756.	745.	733.	724.	719.	723.	729.	735.	737.	738.	740.	744.	747.	748.	750.	756.		775.	776.
ď	83.38	* * * *	MERID.	716.	763.	769.	769.	762.	749.	737.	724.	714.	707.	709.	712.	715.	714.	712.	709.	710.	.902	703.	700.	.869	.969	694.	632.
Fan. De geomet	FLOW	* * * * *	TANG.	108.	103.	100.	102.	103.	106.	110.	115.	121.	128.	141.	155.	169.	182.	197.	212.	223.	24	256.	269.	290.	312.	345.	451.
NASA/AE QHSF Fan. Design Design report geometry. Rotor and stator perform	N 13		RADIAL	-87.	-65.	-50.	-38.	-25.	-2.	18.	38.	57.	78.	100.	122.	144.	166.	188.	207.	216.	237.	251.	262.	278.	9	313.	303.
NASA/A Design Rotor	STATION	VELOCITIES	AXIAL	711.	761.	767.	768.	761.	749.	736.	723.	712.	703.	702.	702.	701.	695.	.989	678.	.979	.999	657.	649.	640.	3	620.	555.
1*VECTORS	IDE ROTOR 1 AT	RADIUS	CORD.	.937-11.78	.783-11.	.638-11.63	.497-11.57	0.35	7-11.	.773-11.3	.470-11.2	1.5	8.829-11.073	.487-11	.130-10	7.754-10.915	.351-10	. 91	.559-10	6.397-10.810	.012 - 10	.749-10.7	5.530-10.762	. 24	4.976-10.740	4.652-10.730	4.283-10.727
	INSIDE	STL	NO.	24	23	22	21	20	19	1	1	1	15	14	13	12	11	10	σ	80	7	9	Ŋ	4	3	2	П
3_212369											n)															

FREE	PS RISE	.0121	.0192	.0265	.0322	.0374	.0471	.0575	.0687	9620.	.0904	.1015	.1130	.1253	.1416	.1614	.1792	.1889	.2074	.2195	.2291	.2370	.2415	.2390	.2455
SPEED EXIT		1474.	1453.	1434.	1415.	1396.	1357.	1317.	1276.	1234.	1190.	1144.	1096.	1045.	991.	932.	884.	862.	810.	775.	745.	707.	671.	627.	577.
WHEEL INLET		1483.	1458.	1436.	1415.	1394.	1352.	1310.	1266.	1221.	1174.	1125.	1073.	1018.	960.	898.	847.	824.	769.	731.	. 669	658.	619.	572.	519.
MERID VEL	RATIO	1.420	1.208	1.144	1.116	1.095	1.066	1.039	1.014	.997	.985	986.	.989	.993	.992	.988	.987	066.	.993	.995	666.	1.008	1.019	1.037	1.045
PS RISE		900.	.016	.025	.032	.038	.051	.063	.077	.091	.105	.119	.135	.153	.174	.200	.223	.235	.259	.275	.288	.301	.311	.315	.338
EXIT		1543.	1551.	1539.	1521.	1500.	1458.	1414.	1369.	1323.	1276.	1228.	1180.	1131.	1079.	1023.	977.	955.	906	874.	847.	813.	783.	749.	645.
REL VELCITY INLET EXIT		1566.	1589.	1585.	1574.	1558.	1524.	1489.	1453.	1415.	1376.	1335.	1292.	1247.	1200.	1151.	1110.	1092.	1047.	1017.	990.	956.	922.	881.	797.
TEMP RISE		.051	.047	.046	.046	.046	.046	.046	.046	.047	.048	.051	.054	.056	.057	.058	.059	.061	.062	.063	.064	.065	990.	690.	.083
EFFIC IENCY		91.49	85.75	86.14	86.35	87.36	89.13	90.06	90.41	90.36	90.23	90.59	91.36	92.56	94.09	95.60	96.56	96.99	97.75	98.09	98.27	98.30	98.23	98.01	89.36
PRESS -URE	RATIO	1.171	1.150	1.145	1.146	1.146	1.150	1.153	1.155	1.158	1.161	1.172	1.183	1.193	1.202	1.209	1.216	1.223	1.230	1.233	1.237	1.242	1.247	1.256	1.283
CH NO EXIT		1.397	1.415	1.406	1.389	1.369	1.329	1.287	1.243	1.200	1.156	1.111	1.067	1.023	.975	.924	.882	.862	.818	.788	.764	.734	.706	.677	.578
REL MACH NO INLET EXIT		1.424	1.462	1.465	1.458	1.444	1.414	1.383	1.350	1.315	1.279	1.241	1.201	1.160	1.116	1.070	1.032	1.015	.972	.943	.919	.885	.853	.814	.731
TURN- ING		8.88	6.03	4.90	4.40	3.98	3.45	2.95	2.50	2.28	2.23	2.68	3.27	3.96	4.59	5.35	6.22	6.98	8.41	9.55	10.73	12.69	14.94	18.43	29.32
ANGLES		62.33	60.52	60.02		59.50	59.09			57.31				50.78	48.53	6.	43.47	41.99	38.80	36.43	34.20	30.84	2	22.11	11.30
REL. P INLET		71.21	96.99	64.92	0.	63.48	62.54	61.56	.5	9	58.56	57.41		54.73	53.12	51.28	49.69	97	47.21	45.98	44.93	43.54	42.20	40.53	40.62
RHO*V RATIO		1.428	1.234	ω	Н		1.145	\vdash	7		1.102	0		9	0		1.132	1.138	1.140	1.141		1.147	7		1.141
SOLID -ITY		1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400
OMEGA BAR		.0202	.0313	.0294	.0294	.0272	.0240	.0226	.0226	.0238	.0254	.0267	.0268	.0249	.0213	.0170	.0141	.0130	.0105	.0094	.0089	.0094	.0106	.0131	.0973
D- FACTOR		.0394	.0467	.0518	.0570	.0607	.0683	.0768	.0863	.0962	.1063	.1181	.1300	.1422	.1564	.1735	.1897	.2000	.2192	.2329	.2455	.2612	.2766	.2952	.4042
STL NO.		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	2	4	33	7	П

TEMP.	EXIT	705.5	700.5	695.8	691.3	8.989	677.9	669.1	660.3	651.4	642.5	633.6	624.6	615.5	606.4	597.0	589.7	586.5	579.3	574.6	570.9	566.2	562.1	557.3	7 0 1
REL.	INLET	707.6	701.5	696.3	691.3	686.5	6.979	667.4	658.0	648.7	639.4	629.9	620.5	611.0	601.4	591.7	584.3	581.1	573.8	569.1	565.3	560.7	556.5	551.9	0 7 7 0
PRESS.	EXIT	31.00	32.37	32.44	32.00	31.41	30.16	28.91	27.63	26.35	25.10	23.89	22.75	21.65	20.59	19.56	18.76	18.43	17.67	17.19	16.81	16.33	15.92	15.45	14 09
REL. P	INLET	31.77	33.29	33.23	32.70	31.98	30.51	29.10	27.71	26.37	25.07	30	. 58	21.39	. 24	19.13	18.30	17.95	17.17	16.69	16.31	15.84	15.44	14.99	14 02
ANGLES	EXIT	8.56	7.67	7.44	7.59	7.69	8.06	8.49	8.99	9.60	10.27	11.26	12.27	13.28	14.33	15.50	16.64	17.46	18.92	20.00	21.05	22.54	24.14	26.42	35, 51
ABS. A	INLET	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00
EFF.	POLY.	25.78	63.43	78.56	84.47	86.93	90.18	92.38	93.06	93.10	2.9	93.26	93.93	95.04	96.50	97.93	98.86	99.24	99.99	100.33	100.48	100.51	100.43	100.20	78 48
CUM.	ADIA.	25.32	62.92	78.22	84.23	86.72	90.03	92.28	92.97	93.01	92.88	93.16	93.83	94.96	96.46	97.92	98.87	99.26	100.04	100.38	100.54	100.57	100.49	100.24	77.82
CUM. TT	RATIO	1.051	1.047	1.046	1.046	1.046	1.046	1.046	1.046	1.047	1.048	1.051	1.054	1.056	1.057	1.058	1.059	1.061	1.062	1.063	1.064	1.065	1.066	1.069	1.083
CUM.PT	RATIO	1.045	1.108	1.131	1.142	1.145	1.151	1.156	1.159	1.163	1.166	1.177	1.188	1.199	1.207	1.215	1.221	1.229	1.236	1.239	1.243	1.248	1.254	1.262	1,244
STL	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	2	4	3	7	_

					•	•	•	•	•	•	•	٠	•	•	•	•	•	•	•	•	٠	•	•	•	•	•	•	
			PERCENT	FLOW	100.00	96.94	93.88	90.83	87.77		75.54	69.42	63.30	57.19	51.07	44.96	38.84	32.72	26.61	21.91	19.91	15.37	12.50	10.25	7.50	5.12	2.50	00.
			PERCENT	SPAN	100.00	97.38	94.86	92.45	90.04	85.22	80.38	75.49	70.56	65.57	60.41	55.01	49.31	43.27	36.85	31.60	29.27	23.72	19.99	16.93	12.99	9.34	5.01	00.
97/03/07 15:44:24		A 1.39934	R*VU		2649.4	2525.0	2501.3	2582.8	2664.7	2779.9	2901.2	3020.1	3140.8	3252.8	3243.0	3213.4	3158.3	3077.2	2993.1	2920.2	2878.6	2814.2	2776.6	2755.1	2740.4	2738.9	2759.7	3238.4
97/03/0 15:44:		6 GAMMA	MERIDINAL	DISTANCE	. 25	21.347	21.434	21.507	21.579	21.717	21.846	21.964	22.071	16	22.258	34	. 42		57	22.658	22.690	22.772	.83	38	22.968	90	22	23.725
	18.40	590.6																										
loss.	PRESSURE	TEMPERATURE	ABSOLUTE	MACH NO	.649	. 668	.663	.661	.656	.643	.637	.636	.641	.652	.652	.658	.667	.679	. 694	.706	. 709	.724	.736	.747	.763	.781	.807	.827
ameter. -60 duct		TOTAL TEMP	VATURE	STATIC	538.7	533.7	533.9	535.8	538.2	542.3	545.5	548.1	549.9	550.8	550.6				539.8	536.8	535.4	532.0	529.7	527.7	525.0	522.4	519.0	525.5
-H			TEMPERATUR	TOTAL		581.3	580.7	582.6	584.4	587.0	589.7	592.4	595.1	597.6	597.4	596.7	595.5	593.6	591.8	590.1	589.2	587.8	586.9	586.4	586.1		586.5	597.3
• 00	v 71.06		SURE	STATIC	11.88	12.20	12.55	12.82	13.09	13.55	13.90	14.15	14.31	14.37	ς.	14.25	14.09	13.87	13.62	13.40	13.30	13.05	12.87	12.71	4.	12.27	11.98	11.50
t. 22-in rotor LE 1 core duct flowpe from DAWES models.		SPEC. FLOW	PRESSUR	TOTAL	15.77	16.46	16.85	17.19	17.47	17.88	18.26	18.57	18.86	19.12	19.10	19.05	18.99	18.89	18.79	18.68	18.61	18.51	18.44	18.40	18.37	18.36	18.39	18.02
0.0	S)	SP	*	ABS.	738.	756.	751.	750.	746.	733.	729.	729.	737.	750.	750.	755.	765.	.977	790.	801.	805.	819.	830.	841.	857.	875.	902.	929.
r. Bir	83.38	265.3	* - * - * -	MERID.	. 769	719.	713.	708.	.669	679.	. 699	.959	652.	653.	646.	645.	.059	658.	. 199	674.	675.	684.	.689	694.	702.	709.	718.	631.
ISF Fan. Des ort geometr stator perf	FLOW	AREA	* * *	TANG.	244.	235.	236.	247.	259.	277.	298.	320.	344.	368.	381.	392.	403.	413.	424.	434.	437.	451.	462.	474.	492.	513.	545.	682.
NE QHSF Fa n report g and state	14		*	RADIAL 7	-85.	-76.	-70.	-65.	-59.	-46.	-31.	-14.	4.	25.	45.	64.	84.	107.	133.	157.	168.	196.	216.	234.	260.	285.	318.	306.
NASA/AE QHSF Fan. Design Design report geometry. Rotor and stator perfor	STATION		VELOCITIES	AXIAL]	692.	715.	709.	705.	. 169	677.	. 699	655.	652.	653.	644.	642.	644.	649.	654.	655.	654.	655.	654.	654.	652.	649.	₹	552.
	1 AT			CORD.	1.267	1.175	11.087	11.013	0.941	0.804	0.677	0.563	0.461	0.371	0.291	0.220	4-10.156	0.099	0.047	10.009	-9.994	-9.961	-9.942	9.928	9.913	9.90	9	9.89
ORS	E ROTOR		RADIUS		10.874-11	10.725-11	.583-	10.443-1	10.303-10	10.020-10	9.733-10	9.441-1	9.142-10.	8.838-1	8.521-10.	.18	.83	7.459-10	.059-	.731-	. 585	. 238	Ŋ	•	5.566 -	5.337 -	5.066 -	4.751 -
1*VECTORS	INSIDE		STL	NO.							18	17	16	15	14	13	12	11	10	0	∞	7	9	Ŋ	4	m	7	Н

FREE	PS RISE	.1143	.1200	.1352	.1503	.1666	.1977	.2274	.2557	.2815	.3042	.3237	.3403	.3542	.3655	.3746	.3788	.3795	.3762	.3694	.3596	.3401	.3135	.2655	.1966
SPEED EXIT		1466.	1446.	1426.	1407.	1389.	1350.	1312.	1272.	1232.	1191.	1148.	1103.	1056.	1005.	951.	907.	887.	841.	809.	783.	750.	719.	683.	640.
WHEEL INLET		1483.	1458.	1436.	1415.	1394.	1352.	1310.	1266.	1221.	1174.	1125.	1073.	1018.	960.	898.	847.	824.	769.	731.	669	658.	619.	572.	519.
MERID VEL	RATIO	1.382	1.137	1.061	1.027	1.005	996.	.938	.918	.910	.910	899	.897	.902	.913	.927	.938	.942	.961	976.	066.	1.013	1.038	1.073	1.044
PS RISE		.101	.111	.128	.144	.162	.196	.229	.262	.292	.321	.348	.373	.397	.419	.440	.455	.461	.472	.475	.475	.470	.458	.430	.401
EXIT		1407.	1407.	1387.	1359.	1329.	1270.	1213.	1156.	1102.	1051.	1003.	.096	921.	885.	850.	823.	812.	787.	772.	760.	748.	738.	731.	633.
REL VELCITY INLET EXIT		1566.	1589.	1585.	1574.	1558.	1524.	1489.	1453.	1415.	1376.	1335.	1292.	1247.	1200.	1151.	1110.	1092.	1047.	1017.	990	926	922.	881.	797.
TEMP RISE		.113	.108	.107	.111	.114	.119	.124	.129	.134	.139	.139	.137	.135	.132	.128	.125	.123	.120	.119	.118	.117	.117	.118	.139
EFFIC IENCY		91.83	86.41	87.00	87.41	88.56	90.37	91.42	91.87	91.98	92.01	91.93	92.24	93.04	94.26	95.56	96.41	96.72	97.47	97.81	97.99	97.99	97.88	97.59	86.82
PRESS -URE	RATIO	1.414	1.366	1.365	1.381	1.400	1.429	1.457	1.481	1.503	1.524	1.522	1.518	1.513	1.506	1.498	1.489	1.483	1.474	1.469	1.466	1.463	1.462	1.464	1.488
CH NO EXIT		1.237	1.243	1.225	1.198	1.169	1.113	1.059	800.1	.959	.914	.873	.836	.804	.775	.747	.725	.716	969.	.684	.675	999.	.659	.655	.563
REL MACH NO INLET EXIT		1.424	1.462	1.466	1.458	1.445	1.414	1.383	1.350	1.316	1.279	1.241	1.202	1.160	1.116	1.070	1.032	1.015	.973	.944	.919	.886	.854	.814	.732
TURN- ING		10.91	7.26	5.84	5.43	5.24	4.86	4.84	5.12	5.86	7.01	7.49	8.38	9.60	11.09	12.95	14.60	15.28	17.53	19.26	20.91	23.36	25.98	29.66	44.36
ANGLES		60.30	59.30	59.08	9.	58.24	57.68	7	55.46	53.74	51.55	49.92	47.77	45.13	42.03	38.33	35.09	33.69	S	/	24.02	\vdash	2	10.87	-3.74
REL. A INLET		71.21	96.99	64.92	64.04	63.48	62.54	61.56	.5	59.60	•	57.41	.15	. 73	53.12	51.28	49.69	48.97	47.21	45.98	44.93	43.54	42.20	40.53	40.62
RHO*V RATIO		1.589	1.334	1.276	0	1.255	1.241	1.231	1.224	1.224	0		9	2	1.207	1.211	1.212	1.210	1.214	1.218	1.221	1.227	1.234	1.244	1.150
SOLID -ITY		1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400
OMEGA BAR		.0407	.0632	.0601	.0603	.0571	.0511	.0486	.0490	.0516	.0548	.0573	.0569	.0528	.0448	.0359	.0299	.0276	.0222	.0198	.0189	.0197	.0220	.0272	.1979
D- FACTOR		.1570	.1669	.1782	.1925	.2063	.2317	.2574	.2831	.3085	.3325	.3512	.3668	.3792	.3881	.3966	.4029	.4049	.4095	.4117	.4129	.4137	.4132	.4098	.5435
STL NO.		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	2	4	33	7	Н

TEMP.	EXIT	703.2	698.4	693.8	689.4	685.0	676.3	667.7	659.3	620.9	642.6	634.3	625.9	617.4	608.7	599.9	593.1	590.2	583.4	579.1	575.7	571.5	567.7	563.4	0
REL.	INLET	707.3	701.2	0.969	691.1	686.2	9.9/9	667.1	657.8	648.5	639.2	629.8	620.3	610.8	601.2	591.6	584.2	581.0	573.7	569.0	565.3	560.6	556.5	551.8	0 777
PRESS.	EXIT	30.23	•	31.43	31.01	30.48	29.37	28.22	27.02	25.82	4.	23.57	2.5	21.55	0	19.72	19.01	18.71	18.03	17.60	17.25	16.81	16.42	15.97	14 27
REL. P	INLET	31.78	33.30	33.24	32.71	31.98	30.51	29.10	27.72	26.37	25.07	23.80	22.58	21.39	20.24	19.13	18.30	17.95	17.17	16.69	16.31	15.84	15.44	14.99	14 02
ANGLES		19.27	18.14	18.34	19.26	20.29	22.23	24.13	26.01	27.79	29.39	30.50	31.30	31.82	32.11	32.45	32.78	32.91	33.43	33.86	34.31	35.06	35.91	37.18	47 19
ABS.	INLET	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00
EFF.	POLY.	61.90	76.68	83.99	86.93	88.68	91.06	92.55	93.10	93.25	ζ.	93.23	93.54	94.31	4.	96.73	97.55	97.88	98.63	98.98	99.16	99.19	99.10	98.84	80.55
COM.	ADIA.	60.73	75.89	83.44	86.48	88.29	90.74	92.29	92.86	92.99	0.	6.	93.29	94.10	95.36	96.69	97.57	97.92	98.72	99.09	99.28	99.31	99.22	98.94	79 64
CUM. TT	RATIO	1.113	1.108	1.107	1.111	1.114	1.119	1.124	1.129	1.134	1.139	1.139	1.137	1.135	1.132	1.128	1.125	1.123	1.120	1.119	1.118	1.117	1.117	1.118	1 139
CUM. PI	RATIO	1.262	1.317	1.349	1.376	1.399	1.431	1.461	1.486	1.510	1.531	1.529	1.525	1.520	1.512	1.504	1.495	1.489	1.481	1.476	1.473	1.470	1.469	1.472	1 442
STL	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	2	4	3	7	,

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			PERCENT	FLOW	100.00	96.94	93.88	90.83	87.77		75.54	69.42	63.30	57.19	51.07	44.96	38.84	32.72	26.61	21.91	19.91	15.37	12.50	10.25	7.50		2.50	00.
			PERCENT	SPAN	100.00	97.11	94.28	91.55	88.85	83.59	78.49	73.47	68.45	63.36	58.12	52.66	46.95	40.96	9.	29.57	27.32	ω	18.45	15.56	11.87	8.51	4.60	00.
97/03/07 15:44:24		٩ 1.39848	R*VU		4481.0	4253.0	4171.5	4261.4	4363.4	4495.4	4583.3	4662.8	4744.1	4813.8	4756.8	4669.7	4553.0	4416.8	4283.4	4171.9	4097.9	3992.8	3933.2	3893.5	3859.0	2	3865.9	4523.7
97/03/ 15:44:		4 GAMMA	MERIDINAL	DISTANCE	.77	21.883	21.992	22.088	22.185	.36	22.517	22.654	22.778	22.896	23.011	23.122	23.230	•	23.454	23.546	23.588	. 69	23.766	23.834	23.933	24.042	24.221	24.746
	21.90	623.4		Ö.	•	•	•	•	•	•••	•	•	•	•	•	•	•	•	•	•	•	•	•••	•	•••	•	•••	•
loss.		TEMPERATURE	ABSOLUTE	MACH NO	.655	.655	.653	.659	.667	.679	. 689	669.	.710	. 724	.725	. 729	. 734	.743	.756	.766	. 769	.785	. 798	.812	.834	.859	868.	.936
ameter. -60 duct			ATURE	STATIC	575.9	571.1	569.8	570.7	571.7	572.8	573.1	573.4	573.3	572.7	571.4	569.1	565.9	•	557.2	553.4	551.5	547.0	543.7	540.8		532.5	526.6	533.0
-H		TOTAL	TEMPERATUR	TOTAL	625.1	620.0	618.2	620.2	622.4	625.4	627.4	629.1	631.0	632.5	631.3	629.3	626.7	623.6	620.7	618.2	616.5	614.2	612.8		611.2		611.3	626.0
σ •	V 61.36		SURE	STATIC	14.77	15.08	15.38	15.60	15.80	16.07	16.20	16.25	16.24	16.16	16.02	15.84	15.63	15.37	15.07	14.80	14.65	14.31	14.04	13.79	13.44	13.06	12.54	11.75
t. 22-in rotor LE 1 core duct flowpe from DAWES models.		SPEC. FLOW	PRESSUR	TOTAL	19.70	20.12	20.47	20.89	21.29	21.87	22.26	22.52	22.73	22.90	22.73	22.55	22.36	22.18	22.01	21.82	21.67		21.36	21.28	21.19	•	21.17	20.66
0.0	9	SP	* *	ABS.	.077	. 191	763.	772.	782.	.967	808.	820.	833.	849.	849.	852.	856.	863.	874.	883.	885.	. 668	912.	926.	946.	972.	.010.	059.
r gi	83.38	250.7	+ + + +	ERID.	649.	655.	653.	654.	.959	657.	.959	654.	653.	654.	644.	638.	637.	640.	645.	649.	650.	629.	. 199	675.	688.		П	613. 1
ISF Fan. Des ort geometr stator perf	FLOW	AREA	* - * - * -	TANG. M	414.	399.	396.	410.	426.		472.	494.			554.				590.	599.	.009				650.		02.	863.
QHSF Fan. report geor nd stator R			*	RADIAL T	-79.	-79.			-73.					-5-		24.				124.							300.	
NASA/AE QHSF Fan. Design Design report geometry. Rotor and stator perfor	STATION 15		VELOCITIES	AXIAL RAD												638.	635.	636.								652. 2		
2 1 12	1 AT S			CORD.	-10.753	0.641	10.531	0.434	0.337	0.160	900.0	9.873	-9.754	-9.644	-9.542	-9.447	-9.360	9.280	9.	.9.152	-9.131	-9.087	-9.061	-9.042	9.022	9.007	8.995	8.998
ORS	E ROTOR		RADIUS		10.811-1	10.669-10	.529-	10.391-10	10.254-10	.982-	9.713-10	9.442 -9	.166	.885		. 284	.959	.616 -	.255 -	1	.831		.317	.149	.934 -	ω	.510 -	.241 -
1*VECTORS	INSIDE		STL	NO.						19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	Ŋ	4	3	7	Н

FREE	PS RISE	.2535	.2483	.2611	.2763	.2927	.3217	.3467	.3698	.3909	.4096	.4267	.4432	.4589	.4726	.4837	.4876	.4857	.4768	.4626	.4430	.4068	.3555	.2664	.1241
SPEED EXIT		1457.	1438.	1419.	1400.	1382.	1345.	1309.	1272.	1235.	1197.	1158.	1116.	1073.	1026.	978.	938.	921.	879.	851.	829.	800.	773.	743.	.902
WHEEL INLET		1483.	1458.	1436.	1415.	1394.	1352.	1310.	1266.	1221.	1174.	1125.	1073.	1018.	960.	898.	847.	824.	769.	731.	669	658.	619.	572.	519.
MERID VEL	RATIO	1.287	1.037	.971	.949	.943	.935	.926	.917	.912	.910	968.	.887	.884	.888	968.	.903	806.	.926	.944	.963	.993	1.030	1.086	1.014
PS RISE		.231	.232	.247	.264	.282	.316	.346	.376	.406	.434	.463	.493	.524	.555	.588	.611	.618	.633	.637	.633	.620	.593	.537	.459
EXIT		1228.	1229.	1213.	1187.	1160.	1110.	1064.	1017.	970.	926.	883.	844.	810.	780.	752.	732.	725.	711.	705.	703.	704.	711.	728.	633.
REL VELCITY INLET EXIT		1566.	1589.	1585.	1574.	1558.	1524.	1489.	1453.	1415.	1376.	1335.	1292.	1247.	1200.	1151.	1110.	1092.	1047.	1017.	990	956.	922.	881.	797.
TEMP RISE		.192	.182	.178	.182	.187	.192	.196	.199	.203	.206	.203	.200	.195	.189	.183	.178	.175	.171	.168	.166	.165	.164	.165	.193
EFFIC IENCY		92.10	86.78	87.19	87.39	88.36	90.04	90.94	91.25	91.22	91.09	96.06	91.29	92.17	93.55	95.03	95.98	96.33	•	97.56	97.75	91.16	97.65	97.33	85.28
PRESS -URE	RATIO	1.766	1.670	1.659	1.677	1.706	1.748	1.776	1.795	1.812	1.825	1.812	1.797	1.782	1.767	1.754	1.739	1.726	1.711	1.702	1.695	1.688	1.684	1.686	1.707
CH NO EXIT		1.045	1.049	1.037	1.014	066.	.947	.907	.867	.827	.790	.754	.723	.695	.672	.650	.635	.630	.620	.617	.617	.620	.629	.647	.560
REL MACH NO INLET EXIT		1.424	1.463	1.466	1.458	1.445	1.414	1.383	1.351	1.316	1.279	1.241	1.202	1.160	1.117	1.070	1.032	1.015	.973	.944	.919	.886	.854	.814	.732
TURN- ING		13.12	8.79	7.45	7.47	7.92	8.81	99.6	10.63	11.90	13.47	14.25	15.27	16.56	18.20	20.28	22.11	22.72	25.16	27.05	28.78	31.27	33.85	37.31	54.95
ANGLES		58.09	57.76	57.46	56.57	55.56	53.73	9	49.95	47.70	45.09	۲.	40.88	38.17	34.92	31.00	27.58	26.25	22.05	18.93	16.15	12.27	8.35	3.22	-14.33
REL. A INLET		71.21	96.99	64.92	64.04	63.48	62.54	61.56	.5	59.60	58.56	57.41	56.15	54.73	53.12	51.28	49.69	48.97	47.21	45.98	44.93	43.54	42.20	40.53	40.62 -
RHO*V RATIO		1.721	1.405		0	1.337	1.348		1.342	1.336	0	Н	0	2	1.259	1.255	1.250	1.247	1.248	1.252	1.258	1.266		1.299	1.124
SOLID -ITY		1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400
OMEGA BAR		.0613	.0955	.0913	.0922	.0879	.0793	.0755	.0763	.0803	.0855	.0894	.0888	.0822	.0699	.0561	.0468	.0432	.0348	.0311	.0296	.0309	.0343	.0421	.3054
D- FACTOR		.3095	.3157	.3235	.3386	.3528	.3768	.3988	.4218	.4457	.4693	.4886	.5054	.5188	.5285	.5374	.5434	.5429	.5439	.5421	.5378	.5302	.5172	.4949	.6517
STL NO.		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	Q	∞	7	9	Ŋ	4	3	2	IJ

TEMP.	EXIT	701.0	696.4	692.0	687.6	683.3	675.1	667.1	659.2	651.5	643.8	636.1	628.3	620.3	612.3	604.2	597.9	595.2	589.0	585.0	581.8	577.9	574.5	570.6	566.2
REL.	INLET	707.0	701.0	695.7	8.069	682.9	676.4	6.999	657.6	648.3	639.0	629.6	620.2	610.7	601.1	591.5	584.1	580.9	573.6	568.9	565.2	560.6	556.4	551.8	546.9
PRESS.	EXIT	9.4	30.25	30.41	30.01	29.55	28.60	27.60	26.53	25.43	24.37	23.35	22.42	21.57	20.79	20.02	19.41	19.15	18.54	18.15	17.83	17.41	17.04	16.62	14.53
REL. P	INLET		33.30	33.24	32.71	31.99	30.52	29.11	27.72	26.37	25.07	23.81	22.58	21.39	20.24	19.13	18.30	17.95	17.17	16.69	16.31	15.84	15.44	14.99	14.02
ANGLES	EXIT	32.56	31.31	31.27	32.10	32.98	34.44	35.71	37.04	38.39	39.68	40.67	41.44	41.94	42.20	42.48	42.73	42.68	42.89	43.05	43.16	43.40	43.61	44.00	54.60
ABS. F	INLET	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
EFF.	POLY.	74.03	81.24	85.73	87.49	88.82	90.80	91.97	92.35	92.36	92.26	92.15	92.46	93.28	94.56	95.93	96.83	97.17	97.97	98.34	98.54	98.57	98.47	98.19	81.11
CUM.	ADIA.	.5	80.16	84.93	86.79	88.19	90.28	91.52	91.92	91.92	91.80	91.69	92.04	92.95	94.35	95.85	96.83	97.21	98.09	98.49	98.70	98.74	98.64	98.34	79.93
CUM. TT	RATIO	1.192	1.182	1.178	1.182	1.187	1.192	1.196	1.199	1.203		1.203		1.195	1.189	1.183	1.178	1.175	1.171	1.168	1.166	1.165	1.164	1.165	1.193
CUM. PT	RATIO	1.577	1.610	1.638	1.672	1.704	1.751	1.782	1.802	1.819	1.833	1.820	1.805	1.790	1.775	1.761	1.747	1.734	1.719	1.710	1.703	1.696	1.692	1.694	1.654
STL	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	5	4	3	7	Н

		O.	.9510	.9607	70	.9726	.9767	.9792	0086.	.9798	.9795	9426.	6086.	.9834	.9870	.9905	.9927	.9936	.9954	.9963	.9967	8966.	.9967	.9961	.9612
		PERCENT F L O W	0	96.94	∞	87.77		75.54	69.42	63.30	57.19	51.07	44.96	38.84	32.72	26.61	σ	19.91	15.37	12.50	10.25	7.50	5.12	2.50	00.
		PERCENT S P A N	0.0	95.96	9.2	Ξ.	4.	•	69.97	•	9	54.15	48.63	42.98	37.22	31.35	9	24.69	19.96		14.34	11.10	8.13	4.59	00.
97/03/07 15:44:24	15444. A 1.39816	R*VU		4955.3	938.	5016.6	5095.1	5110.9	5124.9	5146.7	5158.9	5146.1	5099.4	5020.8	4921.9	4823.5	4755.5	4725.4	4654.8	4613.2	4583.5	4558.3	4550.6	4572.2	5319.7
97/0	3 RPM 7 GAMMA	MERIDINAL DISTANCE	22.293	22.492	2.78	22.905	23.113	23.274	23.410	ς.	23.658	23.779	23.898	24.019	4.	24.276	4.	24.431	24.545	24.628	24.700	24.800	24.908	25.079	25.571
	23.03 RE 635.7	NO.	0	601		664	695	712	.724				774	. 790				854		887	668	915	933	959	
ct loss	PRESSURE TEMPERATURE	ABSOL	٠	•		•	. 7 .	٠		•	•	•	•			•	•	•	•	•	•	٠	0	•	8
tip diameter. th w/ -60 duct	TOTAL PR	RATURE STATI	598	593.	87.		8 582.		5 579.0			.0 574.0		2 566.7	561	22	552	550	Ŋ	.0 543.0	.4 540.	.8 537.	534.	1 530.	8 542.
		TEMPE C TOTAL	638.	635.7	35	637.1	638.	639.	639.5	640.	640.	640.	638.	637.2	634.	632.	631.	630.	629.0	628.	627.	626.	626.	627.	643.
n rotor LE tip duct flowpath WES models.	W 58.90 W 34.27	PRESSURE TAL STATIO	16.66	16.70	6.7	9.	9.	.5	4.	16.32	16.14	15.94	15.70	15.45	Η.	14.87	9.	14.48	14.15	13.92	13.71	13.43	13.15	12.78	12.26
t. 22-in rotor LE 1 core duct flowps from DAWES models.	CORR. FLOW SPEC. FLOW	PRES	20.92	21.30		22.42	22.97	23.22	23.33	23.38	•	23.35	23.33	23.31	•	•	ς.	23.30	23.25	23.20	23.15	23.08	23.05	23.07	22.32
nt. al c fro	8 COO	* - * - * - * - * - * - * - * - * - * -	695.	716.	762.	787.	822.	841.	854.	867.	879.	892.	906	922.	939.	958.	974.	981.	00	01	02	1039.	1057.	1082.	1104.
	83.3	*-*-*- MERID	508.	542.	592.	613.	640.	652.	655.	.959	656.	657.	662.	671.	683.	695.	704.	708.	718.	724.	729.	735.	740.	747.	559.
Fan. Design t geometry. ator performa	FLOW AREA	-*-*-*- TANG.	. 474.	468.	480.	494.	. 516.	531.	. 548.	. 566.	. 585.	. 603.	. 618.	632.	. 645.	. 659.	. 673.	. 679	. 695.	. 707.	718.	. 735.	. 754.	. 783.	. 951.
OH G D		ITIES	-52	158	-61	-59	-51	-42	-31	-21	-1			37		92			159		197	220	242	271	229
NASA/AE Design r Rotor an	N 16	VELOC	506.	539.	589.	610.	638.	650.	654.	656.	656.	657.	662.			689.	694.	.969	700.	702.	702.	701.	.669	.969	510.
	STATION	CORD.	0	-10.038	9.75	-9.625	-9.416	-9.256	-9.121	ο.	•	-8.775	-8.672	-8.572	-8.475	-8.384	-8.319	-8.297	-8.250	-8.224	-8.208	-8.197	-8.195	2	-8.252
ORS	1 AT	RADIUS	.748	10.584-	10.289	10.149	9.880	9.620	9.360	9.094	8.820	8.538	8.246	7.945	7.636	7.319	7.066	6.955	6.695	•	.38	. 20	6.037		5.591
1*VECTORS	ROTOR	STL 1	4	23	ı ⊢		19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	2	4	ĸ	7	П

FREE	PS RISE	.3480	.3268	.3284	.3353	.3442	.3610	.3766	.3916	.4053	.4178	.4292	.4390	.4469	.4518	.4525	.4470	.4421	.4228	.4029	.3811	.3441	.2986	.2265	.1390
SPEED	EXT.I.	1449.	1427.	1406.	1387.	1368.	1332.	1297.	1261.	1226.	1189.	1151.	1111.	1071.	1029.	.986	952.	937.	902.	879.	860.	836.	814.	787.	754.
WHEEL	LET	1483.	1458.	1436.	1415.	1394.	1352.	1310.	1266.	1221.	1174.	1125.	1073.	1018.	960.	898.	847.	824.	769.	731.	. 669	658.	619.	572.	519.
MERID	VEL RATIO	1.008	.858	.850	.860	.881	.911	.919	.918	.916	.914	.914	.920	.931	.948	996.	.981	.988	1.010	1.025	1.040	1.061	1.084	1.116	.924
т В Б В	RI SE	.317	.300	.303	.311	.321	.342	.364	.387	.410	.433	.457	.482	.509	.537	.567	.589	.598	.613	.620	.622	.618	.608	.582	.580
LCITY	EXIT.	1099.	1101.	1100.	1083.	1067.	1037.	1005.	.696	930.	892.	856.	825.	802.	784.	769.	758.	754.	747.	744.	743.	742.	743.	747.	593.
REL VELCITA	L L L L L	1566.	1589.	1585.	1574.	1558.	1524.	1489.	1453.	1415.	1376.	1335.	1292.	1247.	1200.	1151.	1110.	1092.	1047.	1017.	.066	926	922.	881.	797.
TEMP	RISE E	.218	.212	.208	.211	.214	.218	.218	.219	.220	.220	.220	.218	.215	.210	.206	.203	.202	.199	.197	.196	.195	.195	.195	.227
EFFIC	LENCY	90.36	83.50	83.79	83.83	84.90	86.99	88.14	88.51	88.43	88.23	88.25	88.87	90.17	92.05	93.99	95.22	95.70	96.74	97.22	97.46	97.50	97.40	97.08	83.92
PRESS	-URE RATIO	1.876	1.769	1.754	1.769	1.796	1.836	1.853	1.860	1.863	1.864	1.861	1.859	1.858	1.859	1.859	1.858	1.857		1.848	1.844	1.839	1.836	1.837	1.843
ON H	EXT.I.	.917	.923	.926	.912	006.	.877	.852	.822	.790	.759	.729	.705	.687	.675	.665	.658	.656	.653	.652	.652	.653	.656	.662	.520
REL MACH NO	INLET.	1.425	1.463	1.466	1.459	1.445	1.415	1.383	1.351	1.316	1.280	1.242	1.202	1.160	1.117	1.071	1.033	1.015	.973	.944	.919	988.	.854	.814	.732
TURN-	D L NC	8.76	90.9	6.22	7.19	8.54	10.66	11.97	13.12	14.45	15.94	17.60	19.48	21.54	23.74	26.08	28.06	28.96	31.14	32.63	33.90	35.71	37.59	40.21	60.10
ANGLES	EXI.I.	62.45	60.49	58.70	56.84	54.94	51.88	49.59	47.46	45.15	9.	39.81	36.67	۲.	29.38	25.21	21.63	20.01	16.08	13.35	11.03	7.82	4.62	.33	-19.48
REL. A	L L L L L L L	71.21	96.99	64.92	64.04	63.48	62.54	61.56	.5	59.60	58.56	4.	56.15	54.73	53.12	51.28	49.69	48.97	47.21	45.98	44.93	43.54	42.20	40.53	40.62 -
RHO*V	RATIO	1.463			3		1.338	1.351		1.338	9	Ŋ	1.311		1.328	1.338	1.342	1.344	1.348	1.350		1.351	Н	1.352	1.050
SOLID	T.T.T –	1.490	1.509	1.526	1.543	1.559	1.587	1.607	1.626	1.649	1.683	1.727	1.782	1.848	1.925	2.012	2.088	2.123	2.214	2.282	2.345	2.437	2.533	2.667	2.849
OMEGA	BAR	.0819	.1321	.1279	.1300	.1245	.1121	.1057	.1061	.1110	.1175	.1221	.1205	.1110	.0941	.0752	.0628	.0578	.0464	.0414	.0393	.0408	.0451	.0550	.3923
- D	FACTOR	.3987	.4035	.4014	.4097	.4159	.4252	.4355	.4490	.4641	.4789	.4912	.4980	.4979	.4913	.4811	.4713	.4657	.4483	.4341	.4204	.4004	.3778	.3450	.5039
STL		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	9	∞	7	9	Ŋ	4	33	7	Н

TEMP.	EXIT	0.669	693.7	688.9	684.4	680.1	672.0	664.3	6.959	649.5	642.1	634.7	627.3	620.0	612.8	605.6	600.1	597.8	592.4	589.0	586.3	582.9	579.8		572.0
REL.	INLET	706.9	700.9	695.6	690.7	682.9	676.3	8.999	657.5	648.2	638.9	629.5	620.1	610.6	601.1	591.5	584.1	580.9	573.6	568.9	565.2	560.5	556.4	551.8	546.9
PRESS.	EXIT	8.6	28.95	29.04	28.61	28.20		26.60	25.63	24.62	23.62	22.69	21.88	21.18	20.59	20.00	19.53	19.32	18.84	18.52	18.24	17.88	17.54	17.14	14.73
REL. P	INLET	31.79	33.31	33.25	32.72	31.99	30.52	29.11	27.72	26.37	25.07	23.81	22.58	21.39	20.24	19.13	18.30	17.95	17.17	16.69	16.31	15.84	15.44	14.99	14.02
ANGLES	EXIT	43.01	40.80	39.19	39.02	38.89	38.85	39.19	39.89	40.78	41.70	42.52	43.05	43.29	43.34	43.46	43.70	43.81	44.07	44.32	44.56	45.00	45.52	46.35	59.56
ABS. A	INLET	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
EFF.	POLY.	74.53	79.11	82.94	84.35	85.72	8.0	ω.	89.80	. 7	89.59	89.62	90.20	91.40	93.13	94.91	96.05	96.49	97.47	97.91	98.15	98.20	98.12	97.84	80.68
CUM.	ADIA.	72.84	77.73	81.82	83.31	84.75	87.21	88.66	89.13	80.68	88.90	88.93	89.56	90.89	92.78	94.73	95.98	96.47	97.55	98.03	98.29	98.35	98.26	97.95	79.27
CUM. TT	RATIO	1.218	1.212	1.208	1.211	1.214	1.218	1.218	1.219	1.220	1.220	1.220	1.218	1.215	1.210	1.206	1.203	1.202	1.199	1.197	1.196	1.195	1.195	1.195	1.227
	RATIO		1.705	1.733	1.763	1.794	1.838	1.859	1.867	1.871	1.872	1.869	1.867	1.866	1.867	1.868	1.867	1.865	1.861	1.857	1.853	1.848	1.845	1.847	1.787
STL	NO.	24	23	22	21	20	19			16	15	14	13	12	11	10	0	∞	7	9	2	4	3	7	П

		Ø	.9510	89	.9705	.9726	.9767	.9792	0086.	.9798	.9795	9426.	6086.	.9834	.9870	.9905	.9927	.9936	.9954	.9963	.9967	8966.	.9967	.9961	.9612
		PERCENT F L O W	0.0	93.88	0	87.77	i,	75.54	69.42	.30	7.19	.07	44.96	38.84	.72	.61	.91	19.91	15.37	•	10.25	7.50	5.12	2.50	00.
	10	PERCENT S P A N	0.0	93.76	91.02	8.4	3.4	78.67	73.83	68.86	63.73	58.38	52.81	47.01	40.99	٠.	29.67	27.44	22.19	18.70	15.87	12.20	8.84	4.87	00.
7/03/07 5:44:24	A 1.39816	R*VU	5096.5	620		5016.6		5110.9	5124.9	5146.6	5158.9	5146.1		5020.8		23.	4755.5	4725.4	4654.8	4613.2	4583.5	4558.3	50.	4572.2	5319.7
97/	.03 5.7 GAMMA	MERIDINAL DISTANCE	ω,	27	4.	24.502	4.67	24.785	24.862	4.	24.986	25.040	ω.	25.133	δ.	25.222	5.	25.274	25.318	25.354		25.447	5.51	25.652	26.123
loss.	23 URE 63	ABSOLUTE MACH NO.	.570	.617	.641	.664	669.	.718	.731	.742	.752	.763	.775	. 789	908.	•	•	•	.869	.884	868.	.917	.937	996.	696.
tip diameter. .th w/ -60 duct		STATIC	0.009	588.9	7	585.6	582	579		576.7	575.4	573.5	570.8	566.9	562.3	557.4	553.5	551.6	546.8	543.4	\circ	536.9	533.3	528.8	542.4
tip dia .h w/ -	TOTAL	TEMPERATURE TOTAL STATI	638.8	') (')	635.3	637.1		639.2	639.5	640.0	640.2	640.0	638.9	637.2	634.9			630.5	629.0	0.	627.4	626.8	626.6	627.1	643.8
· m	58.90	URE STATIC	16.79	` .	. 7	9.	16.58	16.48	16.36	•	16.07	15.90	15.70	15.47	•	14.95	•	14.57	14.22	13.96	13.72	13.40	13.09	12.68	12.22
t. 22-in rotor LE 1 core duct flowpe from DAWES models	R. FLOW	PRESSURE TOTAL STA	20.92	1.65	2.03	22.42	_		~	3.38	23.38	23.35	23.33	23.31	23.32	23.33	3.3	23.30	23.25			23.08		23.07	22.32
ut. al c fro	CORR.	.*-* ABS.	684.	734.	761.	788.	826.	847.	861.	873.	884.	895.	. 906	920.	936.	953.	. 896	975.	995.	01	.023.	1041.	90	.088.	.106.
m	83.38	-*-*-*- MERID.	490.	567.	590.	614.	647.	663.	.699	671.	671.	671.	674.	681.	692.	703.	712.	717.	730.		. &			2.	
an. Deg geomet: or per:	FLOW	-*-*- FANG.	477.	467.	480.	493.	513.	527.	542.	559.	576.	592.	.909	618.	630.	643.	656.	662.	677.	688.	.869	713.	731.	757.	915.
E QHSF Fan. Design Poir report geometry. Fins and stator performance		* - \	7.	16.	20.	24.	32.	38.	43.	48.	53.	59.	. 99	74.	84.	97.	109.	115.	133.	146.	159.	177.	197.	226.	215.
NASA/AE QHSF Fan. Design Design report geometry. Rotor and stator perform		VELOCITIES AXIAL RADI	490.	566.	590.	613.	646.	662.	.899	.699	.699	.899	671.	677.	687.	697.	704.	707.	718.	725.	731.	737.	743.	749.	583.
		CORD.	-8.541	-8.252	.13	0.	•	-7.746	-7.673	-7.613	. 56	-7.523	-7.493	.47		-7.457	-7.466	-7.475		. 52	2	-7.578	.61	9	-7.745
RS	N 17	RADIUS	10.690	. 41	. 29	10.172 -					8.956					.504	.252	.141	.881	.709	.570	.391	.229	6:039	5.812
1*VECTORS	STATION	STL R	4, 0	22 1		20 1					15					10	σ	∞	7	9	2	4	Ж	7	П

		9510	.9607	.9683	9705	.9726	.9767	.9792	0086.	.9798	.9795	9426.	6086.	9834	.9870	.9905	.9927	9866	.9954	.9963	.9967	8966	.9967	.9961	9612
		100.00	96.94	93.88	90.83	87.77	81.65	75.54	69.42	63.30	57.19	51.07	44.96	38.84	.72	26.61	21.91	19.91	15.37	12.50	10.25	7.50	5.12	2.50	. 00.
		100.00	97.40	95.04	92.82	99.06	86.32	81.63	76.68	71.56	66.23	60.67	54.86	48.81	42.52	35.96	30.67	28.34	22.83	19.16	16.18	12.33	8.83	4.74	00.
	1.39816	5096.5	4955.3	4862.6	4938.0	5016.6	5095.1	5110.9	5124.9	5146.7	5158.9	5146.1	5099.4	5020.8	4921.9	4823.5	4755.5	4725.4	4654.8	4613.2	4583.5	4558.3	4550.6	4572.2	5319.7
~	' GAMMA	25.729	25.829	25.923	26.006	26.080	26.176	26.190	26.170	26.143	26.110	26.071	26.028	25.981	25.933	25.886	25.853	25.841	25.821	25.818	25.824	25.844	25.887	25.993	26.449
23.03	R 635.7	545 2	572 2	595	619 2	643 2	678 2	700	715 2	730 2	.744	.759	.775	. 794	815 2	839 2	857 2	866	885	899	910 2	926	944 2	971 2	981 2
PRESSURE	TEMPERATUR	.2	o.	6.	.2	. 9.	.2	4.	.4	. 9.	.7	.2	.7	.1	. 7.	. 0.	. 9.	. 7.	.1	. 0.	. 9.	.4	.2	. 0.	٣.
TOTAL PI	TOTAL T	8 603	2969	5 591	590	. 588	585	582	580	578	576	574	570	266	560	552	550	548	544	541	538	535	532	528	540
		638.8	635.7	633.6	635.3	637.1	638.8	639.2	639.5	640.0	640.2	640.0	638.9	637.2	634.9	632.7	631.2	630.5	629.0	628.0	627.4	626.8	626.6	627.1	643.8
W 58.90	W 34.70	17.11	17.07	17.04	17.01	16.98	16.88	16.75	16.59	16.41	16.19	15.96	15.69	15.40	15.07	14.73	14.43	14.30	13.97	13.74	13.54	13.27	12.99	12.61	12.06
CORR. FLOW	SPEC. FLOW	20.92	21.30	21.65	22.03	22.42	22.97	23.22	23.33	23.38	23.38	23.35	23.33	23.31	23.32	23.33	23.32	23.30	23.25	23.20	23.15	23.08	23.05	23.07	22.32
	4 SI	655.	684.	709.	737.	765.	804.	827.	844.	860.	875.	891.	907.	925.	946.	968.	986.	993.	1012.	1024.	1035.	1050.	1067.	1093.	1117.
83.38		455.	502.	538.	564.	588.	623.	642.	651.	658.	663.	.699	678.	692.	710.	729.	741.	746.	758.	766.	771.	778.	786.	799.	663.
FLOW	AREA	472.	465.	462.	475.	488.	508.	522.	537.	554.	572.	588.	602.	614.	625.	637.	650.	655.	670.	680.	.069	705.	721.	746.	900.
		26.	27.	28.	28.	28.	28.	29.	30.	32.	34.	38.	43.	49.	58.	71.	83.	89.	105.	116.	127.	141.	156.	177.	169.
		454.	501.	538.	563.	588.	622.	641.	651.	657.	662.	.899	. 779	691.	708.	725.	737.	741.	751.	757.	761.	.992	770.	.677	641.
		-6.808	-6.705	-6.611	-6.527	-6.453	-6.358	-6.344	-6.367	-6.400	-6.442	-6.494	-6.555	-6.625	-6.705	-6.796	-6.875	-6.912	-7.002	-7.065	-7.117	-7.188	-7.256	-7.337	-7.435
ION 18		.804	10.660	.526	.399	.274	.031	.788	.540	.286	.024	.753	.471	.180	.879	.568	.318	.209	.953	.783	.645	.470	6.310	6.126	
STATION		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	9	∞	7	9	വ	4	m	7	П

		.9510	.9607	.9683	.9705	.9726	.9767	.9792	.9800	.9798	.9795	9426.	6086.	.9834	.9870	.9905	.9927	.9936	.9954	.9963	.9967	.9968	.9967	.9961	.9612
		100.00	96.94	93.88	90.83	87.77	81.65	75.54	69.42	63.30	57.19	51.07	44.96	38.84	32.72	26.61	21.91	19.91	15.37	12.50	10.25	7.50	5.12	2.50	00.
	9	100.00	97.57	95.41	93.46	91.48	87.12	82.15	76.92	71.57	66.05	60.32	54.37	48.21	41.83	35.22	29.92	27.59	22.13	18.51	15.57	11.83	8.43	4.49	00.
	1.3981	5096.5	4955.3	4862.6	4938.0	5016.6	5095.1	5110.9	5124.9	5146.6	5158.9	5146.1	5099.4	5020.8	4921.9	4823.5	4755.5	4725.4	4654.8	4613.2	4583.5	4558.3	4550.6	4572.2	5319.7
)3	.7 GAMMA	27.464	27.492	27.521	27.556	27.577	27.546	27.433	27.298	27.162	27.026	26.887	26.748	26.609	26.473	26.342	26.249	26.212	26.138	26.102	26.083	26.076	26.096	26.179	26.617
23.0	£ 635.	16	47	74	00	27	55	89	.706	.722	3.7	53	71	91	814	839	858	867	887	006	911	56	42	99	8
SURE	TEMPERATURE	. 51	.54	. 57	. 60	.62	99.	. 68	. 7	. 7.	. 7.	. 753	.771	.791		80	80	8.	8.	9.	6	. 92	. 94	96.	96.
TOTAL PRESSURE		606.7	0.009	594.7	592.8	590.8	587.1	584.0	581.8	579.8	577.7	575.0	571.3	566.5	560.9	555.0	550.5	548.5	543.9	540.9	538.5	535.5	532.6	528.8	542.6
TOT	TOTAL	638.8	635.7	633.6	635.3	637.1	638.8	639.2	639.5	640.0	640.2	640.0	638.9	637.2	634.9	632.7	631.2	630.5	629.0	628.0	627.4	626.8	626.6	627.1	643.8
58.90	34.31	17.45	7.39	7.33	7.27	7.21	7.07	6.92	6.74	6.53	6.30	6.04	5.75	5.43	5.09	4.73	4.42	4.28	3.95	3.73	3.54	3.28	3.02	2.68	2.24
. FLOW	. FLOW	0.92	1.30 1	1.65 1	2.03 1	2.42	2.97	3.22 1	3.33 1	3.38 1	3.38 1	3.35 1	3.33 1	3.31 1	3.32 1	3.33 1	3.32 1	3.30 1	3.25 1	3.20 1	3.15 1	3.08 1	3.05 1	3.07 1	2.32
CORR.	SPEC		56. 2	∞		4	90. 2	15. 2	34. 2	51. 2	68. 2	∞	03. 2	23. 2	45. 2	68. 2	87. 2	94. 2	13. 2	25. 2	35. 2	49. 2	65. 2	88. 2	.05. 2
83.38	247.2	411. 6	466. 6	509.	538. 7	567. 7	606. 7	628.	639.	648.	656. 8	664. 8	676. 9	692.	712. 9	732. 9	746. 9	751. 9	764. 10	771. 10	776. 10	782. 10	788. 10	797. 10	650. 11
																									894.
																									149.
		411.	466.	508.	538.	567.	.909	628.	639.	648.	.959	664.	675.	691.	710.	730.	742.	747.	758.	763.	767.	771.	774.	780.	632.
1 19		0.889 -5.075	0.730 -5.043).586 -5.014	0.451 -4.978	0.320 -4.957	0.068 -4.988	9.824 -5.102	9.578 -5.240	3.326 -5.381	9.066 -5.527	3.795 -5.679	3.515 -5.836	3.224 -5.999	7.924 -6.167	7.612 -6.342	7.362 -6.482	7.253 -6.544	5.995 -6.688	6.825 -6.784	5.686 -6.861	5.510 -6.960	5.349 -7.050	5.164 -7.154	5.952 -7.272
STATION 19		24 10																		9					

		Q	.9558	.9605	.9665	.9689	.9743	.9771	.9783	.9785	.9783	.9785	.9797	.9822	.9857	.9891	.9911	.9919	.9937	.9945	.9950	.9954	.9953	.9932	.9649
		PERCENT F L O W	00	96.94	0.83	87.77	1.65	75.54	69.42	63.30	57.19	27	96	38.84	32.72	26.61	σ.	19.91	15.37	.5	10.25	7.50	5.12	2.50	00.
	10	PERCENT S P A N	0.00	97.45		90.03		79.99	74.77	69.46	63.99	58.30	52.41	•	•	•	•	26.02	20.70	17.20	•	∞.	7.62	3.92	00.
97/03/07 15:44:24	4 1.39816	R*VU	555	2498.0	475	508.	557.	2568.6	2569.3	2576.4	2582.2	578	2558.5	2521.2	2468.9	2411.3	2367.6	2349.4	2311.0	2289.1	2273.5	2260.1	2256.1	2269.3	2628.2
97/0	22.90 635.7 GAMMA	MERIDINAL DISTANCE	28.360	28.342		28.298	28.210	28.075	27.931	27.786	27.640	27.492	27.343	27.194	27.048	26.908	26.808	26.768	26.689	26.651	26.630	26.624	26.645	26.730	27.181
loss.	URE	ABSOLUTE MACH NO.	. 469	. 469	.502	.530	.572	.591	.601	609.	.617	.624	.632	.644	.658	. 677	.694	.703	.723	.737	.748	.763	. 777	. 789	.786
tip diameter. ith w/ -60 duct	TOTAL PRESSURE TOTAL TEMPERAT	TEMPERATURE TOTAL STATIC	612	0.609	605		50	597.7	596	596.0	595.2				584		575	574.1		566.	564.	561.6	52	557.9	573.4
tip di. .th w/		TEMPE	638.8	635.7	635.3	637.1	638.8	639.2	639.5	640.0	640.2	640.0	638.9	637.2	634.9	632.7	631.2	630.5	629.0	628.0	627.4	626.8	626.6	627.1	643.8
<i>w</i> •	59.23 38.76	URE STATIC	18.32	18.31	. 4	18.27		18.21	18.17	18.11	18.02	17.90	17.75	17.58	17.36	17.08	16.81	16.67	ς.	0.	∞.	•	15.39	15.14	15.06
t. 22-in rotor LE 1 core duct flowp: from DAWES models	R. FLOW	PRESSURE TOTAL STA		21.29	; ;	22.12	22.77	23.05	.18	. 26	. 28	. 26	23.23	. 22		. 22	.19	23.17	23.11	.05	23.01	22.97	22.94	22.83	22.62
nt. al c fro	CORR.	*-*- ABS.	568.	567.	605.	638.	. 789	707.	719.	728.	737.	745.	754.	765.	780.	. 662	816.	825.	845.	859.	871.	. 988	901.	913.	922.
sign Pc ry. Fi formanc	83.38	-*-*-*- MERID.	518.	518.	556.	589.	638.	657.	. 199	674.	.679	685.	691.	701.	715.	734.	751.	759.	.677	792.	803.	817.	830.	839.	814.
ISF Fan. Design Poi oort geometry. Fin stator performance	FLOW	-*-*-*- L TANG. 1		233.	237.	244.	255.	262.	269.	277.	285.	293.	300.	306.	311.	315.	320.	322.	328.	332.	336.	343.	350.	362.	432.
AE QHSF F n report and stat	20	◁	0	Ω	1 4 	-12.	-21.	-24.	-24.	-21.	-16.	-10.	-4.	2	o	19.	29.	34.	50.		73.	90.	108.	130.	162.
NASA/AE QHSF Fan. Design Design report geometry. Rotor and stator performa	STATION 2	VELOCITIES	518.	518.	556.	589.	637.	656.	.999	673.	679.	684.	691.	701.	715.	734.	751.	758.	777.	790.	800.	812.	823.	829.	798.
	1 AT S	CORD.	-4.180	-4.194	-4.216	-4.236	-4.324	-4.460	-4.607	-4.758	-4.913	-5.074	-5.241	-5.414	-5.593	-5.777	-5.924	-5.989	-6.139	-6.238	-6.318	-6.419	-6.510	-6.614	-6.725
RS	VANE	RADIUS	0	739	441	10.300				9.310							7.402	7.297	7.050	6.888	6.757	6.592	6.444	6.273	060.9
1*VECTORS	INSIDE	STL R		23	ı ⊢	0				16											വ	4	ĸ	7	П
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MERID VEL RATIO	1.260	1.110	1.054	1.034	1.040	1.051	1.047	1.042	1.039	1.036	1.031	1.023	1.013	1.005	1.003	1.007	1.011	1.020	1.028	1.035	1.046	1.054	1.053	1.254
PS RISE	.251	.237	.224	.213	.204	.198	.205	.218	.231	.243	.255	.265	.272	.276	.274	.268	.265	.255	.249	.244	.239	.237	.237	.280
VELOCITY T EXIT	568.	567.	584.	605.	638.	687.	707.	719.	728.	737.	745.	754.	765.	780.	799.	816.	825.	845.	859.	871.	886.	901.	913.	922.
ABS VEI INLET	623.	.959	685.	716.	747.	790.	815.	834.	851.	868.	885.	903.	923.	945.	968.	987.	994.	1013.	1025.	1035.	1049.	1065.	1088.	1105.
TEMP RISE	.218	.212	.208	.211	.214	.218	.218	.219	.220	.220	.220	.218	.215	.210	.206	.203	.202	.199	.197	.196	.195	.195	.195	.227
EFFICIENCY	93.14	83.42	82.48	81.54	82.77	85.64	96.98	87.55	87.66	87.57	87.62	88.23	89.51	91.32	93.15	94.28	94.71	95.68	96.15	96.43	96.64	96.56	95.28	85.92
PRESS 1 -URE :	1.909	1.768	1.740	1.744	1.772	1.820	1.839	1.848	1.854	1.856	1.854	1.851	1.850	1.850	1.850	1.848	1.846	1.841	1.837	1.833	1.830	1.827	1.818	1.868
EXIT	.469	.469	.484	.502	.530	.572	.591	.601	609.	.617	.624	.632	.644	.658	.677	.694	.703	.723	.737	.748	.763	.777	.789	.786
ABS MACH INLET EX	.516	.547	.574	.600	.627	.665	.689	.706	.722	.737	.753	.771	.791	.814	.839	.858	.867	.887	006.	.911	.926	.942	996.	.968
TURN- ING	24.33	20.52	18.73	18.20	18.18	18.07	17.90	17.95	18.07	18.19	18.20	18.07	17.83	17.63	17.63	17.83	17.95	18.25	18.50	18.74	19.10	19.43	19.63	26.07
ABS. ANGLES INLET EXIT	48.72 24.38	44.72 24.20	42.09 23.36	41.28 23.08	40.63 22.45	39.84 21.77	39.66 21.75	39.92 21.97	40.41 22.33	40.95 22.77	41.39 23.19	41.55 23.48	41.42 23.59	41.11 23.49	40.89 23.26	40.90 23.07	40.93 22.98	41.07 22.82	41.26 22.76	41.47 22.73	41.86 22.76	42.30 22.88	42.96 23.33	53.99 27.92
RHO*V RATIO	1.311	1.152	1.093	1.073	1.082	1.100	1.101	1.103	1.108	1.112	1.114	1.113	1.111	1.109	1.114	1.122	1.127	1.139	1.149	1.159	1.173	1.186	1.192	1.459
SOLID -ITY	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400
OMEGA BAR	1073	.0028	.0405	.0663	.0579	.0336	.0276	.0218	.0168	.0141	.0129	.0124	.0124	.0129	.0137	.0146	.0151	.0155	.0151	.0142	.0116	.0111	.0228	0297
D- FACTOR	.2213	.2600	.2671	.2730	.2620	.2448	.2459	.2524	.2603	.2683	.2763	. 2833	.2886	.2914	.2915	.2895	.2880	. 2833	.2799	.2774	.2747	.2746	. 2823	.3104
STL NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	2	4	3	2	IJ

ഥ	OLY	7.0	9.0	1.7	2.2	3.7	6.7	8.2	8.9	9.0	8.9	9.0	9.6	0.7	2.4	4.1	5.1	5.5	6.5	6.9	7.2	7.4	7.3	96.19	2.5
CUM.	ADIA.	75.52	77.64	80.52	81.02	82.62	85.85	87.49	88.17	88.31	88.23	88.30	88.93	90.22	92.05	93.89	95.04	95.49	96.48	96.97	97.26	97.49	97.42	96.15	81.25
B	AT			7		7					7	7			7		7	7	۲.	⊣:	Η.	Η.	⊣.	1.195	
M M	ATI	.70	.70	.71	.73	.77	.82	.84	.85	.86	.86	.86	.86	.85	.85	.85	.85	.85	.84	.84	.84	.83	.83	1.828	.81
STL	0	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	Ŋ	4	ĸ	7	П

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		PERCENT		100.00	. m	90.83	87.77		75.54	69.42	63.30	57.19	51.07	44.96	38.84	32.72	26.61	21.91	19.91	15.37	12.50	10.25	7.50	5.12	2.50	00.
		PERCENT	S P A N	100.00	94.85	91.94	80.68	83.63	78.30	72.99	99.19	62.21	56.56	50.71	44.63	38.35	31.88	26.76	24.53	19.36	15.99	13.28	9.90	6.90	3.46	00.
97/03/07 15:44:24	1.39816	R*VU		785.4	674.7	678.8	684.1	697.1	701.5	700.6	701.7	703.6	704.3	702.1	695.3	681.8	661.7	642.3	634.5	620.8	613.6	8.809	605.3	605.0	610.2	705.8
97/0	7 GAMMA	MERIDINAL	STAN	29.197	9.13	29.076	29.014	28.875	28.719	28.563	28.407	28.252	28.095	27.936	27.77	27.619	27.469	27.361	27.319	27.235	27.194	27.172	27.164	27.187	27.274	27.735
loss.	22.78 TURE 635.7	ABSOLUTE M	NO.	.454 425	. 427	.435	.468	.520	.539	.550	.559	.562	.563	.563	.565	.573	.586	.597	.603	.617	.628	.639	.656	.673	.684	.722
· t)	. PRESSURE AL TEMPERATURE	·	ric	613.7	611.5	612.2	610.4	606.2	604.2	603.1	602.5	602.3	602.0	601.0	599.0	595.9	592.3	589.4	588.0	584.6	582.3	580.2	577.3	574.8	573.7	583.3
크.	TOTAL F TOTAL	TEMPERATURE	Ţ	638.8	633.6	635.3	637.1	638.8	639.2	639.5	640.0	640.2	640.0	638.9		634.9		631.2	630.5	629.0	628.0	627.4	626.8	626.6	627.1	643.8
· m	59.56 W 38.96	SURE	ΑT	18.80	. ∞	18.79	18.78	18.77	18.78	18.76	18.73	18.71	18.69	18.67	•	18.50	18.31	18.12	18.02	17.76	17.56	17.38	17.12	•	16.53	16.20
	CORR. FLOW SPEC. FLOW	PRESSURE	TOTAL	21.64	1 [\vdash	21.82	22.57	22.87	23.04	23.15	23.18	23.17	23.14	23.12	23.11	23.10	23.06	23.03	22.96	22.91	22.87	22.86	22.83	22.60	22.91
Poin Fina nce	Ð	* * *	. AB	. 551.	517.	. 528.	. 567.	. 627.	. 649.	. 662.	672.	. 676.	. 676.	. 676.	. 678	. 686.	. 698.	. 710.	. 716.	. 731.	. 743.	. 754.	. 772.	. 190.	. 803.	. 854.
'an. Design geometry. or performa	83.38 A 220.2	* * *	Σ.	. 546		. 52	_,	. 62				. 672		. 671		. 680	. 69	. 705	. 711	. 726	. 738	. 74	. 767	. 78	. 79	. 846
ISF Fan. oort geom stator p	FLOW AREA	* *	Ţ	. 72															. 87			6		. 93	6	. 114
reg reg	21	VELOCITIES	. RADIAL												-16								. 13	. 2	. 48	84
NASA/AE Design Rotor a	STATION	VELOC	AXIAL	546.	513.	524.	563.									680.	693.	705.	710.	726.			767.	784.		
	1 AT S'		ORD	-3.342	-3.401	-3.458	-3.520	-3.659	-3.816	-3.976	-4.137	-4.302	-4.472	-4.648	-4.832	-5.021	-5.217	-5.371	-5.438	-5.594	-5.696	•	-5.880	-5.970	-6.074	-6.178
TORS	E VANE	RADIUS		10.889	10.592	10.439	10.292	10.024	9.777	9.533	9.287	9.037	8.777	8.508	8.228	7.939	7.641	7.406	7.303	7.065	6.910	6.786	6.630	6.492	6.334	6.174
1*VECTORS	INSIDE	STL	NO.	24 24	2 2 2 2 2 3	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	S	4	m	2	П
3-212369									7	220)															

MERID	VEL	RATIO	1.328	1.096	1.009	.973	.993	1.028	1.028	1.029	1.030	1.024	1.012	.993	.972	.956	.947	.945	.946	.951	.957	.965	.981	966.	1.000	1.303
PS	RISE		.388	.361	.340	.319	.302	.289	.295	.306	.321	.340	.363	.385	.404	.414	.417	.416	.415	.410	.405	.400	.392	.383	.370	.393
ABS VELOCITY	EXIT		551.	516.	517.	528.	567.	627.	649.	662.	672.	.929	.929	.979	678.	.989	.869	710.	716.	731.	743.	754.	772.	790.	803.	854.
ABS VE	INLET		623.	656.	685.	716.	747.	790.	815.	834.	851.	868.	885.	903.	923.	945.	968.	987.	994.	1013.	1025.	1035.	1049.	1065.	1088.	1105.
TEMP	RISE		.218	.212	.208	.211	.214	.218	.218	.219	.220	.220	.220	.218	.215	.210	.206	.203	.202	.199	.197	.196	.195	.195	.195	.227
EFFIC	IENCY		95.70	83.34	81.16	79.23	80.64	84.28	85.78	86.58	86.90	86.90	87.00	87.60	88.84	90.58	92.32	93.35	93.74	94.63	95.10	95.41	95.79	95.72	93.50	87.86
PRESS	-URE	RATIO	1.941	1.767	1.726	1.719	1.748	1.804	1.825	1.837	1.845	1.848	1.846	1.844	1.842	1.842	1.841	1.837	1.835	1.829	1.825	1.822	1.821	1.818	1.800	1.892
ON H	EXIT		.454	.425	.427	.435	.468	.520	.539	.550	.559	.562	.563	.563	.565	.573	.586	.597	.603	.617	.628	.639	.656	.673	.684	.722
ABS MACH NO	INLET		.516	.547	.574	.600	.627	.665	.689	.706	.722	.737	.753	.771	.791	.814	.839	.858	.867	.887	006.	.911	.926	.942	996.	.968
TURN-	ING		41.19	37.12	35.01	34.21	33.89	33.48	33.31	33.55	33.95	34.34	34.58	34.54	34.26	33.92	33.77	33.88	33.96	34.17	34.40	34.64	35.07	35.53	36.07	46.29
ANGLES	EXIT		7.53	7.59	7.08	7.08	6.74	6.37	6.35	6.37	6.46	6.61	6.81	7.01	7.16	7.19	7.12	7.01	6.97	06.9	98.9	6.83	6.19	6.77	68.9	7.69
ABS. AN	INLET		48.72	44.72	42.09	41.28	40.63	39.84	39.68	39.92	40.41	40.95	41.39	41.55	41.42	41.11	40.89	40.90	40.93	41.07	41.26	41.47	41.86	42.30	42.96	53.99
RHO*V	RATIO		1.414	1.158	1.064	1.025	1.049	1.095	1.102	1.112	1.123	1.128	1.126	1.119	1.109	1.103	1.104	1.110	1.113	1.126	1.138	1.151	1.174	1.195	1.202	1.604
SOLID	-ITY		1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400	1.400
OMEGA	BAR		2075	.0056	.0811	.1323	.1153	.0672	.0553	.0435	.0335	.0282	.0257	.0248	.0248	.0256	.0272	.0291	.0299	.0308	.0299	.0283	.0230	.0220	.0454	0588
- D	FACTOR			.4285	.4519	.4664	.4420	.4039	.4011	.4042	.4112	.4238	.4396	.4557	.4688	.4762	.4798	.4817	.4819	.4803	.4781	.4751	.4688	.4641	.4706	.4731
STL	NO.		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	2	4	3	7	П

)LY	4.	9.	4.	0.	. 7	.5	.1	0.			4.	0.	.1			.3	9	.5	9	. 2	9	.5	94.54	.3
CUM.	ADIA.	78.01	77.57	79.21	78.71	80.49	84.50	86.31	87.20	87.55	87.57	87.68	88.29	89.55	91.31	93.06	94.11	94.51	95.43	95.91	96.24	96.64	96.58	94.37	83.17
٠.	ATI(.21	.21	.20	.21	.21	.21	.21	.21	. 22	. 22	. 22	.21	.21	.21	.20	.20	.20	.19	.19	.19	.19	.19	1.195	. 22
Σ	ATI(.73	.70	.70	.71	.74	.80	.83	.84	.85	.85	.85	.85	.85	.85	.84	.84	.84	.83	.83	.83	.83	.82	1.809	.83
STL	0	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	2	4	3	7	Η

		Ы	N		•	-																			
		PERCENT	FLOV	96.94	93.88	90.83	7	i.	5.5	4.	63.30	57.19	51.07	44.96	38.84	32.72	26.61	21.91	19.91	15.37	ζ.	10.25	7.50	5.12	2.50
		PERCENT	SPAN	97.19	(٧)	9.		1.3	ь.	70.57	65.25	59.89	54.40	48.74	42.85	9	•	25.01	22.78	17.72	14.48	11.91	8.77	6.03	2.94
4:24	1.39816	R*VU	c	0.0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
15:44	5 7 GAMMA	MERIDINAL	DISTANCE	30.056	9.	9	9.73	о О	29.366	о О	ο.	φ.	28.700	ω.	.36	•	•	7.91	27.866	27.774	27.729	27.706	27.698	27.722	27.812
	22.65 635.7		. D	n m	2	2	2	2	7	7	7	7	7	7	7	(1	(1	7	2	7	(1	(1	2	2	2
loss.	URE	ABSOLUTE	MACH NO	.387	.372	.369	.411	.486	.516	.536	.549	. 555	.556	.556	.549	.536	.527	.519	.520	.537	.551	. 564	.585	.604	.610
-60 auct		ATURE	STATIC	617.3	616.6	618.6	616.3			604	603.7	603.2	602.8	601.9	601.1	9.009	9.669	599.1	598.4	594.8	592.2	590.0	586.8	584.2	583.9
	TOTAL	TEMPERATUR	TOTAL :	635.7	633.6	635.3	637.1	638.8	•	639.5	640.0	640.2	640.0	638.9	637.2	634.9	632.7	631.2	630.5	629.0	628.0	627.4	626.8	626.6	627.1
core duct flowpath W/	59.89 37.31	URE	STATIC	፣ ፫	19.20	19.20	19.16	0.	18.93	18.83	18.77	18.73	18.70	18.69	18.76	18.92	0.	19.09	19.05	18.75	18.53	18.33	18.04	17.76	17.40
ormance from DAWES models	CORR. FLOW SPEC. FLOW	PRESSUR	TOTAL	21.27	i.	21.08	i.	2	ď.	Ċ.		3.08	3.07			23.01	•	22.93	22.90	22.82	22.77	22.74	22.75	22.72	22.37
nce fro	CORR	*	ABS.	471.	453.	449.	500.	588.	623.	646.	661.	. 899	. 699	. 899	629.	644.	632.	622.	623.	642.	657.	671.	695.	716.	722.
formance	83.38	* *	MERID.												629.			2.	623.			1.	695.	716.	722.
or perfe	FLOW AREA	* * * *			0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Rotor and stator			RADIAL TANG	7.	19.	26.	32.	39.	40.	37.	33.	30.	29.	29.	28.	25.	20.	13.	-20.	-17.	-14.	-13.	-10.	-5-	1.
Rotor a		VELOCITIES	AL.	471.	452.	448.	499.	587.	622.	645.	.099	. 199	.699	. 199	629.	643.	632.	622.	623.	642.	657.	671.	695.	716.	722.
	10N 22		Ūς	.48	. 58	•	-2.801	•	-3.170	•	•	•	-3.867	-4.050	-4.241	-4.442	-4.651	-4.820	-4.892	-5.056	.16	.24	.34	.43	-5.535
	1 STATION	RADIUS		о С	.60	. 44	. 29	.01	9.760	•	•	00.	8.751	8.486	•	•	7.616	•	7.268	•	•	6.759	6.611	6.483	6.338
	VANE	STL	NO.	23	22	21	20	19	18	17	16	15	14	13	12	11	10	σ	∞	7	9	Ŋ	4	ĸ	7

MERID	VEL	RATIO	1.334	1.010	.890	.835	.882	.970	.993	1.010	1.020	1.019	1.008	.988	.953	.904	.864	.835	.829	.841	.853	.865	.889	606.	906.	1.246
PS S	RISE		.494	.459	.433	.405	.374	.333	.319	.318	.326	.343	.364	.388	.422	.465	.499	.525	.529	.516	.507	.499	.486	.472	.455	.474
LOCITY	EXIT		548.	471.	453.	449.	500.	588.	623.	646.	661.	668.	.699	.899	629.	644.	632.	622.	623.	642.	657.	671.	695.	716.	722.	810.
ABS VELOCITY	INLET		623.	.959	685.	716.	747.	790.	815.	834.	851.	868.	885.	903.	923.	945.	968.	987.	994.	1013.	1025.	1035.	1049.	1065.	1088.	1105.
TEMP	RISE		.218	.212	.208	.211	.214	.218	.218	.219	.220	.220	.220	.218	.215	.210	.206	.203	.202	.199	.197	.196	.195	.195	.195	.227
EFFIC	IENCY		98.58	83.25	79.82	76.88	78.48	82.91	84.59	85.61	86.13	86.23	86.37	96.98	88.16	89.84	91.48	92.42	92.76	93.58	94.05	94.41	94.95	94.89	91.72	94.68
PRESS E		RATIO	1.976	1.766	1.712	1.693	1.724	1.788	1.811	1.826	1.836	1.840	1.839	1.836	1.834	1.833	1.831	1.827	1.825	1.818	1.814	1.812	1.812	1.810	1.781	1.916
ON H	EXIT		.452	.387	.372	.369	.411	.486	.516	.536	.549	.555	.556	.556	.549	.536	.527	.519	.520	.537	.551	.564	.585	.604	.610	.681
ABS MACH NO	INLET		.516	.547	.574	.600	.627	.665	.689	.706	.722	.737	.753	.771	.791	.814	.839	.858	.867	.887	006.	.911	.926	.942	996.	.968
TURN-	ING		48.72	44.72	42.09	41.28	40.63	39.84	39.68	39.92	40.41	40.95	41.39	41.55	41.42	41.11	40.89	40.90	40.93	41.07	41.26	41.47	41.86	42.30	42.96	53.99
ANGLES	EXIT		00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.	00.
ABS. AN	INLET		48.72	44.72	42.09	41.28	40.63	39.84	39.68	39.92	40.41	40.95	41.39	41.55	41.42	41.11	40.89	40.90	40.93	41.07	41.26	41.47	41.86	42.30	42.96	53.99
RHO*V	RATIO		1.448	1.084	.951	.889	.942	1.041	1.069	1.093	1.112	1.121	1.121	1.113	1.092	1.059	1.033	1.015	1.014	1.034	1.051	1.069	1.102	1.130	1.127	1.596
SOLID	-ITY		2.191	2.107	2.014	1.905	1.818	1.717	1.689	1.679	1.673	1.670	1.674	1.686	1.709	1.745	1.799	1.854	1.882	1.954	2.008	2.057	2.127	2.197	2.279	2.381
OMEGA	BAR		3218	.0085	.1222	.1988	.1730	.1010	.0832	.0654	.0503	.0423	.0386	.0373	.0374	.0386	.0409	.0435	.0447	.0459	.0446	.0421	.0343	.0328	.0677	0876
D-	FACTOR		. 2914	.4488	05	.5460	.5096	.4424	.4252	.4172	.4178	.4274	.4418	.4573	.4793	.5071	.5286	.5457	.5475	3	$^{\circ}$.5118	.4937	.4796	.4841	.4337
STL	NO.		24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	Q	∞	7	9	2	4	3	7	1

Ē	ХTC	2.0	8.0	9.2	7.8	9.7	4.2	6.0	7.1	7.6	7.7	7.8	8.4	9.5	1.0	2.6	3.4	3.7	4.5	5.0	5.3	5.8	5.8	92.90	6.0
	DIA	0.8	7.4	7.8	6.3	8.3	3.1	5.1	6.2	6.7	6.9	7.0	7.6	8.8	0.5	2.2	3.1	3.5	4.3	4.8	5.2	5.7	5.7	92.58	5.0
E	AT	2	2	.2	2	2	2	7	2	7	7	7	2	2	2	.2	.2	2	i,	Η̈́.	i.	Η̈́.	i.	1.195	7
M M	ATI	.76	.70	.69	.68	.72	.79	.81	.83	.84	.84	.84	.84	.84	.84	.84	.83	.83	.82	.82	.82	.82	.81	1.790	.85
\vdash	NO.	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	0	∞	7	9	Ŋ	4	ĸ	7	Н

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Design and Test of Fan/Nacelle	Models Quiet High-Speed	Fan Design	
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Russ Repp, David Gentile, Davi	a Hanson, and Stinivas Cit	ınduru	
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13. ABSTRACT (Maximum 200 words)			
The primary objective of the Qu	iet High-Speed Fan (OHSF) program was to develop an ac	lyanced high-speed fan
design that will achieve a 6 dB r			
performance. The program appl			
developed by NASA, U.S. indus		hnology is needed to maintain	U.S. industry leadership in
the regional turbofan engine ma	·ket.		
14. SUBJECT TERMS			15. NUMBER OF PAGES
Jet aircraft noise; Compressor b	ades; Turbofans; Turbomac	chinery	245 16. PRICE CODE
17 SECUDITY OF ASSISTATION 19 9	FOURITY OF ACCUSES	Table of our province and our province a	20 I IMITATION OF ARSTRACT
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